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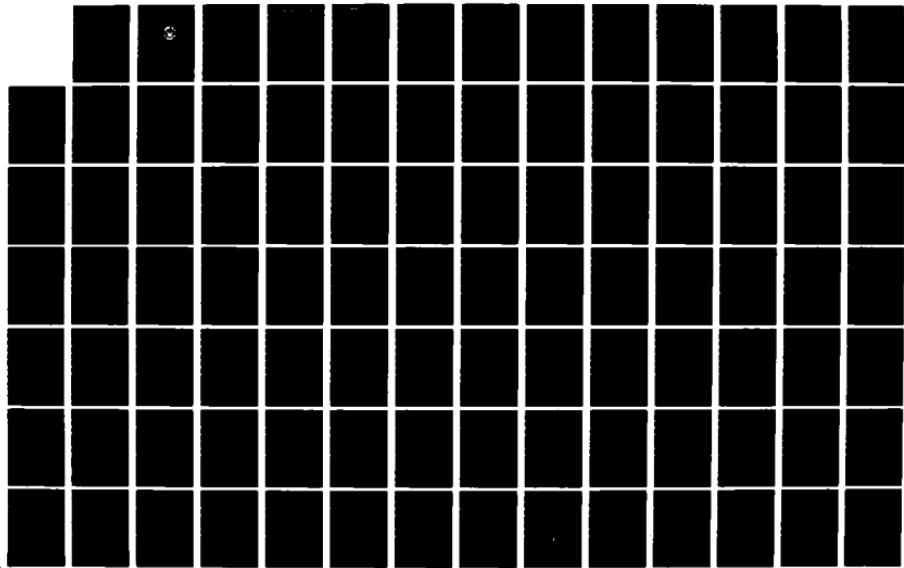
TURBOMACHINES WITH RESULTS FOR NASA TASK-1 COMPRESSOR

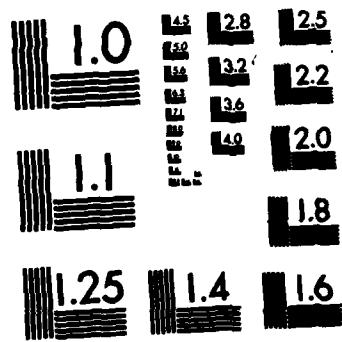
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# THESIS

FINITE ELEMENT PROGRAM FOR CALCULATING  
FLOWS IN TURBOMACHINES WITH RESULTS  
FOR NASA TASK-1 COMPRESSOR

BY

Julian A. Ferguson III

October 1982

Thesis Advisor:

Raymond P. Shreeve

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# Finite Element Program for Calculating Flows in Turbomachines with Results for NASA Task-1 Compressor

by

Julian A. Ferguson III  
Lieutenant, United States Navy  
B.S., Auburn University, 1975

Submitted in partial fulfillment of the  
requirements for the degree of

## **MASTER OF SCIENCE IN AERONAUTICAL ENGINEERING**

and

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## **ABSTRACT**

A general mesh generation code (MESHGEN) and finite element flow solver (TURBO) for calculating the flow development through axial turbomachines are fully documented. The finite element approach followed Hirsch and Warzee. Excellent results were obtained for the NASA Task-1 compressor operating with subsonic flow conditions. Construction of the code will allow straightforward extension to transonic flows, turbine stages and multiple stage machines.

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### LIST OF SYMBOLS

| <u>Symbol</u>    | <u>Description</u>   | <u>Units</u>        |
|------------------|--|---------------------|
| $\psi$           | Stream function  | lbm/sec             |
| $\alpha$         | Absolute flow angle whose tangent is the ratio of the absolute tangential-to-meridional velocity   | degrees             |
| $\beta$          | Relative flow angle whose tangent is the ratio of the absolute tangential-to-meridional velocity   | degrees             |
| $\delta$         | Deviation angle, difference in flow angle and camber-line angle at trailing edge in cascade projection tangential-to-meridional velocity | degrees             |
| $\kappa_1$       | Angle between tangent to the blade meanline and the axial direction  | degrees             |
| $\phi$           | Camber angle, difference between angles in cascade projection of tangents to camber line at extremes of camber-line arc                  | degrees             |
| $\sigma$         | Solidity, ratio of chord to blade spacing  | dimensionless       |
| $\tilde{\omega}$ | Total-pressure-loss coefficient  | dimensionless       |
| T                | Temperature  | °R                  |
| P                | Pressure   | psia                |
| $\rho$           | Density  | lbm/ft <sup>3</sup> |

**Subscripts**

t      Total conditions  
1      blade leading edge  
2      blade trailing edge  
E      Over an element

**Superscripts**

e      For an element

## I. INTRODUCTION

### A. STATEMENT OF TASK

The original task of this research project was to continue the work of Macchi [Ref. 1] and Cirone [Ref. 2] in the development of a turbine prediction computer code for the Turbopropulsion Laboratory at the Naval Postgraduate School. An analysis of the referenced code by Ferguson [Ref. 3] indicated that a significant amount of work remained to be done in order to make the program operational. In the author's opinion the task could be better accomplished through the use of a different solution technique. After additional study and review of similar work [Refs. 4, 5, 6 and 7] it was decided that a finite element approach to the problem would be adopted. A program developed by Gavito [Ref. 8], which followed the work of Hirsch and Warzee [Ref. 4], was selected as the basis for development of the computer code described in the sections that follow. However, Gavito's program was formulated as a compressor performance prediction which, as it was reported, had not given results similar to those obtained by Hirsch and Warzee. Thus the first goal of the project became the development of an axial compressor prediction code that could produce results comparable to those published by Hirsch and Warzee. The second goal was to revise and document the program so that its

application to any compressor and its extension to turbine analysis could be carried out without excessive difficulty.

#### B. DESCRIPTION OF THE PROBLEM

One purpose of conducting a through-flow analysis is to predict the performance of a turbomachine under design and off-design operating conditions. Through the combination of a mathematical model and empirically determined correlations, it is possible to provide the engineer with a tool that will determine the effects of variations in design parameters and analyze the performance of an existing machine.

The first problem addressed in the formulation of a performance prediction code is that of expressing the analysis in a form that can be accurately and efficiently solved by computer methods. Most methods for through-flow calculations are based on the classic work of Wu [Ref. 9] which stated that the equations of fluid flow in turbomachines can be solved on the intersecting sets of stream surfaces known as the S1 family and S2 family of stream surfaces (Fig. 1). In general the intersection of a S1 surface and a S2 surface is a twisting line with three dimensional variations. However, if an axisymmetric assumption is made, the S2 surface will lie on a meridional plane and the directional derivatives on the S2 surface become the  $\partial(\ )/\partial r$  and  $\partial(\ )/\partial z$  in cylindrical coordinates. As shown by Smith [Ref. 10], circumferentially averaged equations with an axisymmetric

flow assumption can be used to a good first approximation for the through-flow analysis.

Three general methods of solving the so-called radial equilibrium equation of flows in turbomachines which results from the axisymmetric assumption can be found in the literature. The first is called the streamline curvature method [Refs. 1, 2 and 11]. The method derived its name from the fact that the radius of curvature of the streamline is an integral part of the formulation. The second, a matrix method, applies a finite differences technique to the radial equilibrium equation, normally after it has been reduced to a Poisson form [Refs. 12 and 13]. The third method is the finite element method which was used in the present work.

In the mid-1970's, Hirsch and Warzee [Ref. 4] first reported the application of the finite element method to solution of the axisymmetric radial equilibrium equation. They applied the finite element technique to solve the equation expressed in quasi-harmonic form in terms of the stress function. They published extensive comparisons of the predictions obtained using their method with measurements obtained on several machines under various operating conditions. In general, the method produced excellent results for compressors and turbines of single and multi-stage configurations. It was this demonstrated ability of the method over

such a wide range of parameters that led to its selection for use in the present project.

In the sections which follow, the development of programs MESHGEN and TURBO, which are based on the work of Hirsch and Warzee [Ref. 4], is documented. Comparisons are given of the results obtained when the program was applied to analyze the NASA Task-1 compressor with results obtained by the referenced authors.

## II. MATHEMATICAL MODEL

The equation of motion for a fluid has the general form  
 (Vavra [Ref. 14])

$$(\partial \vec{V}/\partial t) + (\vec{V} \cdot \nabla) \vec{V} = -\nabla p/\rho + \vec{\epsilon}_f - \nabla(gz) \quad (1)$$

Using the vector identity

$$(\vec{V} \cdot \nabla) \vec{V} = \nabla(v^2/2) - (\vec{V} \times \nabla \times \vec{V}) \quad (2)$$

Eq. (1) can be written as

$$\partial \vec{V}/\partial t + \nabla(v^2/2 + gz) = -\nabla p/\rho + \vec{\epsilon}_f + (\vec{V} \times \nabla \times \vec{V}) \quad (3)$$

The first law of thermodynamics for a fluid particle can be written as

$$Tds = dh - dp/\rho \quad (4)$$

which, along an elemental path length  $d\vec{r}$  in a fluid field implies that

$$T(d\vec{r} \cdot \nabla s) = (d\vec{r} \cdot \nabla) h - (d\vec{r} \cdot \nabla) p/\rho \quad (4a)$$

or

$$d\vec{r} \cdot (\nabla h - T\nabla s - \nabla p/\rho) = 0 \quad (5)$$

Since  $d\vec{r}$  is not equal to zero in general, in a homogeneous fluid flow the first law may be expressed as

$$\nabla h - T\nabla s - \nabla p/\rho = 0 \quad (6)$$

Substituting Eq. (6) into Eq. (3) yields

$$\frac{\partial \vec{V}}{\partial t} + \nabla(h + V^2/2 + gz) = T\nabla s + \vec{F}_f + \vec{V} \times \nabla \times \vec{V} \quad (7)$$

For steady, inviscid flow Eq. (7) reduces to

$$\nabla H = T\nabla s + (\vec{V} \times \nabla \times \vec{V}) \quad (8)$$

As shown by Hirsch and Warzee [Ref. 15], Eq. (8) can be revised to describe the flow through blade rows by introducing a circumferential averaging process and assuming that the flow is axisymmetric at the averaged condition. The averaged equation can be expressed as

$$-(\vec{V} \times \nabla \times \vec{V}) = T\nabla s - \nabla H + \vec{F}_b + \vec{F}_d \quad (9)$$

where  $F_b$  is the body force representing the action of the blades on the flow and  $F_d$  represents the dissipative force whose work generates the irreversible entropies. The  $F_d$  forces are considered to be uniformly distributed in the tangential direction and proportional to the loss coefficients. Equation (9) leads to the following three equations in cylindrical coordinates (with  $\partial(\ )/\partial\theta = 0$ )

$$(V_u/r)(\partial(rV_u)/\partial r) - V_z(\partial V_r/\partial z) - (\partial V_z/\partial r) = \partial H/\partial r - T(\partial s/\partial r) - F_{b,r} - F_{d,r} \quad (10a)$$

$$(V_z/r)(\partial(rV_u)/\partial z) + (V_r/r)(\partial(rV_u)/\partial r) = F_u \quad (10b)$$

$$V_r(\partial V_r/\partial z) - (\partial V_z/\partial r) - (V_u/r)(\partial(rV_u)/\partial z) = \partial H/\partial z - T(\partial s/\partial z) - F_z \quad (10c)$$

Equation (10a) expresses the radial equilibrium of the meridional through-flow and it is the governing equation to be solved for the velocity components. Equations (10b) and (10c) determine the tangential and axial components of the forces once Eq. (10a) has been solved.

For the solution of Eq. (10a) to have physical meaning, care must be taken to ensure that continuity is satisfied throughout the field. In general, the continuity equation can be expressed as

$$\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \vec{V}) = 0 \quad (11)$$

which for steady, circumferentially averaged flow can be written as

$$(\frac{\partial}{\partial r})(\rho b V_r) + (\frac{\partial}{\partial z})(\rho b V_z) = 0 \quad (12)$$

where  $b$  is a blockage factor describing the reduction in the flow area in the tangential direction due to the presence of rotor and stator blades. The tangential blockage is approximated by

$$b = 1 - t/s \quad (12a)$$

where  $t$  is the blade thickness and  $s$  is the blade spacing. As will be discussed in the description of subroutine INPUT, this factor will be modified to account for the end-wall boundary layer contractions. A stream function,  $\psi$ , can be introduced and defined so that Eq. (12) is automatically satisfied as follows:

$$\frac{\partial \psi}{\partial r} = \rho rbv_z \quad (13a)$$

$$\frac{\partial \psi}{\partial z} = -\rho rbv_r \quad (13b)$$

After the substitution of Eqs. (13a) and (13b) into Eq. (10a) one is assured of the implicit satisfaction of continuity as the radial equilibrium equation is solved explicitly. Equation (10a) may now be written as

$$\begin{aligned} \left( \frac{\partial}{\partial r} \right) \left( \frac{1}{\rho rb} \left( \frac{\partial \psi}{\partial r} \right) \right) + \left( \frac{\partial}{\partial z} \right) \left( \frac{1}{\rho rb} \left( \frac{\partial \psi}{\partial z} \right) \right) = \\ \left( \frac{1}{V_z} \right) \left( \frac{\partial H}{\partial r} \right) - T \left( \frac{\partial s}{\partial r} \right) + \\ \left( V_u/r \right) \left( \frac{\partial (rv_u)}{\partial r} \right) - F_{b,r} - F_{d,r} \end{aligned} \quad (14)$$

The equation is written in a slightly different form for solution in rotor regions where rothalpy remains constant along a streamline. The definition of rothalpy,  $H_R$ , given by

$$H_R = H - UV_u \quad (15)$$

is substituted into Eq. (14) and the terms in brackets on the right-hand side become

$$\left[ \frac{\partial H_R}{\partial r} - T \left( \frac{\partial s}{\partial r} \right) - \left( V_u/r \right) \left( \frac{\partial (rv_u)}{\partial r} \right) - F_{b,r} - F_{d,r} \right] \quad (16)$$

The significance of the  $F_{b,r}$  and  $F_{d,r}$  terms can be analyzed in the following manner. The body force  $F_b$  acts in a direction normal to the mean blade surface, which for radial blading is in the circumferential direction. The term  $F_{b,r}$  accounts for the body force component in the radial direction that results when blade lean is present. For most blading this is a small term that may be neglected.

Similarly,  $F_{d,r}$  is the contribution of the dissipative forces in the radial direction for non-cylindrical stream surfaces. This contribution can normally be neglected for axial-flow machines, which is the case treated here. (Note that the two body force terms are included in the analysis for machines in which the magnitudes of these forces are significant.) With these simplifications the radial equilibrium equation may be written in the form

$$\begin{aligned} \left(\frac{\partial}{\partial r}\right)\left(\frac{1}{\rho rb}\left(\frac{\partial \psi}{\partial r}\right)\right) + \left(\frac{\partial}{\partial z}\right)\left(\frac{1}{\rho rb}\left(\frac{\partial \psi}{\partial z}\right)\right) = \\ \left(1/V_z\right)\left(\frac{\partial H}{\partial r}\right) - T\left(\frac{\partial s}{\partial r}\right) + \left(V_u/r\right)\left(\frac{\partial(rV_u)}{\partial r}\right) \end{aligned} \quad (17)$$

with the appropriate modifications for solution in a rotor region.

### III. FINITE ELEMENT METHOD

#### A. INTRODUCTION

The finite element method is a numerical procedure that is particularly well suited to solving problems in continuum mechanics, which invariably involve equations expressed in differential form. As stated by Cook [Ref. 16], the essence of the finite element method is the "piecewise approximation of a function  $\phi$ , by means of polynomials, each defined over a small region (element) and expressed in terms of nodal values of the function."

In order to understand the finite element method and the solution techniques employed in the computer program reported herein, one must first have a complete understanding of the basic element, the terminology used to describe the element, and the relationship between the element and the remainder of the solution domain. The complete problem is solved in a piecewise manner, in which the solution of the derived governing relationship over the discrete region of an element is sought to determine the contribution of the element to the overall solution. Figure 2 illustrates the single element as it is used in the present work and the nomenclature for the element on what is referred to as the "local domain". The number scheme to employ is arbitrary, limited only by the requirement that the system remain

consistent from element to element. Figure 3 shows a vertical stack of three elements to show how elements are connected in what is known as a "global domain". Table 1 lists the relationship between the two reference systems, known as the connectivity. The connectivity is important because the solution of a problem over a computational region involves a careful summation of the local contributions of each element to the global equations. The summation process is tracked by the connectivity. Again the global numbering scheme is arbitrary, influenced mainly by considerations of computer storage and computational efficiency.

The key concept to be grasped is that the finite element method is a series of local solutions that are coupled together through the connectivity relationships to form a solution for the entire computational domain. A more detailed description of the finite element method is contained in Refs. 16 through 18.

#### B. APPLICATION OF THE WEIGHTED RESIDUAL PROCESS

A standard weighted residuals process was used to transform Eq. (17) into a form that can be solved by numerical techniques. As a first step, the equation was written in a more compact form as

$$(\partial/\partial r)[k(\partial\psi/\partial r)] + (\partial/\partial z)[k(\partial\psi/\partial z)] + f = 0 \quad (18)$$

where

$$k(r,z) = (1/\rho rb) \quad (19)$$

$$f(r,z) = (1/V_z) [T(\partial s/\partial r) - \partial H/\partial r + (V_u/r)(\partial(rV_u)/\partial r)] \quad (20)$$

with the boundary conditions

$$k(\partial\psi/\partial n) + \alpha_1(\psi - \psi_0) = 0 \quad (21)$$

on the associated exterior surface S. Equations (18) and (21) may be rewritten as

$$(1/r)\{(\partial/\partial r)[k(\partial\psi/\partial r)] + (\partial/\partial z)[k(\partial\psi/\partial z)] + f\} = 0 \quad (22)$$

in the volume, V, and

$$(1/r)[k(\partial\psi/\partial n) + \alpha_1(\psi - \psi_0)] = 0 \quad (23)$$

on the surface, S. An approximation,  $\tilde{\psi}(r,z)$ , of the unknown solution is searched for such that the corresponding weighted residual,  $\bar{R}$ , is equal to zero. Analytically this is expressed as

$$\bar{R} = \int_V W(r,z) R_V(r,z) dV + \int_S W(r,z) R_S(r,z) dS = 0 \quad (24)$$

where  $W(r,z)$  is the (known) weight function and  $R_V$  and  $R_S$  are the so-called "residuals" in the volume and on the surface, respectively. As the sum of  $R_V$  and  $R_S$  approaches zero, the approximation,  $\tilde{\psi}$ , approaches the exact solution,  $\psi$ , with  $R_V$  and  $R_S$  defined to be identically zero if  $\tilde{\psi} = \psi$ . By defining  $R_V$  to be equal to the left-hand side of Eq. (22) and  $R_S$  to be equal to the left-hand side of Eq. (23), Eq. (24) can be written as

$$\begin{aligned} & \int_{\Omega} [-W(r,z)\{(\partial/\partial r)\{k(\partial\psi/\partial r)\} + (\partial/\partial z)\{k(\partial\psi/\partial z)\} + f\} 2\pi d\Omega \\ & + \int_C W k(\partial\psi/\partial n) 2\pi dC = 0 \end{aligned} \quad (25)$$

Integration of the first term of Eq. (25) by parts yields

$$\int_{\Omega} [k \{ (\partial \psi / \partial r) (\partial W / \partial r) + (\partial \psi / \partial z) (\partial W / \partial z) \} - W_f] d\Omega = 0 \quad (26)$$

if  $\psi$  is selected to equal  $\psi_0$  along the corresponding part of the boundary. The second term of Eq. (26) reduces to zero through the proper application of the boundary conditions.

The boundary conditions must be satisfied in different ways for different portions of the boundary. Along the inlet where  $\alpha_1 = 0$  the conditions may be satisfied by specifying  $(\partial \psi / \partial n)$  to be zero or by specifying the nodal values of  $\psi$  if the inlet conditions are conducive to calculating  $\psi$  for each node. Along the shroud and along the hub the value of  $\psi$  must be specified as  $\psi = (m/2\pi)$  at the shroud and  $\psi = 0$  at the hub. For nodes at the exit plane, the condition that  $(\partial \psi / \partial n) = 0$  is required.

#### C. APPLICATION OF THE FINITE ELEMENT METHOD

The first step is to discretize the region into sub-regions or elements. Within each element the unknown stream function,  $\psi$ , and the coordinates  $r$  and  $z$  are assumed to have the following polynomial variations:

$$\psi(r, z) = \sum_i^n \psi_i N_i(\xi, \eta) \quad (27a)$$

$$r = \sum_i^n r_i N_i(\xi, \eta) \quad (27b)$$

$$z = \sum_i^n z_i N_i(\xi, \eta) \quad (27c)$$

where  $n$  = number of nodes in the element

$N_i$  = the shape or (trial) function for node  $i$

$\psi_i$  = value of  $\psi$  at node  $i$

$r_i$  = value of  $r$  at node  $i$

$z_i$  = value of  $z$  at node  $i$

Equations (27b) and (27c) imply a geometrical as well as functional transformation or mapping, as shown for the present case in Fig. 4.

The second step in the process is the selection of the weight and shape functions. The shape functions are defined when the particular type of finite element is selected for the computational grid [Ref. 16]. The eight-noded quadrilateral was used in the present program and its associated shape functions were entered in a subroutine. The weight function is independent of the shape function and may be chosen at the discretion of the individual. In the present case the standard Galerkin technique was employed and therefore, the weight functions were defined as being equal to the shape functions, so that

$$W(r,z) = N(r,z) \quad (28)$$

Equation (26) may now be expressed in the following form:

$$\int_E \left\{ k \left[ (\partial N_j / \partial r) \sum_i^n (\partial N_i / \partial r) + (\partial N_j / \partial z) \sum_i^n (\partial N_i / \partial z) \right] - N_j f(r,z) \right\} d\Omega = 0 \quad (29)$$

where  $\int_E$  represents the integral over an element. In matrix notation this becomes

$$[K]^e \{\delta\}^e = \{f\}^e \quad (30)$$

where

$$k_{ij}^e = \int_E k(r, z) [(\partial N_j / \partial r)(\partial N_i / \partial r) + (\partial N_j / \partial z)(\partial N_i / \partial z)] d\Omega \quad (31a)$$

$$f_i^e = \int_E N_i f(r, z) d\Omega \quad (31b)$$

and

$$\delta_i = \psi_i \quad (31c)$$

The summation of the elemental contributions over the entire region yields the global system of equations needed to solve the problem. In matrix notation the global system of equations is expressed as

$$[K]\{\delta\} = \{f\} \quad (32)$$

where

$$[K] = \sum_i^m [K]_i^e \quad (33a)$$

$$\{\delta\} = \sum_i^m \{\delta\}_i^e \quad (33b)$$

$$\{f\} = \sum_i^m \{f\}_i^e \quad (33c)$$

and  $m$  = number of elements in the mesh

$$\delta_i = \psi_i$$

$[K]$  = system's stiffness matrix

$\{f\}$  = system's right-hand side vector

Since  $k_{ij}$  and  $f_i$  depend on the unknown solution  $\psi$ , Eq. (32) is a nonlinear differential equation to be solved by an iterative procedure. The details of the procedure are contained in section V.

#### D. NUMERICAL INTEGRATION TECHNIQUE

In general, problems are analyzed using a coordinate system in which the boundary conditions can be written and satisfied in the simplest possible way. For problems with irregularly shaped boundaries and/or mixed conditions along different portions of the boundary, obtaining numerical solutions in the original coordinate system can be a formidable task. Very often a scheme must be found to transform the derived equations into a new coordinate system that conforms to the requirements of standard numerical techniques. Traditional transformation techniques tend to be complicated exercises in algebra that require extensive reformulation for each geometry or type of boundary condition. The power of the finite element method is the automatic inclusion of a transformation of the geometry and the function to a rectangular domain where a variety of integration techniques may be employed. This can be seen in Fig. 4, which illustrates what is implied by Eqs. (27a), (27b), and (27c). The method can handle extremely complicated boundary conditions

and shapes with ease and is limited only by the type of element selected by the individual.

The quadratic properties of the eight-node element allows curved boundaries in the physical domain so long as the section of the boundary included within a single element does not have a point of inflection. The use of a quadratic element also ensures continuity of the approximated function along the elemental boundaries regardless of the direction of approach from within the mesh. The specific numerical technique used in the program is discussed in the following section.

### 1. Stiffness Matrix Evaluation

In section C the following relationship was derived for the individual elements of the stiffness matrix, [K]:

$$k_{ij}^e = \int_E k(r,z) [(\partial N_j / \partial r)(\partial N_i / \partial r) + (\partial N_j / \partial z)(\partial N_i / \partial z)] d\Omega \quad (31a)$$

In order to take advantage of well established numerical integration techniques, Eq. (31a) must be transformed from the (r,z) domain and its irregular elemental boundaries to the rectangular ( $\xi, \eta$ ) domain. Equations (27a) through (27c) describe the variation of the function and the (r,z) coordinate values in the ( $\xi, \eta$ ) plane. In order to transform Eq. (31a) it is necessary to determine the relationship of the variations of the derivatives in the two domains. These relationships can be derived in a straightforward manner through the use of the chain rule as follows:

$$(\partial N_i / \partial \xi) = (\partial N_i / \partial z) (\partial z / \partial \xi) + (\partial N_i / \partial r) (\partial r / \partial \xi) \quad (34)$$

and

$$(\partial N_i / \partial \eta) = (\partial N_i / \partial z) (\partial z / \partial \eta) + (\partial N_i / \partial r) (\partial r / \partial \eta) \quad (35)$$

Equations (34) and (35) can be combined in matrix form as

$$\begin{Bmatrix} \frac{\partial N}{\partial \xi} \\ \frac{\partial N}{\partial \eta} \end{Bmatrix} = \begin{bmatrix} \frac{\partial z}{\partial \xi} & \frac{\partial r}{\partial \xi} \\ \frac{\partial z}{\partial \eta} & \frac{\partial r}{\partial \eta} \end{bmatrix} \begin{Bmatrix} \frac{\partial N_i}{\partial z} \\ \frac{\partial N_i}{\partial r} \end{Bmatrix} \quad (36)$$

Through the selection of the type element to be used in the mesh,  $N_i(\xi, \eta)$  is a known function [Ref. 16], which makes possible the computation of the left-hand side vector for any point within the element boundaries. Similarly, by taking the appropriate derivatives of Eqs. (27b) and (27c) the 2x2 matrix, known as the Jacobian matrix [J], can be determined. It follows that the r and z derivatives of the shape/weight functions can be determined for any point of an element from

$$\begin{Bmatrix} \frac{\partial N_i}{\partial z} \\ \frac{\partial N_i}{\partial r} \end{Bmatrix} = \begin{bmatrix} \frac{\partial z}{\partial \xi} & \frac{\partial r}{\partial \xi} \\ \frac{\partial z}{\partial \eta} & \frac{\partial r}{\partial \eta} \end{bmatrix}^{-1} \begin{Bmatrix} \frac{\partial N_i}{\partial \xi} \\ \frac{\partial N_i}{\partial \eta} \end{Bmatrix} \quad (37)$$

An examination of Eqs. (27a) through (27c) shows that once the derivatives of the shape/weight functions are known for a point then it is a simple procedure to determine the

derivatives of any other elemental property that has a value specified at the nodes.

The final relationship that is needed for the transformation is the relationship between the differential change in the coordinate directions of the  $(r,z)$  plane and the  $(\xi,\eta)$  plane. As shown by Kaplan [Ref. 19], the required relationship is

$$dz/dr = |J|d\xi/dn \quad (38)$$

Equation (31a) can now be transformed for integration in the  $(\xi,\eta)$  plane to the form

$$k_{ij}^e = \int_{-1}^1 \int_{-1}^1 \sum_k^n N_k k_k ([B]^T [B]) |J| d\xi d\eta \quad (39)$$

where

$$[B] = [J]^{-1} \begin{Bmatrix} \frac{\partial N}{\partial \xi} \\ \frac{\partial N}{\partial \eta} \end{Bmatrix} \quad (40)$$

and  $k = (1/\rho r b)$  (41)

The Gauss-Legendre method of numerical integration was used to obtain a solution to Eq. (39). It was selected because its determined accuracy was easily determined and the simple summing procedure used in the solution could be efficiently coded. A one dimensional example is used here to illustrate the use of the method.

The definite integral

$$I = \int_{-1}^1 \phi(\xi) d\xi \quad (42)$$

may be written in the form

$$I = w_1 \phi_1 + w_2 \phi_2 + w_3 \phi_3 + \dots + w_n \phi_n \quad (43)$$

where  $\phi_i = \phi(\xi_i)$

$w_i$  = Gaussian weight function for  $\xi_i$

The values of the points, called abscissas, and their corresponding weighting function values are catalogued for use. The number of points to be used is determined by the order of the function to be approximated. In general, a polynomial of  $(2n - 1)$  is integrated exactly by the use of  $n$  Gauss points. In two dimensions the Gauss method can be written as

$$I = \int_{-1}^1 \int_{-1}^1 \phi(\xi, \eta) d\xi d\eta \quad (44)$$

which can be written as

$$I = \sum_i^n \sum_j^m w_i w_j \phi(\xi_i, \eta_j) \quad (45)$$

Equation (39) can now be written in a form that can be coded for solution by the computer as

$$k_{ij}^e = \sum_k \sum_l w_k w_l \sum_m N_m k_m ([B]^T [B]) |J| \quad (46)$$

The scheme used in the program is a two dimensional, three-point Gauss integration. A detailed description of the evaluation of  $[B]^T [B]$  and the method used to obtain  $k_m$  is contained in the description of subroutine STIFF.

## 2. Right-hand Side Vector Evaluation

In Section C the following relationship for  $f$  was derived:

$$f_i^e = \int_E N_i f(r, z) d\Omega \quad (31b)$$

By using the techniques of Section D1, Eq. (31b) can be replaced by

$$f_i^e = \sum_j \sum_k w_j w_k \sum_i N_i \sum_\ell N_\ell F_\ell |J| \quad (47)$$

A more detailed description of the specific methods used to solve Eq. (47) is contained in the description of subroutine FCAL.

#### **IV. DESCRIPTION OF PROGRAM MESHGEN**

##### **A. MAIN PROGRAM DESCRIPTION**

MESHGEN was developed in order that the required inputs for the program TURBO could be generated in a fast, accurate, and conceptually correct manner. The program generates an eight-node isoparametric element mesh, computes the related connectivity matrix, defines the type of region enclosed by each element, computes the tangential blockage factor and an estimate of the stream function for each node in the mesh, and computes the thermodynamic conditions and velocity at the inlet. The inputs required to operate the program are the mass flow rate, total temperature, and total pressure at the inlet, the blading and machine geometries, RPM,  $C_p$ ,  $\gamma$ ,  $R$ , and scaling constants for the plot of the mesh. The blading geometries must be coded in the program as a subroutine that uses approximations or design information to express the blade variables as functions of radius. The user provides the other information in response to interactive prompts. The program has two modes of operation, one which generates a complete mesh and all of the associated information and another which uses a previously generated mesh to compute the changes in specific arrays that result from a change in the inlet conditions.

The program is completely general and may be used for either single-stage axial compressor or turbine, and, with very minor modifications, can be expanded for use with multi-stage machines. The mesh size that can be generated by the program is limited only by computer storage considerations. To use the program for another machine, the user is required to replace subroutine TASK1 with a subroutine that can compute the tangential blockage factor, b, for the desired blading. The functioning of the program and its subroutines for both modes of operation is described in the section B.

The program's algorithm in outline form is as follows:

Algorithm:

Determine the value of the appropriate operating conditions and whether a new mesh is desired (Subroutine INIT1).

Obtain the coordinates of the super element corners and a description of the division of the super elements into the final mesh. Compute the storage allocation parameters (Subroutine INPUT).

Compute the (r,z) coordinates for all nodes in the mesh (Subroutine TMESH).

Compute the connectivity relationships for the mesh and determine the beginning and ending node numbers for the rotor and stator (Subroutine CONEC).

Compute the array of node numbers where the value of  $\psi$  is to be specified. Compute an initial estimate of the nodal stream function distribution and call subroutine FLOFCT to determine the inlet conditions (Subroutine INIT2).

Compute the nodal tangential blockage factor, b, for the rotor and stator nodes (Subroutine TASK1).

Place the computed values in disk storage (Subroutine FILGEN).

Plot the generated mesh on the Tektronix 618 terminal for inspection (Subroutine MPLOT).

END

## B. SUBROUTINE DESCRIPTIONS

### 1. Subroutine INIT1

Subroutine INIT1 obtains the value of thermodynamic variables and plotting parameters that are required for the program in either mode of operation. The user is required to provide the values of the mass flow (lbm/sec), total temperature ( $^{\circ}$ R), and total pressure (psia), ratio of specific heats ( $\gamma$ ), gas constant (ft-lbf/lbm- $^{\circ}$ R), and the specific heat at constant pressure (BTU/lbm- $^{\circ}$ R). The scaling constants are convenient values of r and z used to frame the plot of the mesh. If a new mesh is not desired, the program exits the subroutine.

Subroutine INIT1 determines if a new mesh is to be generated by the response of the user to an interactive question. If a mesh is to be generated, the subroutine obtains some preliminary information about the flow region. Figure 5 shows how the user must first divide the flow region into a coarse mesh known as super elements, recording the (z,r) coordinates of the corner points. The minimum number of super elements for a single-stage compressor is five so that the three duct regions, the rotor, and the stator may be represented. The maximum number of super elements and the subsequent division into mesh elements is limited only by the

storage limitations of the machine. In practice, the maximum number of elements is limited by the number of equations that may be solved by the program TURBO. It is also limited by the fact that only one super element may be used to describe a rotor or duct region and that the super elements for these regions may only be further subdivided into a single column of elements. The latter restrictions are for compatibility with the computation procedures used in the program TURBO. A decision must then be reached on what subdivision of the super elements will provide a mesh that is sufficiently fine to yield accurate results efficiently. Once the flow region has been divided into super elements and a determination as to the total number of rows and columns of mesh elements has been made, the user can input the appropriate values in response to the prompts provided by the program.

## 2. Subroutine INPUT

Subroutine INPUT uses interactive prompts to obtain a description of the turbomachine's flow passage geometry and the desired mesh characteristics. The user must provide the program with the coordinate values of the super element nodes as shown in Fig. 5. The values are entered as nodal pairs on a station-by-station basis. The first (z,r) coordinate entered is the node on the shroud and the second lies on the hub. The program then asks the user to identify the type of region enclosed in each super element and into how many columns each super element is to be divided.

Enough information is now available for the program to compute and store the information required for the program TURBO. Subroutine INPUT stores the responses to the interactive prompts, determines the values of the storage location pointers, and determines if any storage limitation has been exceeded. If any storage limitation is exceeded, the subroutine calls the appropriate error subroutine to halt execution. The interactive portion of the subroutine is omitted if a new mesh is not desired. However, the values of the pointers are calculated and storage requirements are evaluated as before. A listing of the pointers and the corresponding variables is given in Appendix A.

### 3. Subroutine TMESH

Subroutine TMESH computes the nodal coordinate values from the information obtained in subroutines INIT1 and INPUT. The subroutine uses linear interpolation in the axial direction and a quadratic scheme in the radial direction. The radial interpolation scheme maintains the difference in the squares of the radius of the nodes as a constant. This allows the assumption of equal mass flow between the nodes for uniform axial velocity which is used to determine the initial estimate of the nodal stream function distribution. The nodes are numbered with the assumption that the fluid flow is from left-to-right in the mesh and that the number of columns of elements is greater than or equal to the number of rows of elements. The node at the inlet shroud is labeled

1 and the node at the outlet hub is labeled n for an n node mesh. The numbering proceeds on a column-by-column basis from top-to-bottom. The total number of rows, columns, and mesh elements is displayed to the user. The mesh computations are omitted if a new mesh is not created and elemental count information is displayed as before.

#### 4. Subroutine CONEC

Subroutine CONEC generates the connectivity matrix for the mesh. The connectivity matrix is used to keep track of which nodes are assigned to which elements and the arrangement of the assigned nodes within the element. The connectivity relationships for a three element stack is shown in Fig. 3. Additionally, the subroutine determines the first and last nodes of the rotor and stator.

#### 5. Subroutine INIT2

Subroutine INIT2 stores the node numbers for nodes where the value of  $\psi$  is specified as a known quantity. The array is used in the program TURBO to apply the boundary conditions. The array is not computed if a new mesh is not desired. For either mode of operation, subroutine INIT2 computes the values of the inlet thermodynamic variables, the inlet axial velocity, and an initial estimate of the nodal stream function distribution. Subroutine FLOFCT is used to calculate the inlet conditions and is described in the next section. The initial stream function is computed from the boundary conditions at the shroud and hub. Along

the shroud,  $\psi$  is specified to be equal to  $(\dot{m}/2\pi)$  and along the hub to be zero. The value of  $\psi$  along the inlet is determined by a linear interpolation because of the quadratic node spacing in the radial direction. The remaining nodal values of  $\psi$  are obtained by an assumption that  $\psi$  is a constant along the streamwise boundaries of the elements. The assumption is obviously in error, but it observes the boundary conditions and provides a reasonable first estimate to begin the iteration scheme used in the program TURBO.

#### 6. Subroutine FLOFCT

Subroutine FLOFCT computes the inlet conditions for a passage with a specified geometry, mass flow rate, total temperature, total pressure, and an assumed uniform inlet velocity. The method followed is the "total flow function" formulation proposed by Shreeve [Ref. 20]. The total flow function is defined as the ratio of the mass flux to the limiting or stagnation mass flux. The following definitions and equations are required for the method:

$$v_t = [2H]^{0.5} \quad (48)$$

$$x = v/v_t \quad (49)$$

$$T/T_t = 1 - x^2 \quad (50)$$

$$p/p_t = (1 - x^2)^{\frac{\gamma}{(\gamma-1)}} \quad (51)$$

$$\rho/\rho_t = (1 - x^2)^{\frac{1}{(\gamma-1)}} \quad (52)$$

From the definition of the total flow function,  $\phi$ , it follows that

$$\phi = \rho V / \rho_t V_t = X(1 - X^2)^{\frac{1}{\gamma-1}} \quad (53)$$

The value of  $\phi$  at the inlet can be calculated at the inlet from the assumed uniform conditions by the expression

$$\phi_1 = \dot{m} / (\rho_t V_t A) \quad (54)$$

The value of  $X$  at the inlet is found through the following Newtonian iteration:

$$\phi_1 = \dot{m} / (\rho_t V_t A) \quad (54)$$

Assume  $X = 0.1$  to assure the selection of the subsonic root.

Calculate:  $\phi = X(1 - X^2)^{\frac{1}{\gamma-1}}$  (53)

and  $d\phi/dX = \{1/X - 2X/[(\gamma - 1)(1 - X^2)]\}$

Test  $|\phi_1 - \phi| < \epsilon$

If the test fails then calculate

$$X = X + (\phi_1 - \phi) (d\phi/dX)$$

and recalculate  $\phi$  until convergence is reached. Once convergence is reached the inlet conditions are computed by equations (50) through (52).

#### 7. Subroutine TASK1

Subroutine TASK1 computes the nodal tangential blockage factor for the blading of the NASA TASK1 transonic compressor. The value of the blockage factor is determined by

$$b = 1 - t/s$$

(12a)

The values of  $t$  and  $s$  are obtained by approximations to the known blading geometry that are expressed as functions of radius. The maximum thickness of the blade is artificially defined to be at the mid-point of the chordline to ensure that the factor is accounted for in the calculations in the program TURBO. This artificiality could easily be removed through a modification to the axial interpolation scheme used in rotor and stator super elements. Subroutine TASK1 is the only machine dependent subroutine in use in the program and would need to be replaced with an appropriate substitute in order for the program to be used on another machine. The use of the subroutine is omitted if the user does not desire a new mesh.

#### 8. Subroutine FILGEN

Subroutine FILGEN places the computed mesh parameters on disk storage for use in the program TURBO. If the limited mode of operation was selected by the user, the subroutine only updates the values of the parameters that change for a new inlet condition. A listing of the output parameters and their corresponding storage location is given in Appendix B.

#### 9. Subroutine MPLOT

Subroutine MPLOT provides an on-line plot of the computed mesh on the Tektronix 618 graphics terminal. Figure 6 shows the 63 element, 222 node mesh used in the computations of the test cases. The subroutine displayed the mesh through

direct calls to the subroutines of the library plotting package, GRAFF. The subroutine's algorithm is as follows:

Algorithm:

Form two Real\*4 arrays from the information in the r coordinate and the z coordinate arrays for plotting compatibility.

Sort the arrays and plot the streamwise boundaries of the mesh elements.

Sort the arrays and plot the transverse boundaries of the mesh elements.

END

10. Subroutine ERR1

Subroutine ERR1 is called by subroutine INPUT if the storage limitation for Real\*8 variables has been exceeded. The subroutine displays the amount by which the limitation was exceeded and terminates the program's execution. The user response would be to increase the value of LIMR if possible or reduce the size of the mesh.

11. Subroutine ERR2

Subroutine ERR2 is called by subroutine INPUT if the storage limitation for Real\*4 variables has been exceeded. The subroutine displays the amount by which the limitation was exceeded and terminates the program's execution. The user response would be to increase the value of LIM4 if possible or reduce the size of the mesh.

12. Subroutine ERR3

Subroutine ERR3 is called by subroutine INPUT if the storage limitation for Integer\*4 variables has been exceeded.

The subroutine displays the amount by which the limitation was exceeded and terminates the program's execution. The user response would be to increase the value of LIMI if possible or reduce the size of the mesh.

## V. DESCRIPTION OF PROGRAM TURBO

Program TURBO solves the quasi-harmonic stream function radial equilibrium equation for flow in an axial compressor. The program uses the information computed by the program MESHGEN to calculate the desired thermodynamic information at all nodal points and displays selected values on a graphics terminal for inspection.

### A. MAIN PROGRAM DESCRIPTION

The program obtains a solution of the equation

$$[K]\{\psi\} = \{f\} \quad (32)$$

An iterative scheme was adopted for this nonlinear problem, whereby an estimate of the stream function distribution is used to calculate values of the velocity components and thermodynamic variables at the nodes of the mesh. These computed values are then substituted into Eq. (32) and a new value of the stream function distribution is calculated. The estimate of the distribution is compared to the calculated value to determine if the solution has reached convergence according to the following criterion:

$$\epsilon > \left| \frac{\psi_i^n - \tilde{\psi}_i^{n+1}}{\tilde{\psi}_i^{n+1}} \right| \quad (47)$$

where  $\psi_i^n$  = estimate of  $\psi$  at node i

$\tilde{\psi}_i^{n+1}$  = solution for  $\psi$  at node i

If the maximum difference for all nodes is less than  $\epsilon$ , the procedure is terminated. If the maximum difference exceeds some specified value of  $\epsilon$ , the new estimate of  $\psi$  to be used for the next iteration is determined using

$$\psi_i^{n+1} = \psi_i^n + \alpha [\tilde{\psi}_i^{n+1} - \psi_i^n] \quad (48)$$

where  $\psi_i^{n+1}$  = new estimate of  $\psi$  for the next iteration

$\alpha$  = under relaxation factor required for convergence because of the strong nonlinear properties of Eq. (32).

The process of constructing the inputs required for Eq. (32) is repeated until convergence is obtained. The details of calculating the inputs and constructing the stiffness matrix and the right-hand side vector are contained in descriptions of the program's subroutines.

The program's algorithm follows in outline form:

Obtain the computational constants (SUBROUTINE RDATA).

Determine the values of the pointers used to partition the storage arrays (SUBROUTINE INIT1).

Set the initial values for all storage locations to 0.0 or 0 as appropriate (SUBROUTINE ZEROI).

Recall from storage the externally computed input values and initialize the inlet conditions (SUBROUTINE INPUT).

Calculate a velocity and thermodynamic variable distribution based on the assumed stream function distribution and the inlet conditions (SUBROUTINE DIST).

From the distributions obtained in DIST, calculate the right-hand side vector {f} (SUBROUTINE FCAL).

Using the density and blockage factor distributions, form the stiffness matrix [K] (SUBROUTINE STIFF).

Solve the system of linear equations to obtain a new stream function distribution (SUBROUTINE DSIMQ).

Place the solution vector in its proper storage location (SUBROUTINE REPLA).

Compare the original stream function distribution to the solution vector to determine the maximum difference in the distributions for all nodes (SUBROUTINE TEST).

Determine if the convergence criterion has been satisfied.

If convergence has not been obtained, perform the relaxation iteration to update the estimate of the stream function distribution (SUBROUTINE RELAX), prepare for another cycle (SUBROUTINE NOCON), and then return to SUBROUTINE DIST for further calculations.

If the convergence criterion has been satisfied, print the results (SUBROUTINE OUTPUT) and display selected information on the graphics terminal (SUBROUTINE MPLOT).

END

## B. SUBROUTINE DESCRIPTIONS

Sections 1 through 23 provide a detailed description of the subroutines of program TURBO.

### 1. Subroutine RDATA

Subroutine RDATA is used to store the following computational constants:

- (a) Logical Input/Output variable NREAD and NWRITE.
- (b) Relaxation factor.
- (c) Limits for the storage arrays.

- (d) Three-point Gaussian abscissas and weighting values.
- (e) Constants used for conversions between different units.

## 2. Subroutine INIT1

Subroutine INIT1 determines the values of the pointers used to partition the Real\*8, Real\*4, and Integer\*4 arrays and determines whether the storage limitations for any of the arrays has been exceeded. If the storage limitations have been exceeded the subroutine will halt execution by calling the appropriate error subroutine. A listing of the pointers and their corresponding array names is contained in Appendix C.

## 3. Subroutine ZEROI

Subroutine ZEROI sets the initial value of all arrays equal to 0.0 or 0 as appropriate.

## 4. Subroutine INPUT

Subroutine INPUT retrieves required input information from its corresponding disk storage location. The information must be placed in storage before running program TURBO. The usual method of generating the information and placing it in storage is through the use of the program MESHGEN.

Subroutine INPUT also initializes the inlet conditions to their proper values, and modifies the nodal blockage factors to account for end-wall boundary layer effects. No attempt was made to include a global method for calculating the blockage factors; rather, a method similar to the one used by Hirsch and Warzee [Ref. 4] was used. The full method

used by Hirsch and Warzee was to artificially reduce the size of the flow passage by reducing the boundaries of the mesh, followed by the application of a general blockage factor to the nodes of the mesh. In the program TURBO, no mesh modifications are made. The procedure followed was to apply a general blockage factor to all nodes, followed by the application of an additional blockage factor to nodes in the outer elements in the rotor, stator, and the passage in between. Though reasonable results were obtained by this method, the handling of the end-wall boundary layers remains the most obvious weakness in the code. This is addressed specifically in section VII.

##### 5. Subroutine DIST

Subroutine DIST calculates the distributions of velocity, density, temperature, pressure, fluid flow angles, entropy, and enthalpy using the known blade and machine geometry, inlet conditions, and the assumed distribution of the stream function. Properties of nodes at the mid-line of the rotor or stator blades were assumed to have a value equal to the average of the inlet and exit conditions of the blade. The elemental calculations are accomplished through the control of subroutines SLINE, DUCT, ROTO, and STAT.

The following is the subroutine DIST's algorithm in outline form:

For each element in the mesh.

Determine type element for appropriate computations.

### Duct Elements

For each node at stations 2 and 3:

Determine the location of the streamline and thermodynamic conditions at station one (Subroutine SLINE).

(If the element is along the machine exit plane, ensure that  $(\partial\psi/\partial z) = 0$ .)

Compute the thermodynamic conditions (Subroutine DUCT).

Assign the appropriate values to the proper storage location.

### Rotor Elements

For each node at stations 1:

Determine the location of the streamline at station 3 and the  $\partial\psi/\partial z$  and the  $\partial\psi/\partial r$  at stations 1 and 3 (Subroutine SLINE).

Determine the inlet and outlet relative flow angles and the outlet absolute flow angle.

Compute the total-to-total pressure ratio and the adiabatic efficiency for the streamline (Subroutine ROTO).

Assign the appropriate values to the proper storage location.

For each node at stations 2:

Determine the location of the streamline and the thermodynamic conditions and the  $\partial\psi/\partial z$  and the  $\partial\psi/\partial r$  at stations 1 (Subroutine SLINE).

Determine the location of the streamline and the  $\partial\psi/\partial z$  and the  $\partial\psi/\partial r$  at station 3 (Subroutine SLINE).

Determine the inlet and outlet relative flow angles and the outlet absolute flow angle and the relative deviation angle.

Compute the thermodynamic conditions at station 3 (Subroutine ROTO).

Compute the value of all properties for the node as being the average of the values at station 1 and station 3.

Assign the appropriate values to the proper storage location.

For each node at stations 3:

Determine the location of the streamline and thermodynamic conditions at station 1 and the value of  $\partial\psi/\partial z$  and  $\partial\psi/\partial r$  at stations 1 and 3 (Subroutine SLINE).

Determine the inlet and outlet relative flow angles and the outlet absolute flow angle and compute the thermodynamic conditions (Subroutine ROTO).

Assign the appropriate values to the proper storage location.

#### Stator Elements

The stator algorithm is the same as the rotor algorithm except the outlet absolute flow angle is the only angle calculated. The inlet absolute flow angle is determined through interpolation.

#### 6. Subroutine SLINE

In order to understand the functioning of this subroutine and others to follow, one must refer to the nomenclature used to describe the eight-node element. Figures 2 and 3 show the nomenclature clearly and Table 1 demonstrates the connectivity. All of the calculations in the program for the distributions of velocity, flow angles, and thermodynamic properties are founded on the assumption that the points in question lie on the same stream surface. Thus the objective of the subroutine is to obtain the location of a given value of the stream function at a specified station in the flow region. The location of the streamline is required in order to compute the variables used in [K] and {f}.

Through the application of the boundary conditions, the nodes along the shroud are defined to lie on one stream surface and the nodes along the hub are on another. It is possible for all other nodes in the mesh to be on different stream surfaces. For these nodes an interpolation scheme must be followed to find the  $(\xi, \eta)$  coordinates of a specified stream surface at a given station in the mesh.

The solution sequence that the program follows starts at the top element of the first column of elements in the mesh and solves the thermodynamic and velocity conditions for all nodes in the element using the assumed stream function distribution and the specified inlet conditions. When the calculations for the first element are complete, the program continues down the column until the calculations are complete for the element along the hub. The program then sequences to the top element for the next column and continues until the calculations are complete for the last element in the mesh. In this sequence it is always possible to calculate the conditions at station 1 of an element for any interim nodal values of the stream function.

The process will be described by way of an example for one node as shown in Fig. 3.

Example: Find the  $(\xi, \eta)$  coordinates of the streamline that passes through node 7 of element 1, Node (1,7).

From the connectivity relationships it is known that

(Node (1,7)) = (14)

The value of (14) is known and the search is begun to find two nodal values of  $\xi$  at station 1 that bracket the desired value, (14). The program first tests to see if (Node (1,7)) is greater than (Node (1,5)). In this case it is not and the program would automatically shift and test to see if (Node (1,7)) is greater than (Node (2,5)). In this example the value is larger and the same test would be applied to (node (2,4)). Again the answer would be true. The program would then test to see if the value of (Node (2,3)) were larger than (Node (1,7)). The answer being true would signal the program that the location of the streamline had been bracketed and a half-interval technique would be applied to find the location. As shown in Fig. 4, the value of  $\xi$  for all locations along station 1 is -1. This fact is important for two reasons. One, with  $\xi$  known the program is only required to iterate on  $n$  to obtain convergence. Two, the Kronecker delta property of the shape functions means that only the shape functions at station 1 have nonzero values [Ref. 16]. For the half interval method, the program uses the average  $n$  of the most recent bracketing as its estimate for  $n$ . In this example the first estimate of  $n$  is equal to 0.5 and the solution for  $\psi$  at (-1., 0.5) can be written as

$$\psi(-1., 0.5) = N_3(-1., 0.5)\psi_3 + N_4(-1., 0.5)\psi_4 + N_5(-1., 0.5)\psi_5$$

The solution is then compared to  $\psi$ (Node (1,7)) to determine if the difference is less than some  $\epsilon$ . If the difference exceeds  $\epsilon$ , the new estimate for  $\eta$  becomes 0.25 or 0.75 depending on whether the solution is larger than or less than the value of  $\psi$ (Node (1,7)). The process is continued until convergence is reached. Once  $\xi$  and  $\eta$  for the streamline location at station 1 are known, all of the properties for station 1 can be determined. (The same method is used to find the location of the streamline at station 3 for rotor and stator elements.) Having determined the coordinates of streamlines at all desired locations it is possible to calculate the required  $(\partial N_i / \partial r)$  and  $(\partial N_i / \partial z)$ . The computed inlet and exit coordinates and conditions are then passed to subroutine DIST for use in subroutines DUCT, ROTO, and STAT as appropriate for the calculation of the conditions at Node (1,7).

The following is the subroutine's algorithm in outline form:

Duct Element

If the node being investigated is on station one, exit the subroutine.

For stations two and three, determine the streamline coordinates and the thermodynamic conditions at station one and compute the  $\partial \psi / \partial z$  and  $\partial \psi / \partial r$  at the node. Exit the subroutine.

Rotor/Stator Element

If the node being investigated is on station one, set all inlet thermodynamic variables equal to the corresponding

nodal value and compute the  $\partial\psi/\partial z$  and the  $\partial\psi/\partial r$  at station three. Exit the subroutine.

For station two, determine the streamline coordinates and the thermodynamic conditions at station one and compute the streamline location and the  $\partial\psi/\partial z$  and the  $\partial\psi/\partial r$  at station three. Exit the subroutine.

For station three, determine the streamline coordinates and the thermodynamic conditions at station one and compute the  $\partial\psi/\partial z$  and the  $\partial\psi/\partial r$  at station three. Exit the subroutine.

#### 7. Subroutine DUCT

Subroutine DUCT determines the values of temperature, pressure, and density for the elemental nodes at stations two and three. An iterative procedure is used with the knowledge that angular momentum is a constant in a duct. The initial estimate of the velocity at station 1 is made using the computed values of  $\partial\psi/\partial z$ ,  $\partial\psi/\partial r$ ,  $r$ , and  $b$  at the node (Subroutine SLINE) and by choosing the estimate of the density at the node to be equal to the density at station 1. The following sequence of calculations is repeated until convergence on a value of the exit velocity:

$$v_{m2} = (1/(\rho_2 r_2 b_2)) * \left[ (\partial\psi/\partial z_2)^2 + (\partial\psi/\partial r_2)^2 \right]^{0.5}$$

$$\alpha_2 = \tan^{-1} \left[ (r_1 v_{m1} \tan \alpha_1) / (r_2 v_{m2}) \right]$$

$$T_2 = T_{T1} - (\gamma - 1)/2 * \left( v_{m2}^2 (1 + \tan^2 \alpha_2) \right) / (\gamma R G_C)$$

$$P_2 = P_{T1} (T_2/T_{T1})^{**(\gamma/\gamma-1)}$$

$$\rho_2 = P_2 / (R T_2)$$

$$v_{m2n} = (1/(\rho_2 r_2 b_2)) * \left[ (\partial\psi/\partial z_2)^2 + (\partial\psi/\partial r_2)^2 \right]^{0.5}$$

$$\text{Test if } |v_{m2n} - v_{m2}| < \epsilon$$

The total conditions are then calculated from the static conditions and the computed velocity.

#### 8. Subroutine ROTO

Subroutine ROTO calculates the change in the relative flow angles, the velocity, and the thermodynamic properties along a streamline across a (compressor) rotor element. The program uses the conditions at station one and the location of the streamline and the partial derivatives of the stream function at station 3, all of which were calculated in subroutine SLINE. The relationships in subroutine ROTO are derived from cascade correlations and known property relationships for a streamline in a rotor.

The first step is the calculation of the inlet and exit relative flow angles. For inlet flow without swirl, the relative inlet flow angle can be calculated using

$$\beta_1 = \tan^{-1}(U_1/v_{m1})$$

The incidence angle is calculated using the known blade geometry and inlet angle, since

$$i = \beta_1 - \kappa_1$$

where  $\kappa_1$  is the angle formed between the tangent to the blade chordline and the axial direction. In order to calculate the exit relative flow angle one must determine the deviation angle,  $\delta$ . The program uses the correlations and equations

derived by NASA [Ref. 21]. The specific sequence of equations, using the notation of Ref. 21, is as follows:

$$i_C - i_{2D} = f(M, r)$$

$$i_{2D} = (K_i)_t (K_i)_{SH} (i_0)_{10} + n\phi$$

$$i_{ref} = i_{2D} + (i_C - i_{2D})$$

$$\delta_{2D} = (K_\delta)_t (K_\delta)_{SH} (\delta_0)_{10} + (m/\sigma^b) + (i_C - i_{2D}) (d\delta/di)_{2D}$$

$$\delta_C - \delta_{2D} = f(M_R, r)$$

$$\delta_{ref} = \delta_{2D} + (\delta_C - \delta_{2D})$$

$$\delta = \delta_{ref} + (i - i_{ref}) (d\delta/di)_{2D}$$

$$\beta_2 = \beta_1 - \phi - i + \delta$$

The expressions used to approximate the NASA correlation curves were those obtained by Crouse of NASA Lewis and were provided to the author by Okiishi [Ref. 22].

When the inlet conditions and relative flow angles are known it is possible to determine the conditions at the rotor element exit. The initial exit velocity is obtained in the same way as in subroutine DUCT. The following sequence of equations is solved iteratively until convergence for the exit velocity is reached:

$$\alpha_2 = \tan^{-1} [(\omega r_2 - v_{m2} \tan \beta_2)/v_{m2}]$$

$$D = 1 - (w_2/w_1) + (r_1 w_{u1} - r_2 w_{u2})/(2 w_1 \bar{r})$$

$$\tilde{\omega} = \text{curve fit to } \tilde{\omega}(D, \cos \beta, \sigma)$$

$$T_{E1} = T_{R1} + \omega^2 (r_2^2 - r_1^2) / \text{constant}$$

$$P_{E1} = P_{T1} (T_{E1}/T_{T1}) * (\gamma/\gamma-1)$$

$$P_{R1} = P_{T1} (T_{R1}/T_{T1}) * (\gamma/\gamma-1)$$

$$P_{E2} = P_{E1} - \tilde{\omega} (P_{R1} - P_1)$$

$$w_2 = v_{m2} / \cos \beta_2$$

$$T_2 = T_{E1} - w_2^2 / \text{constant}$$

$$P_2 = P_{E2} (T_2/T_{E1}) * (\gamma/\gamma-1)$$

$$\rho_2 = P_2 / (RT_2)$$

$$v_{\min} = (1/(\rho_2 r_2 b_2)) * \left[ (\partial \psi / \partial z)_2^2 + (\partial \psi / \partial r)_2^2 \right]^{0.5}$$

$$\text{Test if } |v_{\min} - v_{m2}| < \epsilon$$

When convergence is reached the total conditions are calculated from the static conditions and the computed velocity. The value of the entropy change is calculated by:

$$S = R \ln(P_{T2}/P_{T1}) * \text{constant}$$

#### 9. Subroutine STAT

Subroutine STAT calculates the stator element exit absolute flow angle and thermodynamic conditions using the knowledge that the total enthalpy is a constant across the stator. The initial estimate of the exit velocity is obtained in the same way as in subroutine DUCT. The following sequence of equations is used until convergence for the exit velocity is reached:

$$v_1 = \left[ v_{m1}^2 (1 + \tan^2 \alpha_1) \right]^{0.5}$$

$$D = 1 - (v_2/v_1) + (r_1 v_{u1} - r_2 v_{u2}) / (2 \sigma V \bar{r})$$

$\tilde{\omega}$  = curve fit to  $\omega(D, \cos \beta, \sigma)$

$$P_{T2} = P_{T1} - \tilde{\omega}(P_{T1} - P_1)$$

$$T_2 = T_{T1} - (\gamma - 1)/2 * (v_{m2}^2 (1 + \tan^2 \alpha_2)) / (\gamma R G)$$

$$P_2 = P_{T2} (T_2/T_{T2})^{**(\gamma/\gamma-1)}$$

$$\rho_2 = P_2 / (R T_2)$$

$$v_{min} = (1/(\rho_2 r_2 b_2)) * \left[ (\partial \psi / \partial z)_2^2 + (\partial \psi / \partial r)_2^2 \right]^{0.5}$$

$$\text{Test if } |v_{min} - v_{m2}| < \epsilon$$

When convergence is reached the total conditions are calculated from the static conditions and the computed velocity. The value of the entropy change is calculated by the same method used by subroutine ROTO.

#### 10. Subroutine FCAL

Subroutine FCAL uses the previously computed distributions of total temperature, entropy, enthalpy, axial velocity, and tangential velocity to compute the right-hand side vector for the global system of equations. In the absolute frame of reference,  $f(r, z)$  can be expressed in the form

$$f(r, z) = (1/V_z) T(\partial s / \partial r) - \partial H / \partial r + (V_u / r) (\partial (r V_u) / \partial r) \quad (20)$$

The value of  $f(r, z)$  within an element is determined using the following relationships:

$$\begin{aligned}
 T(z, r) &= \sum_i^n N_i(\xi, \eta) T_i & H(z, r) &= \sum_i^n N_i(\xi, \eta) H_i \\
 s(z, r) &= \sum_i^n N_i(\xi, \eta) s_i & V(z, r) &= \sum_i^n N_i(\xi, \eta) V_i \\
 v(z, r) &= \sum_i^n N_i(\xi, \eta) v_i & r &= \sum_i^n N_i(\xi, \eta) r_i
 \end{aligned}$$

where the value of  $N_i(\xi, \eta)$  is determined by the value of  $\xi$  and  $\eta$  for a specified Gaussian integration point within the element. The required partial derivatives are found in a simple and direct way. To illustrate, the  $(\partial H / \partial r)$  is derived as follows:

$$\begin{aligned}
 H(z, r) &= \sum_i^n N_i H_i \\
 (\partial H / \partial r) &= \sum_i^n (\partial N_i / \partial r) H_i + \sum_i^n N_i (\partial H_i / \partial r)
 \end{aligned}$$

and since  $H_i$  is a constant then

$$\partial H / \partial r = \sum_i^n (\partial N_i / \partial r) H_i$$

where  $(\partial N_i / \partial r)$  is found by Eq. (37), and  $H_i$  is the appropriate nodal value of  $H$ . The radial variations in entropy and angular momentum are found in the same manner. The same procedure is followed for the values of rothalpy and relative tangential velocity for rotor elements.

It is now possible to calculate the quantities in the expression for the right-hand side vector at a point. All that remains is to apply an integration technique to

obtain the value over an element and to assemble the resulting local contributions into the global equations. As shown in section III.D.2, the local contribution for node  $i$  in an element can be expressed as:

$$f_i^e = \sum_{jk} w_j w_k \sum_i N_i \sum_\ell N_\ell f_\ell |J| \quad (47)$$

In the program Eq. (47) is modified to

$$f_i^e = \sum_m^9 w_m \sum_i^n N_i \sum_k^n N_k f_k |J|$$

where the Gaussian abscissas and the corresponding product of the weight functions are grouped into three one-dimensional arrays. At the completion of the summing process, the local contribution for  $f$  has been calculated for each node in the element. The global system is then updated by adding the local contributions to the global values through the use of the connectivity relationships.

The following is the subroutine's algorithm in outline form:

#### Algorithm:

Iterate for each element in the mesh.

Iterate for each Gaussian point.

Find shape functions,  $|J|$ , and  $[J]^{-1}$ .

Find  $V_z$ ,  $T_T$ ,  $V_u$ ,  $r$ ,  $rV_u$ ,  $(\partial s/\partial r)$ , and  $(\partial H/\partial r)$ .

Compute the contributions of the value  $f$  at the Gauss point to the value of  $f$  at each node of the element.

Upon completion of the Gaussian integration, add the local contribution to the global system.

END

The same algorithm is followed for rotor elements with the appropriate substitutions of  $H_R$  and  $W_u$ .

### 11. Subroutine STIFF

Subroutine STIFF uses the computed distributions of density and blockage factors to form the stiffness matrix for the global system of equations. It was shown earlier that the contribution to the elemental stiffness is expressed as

$$k_{ij}^e = \int_E k(r, z) [(\partial N_j / \partial r)(\partial N_i / \partial r) + (\partial N_j / \partial z)(\partial N_i / \partial z)] d\Omega \quad (31a)$$

where  $k(r, z) = (1/\rho rb)$  (19)

Again, the elemental properties are considered to have a polynomial variation of the form

$$\rho(z, r) = \sum_i^n N_i \rho_i, \quad r_i = \sum_i^n N_i r_i, \quad \text{and } b(z, r) = \sum_i^n N_i b_i$$

As shown earlier, Eq. (31a) can be converted by the Gauss-Legendre method to

$$k_{ij}^e = \sum_{kl} \sum_m w_k w_l \sum_m N_m k_m ([B]^T [B]) |J| \quad (46)$$

The value of  $k_m$  is determined from the definitions of  $\rho$ ,  $b$ , and  $r$  by the same methods used in subroutine FCAL. The evaluation of  $[B]^T [B]$  is a simple matter to perform. The

matrix  $[B]$  is simply the column vector  $\{(\partial N_i / \partial z), (\partial N_i / \partial r)\}$ . Therefore,  $[B]^T [B]$  can be written  $[(\partial N_i / \partial r)(\partial N_j / \partial r) + (\partial N_i / \partial z)(\partial N_j / \partial z)]$ . The value of  $(\partial N_i / \partial z)$  and  $(\partial N_i / \partial r)$  is found for all nodes in the element in one step through the use of subroutine JACOB. The matrix product can be evaluated at a point  $(z, r)$  as

$$\sum_{i=1}^n \sum_{j=1}^n [(\partial N_i / \partial r)(\partial N_j / \partial r) + (\partial N_i / \partial z)(\partial N_j / \partial z)]$$

By using the same Gaussian weighting scheme used in subroutine FCAL, Eq. (46) may be written in the form

$$k_{ij}^e = \sum_{k=1}^9 w_k \sum_{l=1}^n N_l k_l \sum_{ij}^{nn} [(\partial N_i / \partial r)(\partial N_j / \partial r) + (\partial N_i / \partial z)(\partial N_j / \partial z)] |J|$$

The resulting 8x8 elemental matrix is then added to the global stiffness matrix through the connectivity relationships.

Up to this point  $[K]$  and  $\{f\}$  have been assembled without regard to the boundary conditions except at the exit plane where the  $(\partial \psi / \partial n) = 0$  was enforced explicitly during the procedures used by Subroutine DIST. Care must be taken to ensure that the boundary conditions for the other three segments of the boundary are not violated. As shown in section III.B, the boundary condition for the nodes along the shroud, along the hub and at the inlet plane of the machine is that the value of  $\psi$  is specified. If the value of  $\psi$  is specified at these locations the Eq. (32) must be modified so that  $\psi$  is no longer free at these nodes. A standard technique is

employed to remove individual equations from a system of equations when the degree of freedom represented by the individual equations has been removed.

The following is the subroutine's algorithm in outline form:

Algorithm:

Iterate for each element in the mesh.

Iterate for each Gaussian point.

Find shape functions,  $|J|$ , and  $[J]^{-1}$ .

Find the value of k at the Gauss point.

Compute the elemental stiffness matrix.

Upon completion of the Gaussian integration, add the local contribution to the global system.

Upon completion of the addition of the last element's contribution, modify the system of equations to include the boundary conditions.

END

12. Subroutine DSIMQ

Subroutine DSIMQ is a non-IMSL library, double precision subroutine that solves a set of n simultaneous equations of the form

$$[A]\{X\} = \{B\}$$

where  $[A]$  is an  $n \times n$  matrix

$\{X\}$  and  $\{B\}$  are  $n \times 1$  vectors.

13. Subroutine REPLA

Subroutine REPLA places the solution vector obtained from subroutine DSIMQ into its proper storage location.

#### 14. Subroutine TEST

Subroutine TEST determines the maximum difference in the assumed nodal distribution of the stream function at the beginning of an iteration to the solution of the radial equilibrium equation calculated using the assumed distribution. The difference in the distributions at a node is defined as

$$\text{Diff} = \left| \left[ (\psi_i^n - \tilde{\psi}_i^{n+1}) / \tilde{\psi}_i^{n+1} \right] \right|$$

where  $\psi_i^n$  is the assumed value at node i and  $\tilde{\psi}_i^{n+1}$  is the calculated value for node i. Convergence is considered to be reached when the maximum difference for any node is less than a selected reference value,  $\epsilon$ .

#### 15. Subroutine RELAX

Subroutine RELAX performs the relaxation scheme to obtain an updated estimate of the stream function distribution for the next program iteration. The new estimate for stream function distribution is calculated as follows:

$$\psi_i^{n+1} = \psi_i^n + \alpha [\tilde{\psi}_i^{n+1} - \psi_i^n] \quad (48)$$

#### 16. Subroutine NOCON

Subroutine NOCON prepares the program for the next iteration by setting the elements of the right-hand side vector,  $\{f\}$ , and the stiffness matrix,  $[K]$ , equal to zero.

## **17. Subroutine OUTPUT**

Subroutine OUTPUT prints the computed nodal values of a majority of the velocities and thermodynamic properties. A listing of the values that are printed and the corresponding units is contained in Appendix D. A sample output listing is contained in Appendix G.

## **18. Subroutine MPLOT**

Subroutine MPLOT uses the Tektronix 618 terminal to make an online graphical presentation of selected variables at the rotor inlet, rotor outlet, stator inlet and the stator outlet. Figures 7 through 20 provide examples of the plots available for display to the individual on request. The user is given the option of terminating the plotting sequence at any stage of the presentation through the use of interactive prompts.

The following is the subroutine's algorithm in outline form:

### **Algorithm:**

Convert the appropriate variables to Real 4 for compatibility with the library plotting package GRAFF.

Determine the values of axial velocity, relative flow angles, total-to-total pressure ratio, and the adiabatic efficiency for the rotor inlet, display if requested.

Determine the values of axial velocity, and relative, absolute, and deviation angles for the rotor exit, display if requested.

Determine the values of axial velocity, absolute flow angles, and total-to-total pressure ratio for the stator inlet, display if requested.

Determine the values of axial velocity, and absolute and deviation angles for the stator exit, display if requested.

END

#### 19. Subroutine SHAPE

Subroutine SHAPE calculates the eight nodal shape functions for a given point  $(\xi, \eta)$ . The equations for the nodal shape functions are:

$$N(1) = (\xi\eta + \xi^2 + \eta^2 + \xi^2\eta + \xi\eta^2 - 1)/4$$

$$N(2) = (1 + \eta - \xi^2 - \xi^2\eta)/2$$

$$N(3) = (-\xi\eta + \xi^2 + \eta^2 + \xi^2\eta - \xi\eta^2 - 1)/4$$

$$N(4) = (1 - \eta^2 - \xi + \xi\eta^2)/2$$

$$N(5) = (\xi\eta + \xi^2 + \eta^2 - \xi^2\eta - \xi\eta^2 - 1)/4$$

$$N(6) = (1 - \eta - \xi^2 + \xi\eta^2)/2$$

$$N(7) = (-\xi\eta + \xi^2 + \eta^2 - \xi^2\eta + \xi\eta^2 - 1)/4$$

$$N(8) = (1 - \eta^2 + \xi - \xi\eta^2 - 1)/2$$

The values of the shape functions are stored in the array SF, and are returned to the calling portion of the program.

#### 20. Subroutine JACOB

Subroutine JACOB computes the partial derivatives of the shape functions with respect to  $\xi$  and  $\eta$  and computes the elements of the Jacobian matrix, [J], for a specific point  $(\xi, \eta)$ . The equations for the partial derivatives were obtained directly from the differentiation of the functions shown in the description of subroutine SHAPE. The arrays D

and E store the values of the  $(\partial N / \partial \xi)$  and  $(\partial N / \partial \eta)$  respectively. The Jacobian matrix is calculated by the following sequence of equations:

$$J(1,1) = (\partial z / \partial \xi) = \sum_i^n (\partial N_i / \partial \xi) z_i$$

$$J(1,2) = (\partial r / \partial \xi) = \sum_i^n (\partial N_i / \partial \xi) r_i$$

$$J(2,1) = (\partial z / \partial \eta) = \sum_i^n (\partial N_i / \partial \eta) z_i$$

$$J(2,2) = (\partial r / \partial \eta) = \sum_i^n (\partial N_i / \partial \eta) r_i$$

Arrays D and E and the Jacobian matrix are returned to the calling location in the program.

#### 21. Subroutine ERR1

Subroutine ERR1 is called by subroutine INPUT if the storage limitation for Real 8 variables has been exceeded. The subroutine displays the amount by which the limitation was exceeded and terminates the program's execution. The user response would be to increase the value of LIMR if possible or reduce the size of the mesh.

#### 22. Subroutine ERR2

Subroutine ERR2 is called by subroutine INPUT if the storage limitation for Real 4 variables has been exceeded. The subroutine displays the amount by which the limitation was exceeded and terminates the program's execution. The

user response would be to increase the value of LIM4 if possible or reduce the size of the mesh.

23. Subroutine ERR3

Subroutine ERR3 is called by subroutine INPUT if the storage limitation for Integer 4 variables has been exceeded. The subroutine displays the amount by which the limitation was exceeded and terminates the program's execution. The user response would be to increase the value of LIM1 if possible or reduce the size of the mesh.

## VI. RESULTS AND DISCUSSION

### A. PROGRAM VERIFICATION

Four operating conditions of the NASA TASK-1 compressor were used to test the capabilities of the programs MESHGEN and TURBO. In all cases the 63 element, 222 node mesh with an under-relaxation factor of 0.24 as recommended by Hirsch and Warzee [Ref. 4] were used. Selected portions of the results obtained are presented in Figs. 7 through 76. The points annotated as "observed values" were obtained from the material published in Refs. 23 and 24. The values attributed to Gavito were obtained from Ref. 8 and those attributed to Hirsch from Ref. 4. A discussion of the predictions for the various conditions are presented in the sections that follow.

#### 1. Test Case 1

For test case 1 the operating point was defined as a rotor speed of 50% design speed and an inlet mass flow rate of 107.6 lbm/sec. The author was unable to locate this specific operating point in Ref. 23 or 24 and must assume that in Ref. 4 the mass flow rate was modified to conform to the end-wall boundary layer scheme described in that reference. Therefore, it was necessary to use the observed values published by Hirsch and Warzee in Figs. 21 through 30. The relative differences found between the present predictions and the reported observations are given in Table 1.

a. Comparison to the Work of Gavito

Figures 21, 22, 23 and 24 compare the results obtained by Gavito with the predictions of the present program. A significant improvement has been obtained at all locations. That this was possible was due in large measure to the solid foundation to the present work provided by Gavito's program and to the excellent documentation given in Ref. 8.

b. Comparison to the Work of Hirsch and Warzee

The predictions of the program TURBO compare quite favorably with those of Hirsch and Warzee. As shown in Figs. 25, 26, 29 and 30 the predictions of the two programs have almost identical average relative errors for the velocity profiles. The predictions of Hirsch and Warzee tend to have better agreement in the rotor and stator tip regions, while the program TURBO has slightly better agreement near the hub. The program TURBO's predictions had a 3.7% and 2.5% average error at the rotor inlet and exit respectively with a maximum error of 4.6% at the inlet and 7.0% at the outlet. The stator inlet velocity predictions had an average relative error of 2.6% and maximum error of 5.8% and the outlet predictions had a 2.0% average error with a maximum error of 6.0%. The prediction of both programs for the velocity profiles show excellent agreement with the observed values.

Hirsch and Warzee's program consistently produced flow angle predictions with closer agreement to the observed

values for the published rotor outlet angles. Though the two programs had the same average errors of  $3.3^\circ$  for the relative flow angles and  $4.8^\circ$  for the absolute flow angles, Hirsch and Warzee's program provided better qualitative distributions. The difference is clearly shown in Figs. 27 and 28.

Hirsch and Warzee did not publish predictions of total pressure ratios or adiabatic efficiencies, so the predictions made by TURBO for these parameters were not presented. It is assumed that no significant differences could have occurred because of the similarity of the results for the velocity profile and flow angles discussed earlier.

### 2. Test Case 2

Because of the apparent modification in the mass flow rate which was assumed in test case 1 and the lack of comparative data for all quantities predicted by the program TURBO, another operating point at 50% design speed was compared. The operating condition for case 2 was defined as a mass flow rate of 114.7 lbm/sec at a speed equal to 50% of design, which corresponded to reading 38 of Ref. 24. The results for case 2 are presented in Figs. 31 through 44, and a summary of the relative differences between the predictions and observations is contained in Table 2.

### 3. Test Case 3

Test case 3 corresponded to reading 45 of Ref. 24, which was defined as a speed equal to 70% design and a mass flow rate of 151.55 lbm/sec. When reviewing the results

presented for this case, one should note the decline in the agreement between the program's predictions and the observed values. It is the author's opinion that the degradation is primarily caused by two factors. The first is the formulation used to compute the meridional velocity change across the rotor and the other is the application of a single blockage factor to all nodes to account for the end-wall boundary layers. Both factors are much more significant at 70% design speed than they were at 50%. At 70% design speed the rotor tip relative Mach number is about 0.94. This would require the program to account for transonic effects at the tip. Second, the mass flow and absolute Mach number of the flow in all regions is significantly higher at 70% design speed. Therefore, it is unlikely that a single blockage factor will work satisfactorily in all regions of the machine. It is hoped that both areas will be addressed in any future work on the program.

The results for test case 3 are presented in Figs. 45 through 58 with the corresponding differences between the predictions and observations summarized in Table 3.

#### 4. Test Case 4

The operating condition of test case 4 was at 80% design speed point with a mass flow rate of 174.54 lbm/sec, corresponding to reading 50 of Ref. 24. Figures 59 through 72 and Table 4 present the results of the program's predictions and comparisons to the observed values. As expected,

and for similar reasons to those cited in test case 3, the agreement between the program's predictions and observed values is significantly poorer than any of the three previous cases.

#### B. POINTS OF INTEREST

The results produced by the program are highly dependent on the value of the general blockage factor used to account for the end-wall boundary layers. This factor influences both the quality, in terms of agreement with observations, and the stability of the solution. Figures 73 through 76 show the axial velocity distributions for the rotor and stator for test case 4 with a blockage factor of 9% instead of the 6% factor used to obtain the results shown in Figs. 59 through 72. A comparison of corresponding velocity profiles clearly demonstrates the factor's pronounced influence on the program's solution. The general blockage factor also has a strong influence on the program's convergence rate. In some cases the factor can cause the program to become oscillatory or even divergent.

For the low speed cases of 50% and 70% design, additional blockage factors had to be applied to the duct element between the rotor and stator tips to obtain accurate results. As shown in the program listing for subroutine INPUT of program TURBO, the factors used for 50% design speed were much higher than the factors used for 70% design speed and that

no additional factors were used for 80% design speed. A global method of calculating the end-wall blockage factor must be incorporated if the program is to become independent of inputs other than physical constants.

The deterioration of the program's predictions with increasing Mach number and its failure to run for test cases with strong supersonic relative velocities at the tip demonstrate the need to provide the program with a method of handling supersonic relative velocities. Hirsch and Warzee [Ref. 15] showed a method of extending the radial equilibrium formulation used in the present code to supersonic flow. They presented comparisons of predictions obtained by this method to observations of the NASA TASK-1 transonic compressor at 100% design speed. The results were impressive and clearly showed that the method is valid for relative velocities in excess of Mach 1.4. It is strongly recommended that the first effort at improving the code be an effort to modify TURBO to include the method shown in Ref. 15.

## VII. CONCLUSIONS AND RECOMMENDATIONS

A computer program based on the finite element technique has been developed and has been verified satisfactorily for computing flows through subsonic axial flow compressor stages. Minor modifications have been suggested to allow transonic stages to be calculated.

The code was written in such a way that it could be readily adapted to compute either turbines or compressors with multiple stages. Before such extensions are attempted however, the following specific recommendations are made to improve the present compressor code.

### A. PROGRAM MESHGEN

1. Incorporate some of the two-dimensional techniques of Adamek [Ref. 25] to improve the efficiency of the code.
2. Review the code to find improvements in storage allocations and computational efficiencies.
3. Modify the program to track the first and last nodes of the rotor and stator as subscripted variables so that the program can be used to generate the appropriate mesh parameters for a multi-stage machine.
4. Convert subroutine MPLOT to the DISPLA system.

### B. PROGRAM TURBO

1. Incorporate a method for the global calculation of the blockage and losses created by the end-wall boundary layers.

2. Convert the storage of [K] and the solution technique for the program to at least a symmetric banded scheme or if possible to a skyline equivalent scheme.
3. Test the program on a variety of machines and operating conditions.
4. Obtain expressions that approximate the NASA correlation curves for 65-series blading.
5. Incorporate methods for the prediction of stall/surge.
6. Take advantage of the modular form of the program and include a variety of correlation techniques as a user selected option.
7. Review the program for improved storage and computational techniques. Specifically, determine ways to take fuller advantage of the dynamic dimensioning scheme [Ref. 16] used by the program.
8. Convert the rotor inlet calculations to allow the value of  $\beta_1$  to have nonzero values so that the program can be extended to multi-stage analysis.
9. Modify the use of the values of the beginning nodes of the rotor and stator to subscripted variables so that the analysis can be extended to multi-stage machines.
10. Convert subroutine MPLOT to the DISPLA system.
11. Develop iterative schemes for calculating the flow angle, the velocity distribution changes and the thermodynamic property changes across a turbine rotor and stator for inclusion in subroutines ROTO and STAT.

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TABLE 1  
Connectivity Relationships for Figure Three

| <u>Element Number</u> | <u>Local Node Number</u> | <u>Global Number</u> |
|-----------------------|--------------------------|----------------------|
| 1                     | 1                        | 12                   |
| 1                     | 2                        | 8                    |
| 1                     | 3                        | 1                    |
| 1                     | 4                        | 2                    |
| 1                     | 5                        | 3                    |
| 1                     | 6                        | 9                    |
| 1                     | 7                        | 14                   |
| 1                     | 8                        | 13                   |
| 2                     | 1                        | 14                   |
| 2                     | 2                        | 9                    |
| 2                     | 3                        | 3                    |
| 2                     | 4                        | 4                    |
| 2                     | 5                        | 5                    |
| 2                     | 6                        | 10                   |
| 2                     | 7                        | 16                   |
| 2                     | 8                        | 15                   |
| 3                     | 1                        | 16                   |
| 3                     | 2                        | 10                   |
| 3                     | 3                        | 5                    |
| 3                     | 4                        | 6                    |
| 3                     | 5                        | 7                    |
| 3                     | 6                        | 11                   |
| 3                     | 7                        | 18                   |
| 3                     | 8                        | 17                   |

TABLE 2

Comparison of Program Predictions with NASA Task-1  
Compressor Measurements at 50% Design Speed

|                      | <u>Average Difference</u> | <u>Maximum Difference</u> |
|----------------------|---------------------------|---------------------------|
| <b>Rotor Inlet</b>   |                           |                           |
| Axial Velocity       | 4.6%                      | 7.2%                      |
| Relative Angles      | 1.5°                      | 3.1°                      |
| Total Pressure Ratio | 0.7%                      | 1.4%                      |
| Efficiencies         | 4.3%                      | 11.4%                     |
| <b>Rotor Outlet</b>  |                           |                           |
| Axial Velocity       | 3.2%                      | 6.7%                      |
| Relative Angles      | 1.6°                      | 2.8°                      |
| Absolute Angles      | 1.5°                      | 4.2°                      |
| Deviation Angles     | 2.2°                      | 3.0°                      |
| <b>Stator Inlet</b>  |                           |                           |
| Axial Velocity       | 3.4%                      | 6.8%                      |
| Absolute Angles      | 2.5°                      | 4.2°                      |
| Total Pressure Ratio | 0.3%                      | 1.2%                      |
| <b>Stator Outlet</b> |                           |                           |
| Axial Velocity       | 1.7%                      | 5.7%                      |
| Absolute Angles      | 0.6°                      | 1.5°                      |
| Deviation Angles     | 1.4°                      | 2.6°                      |

TABLE 3

Comparison of Program Predictions with NASA Task-1  
Compressor Measurements at 70% Design Speed

|                      | Average Difference | Maximum Difference |
|----------------------|--------------------|--------------------|
| <b>Rotor Inlet</b>   |                    |                    |
| Axial Velocity       | 4.8%               | 7.4%               |
| Relative Angles      | 1.14°              | 1.8°               |
| Total Pressure Ratio | 1.1%               | 2.0%               |
| Efficiencies         | 3.2%               | 4.8%               |
| <b>Rotor Outlet</b>  |                    |                    |
| Axial Velocity       | 4.1%               | 7.4%               |
| Relative Angles      | 2.0°               | 3.2°               |
| Absolute Angles      | 3.6°               | 5.5°               |
| Deviation Angles     | 1.6°               | 3.5°               |
| <b>Stator Inlet</b>  |                    |                    |
| Axial Velocity       | 4.0%               | 8.8%               |
| Absolute Angles      | 3.9°               | 5.5°               |
| Total Pressure Ratio | 0.5%               | 2.1%               |
| <b>Stator Outlet</b> |                    |                    |
| Axial Velocity       | 3.6%               | 7.4%               |
| Absolute Angles      | 0.6°               | 1.1°               |
| Deviation Angles     | 1.5°               | 2.4°               |

TABLE 4

Comparison of Program Predictions with NASA Task-1  
Compressor Measurements at 80% Design Speed

|                      | Average<br>Difference | Maximum<br>Difference |
|----------------------|-----------------------|-----------------------|
| <b>Rotor Inlet</b>   |                       |                       |
| Axial Velocity       | 7.9%                  | 11.4%                 |
| Relative Angles      | 1.3°                  | 1.5°                  |
| Total Pressure Ratio | 1.5%                  | 2.5%                  |
| Efficiencies         | 3.1%                  | 7.9%                  |
| <b>Rotor Outlet</b>  |                       |                       |
| Axial Velocity       | 6.6%                  | 8.9%                  |
| Relative Angles      | 2.9°                  | 5.5°                  |
| Absolute Angles      | 6.8°                  | 8.5°                  |
| Deviation Angles     | 2.4°                  | 4.7°                  |
| <b>Stator Inlet</b>  |                       |                       |
| Axial Velocity       | 6.9%                  | 13.9%                 |
| Absolute Angles      | 5.6°                  | 7.0°                  |
| Total Pressure Ratio | 0.9%                  | 3.1%                  |
| <b>Stator Outlet</b> |                       |                       |
| Axial Velocity       | 8.0%                  | 20.0%                 |
| Absolute Angles      | 0.8°                  | 1.4°                  |
| Deviation Angles     | 1.5°                  | 2.3°                  |

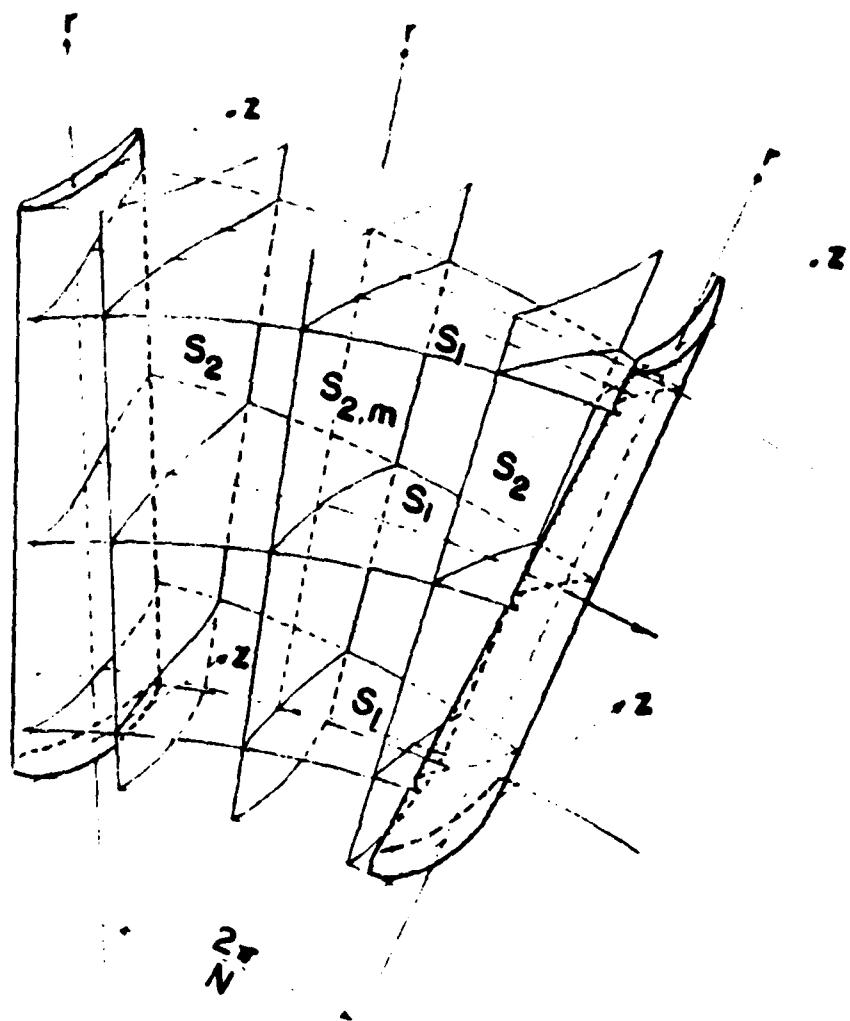


Figure 1. S1 and S2 Stream Surfaces

Figure 2. Nomenclature of an Eight-Node Element

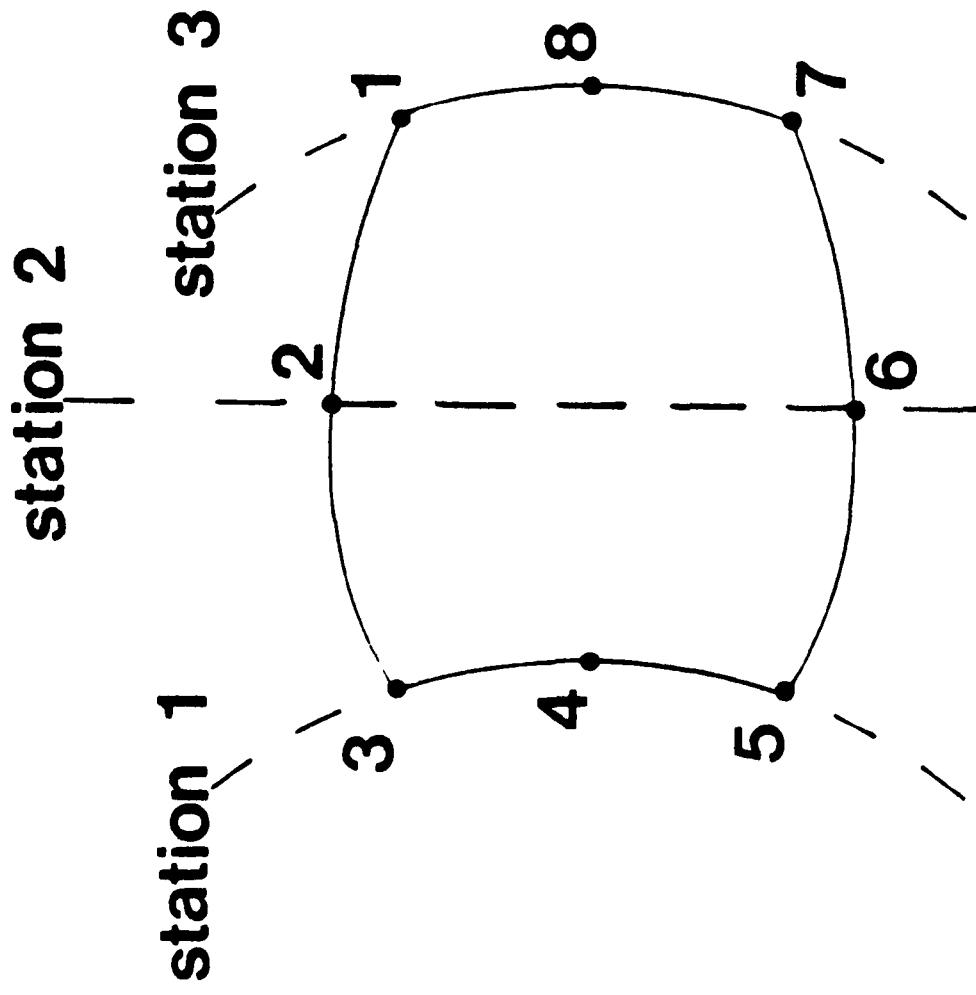
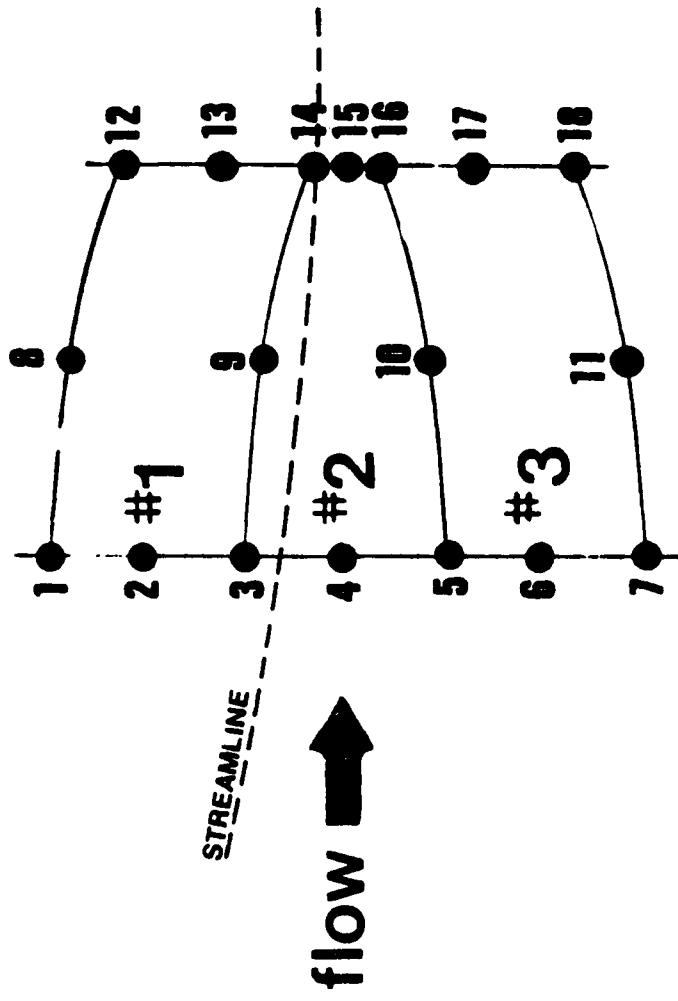


Figure 3. Example of a Three Element Mesh



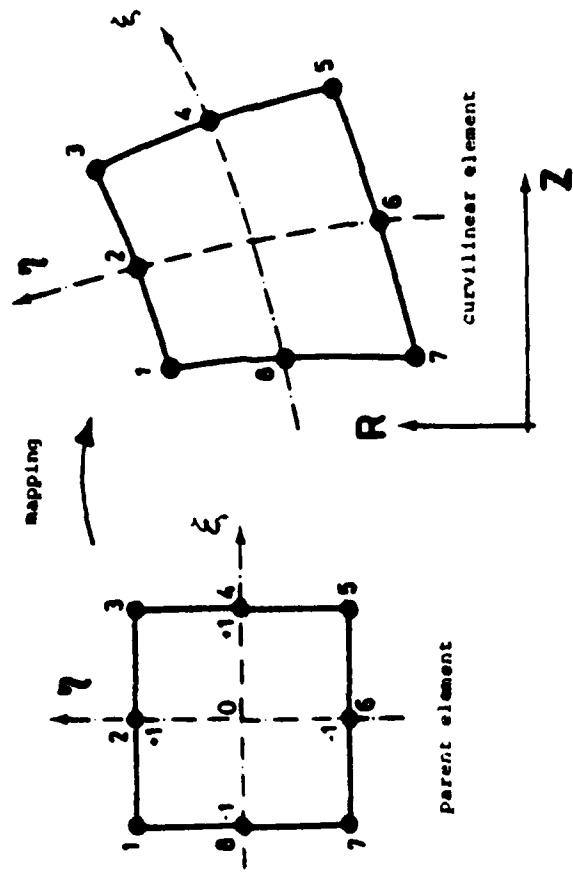
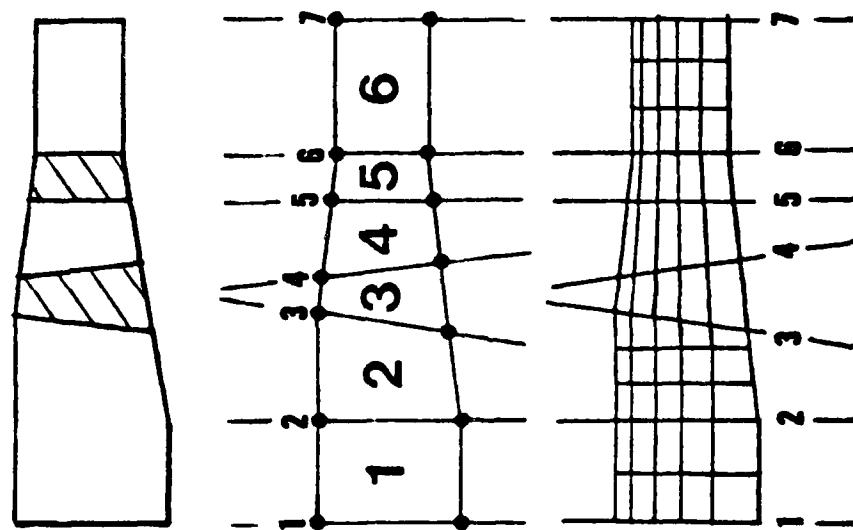


Figure 4. Mapping Relationships from the  $\xi, \eta$  to the  $z, r$  Plane

Figure 5. Mesh Generation



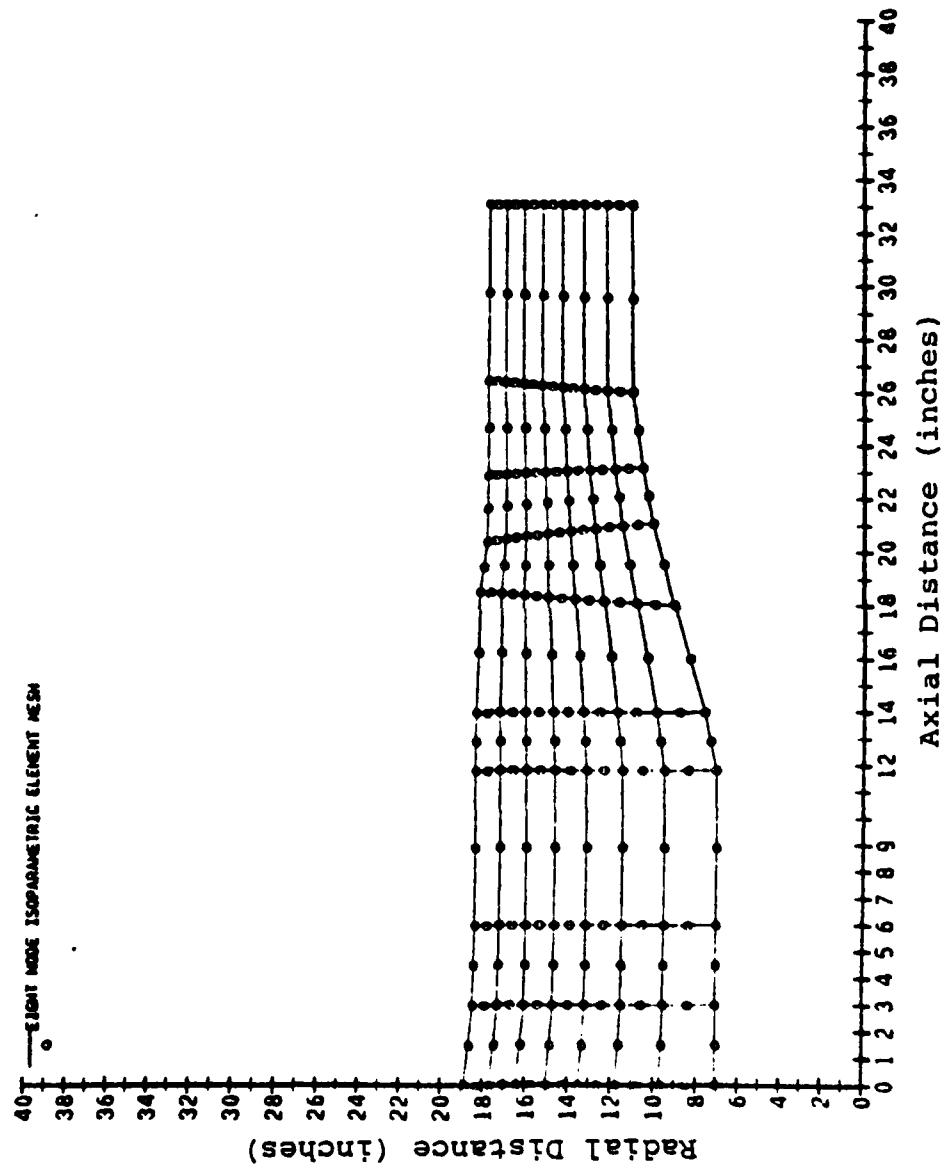


Figure 6. MESGEN Generated Plot of the 222 Node Mesh Used in the Calculations of the Meridional Through-Flow

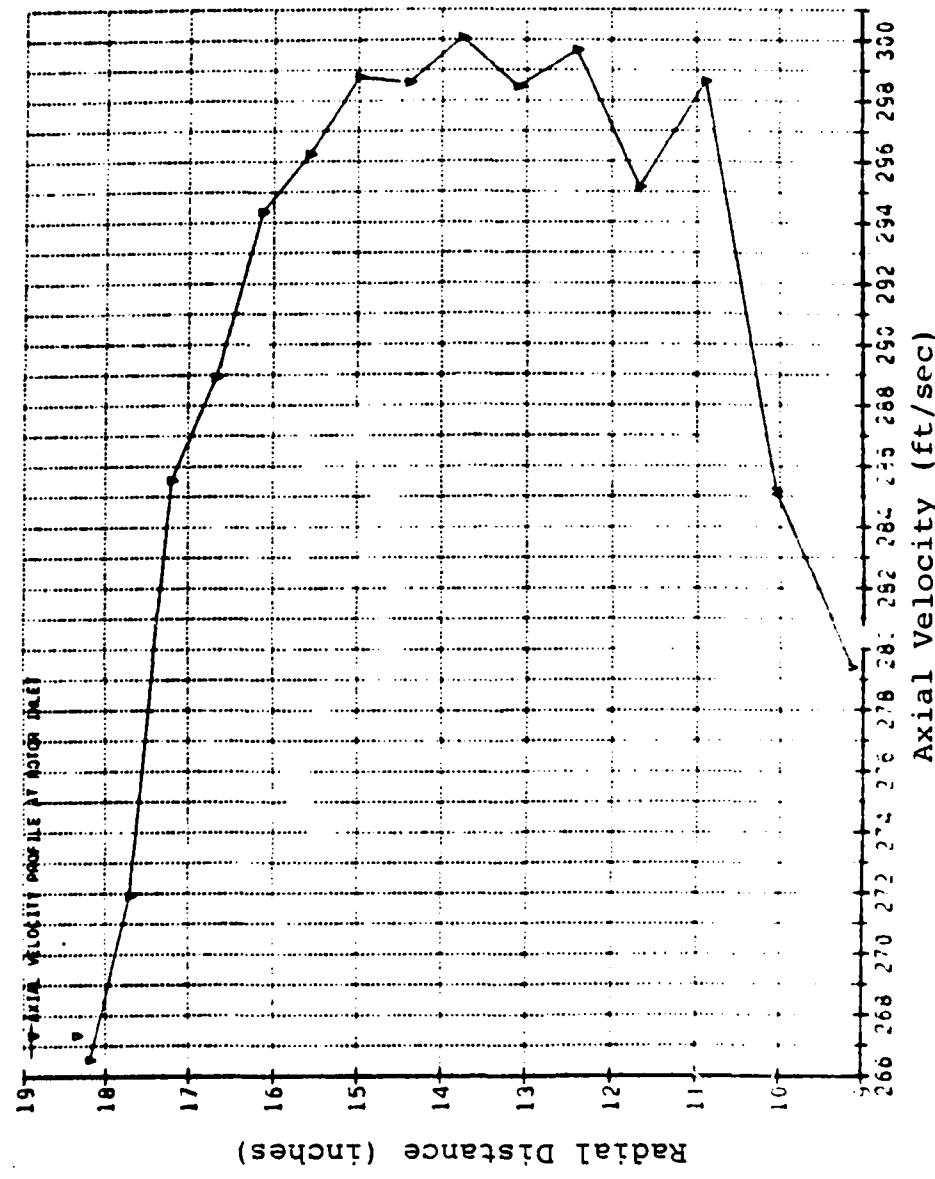


Figure 7. A TURBO Generated Tektonix 618 Plot of the Rotor  
inlet Axial Velocity at 50% Design Speed

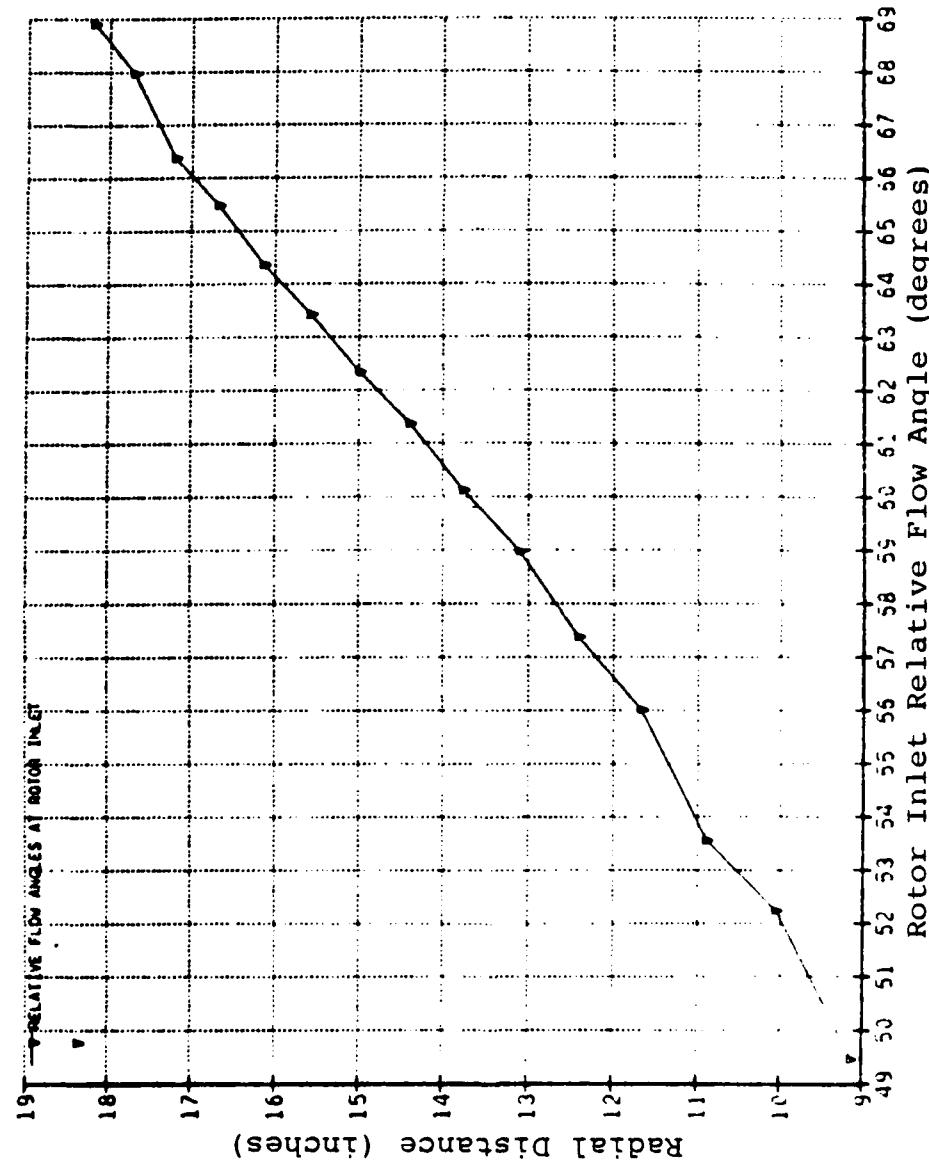
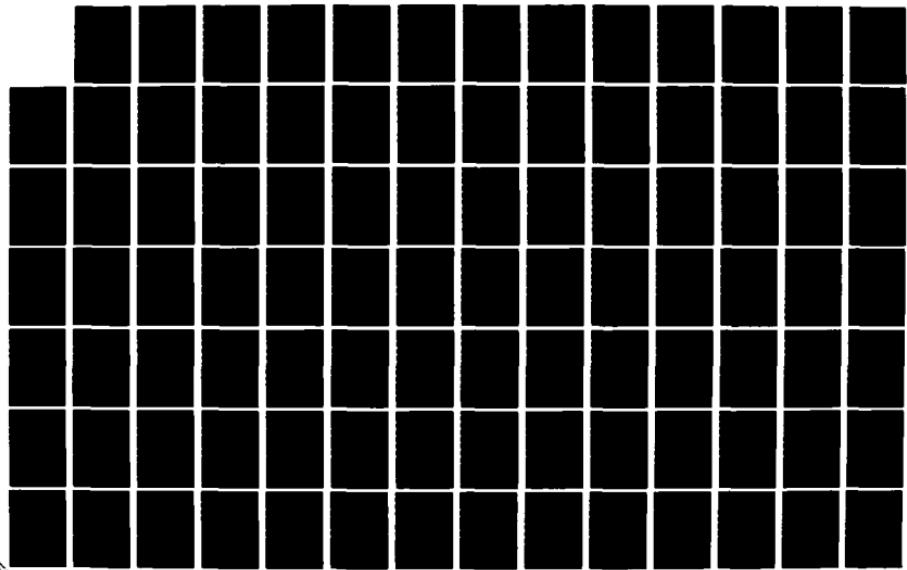
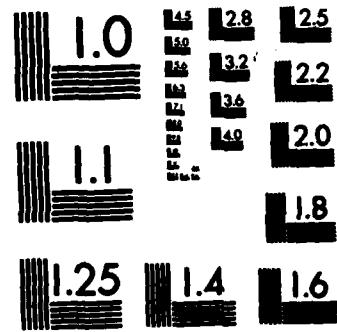


Figure 8. A TURBO Generated Tektonix 618 Plot of the Rotor Inlet Relative Flow Angles

AD-A124 987 FINITE ELEMENT PROGRAM FOR CALCULATING FLOWS IN  
TURBOMACHINES WITH RESULTS FOR NASA TASK-1 COMPRESSOR 2/3  
(U) NAVAL POSTGRADUATE SCHOOL MONTEREY CA J R FERGUSON  
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MICROCOPY RESOLUTION TEST CHART  
NATIONAL BUREAU OF STANDARDS-1963-A

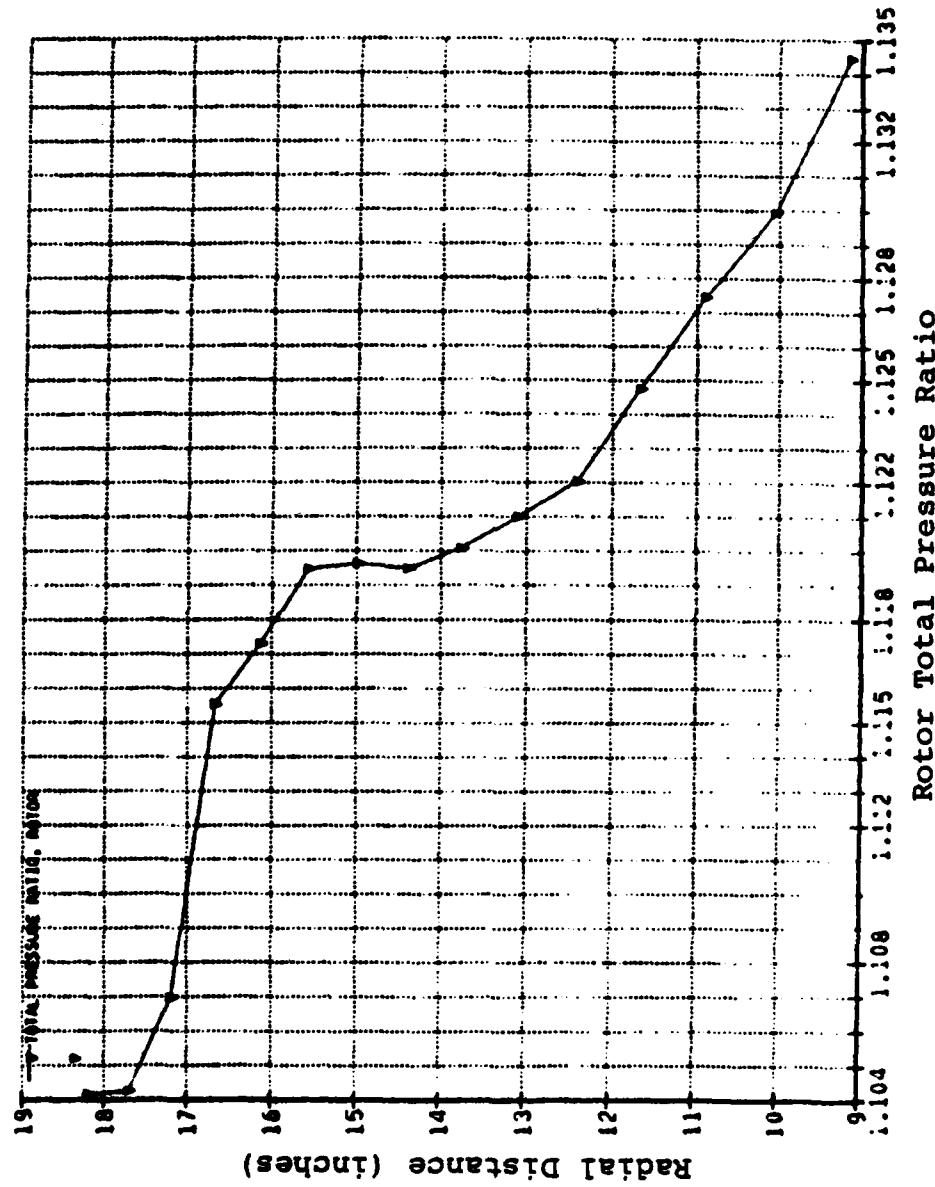


Figure 9. A TURBO Generated Tektronix 618 Plot of the Total Pressure Ratio of the Rotor vs Inlet Radius

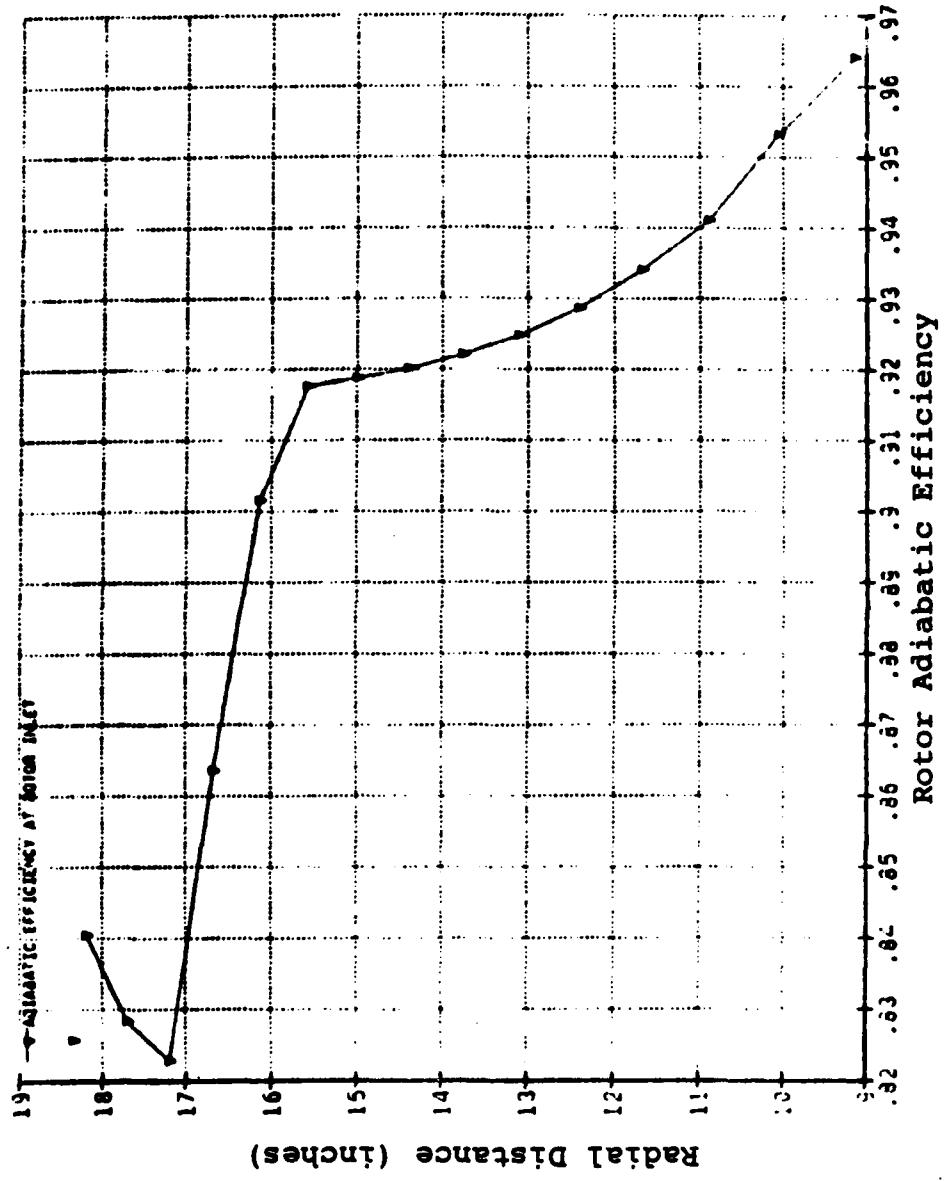


Figure 10. A TURBO Generated Tektronix 618 Plot of the Adiabatic Efficiency of the Rotor vs Inlet Radius

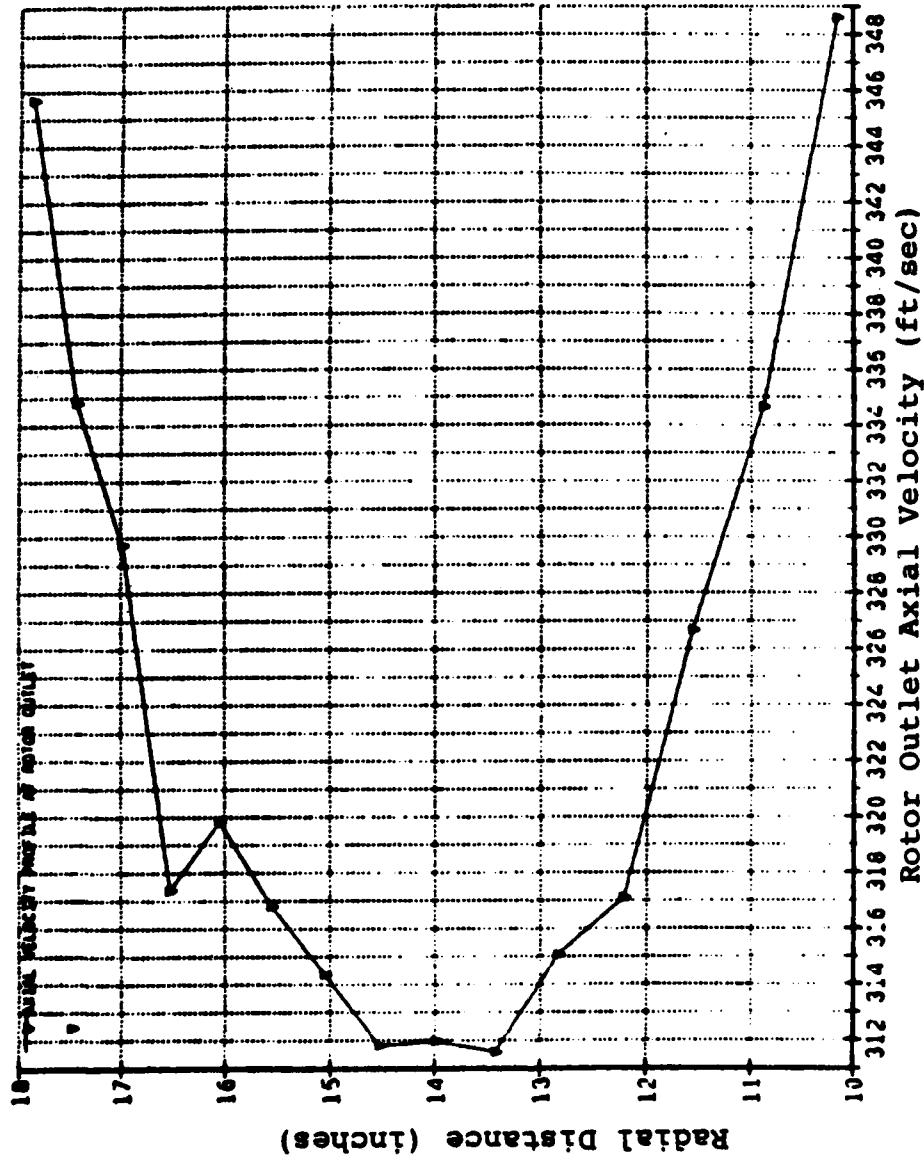


Figure 11. A TURBO Generated Tektronix 618 Plot of the Rotor Outlet Axial Velocity

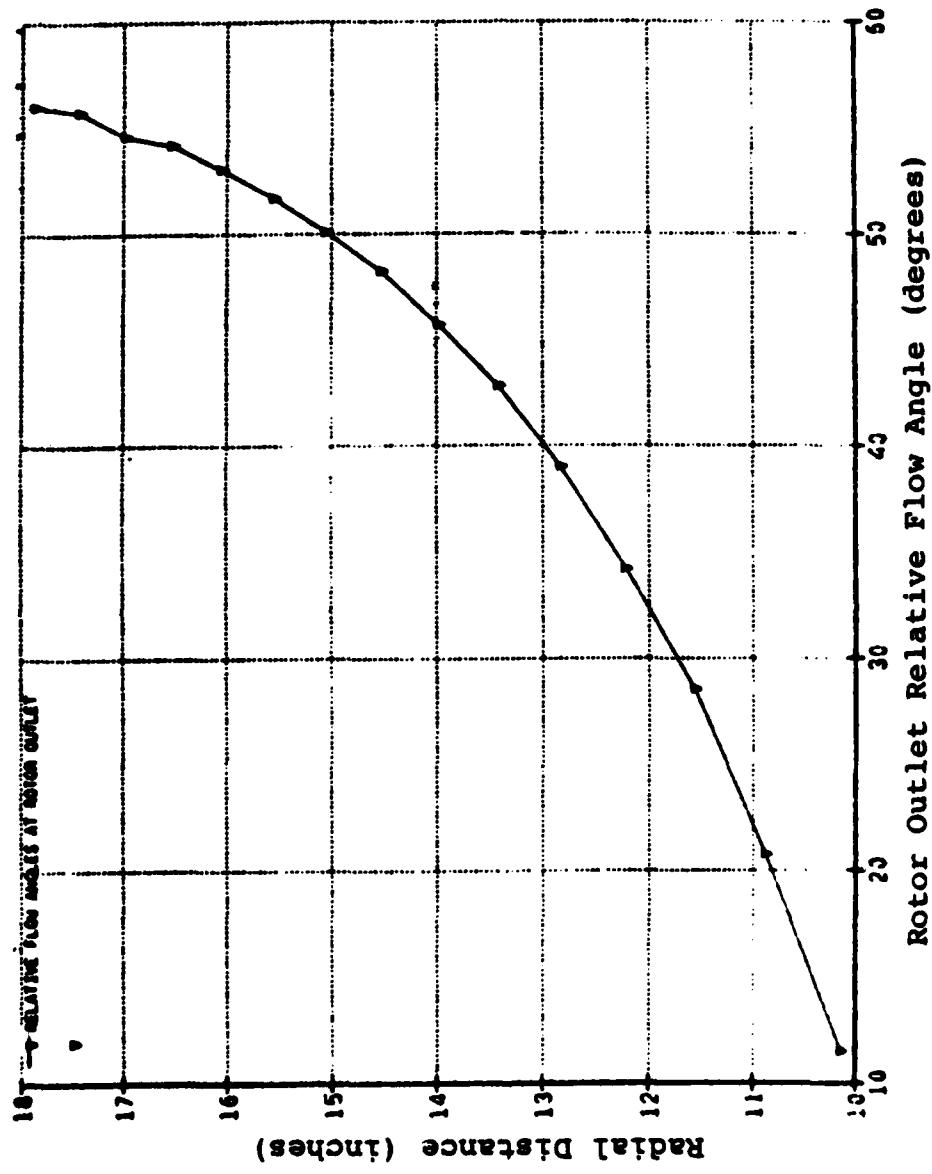


Figure 12. A TURBO Generated Tektronix 618 Plot of the Rotor Outlet Relative Flow Angles

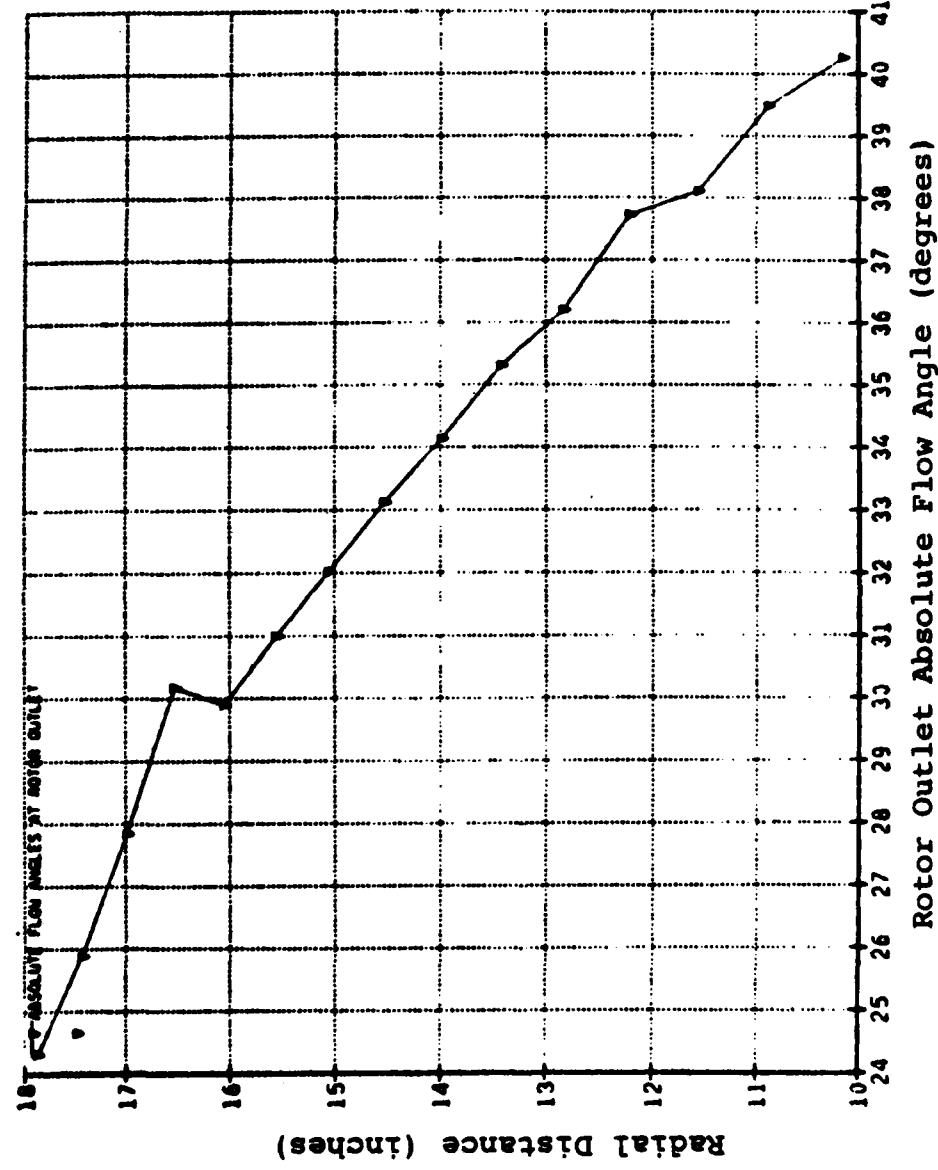


Figure 13. A TURBO Generated Tektronix 618 Plot of the Rotor Outlet Absolute Flow Angles

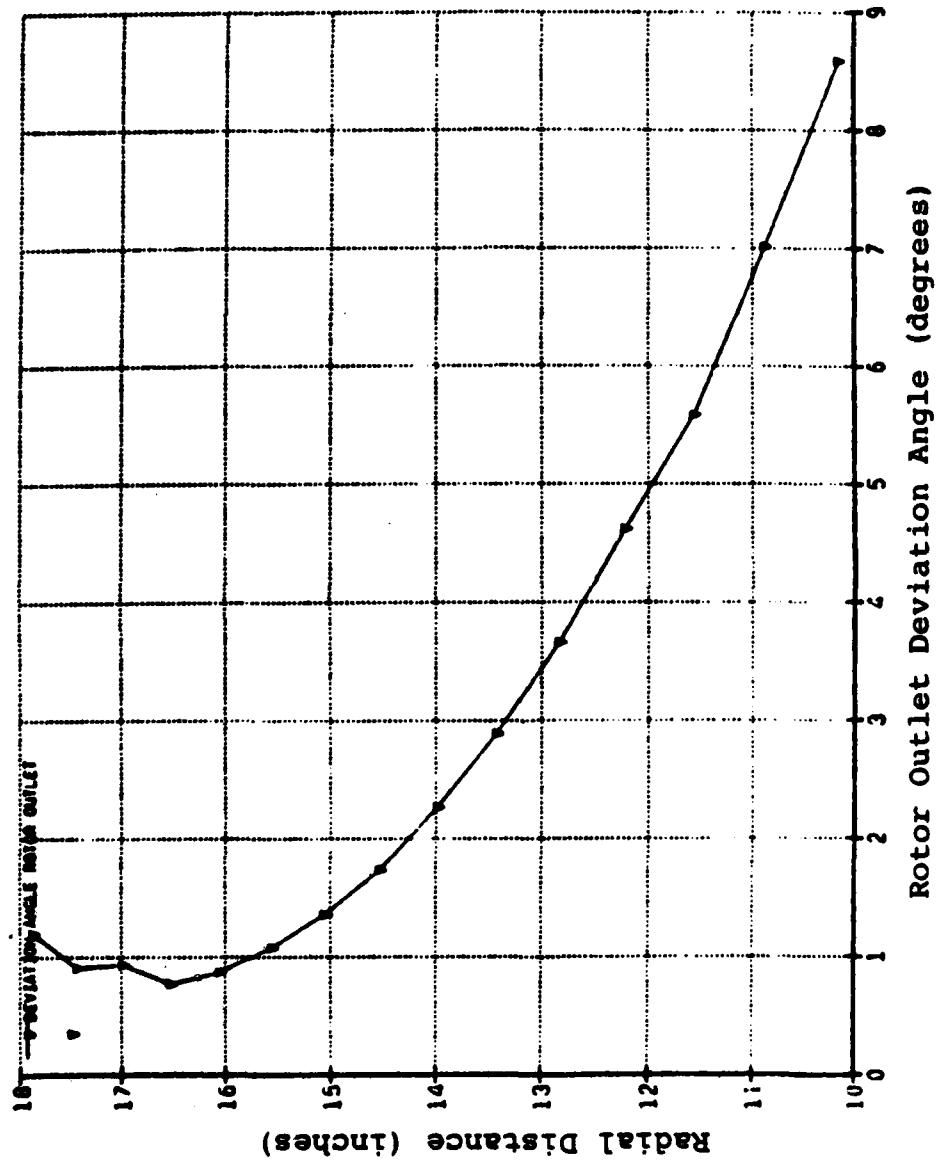


Figure 14. A TURBO Generated Tektronix 618 Plot of the Rotor Outlet Deviation Flow Angles

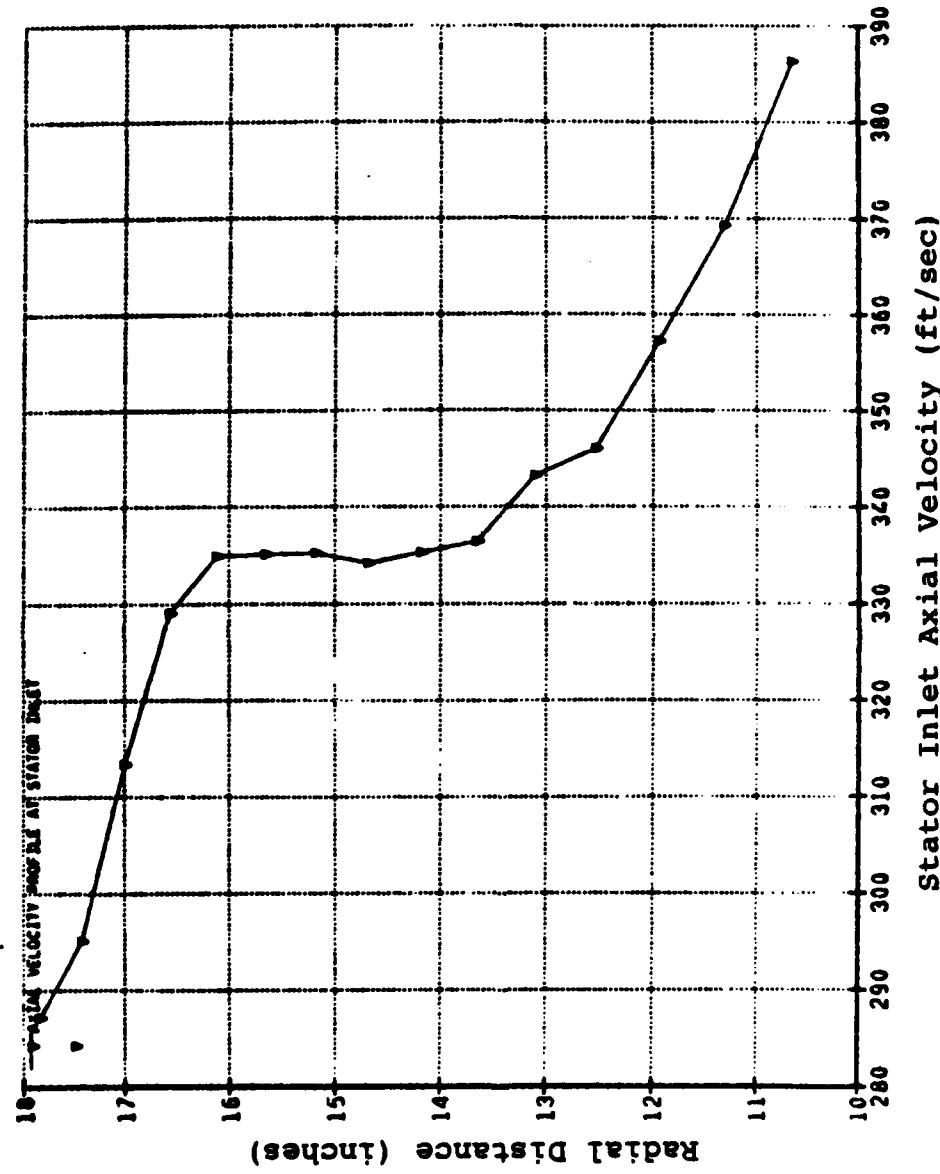


Figure 15. A TURBO Generated Tektronix 618 Plot  
of the Stator Inlet Axial Velocity

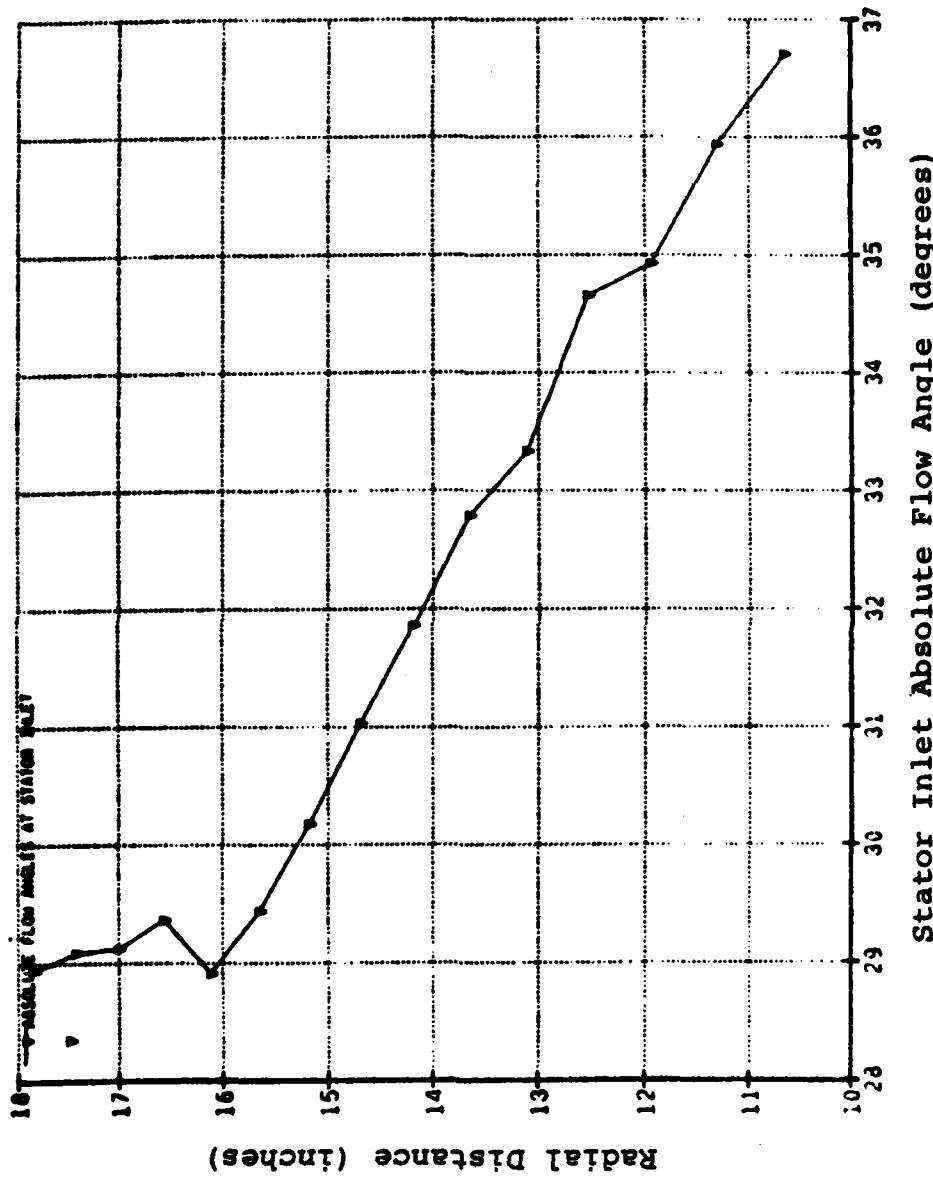


Figure 16. A TURBO Generated tektonix 618 Plot of the Stator Inlet Absolute Flow Angles

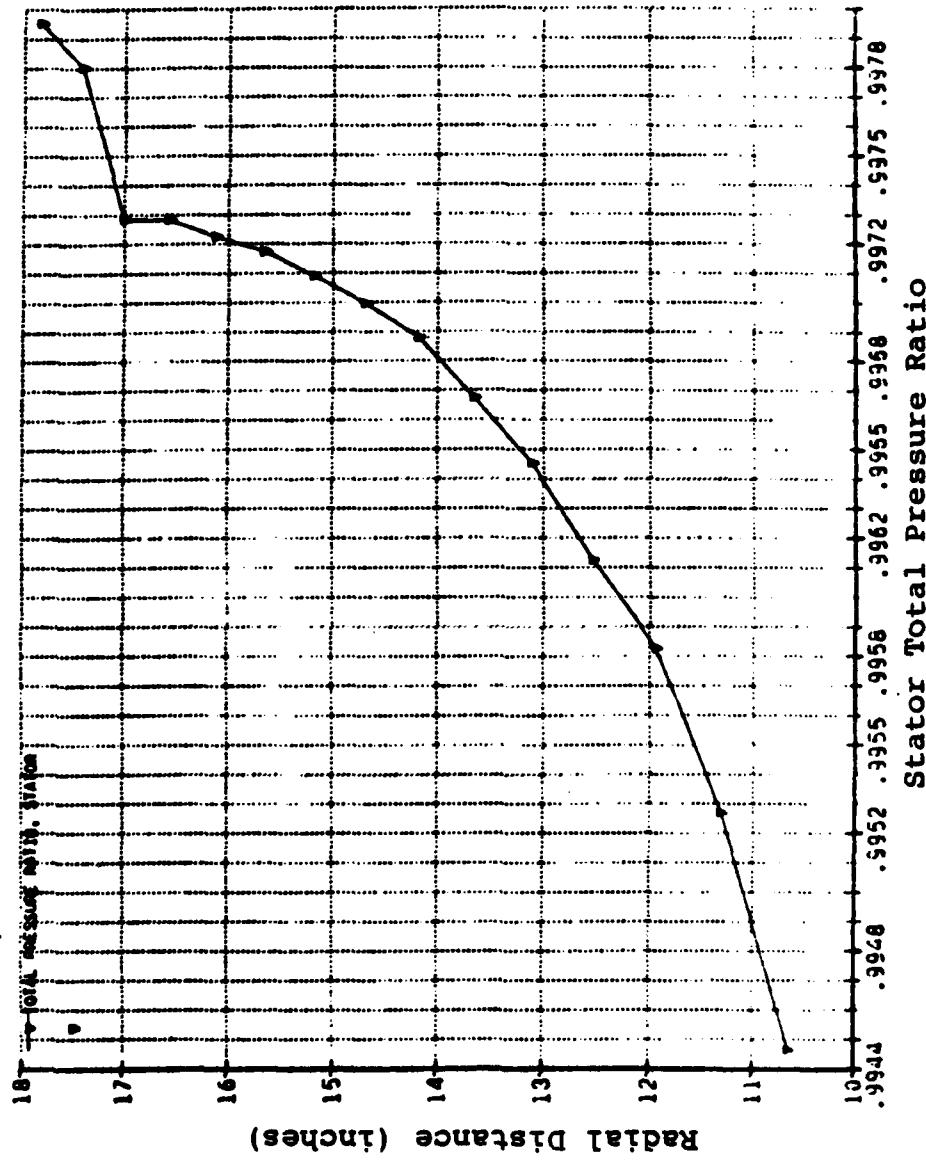


Figure 17. A TURBO Generated Tektronix 618 Plot of the  
Stator Total Pressure Ratio vs Inlet Radius

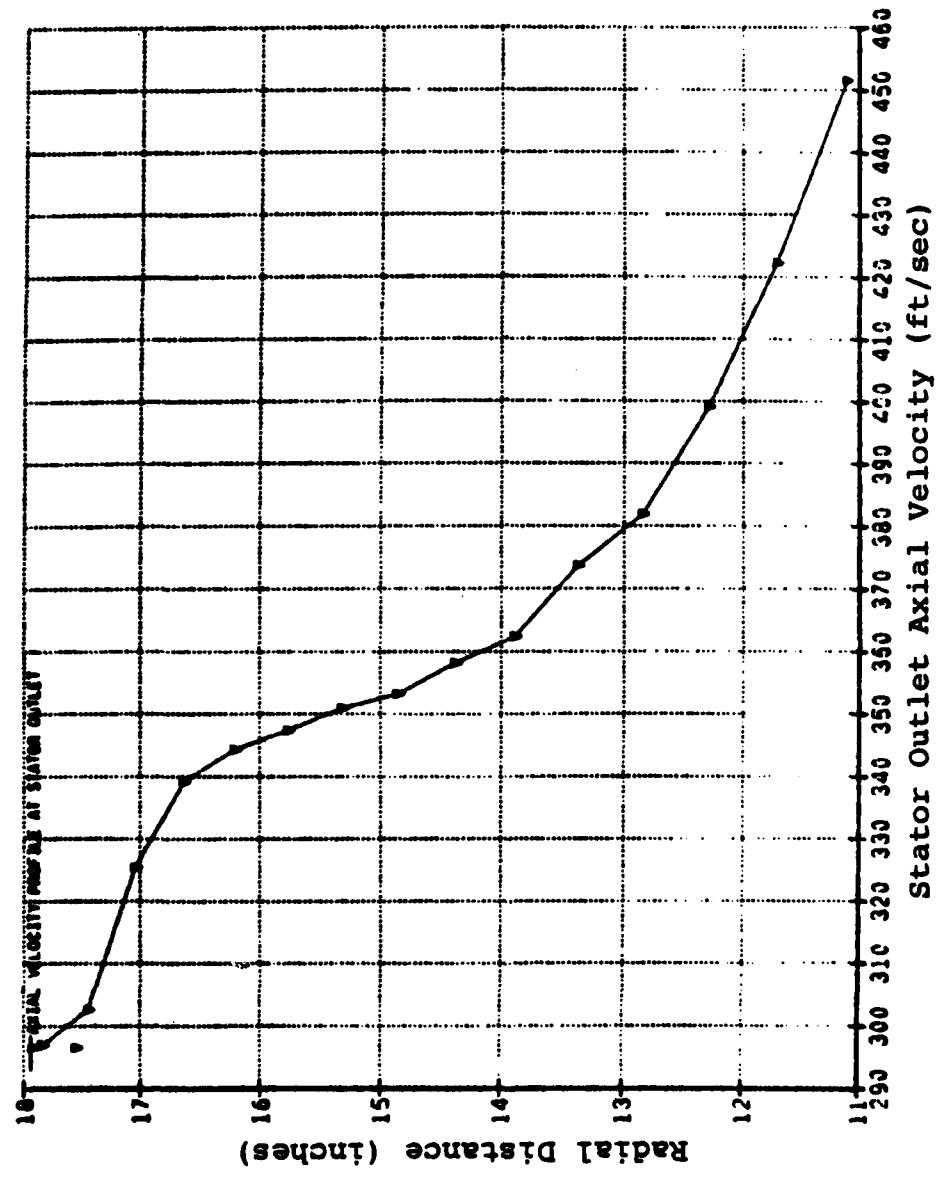


Figure 18. A TURBO Generated Tektronix 618 Plot of the Stator Outlet Axial Velocity

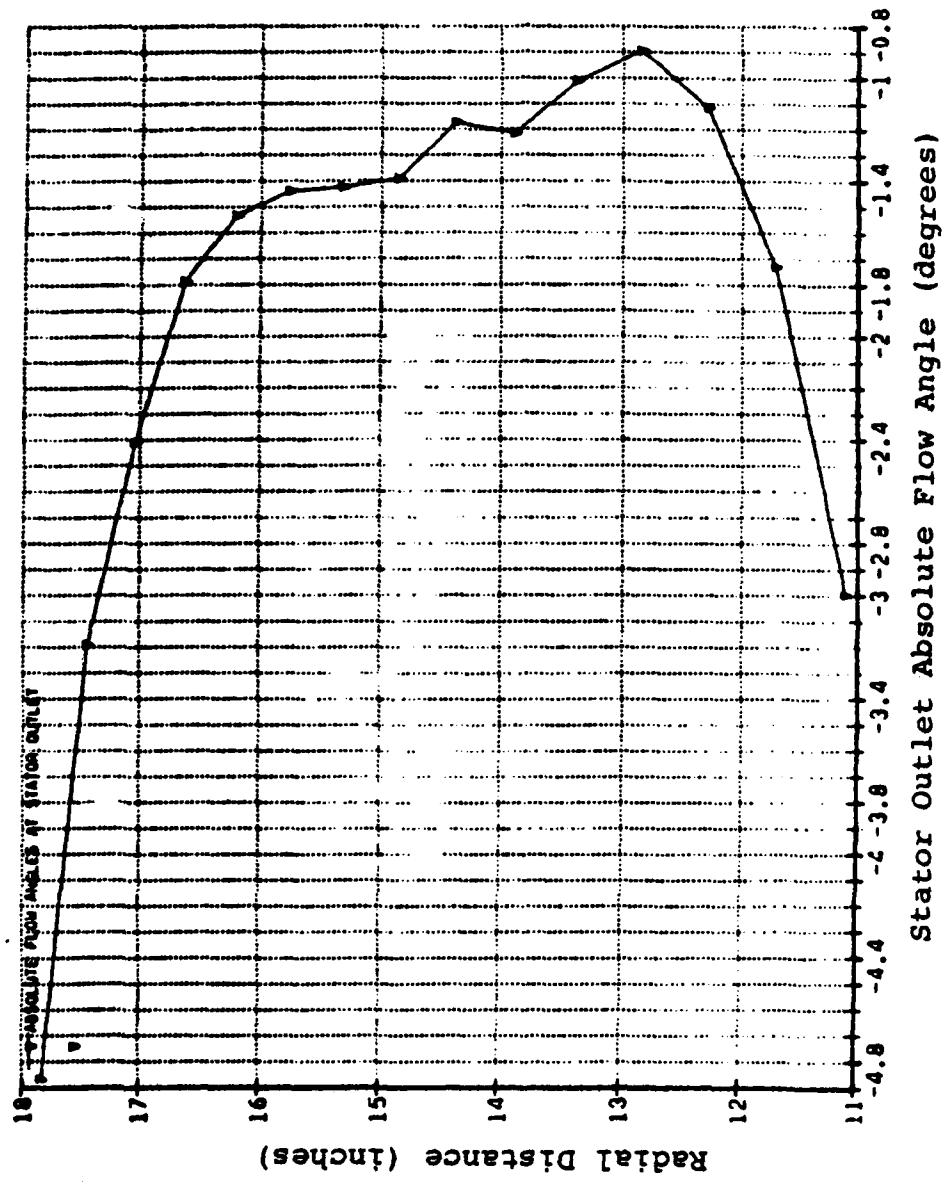


Figure 19. A TURBO Generated Tektronix 618 Plot of the Stator Outlet Absolute Flow Angles

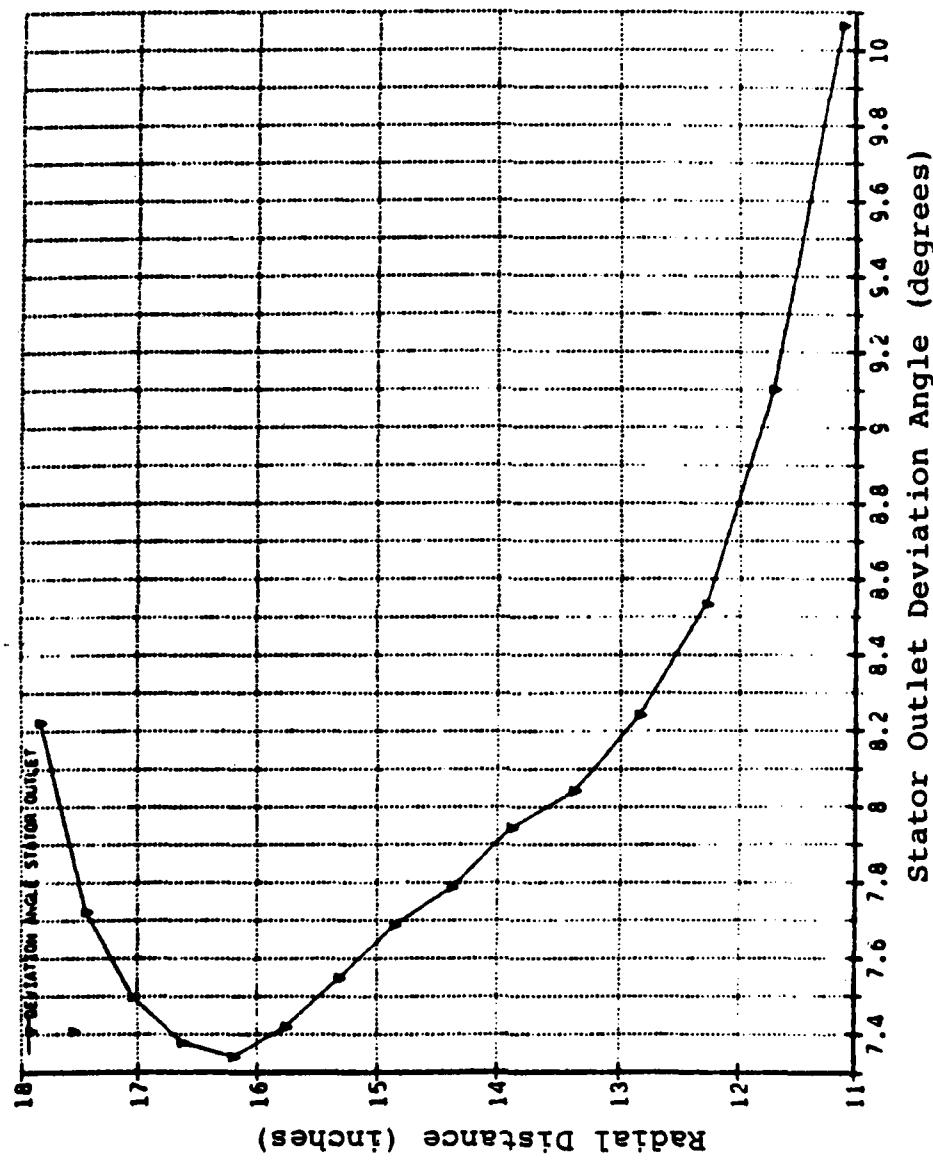


Figure 20. A TURBO Generated Tektronix 618 Plot of the  
Stator Outlet Deviation Flow Angles

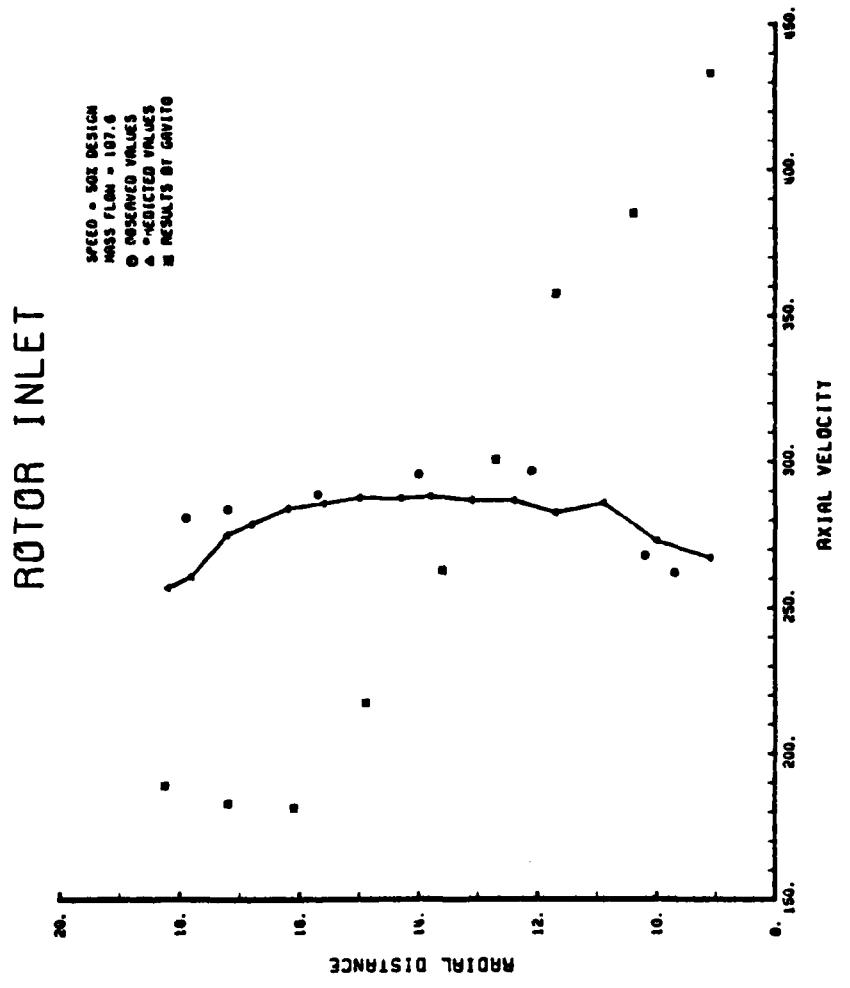


Figure 21. Comparison of Predictions to Gavito and Observations for Rotor Inlet Axial Velocity, 50% Design

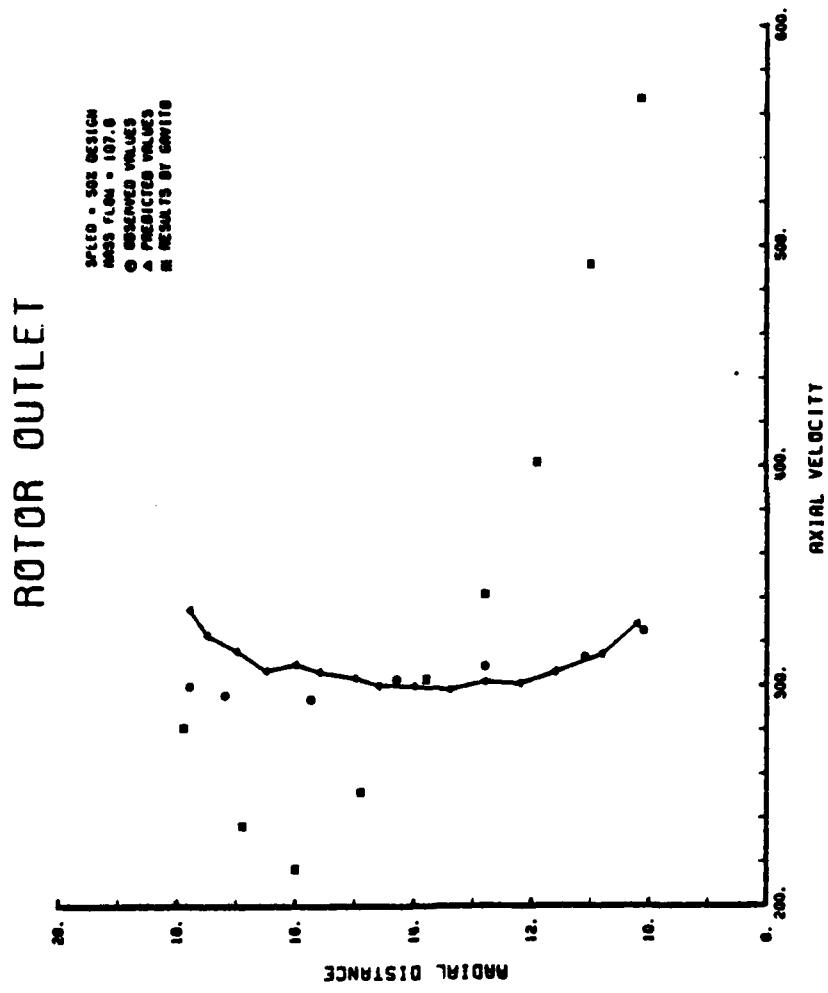


Figure 22. Comparison of Predictions to Gavito and Observations for Rotor Outlet Axial Velocity, 50% Design

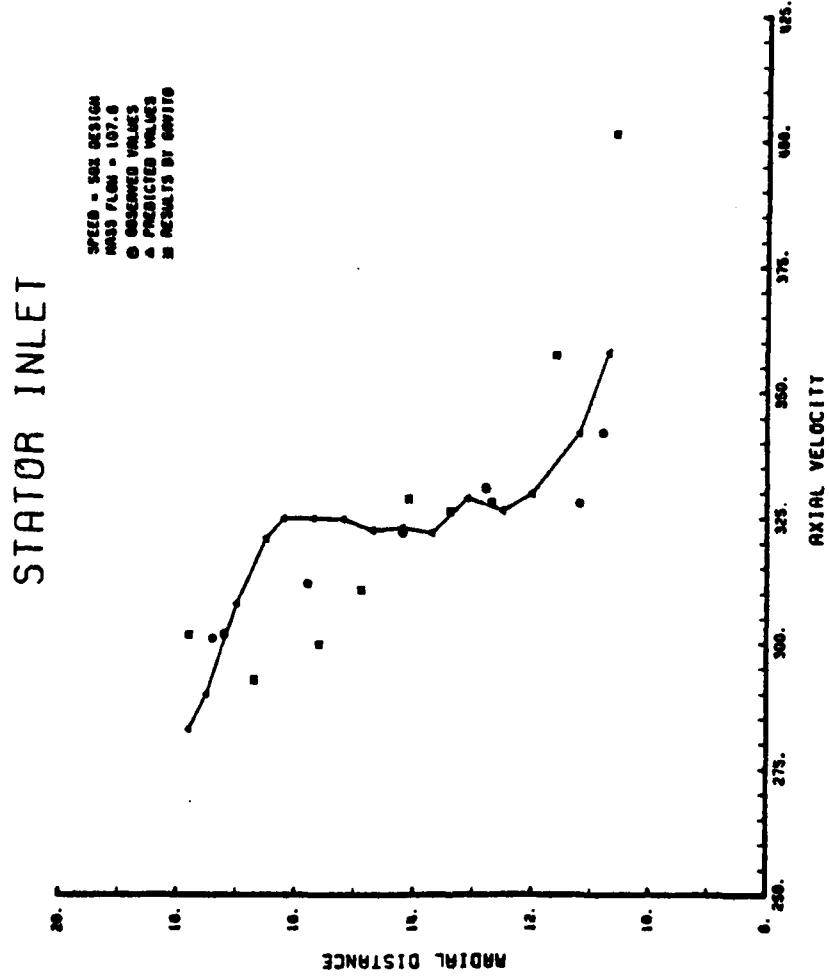


Figure 23. Comparison of Predictions to Gavito and Observations for Stator Inlet Axial Velocity, 50% Design

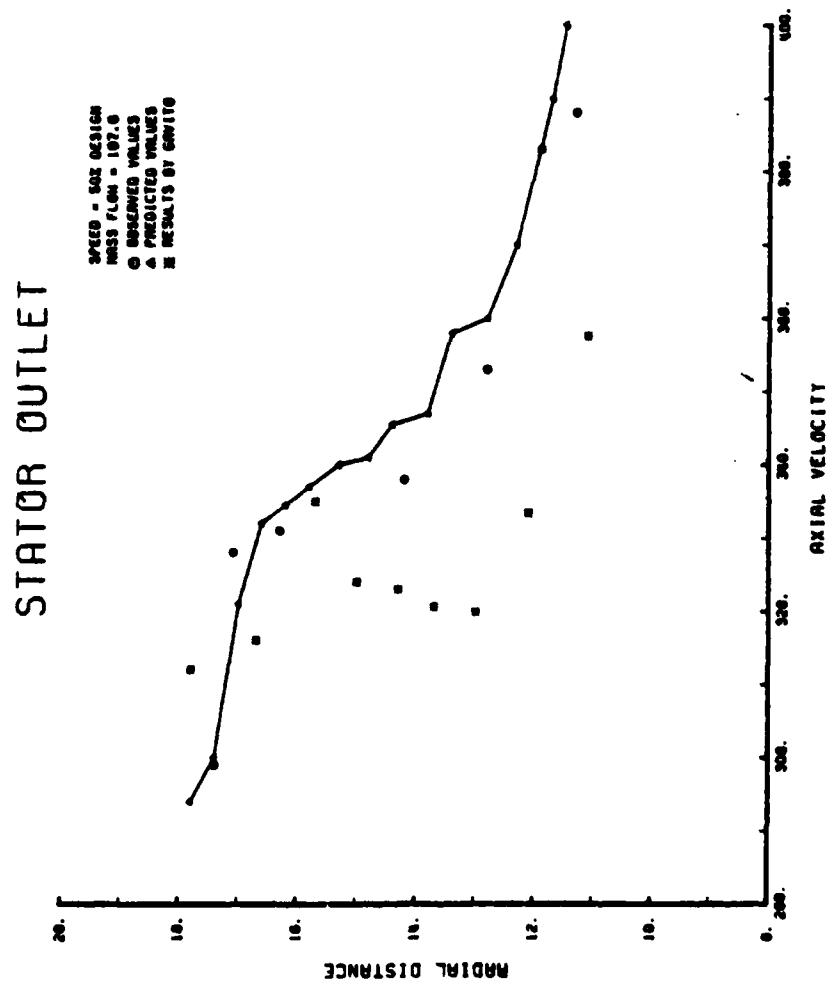


Figure 24. Comparison of Predictions to Gavito and Observations for Stator Outlet Axial Velocity, 50% Design

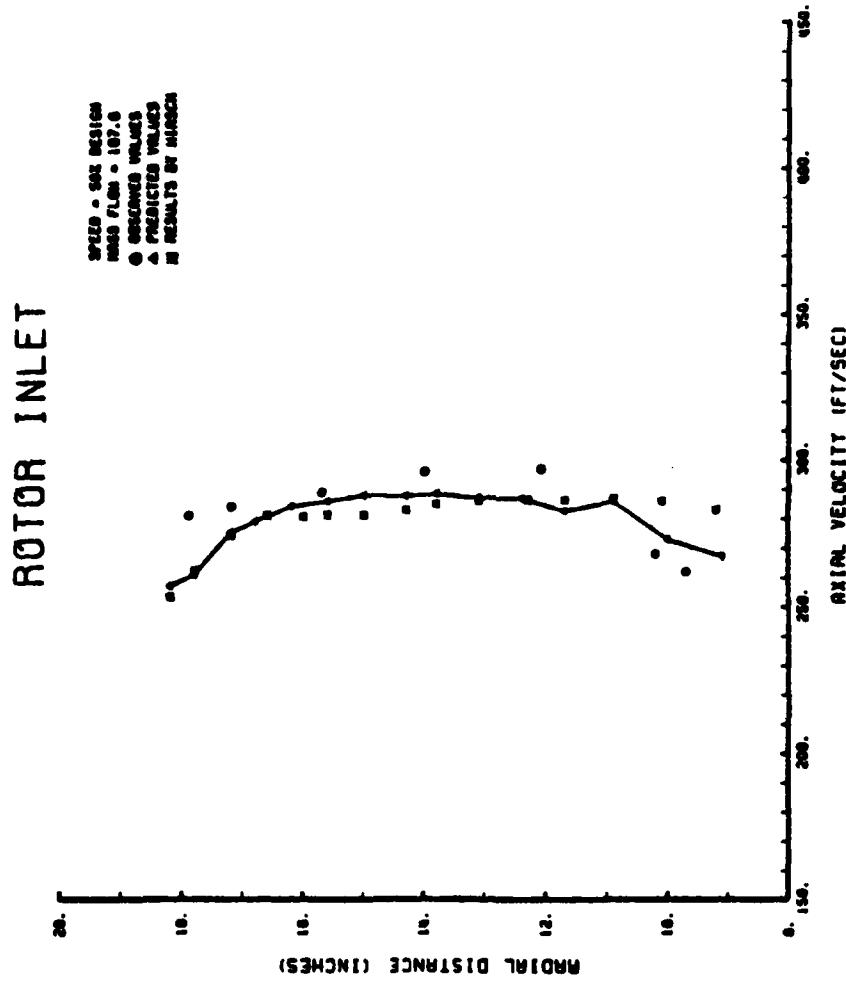


Figure 25. Comparison of Predictions to Hirsch and Observations for Rotor Inlet Axial Velocity, 50% Design

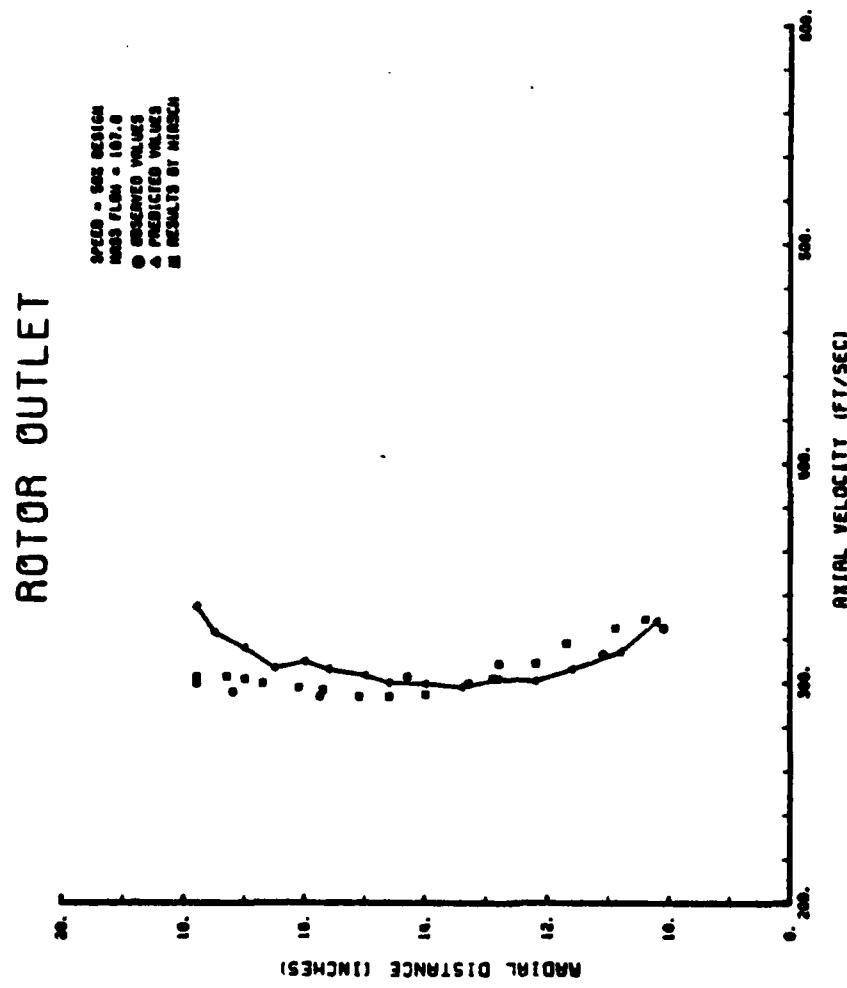


Figure 26. Comparison of Predictions to Hirsch and Observations for Rotor Outlet Axial Velocity, 50% Design

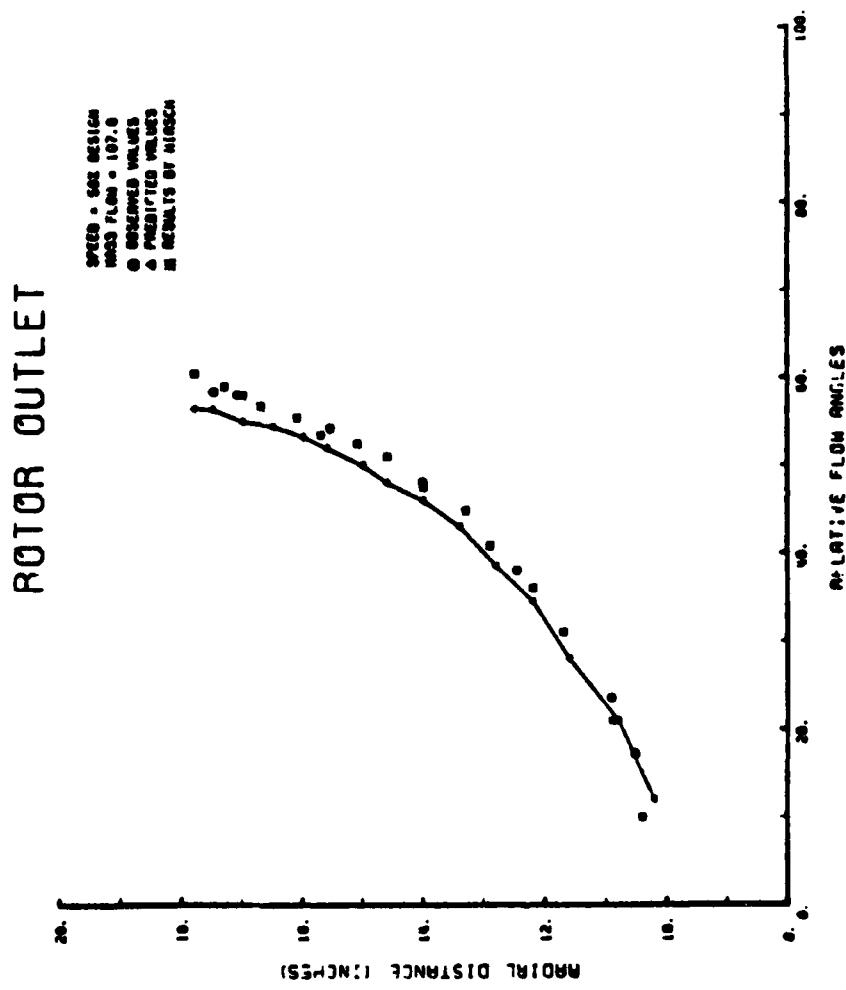


Figure 27. Comparison of Predictions to Hirsch and Observations for Rotor Outlet Relative Flow Angles, 50° Design

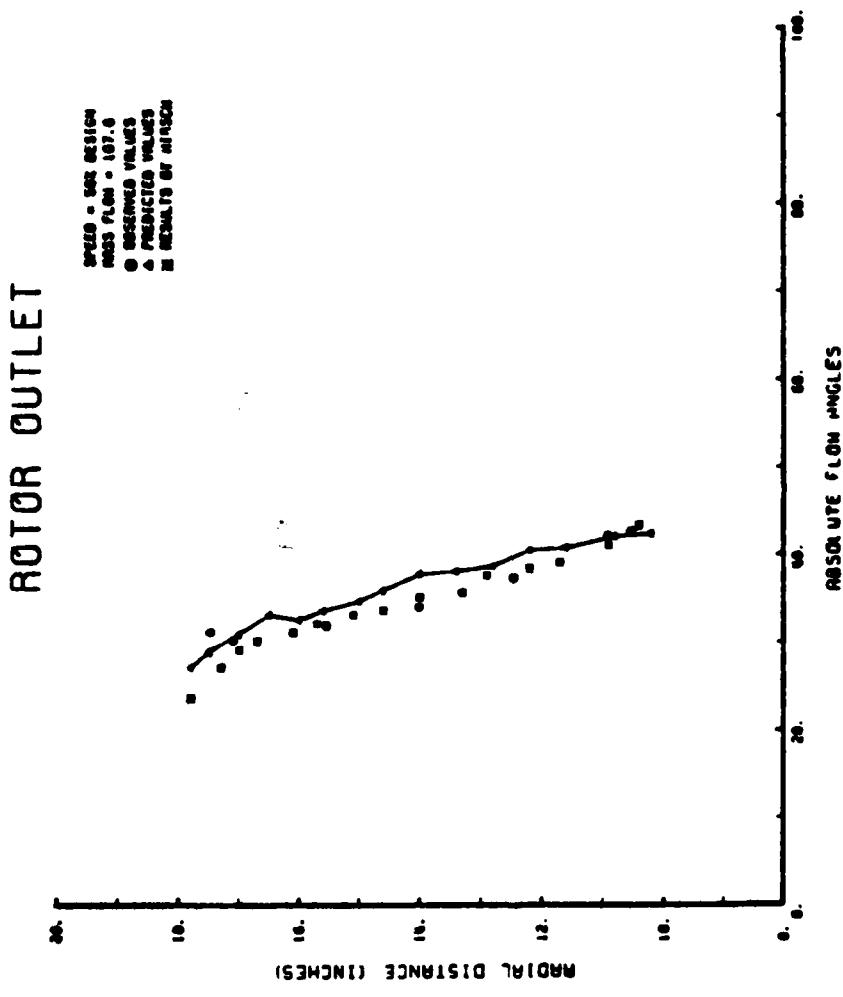


Figure 28. Comparison of Predictions to Hirsch and Observations for Rotor Outlet Absolute Flow Angles, 50% Design

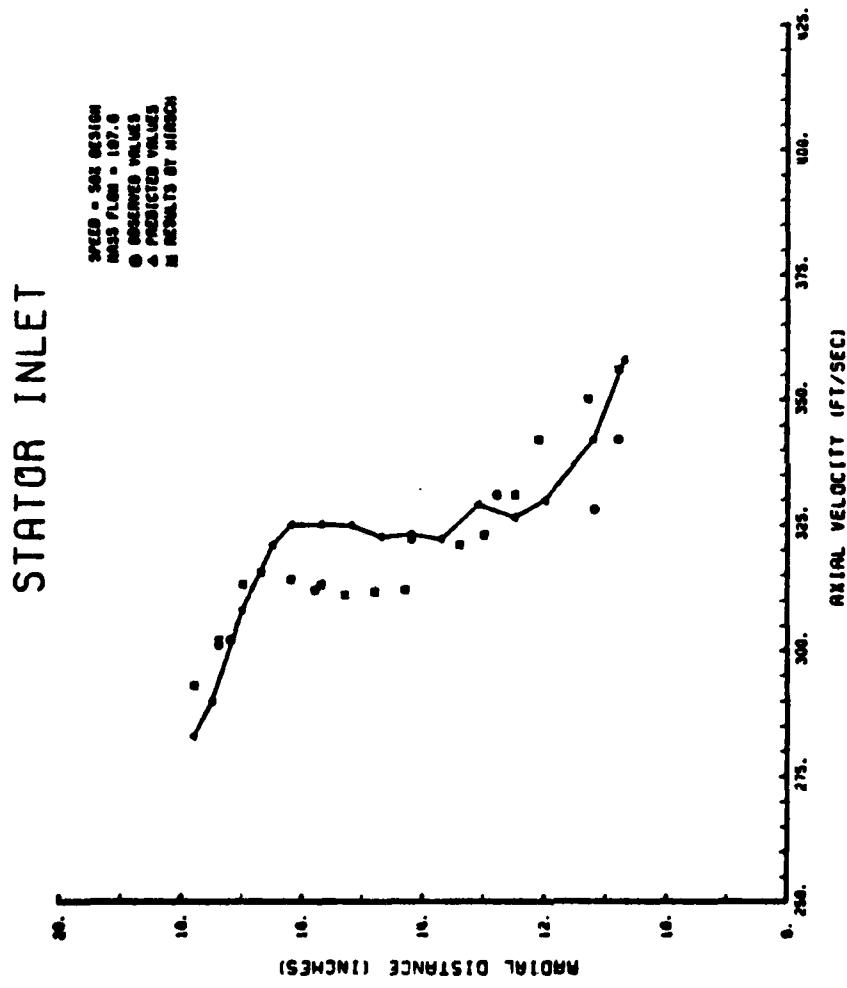


Figure 29. Comparison of Predictions to Hirsch and Observations for Stator Inlet Axial Velocity, 50% Design

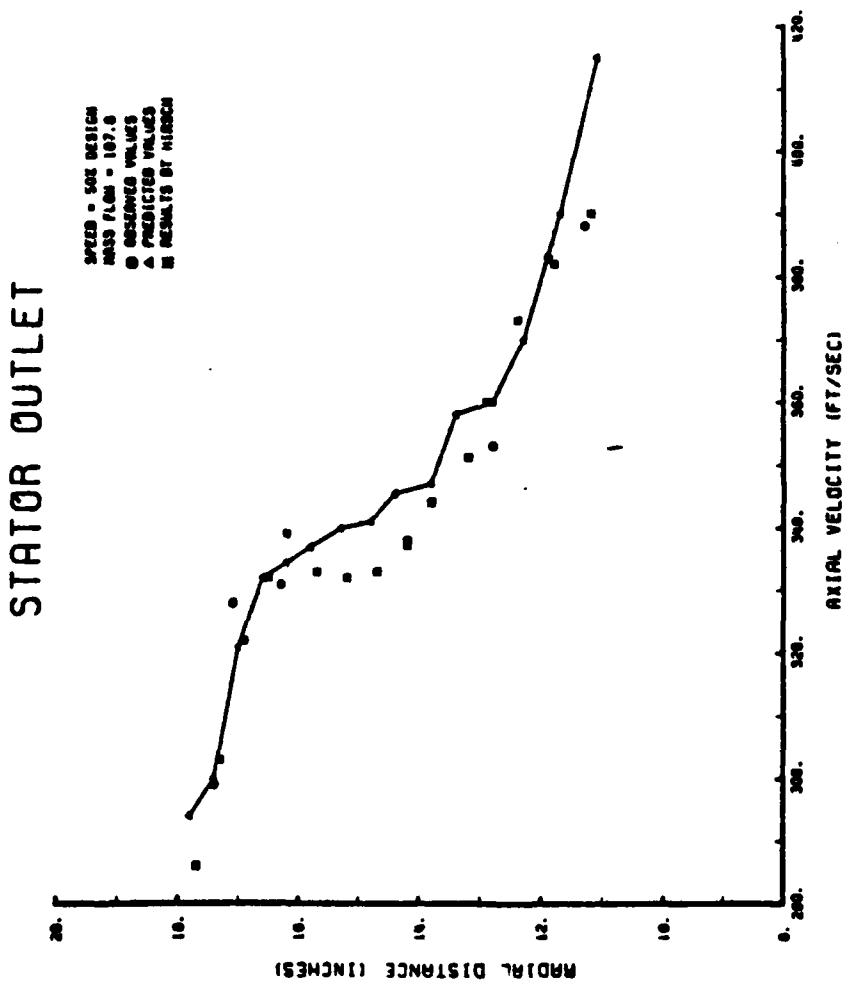


Figure 30. Comparison of Predictions to Hirsch and Observations for Stator Outlet Axial Velocity, 50% Design

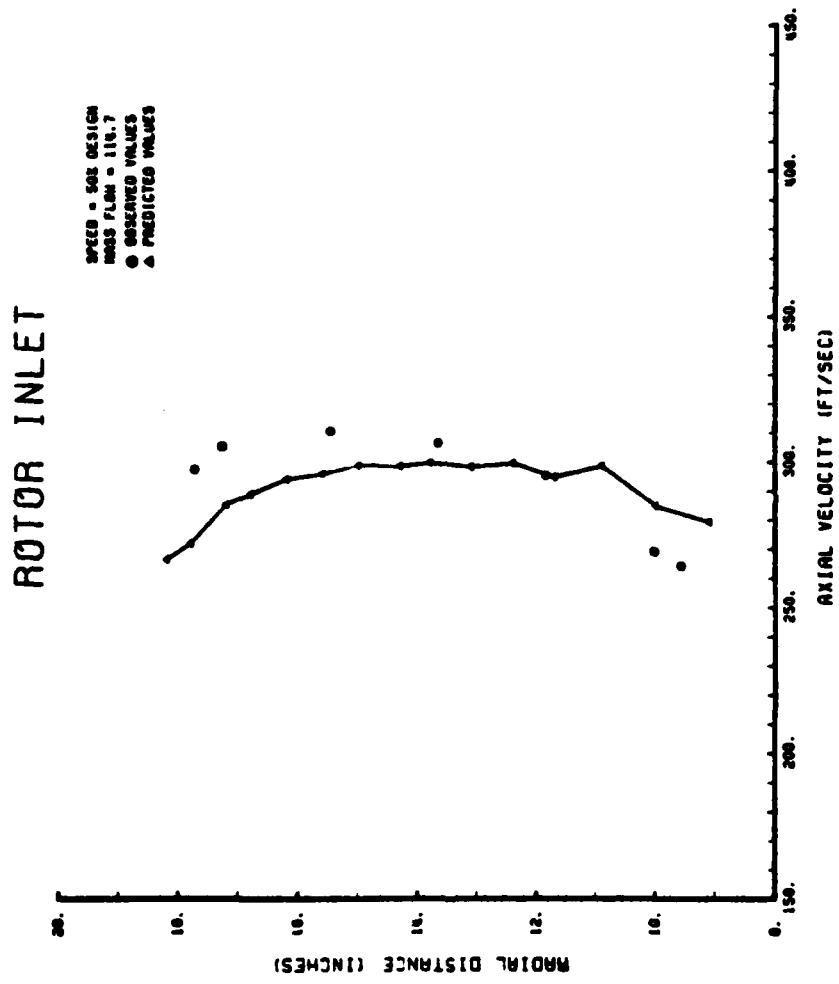


Figure 31. Axial Velocity at the Rotor Inlet, 508 Design

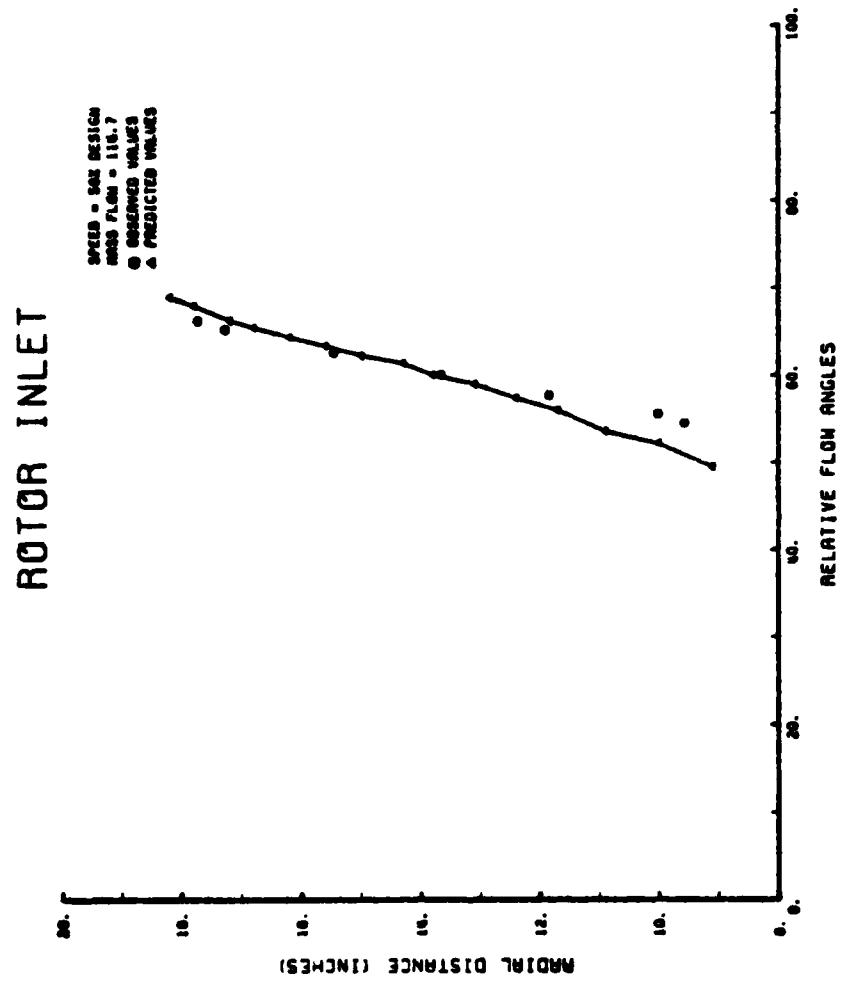


Figure 32. Relative Flow Angles at Rotor Inlet, 50° Design

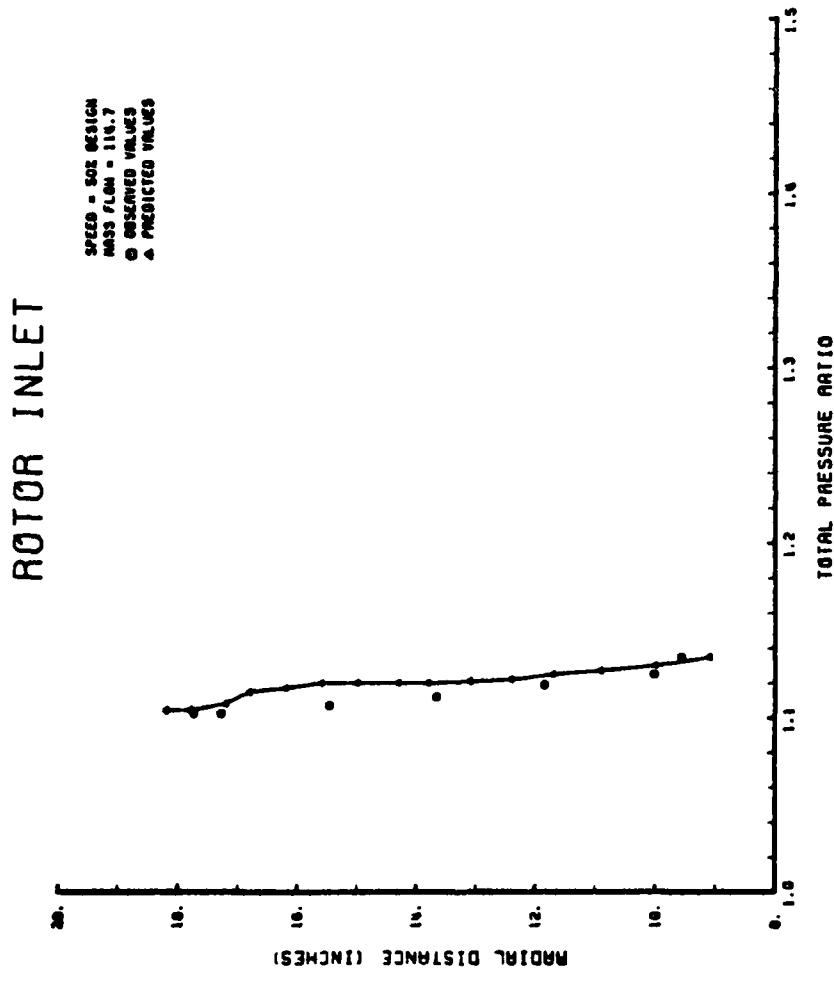


Figure 33. Total Pressure Ratio of the Rotor, 50% Design

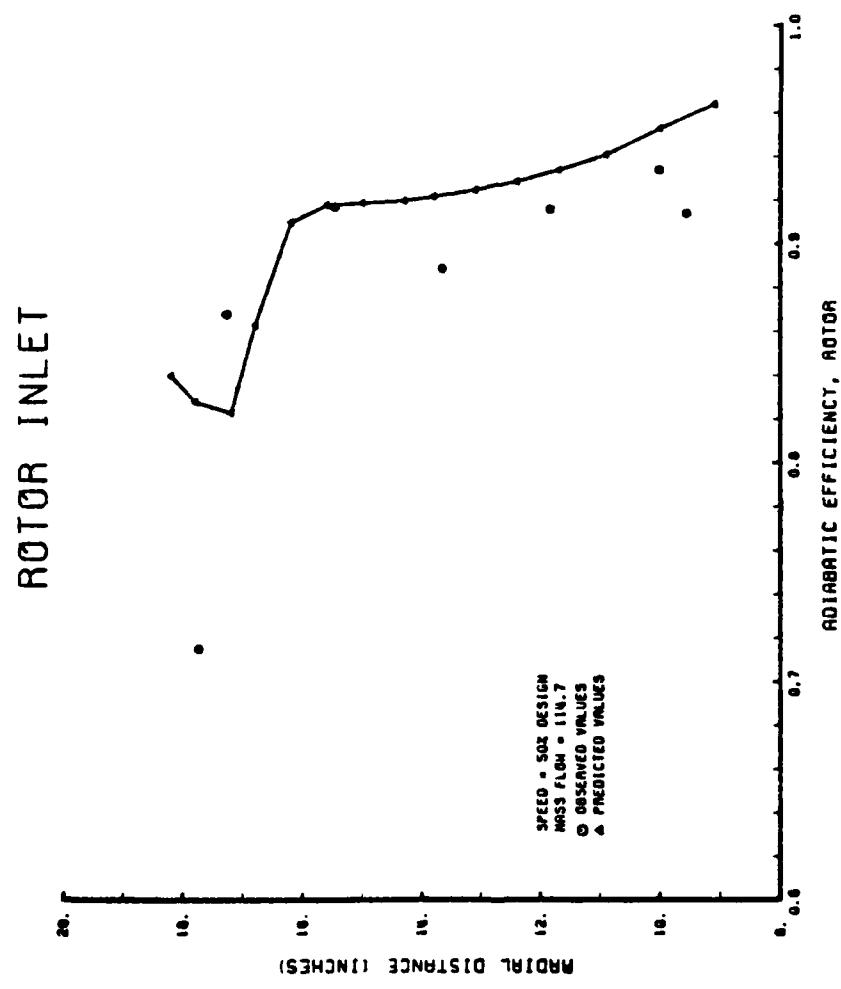


Figure 34. Adiabatic Efficiency of the Rotor, 50% Design

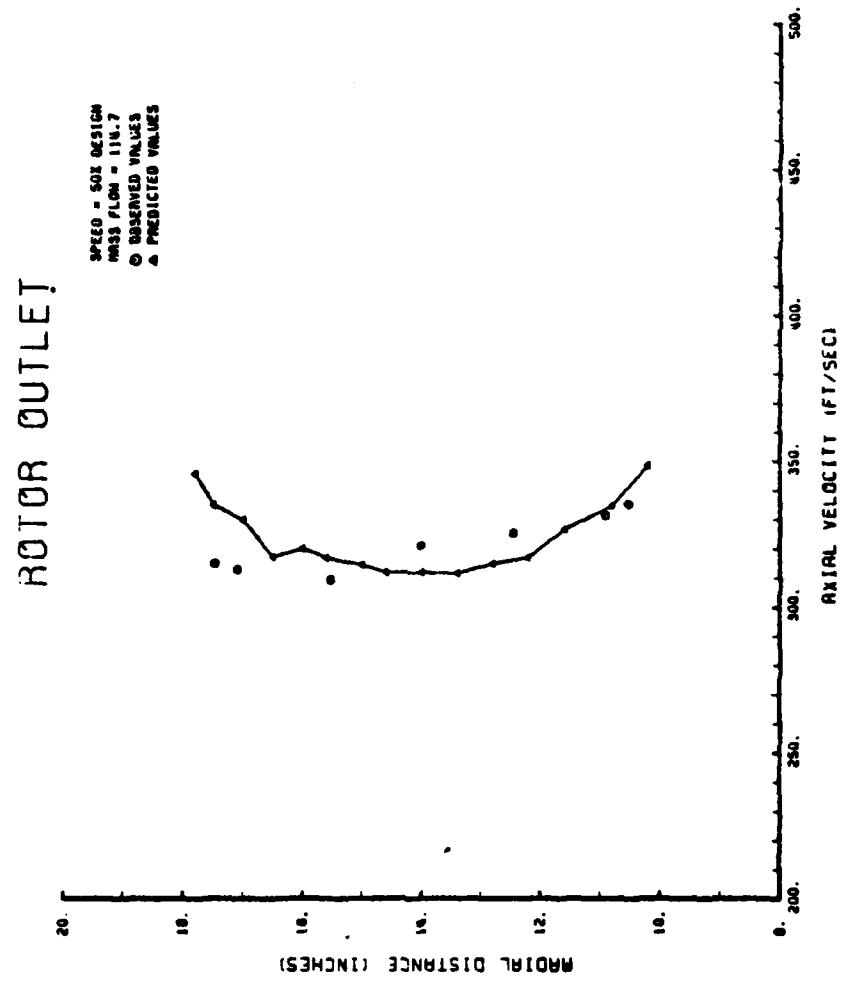


Figure 35. Axial Velocity at the Rotor Outlet, 50% Design

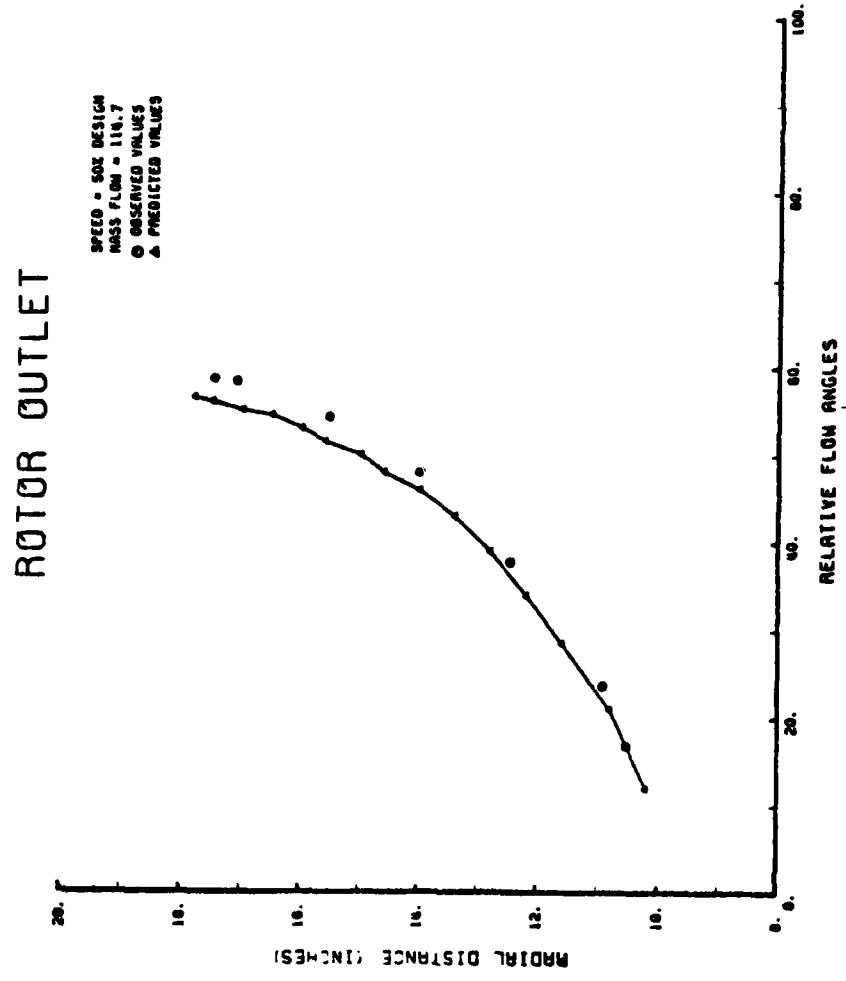


Figure 36. Relative Flow Angles at Rotor Outlet, 50% Design

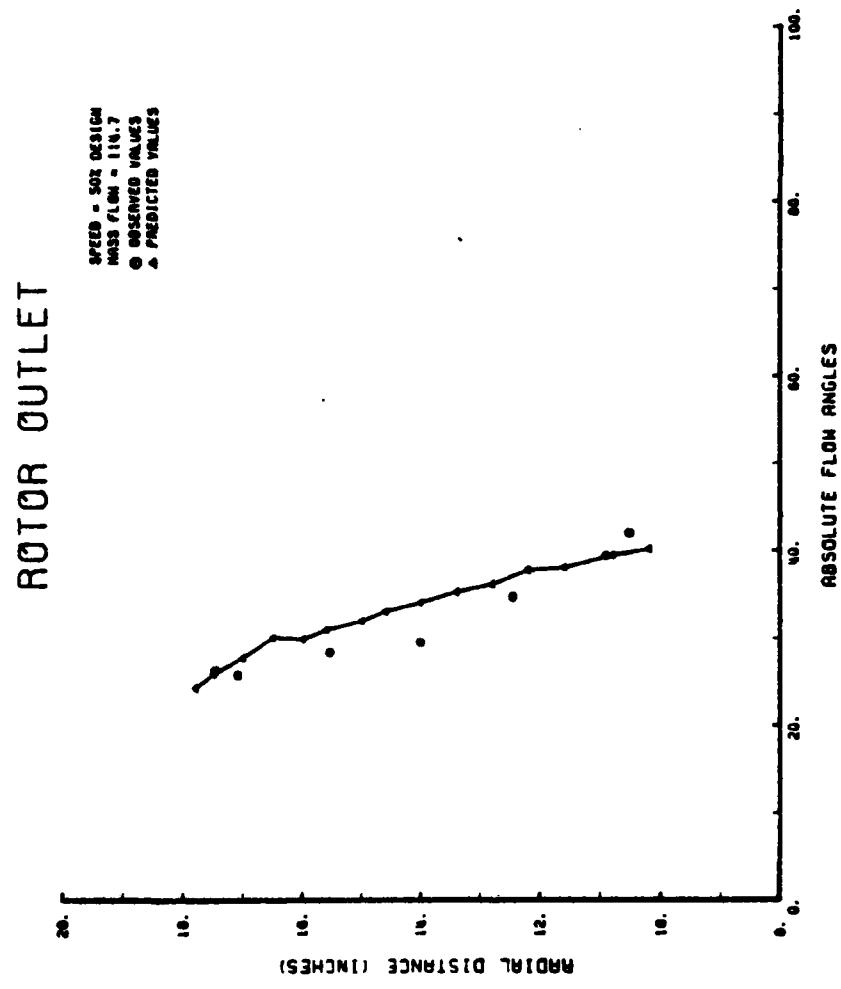


Figure 37. Absolute Flow Angles at Rotor Outlet, 50% Design

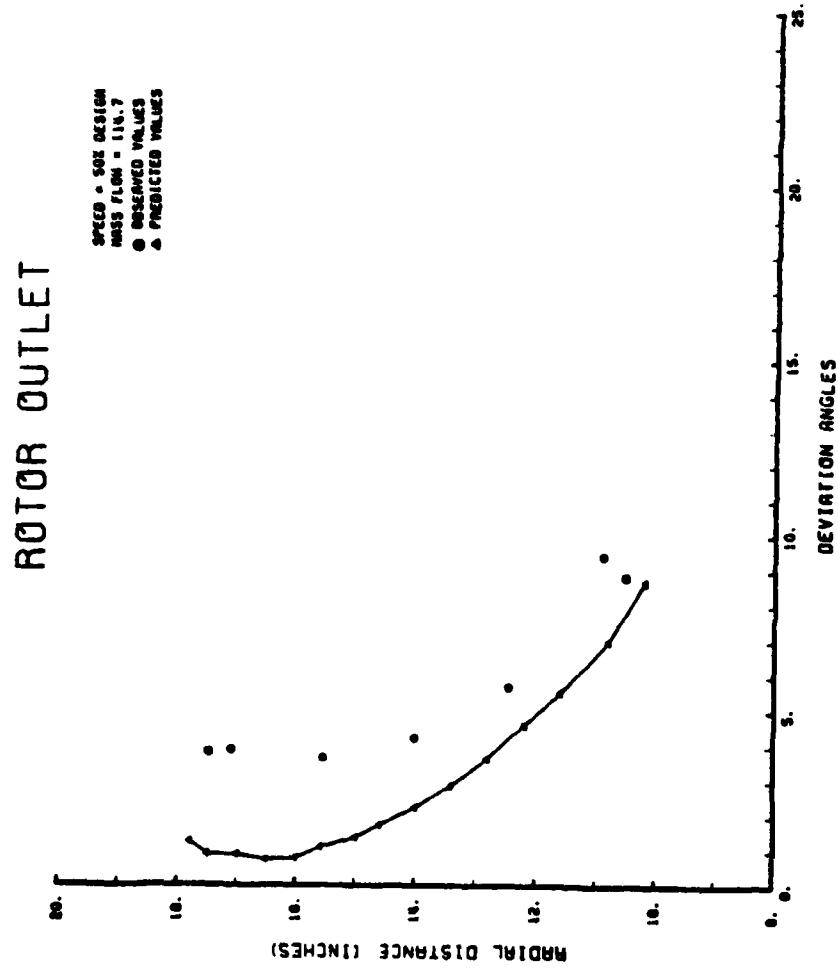


Figure 38. Deviation Flow Angles at Rotor Outlet, 50% Design

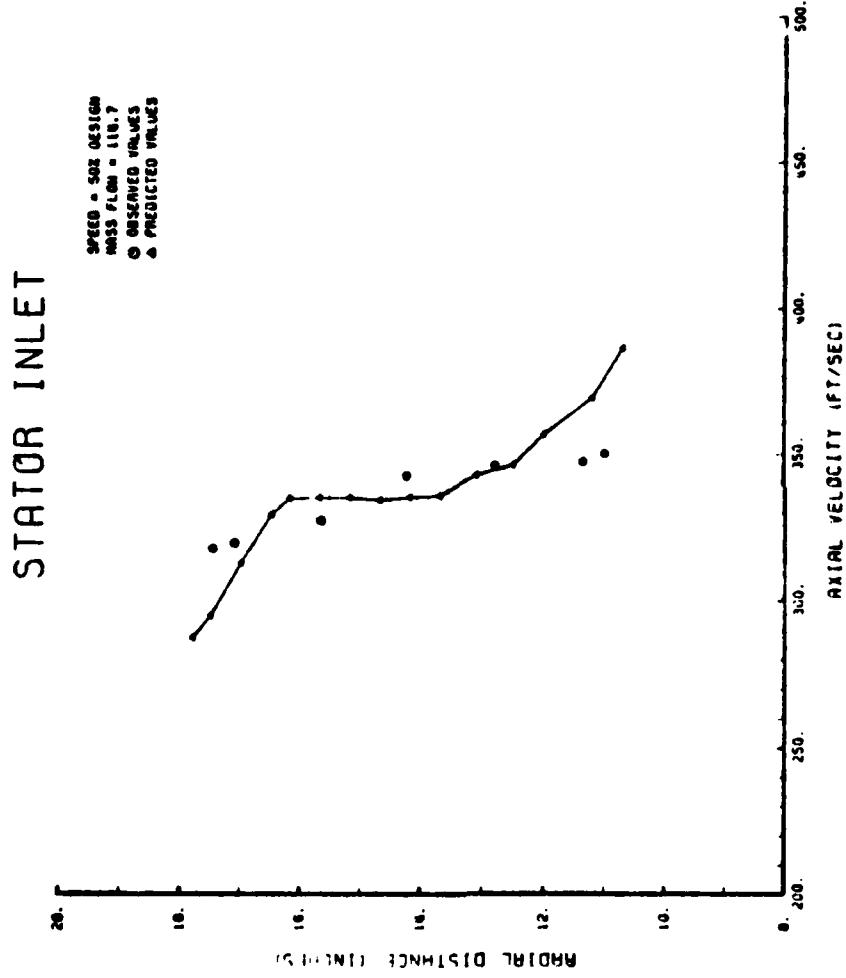


Figure 39. Axial Velocity at the Stator Inlet, 50% Design

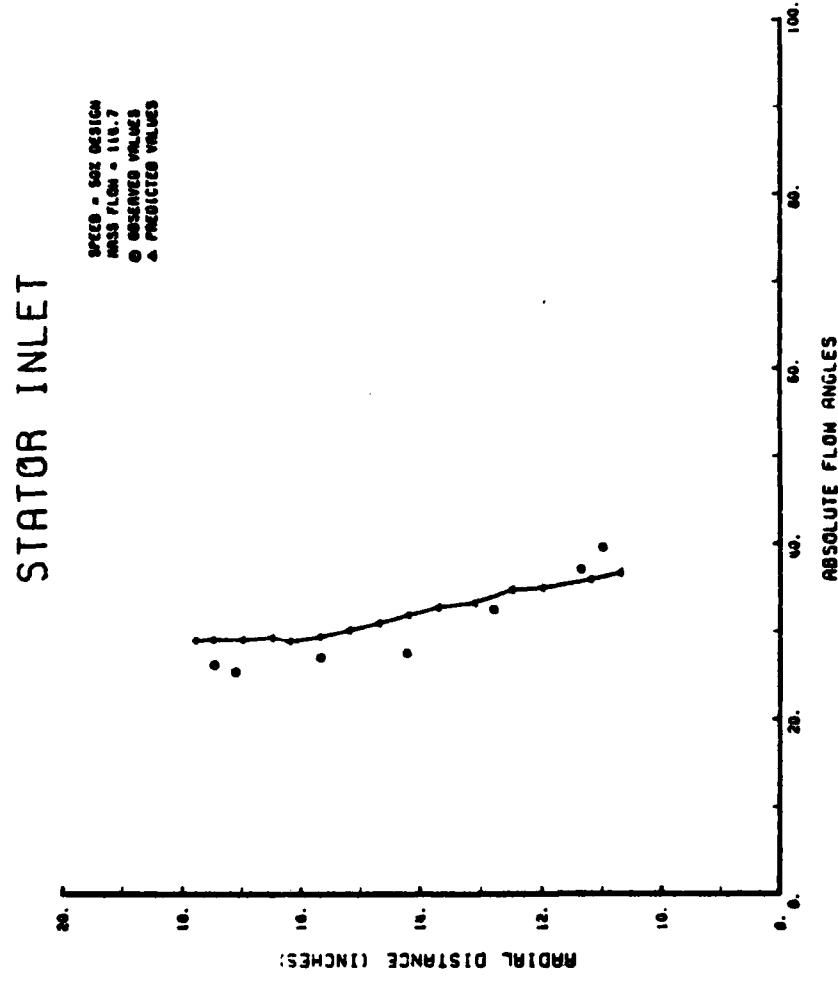


Figure 40. Absolute Flow Angles at Stator Inlet, 50% Design

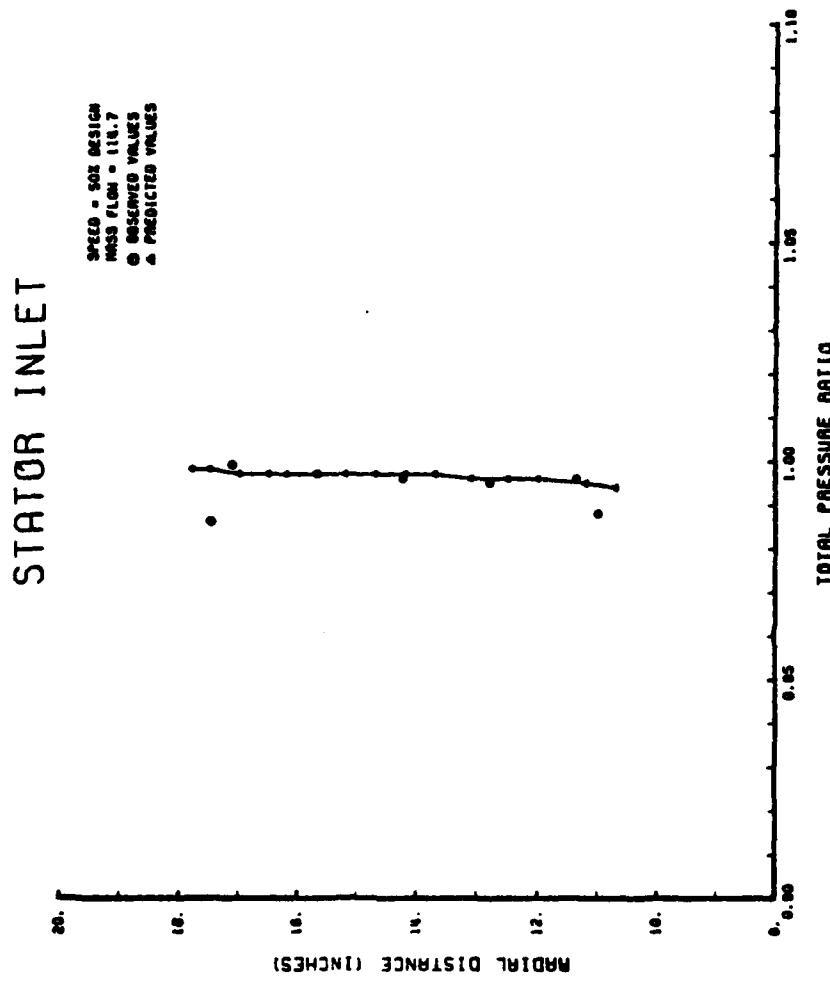


Figure 41. Total Pressure Ratio at Stator Inlet, 50% Design

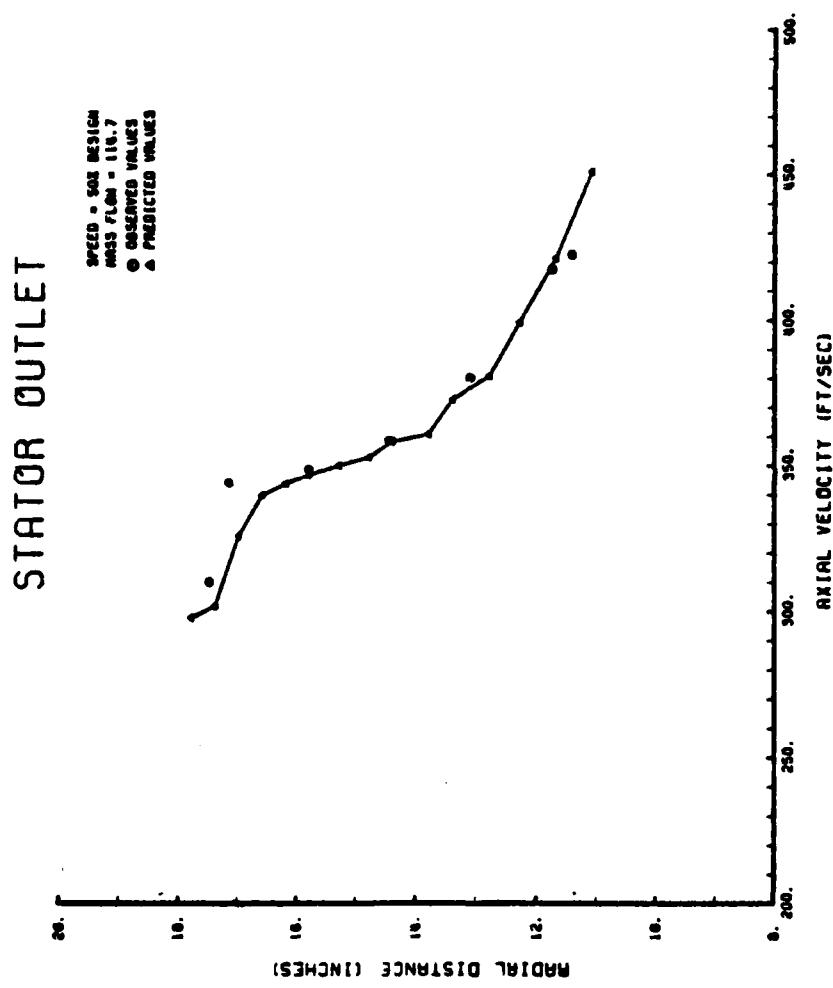


Figure 42. Axial Velocity at the Stator Outlet, 50% Design

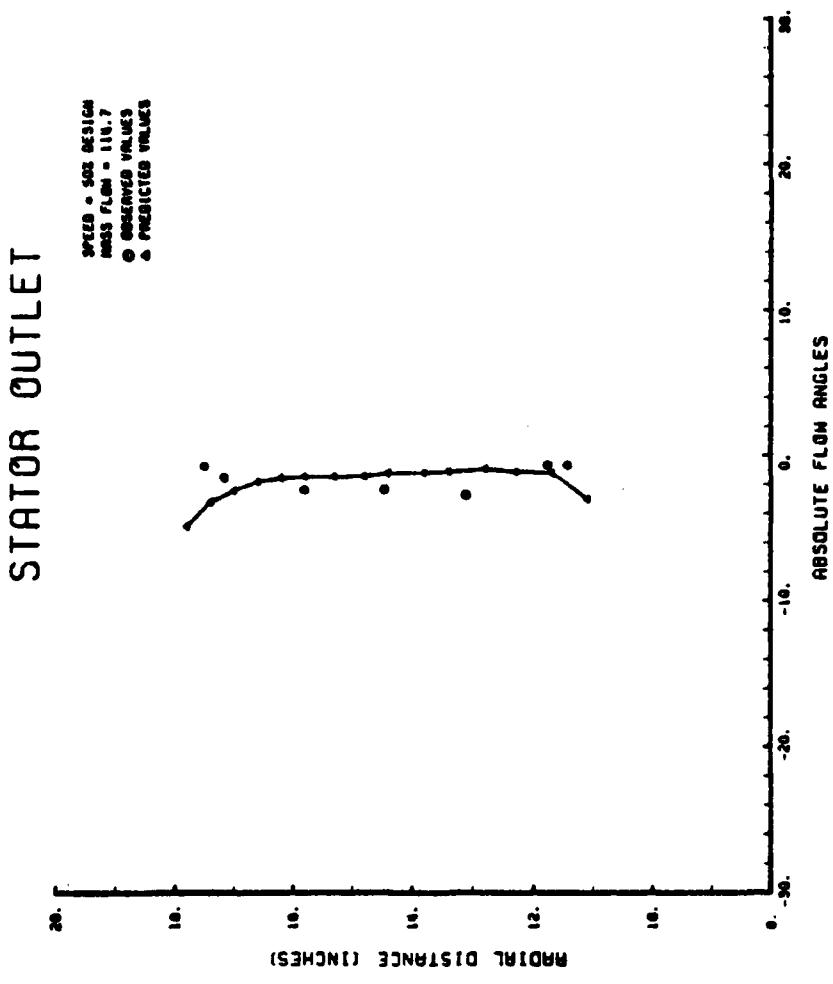


Figure 43. Absolute Flow Angles at Stator Outlet, 50% Design

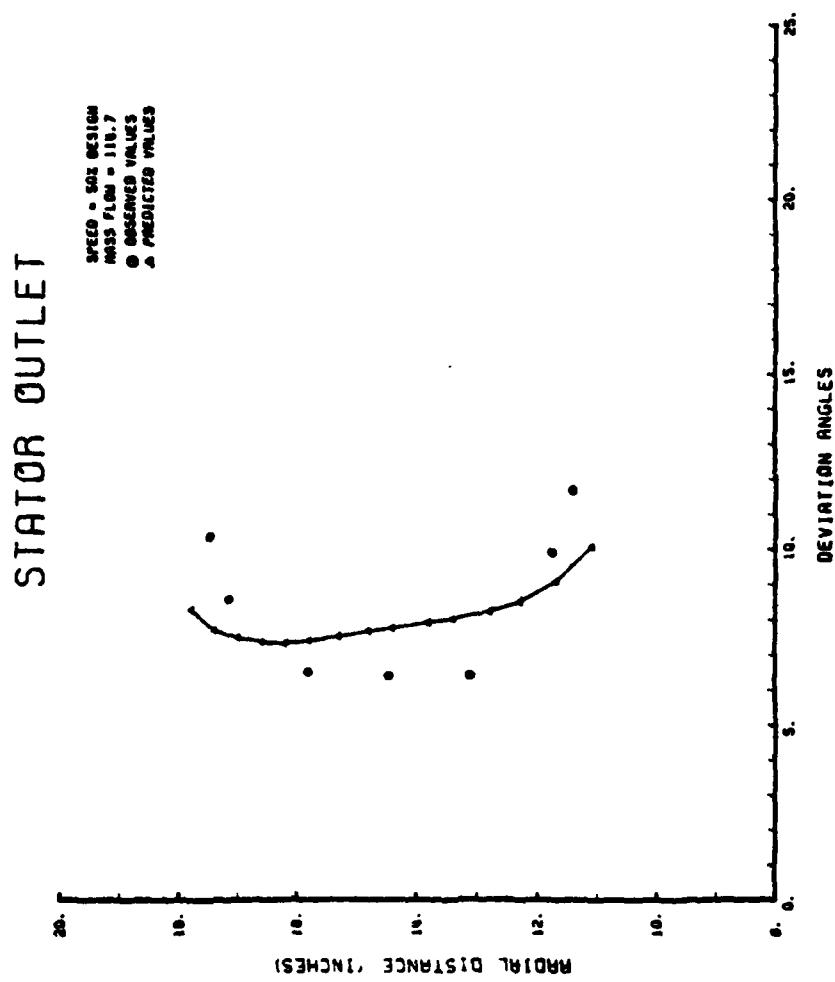


Figure 44. Deviation Flow Angles at Stator Outlet, 50% Design

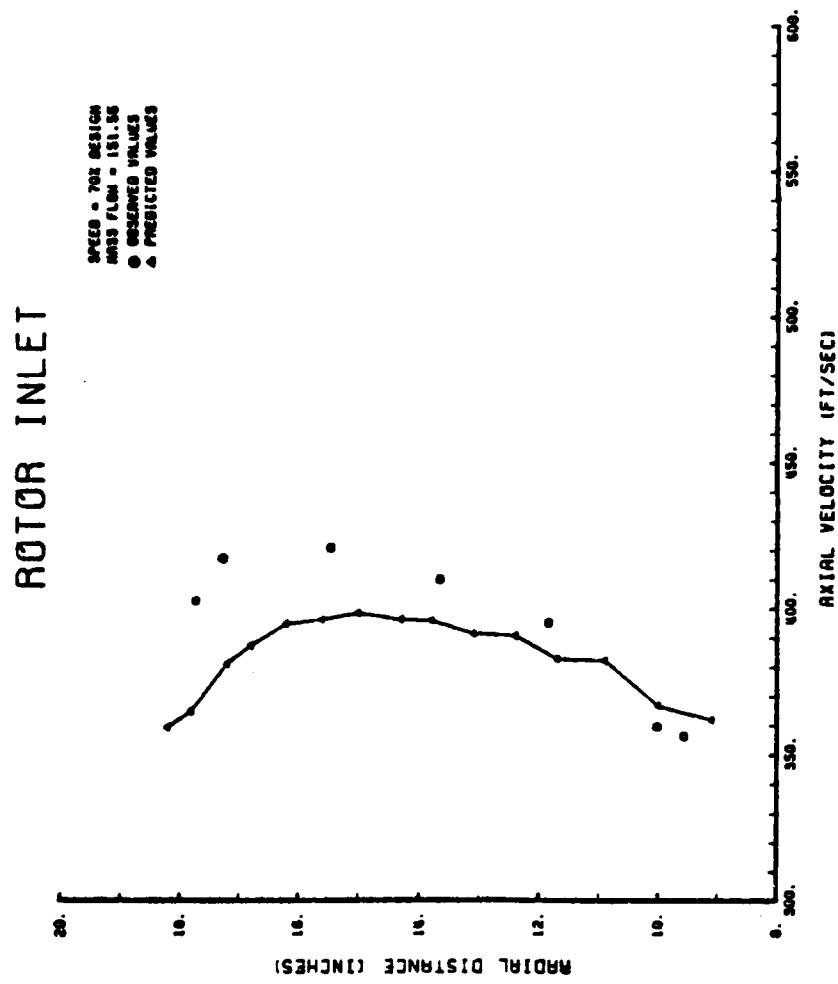


Figure 45. Axial Velocity at the Rotor Inlet, 70% Design

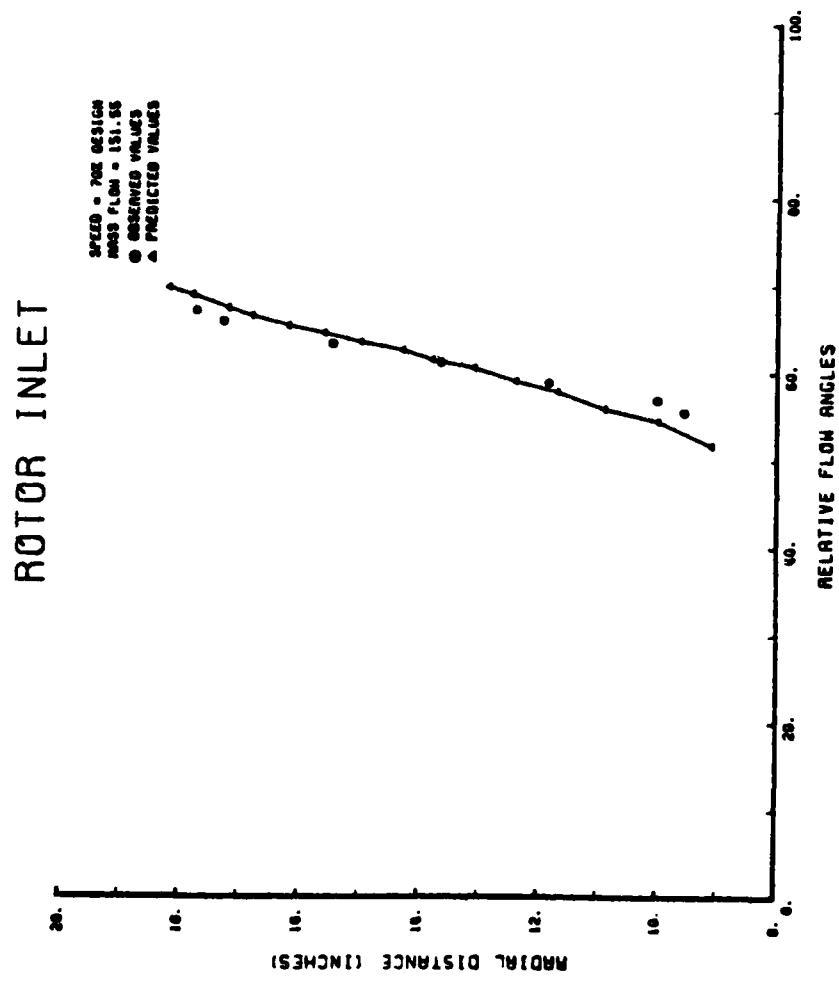


Figure 46. Relative Flow Angles at Rotor Inlet, 70% Design

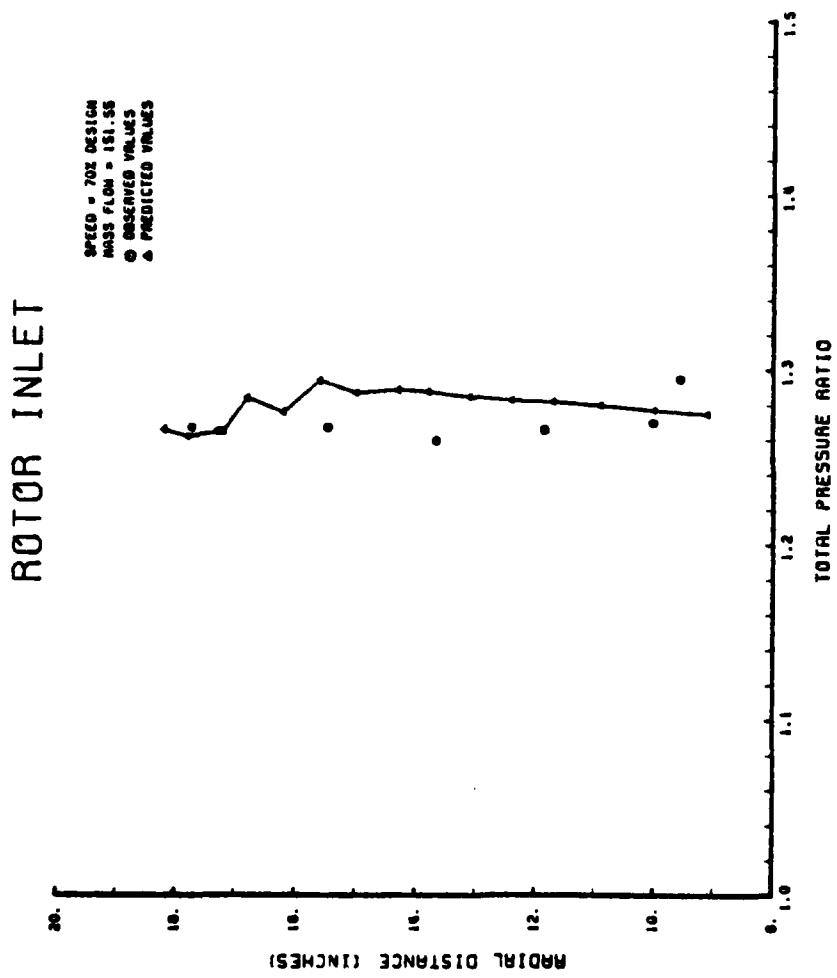


Figure 47. Total Pressure Ratio of the Rotor, 70% Design

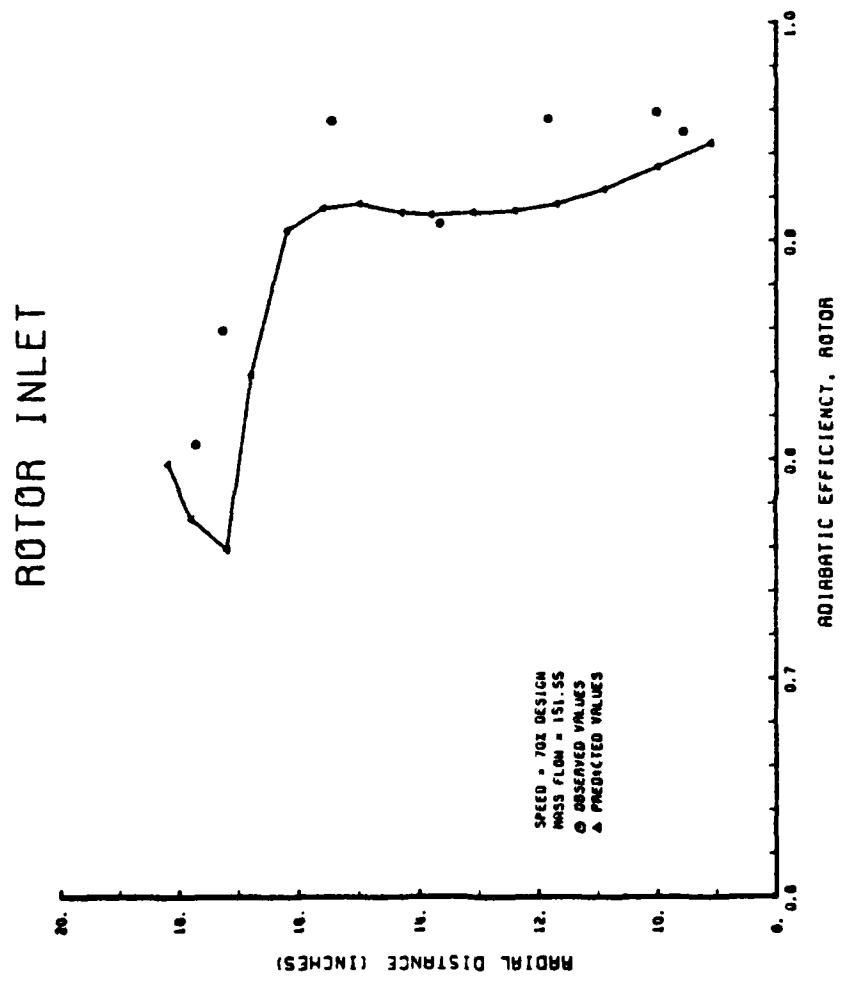


Figure 48. Adiabatic Efficiency of the Rotor, 70% Design

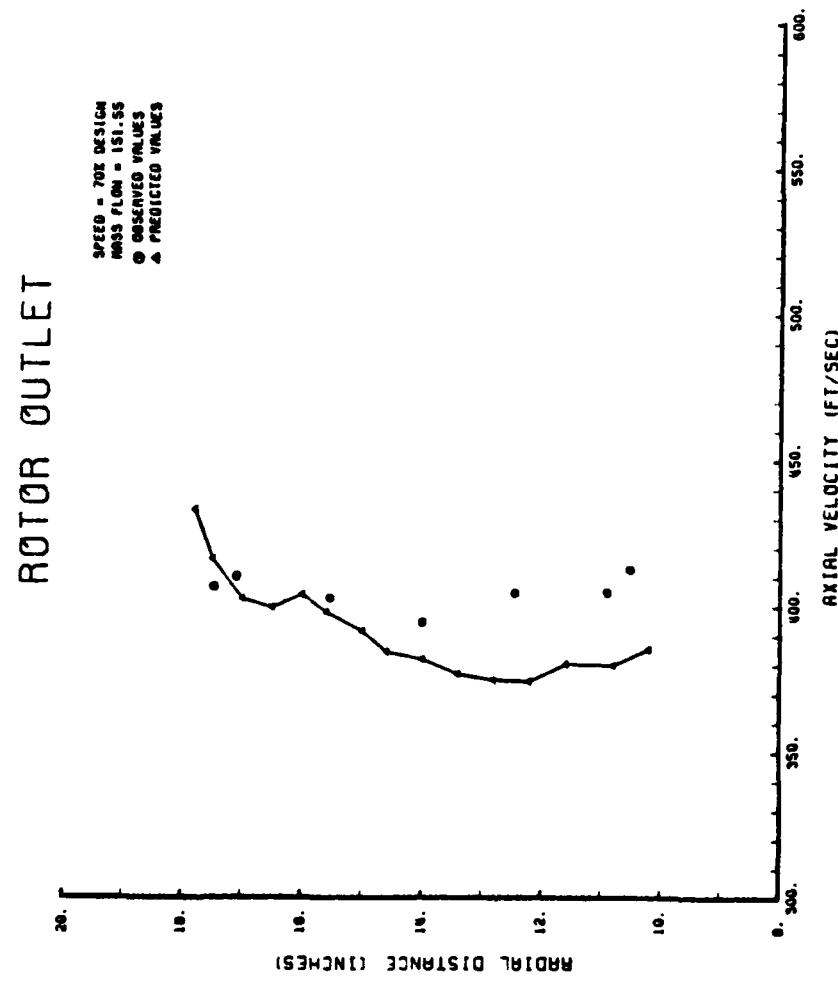


Figure 49. Axial Velocity at the Rotor Outlet, 70% Design

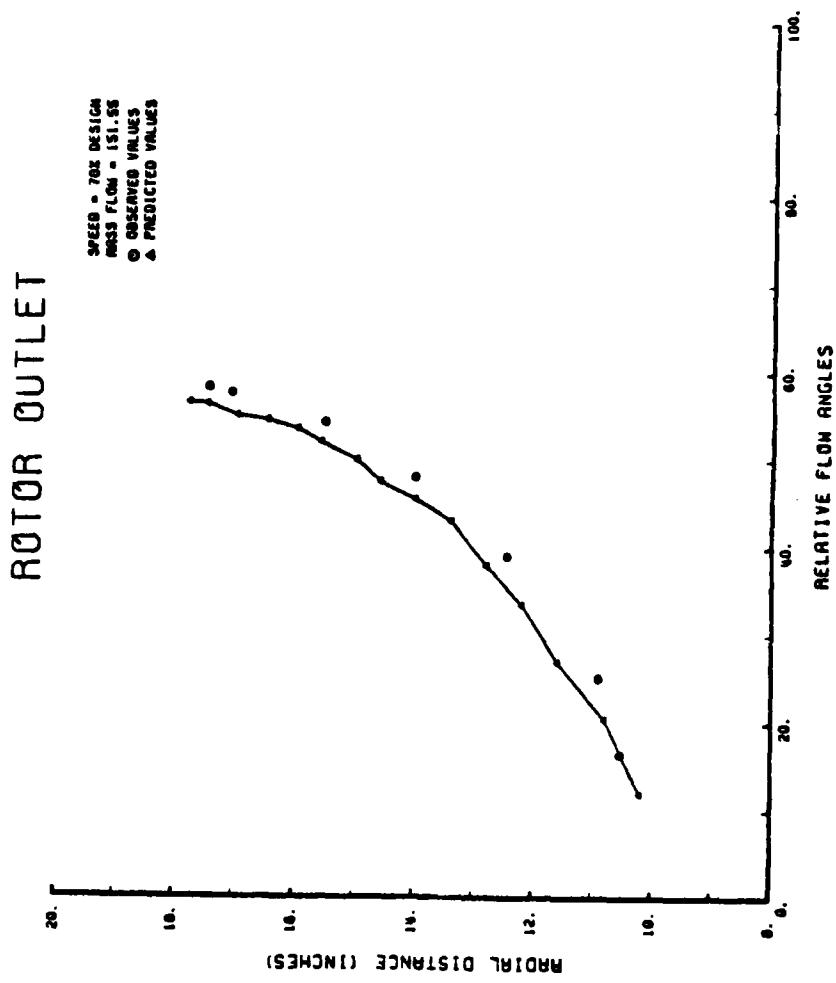


Figure 50. Relative Flow Angles at Rotor Outlet, 70% Design

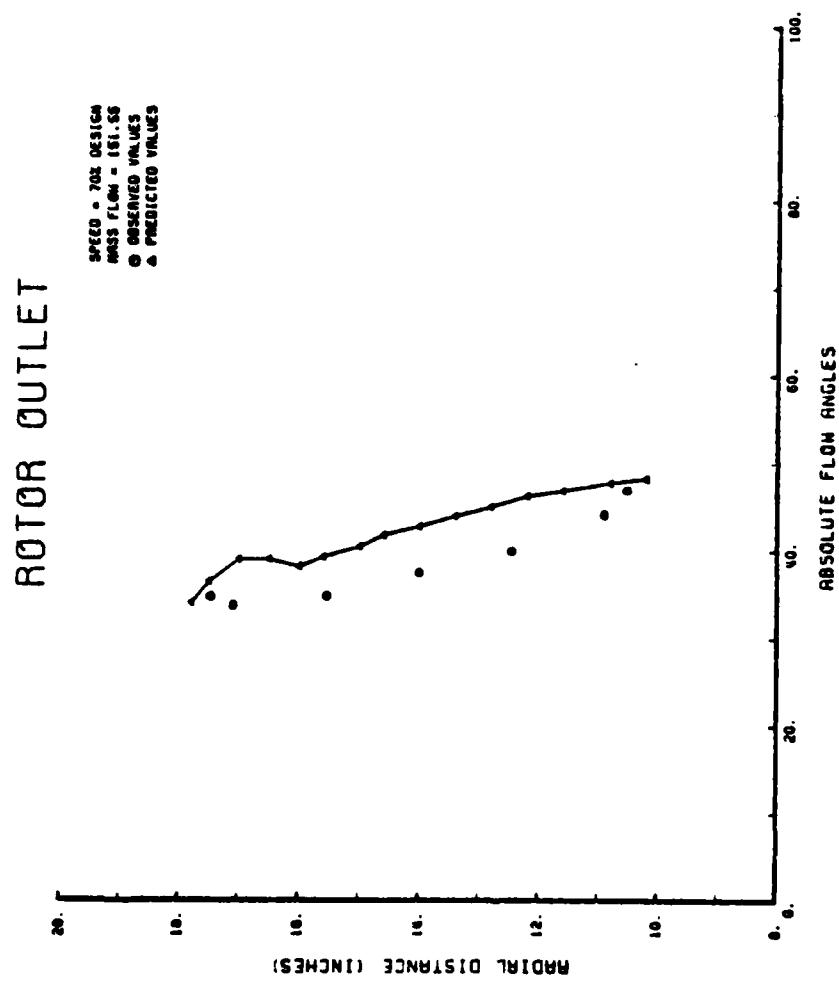


Figure 51. Absolute Flow Angles at Rotor Outlet, 70% Design

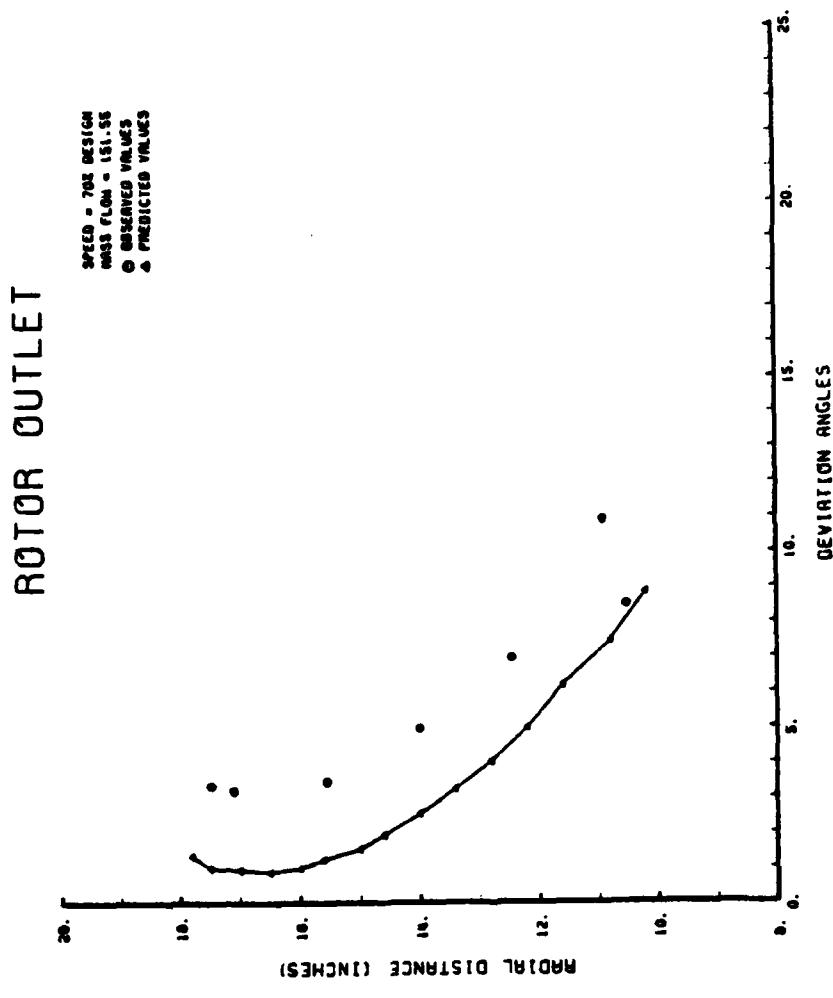


Figure 52. Deviation Flow Angles at Rotor Outlet, 70% Design

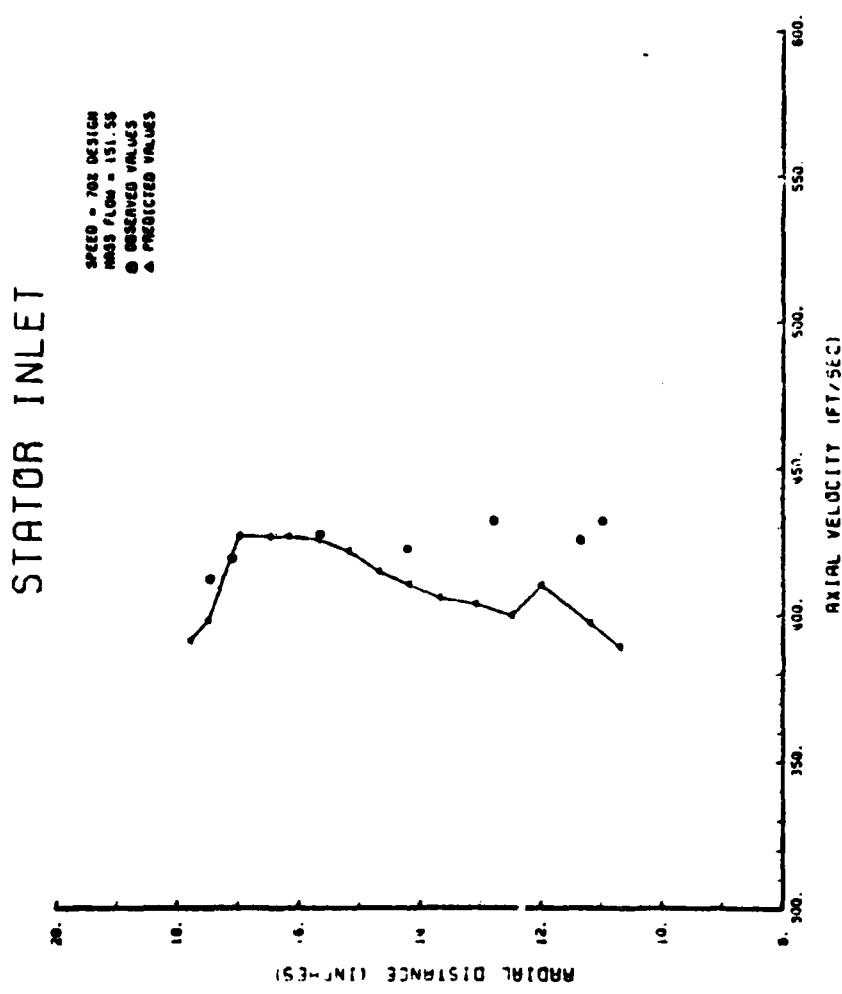


Figure 53. Axial Velocity at the Stator Inlet, 70% Design

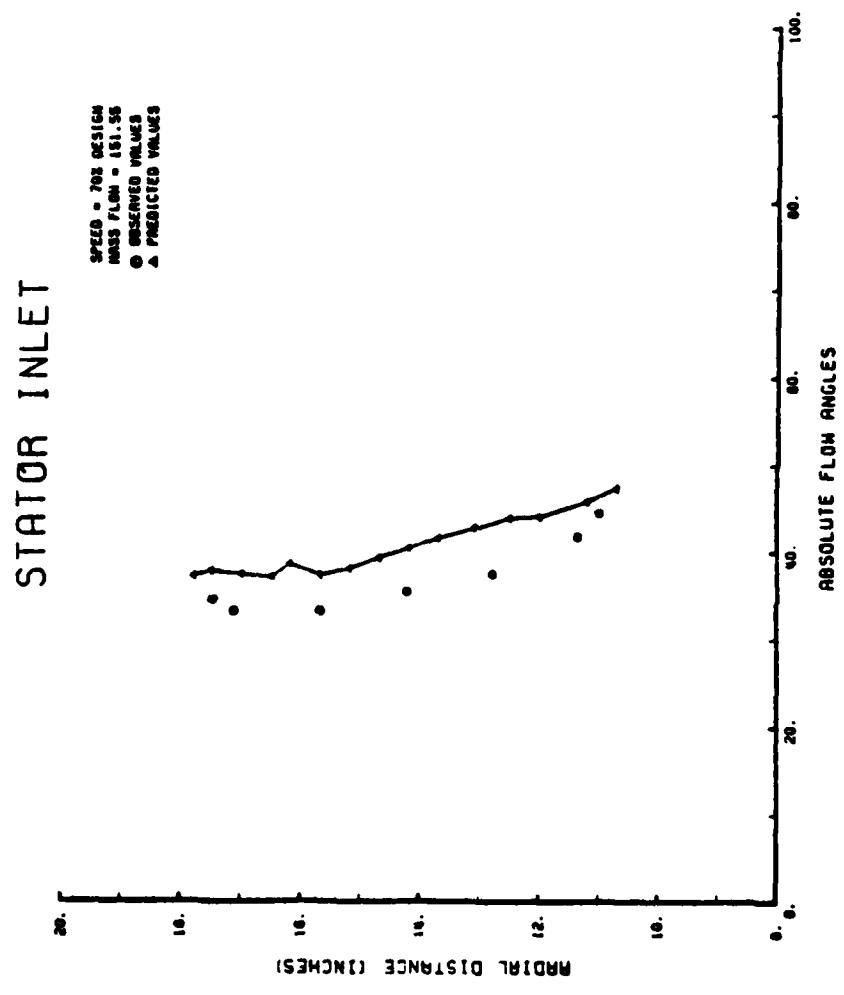


Figure 54. Absolute Flow Angles at Stator Inlet, 70% Design

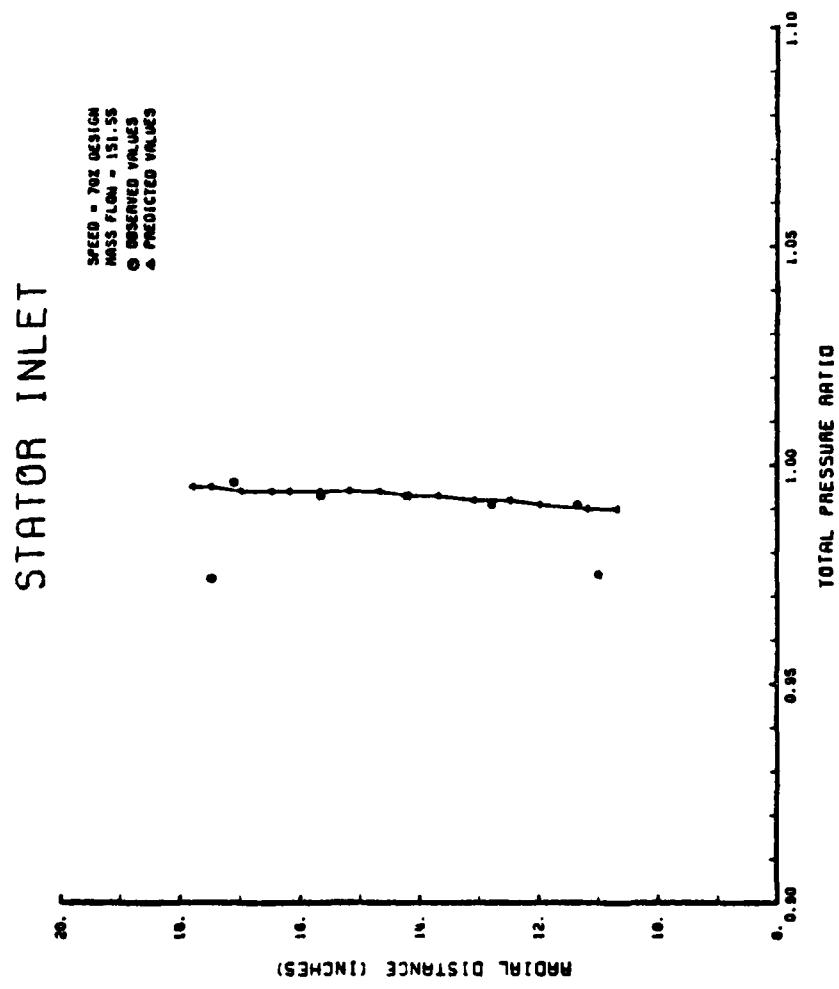


Figure 55. Total Pressure Ratio at Stator Inlet, 70% Design

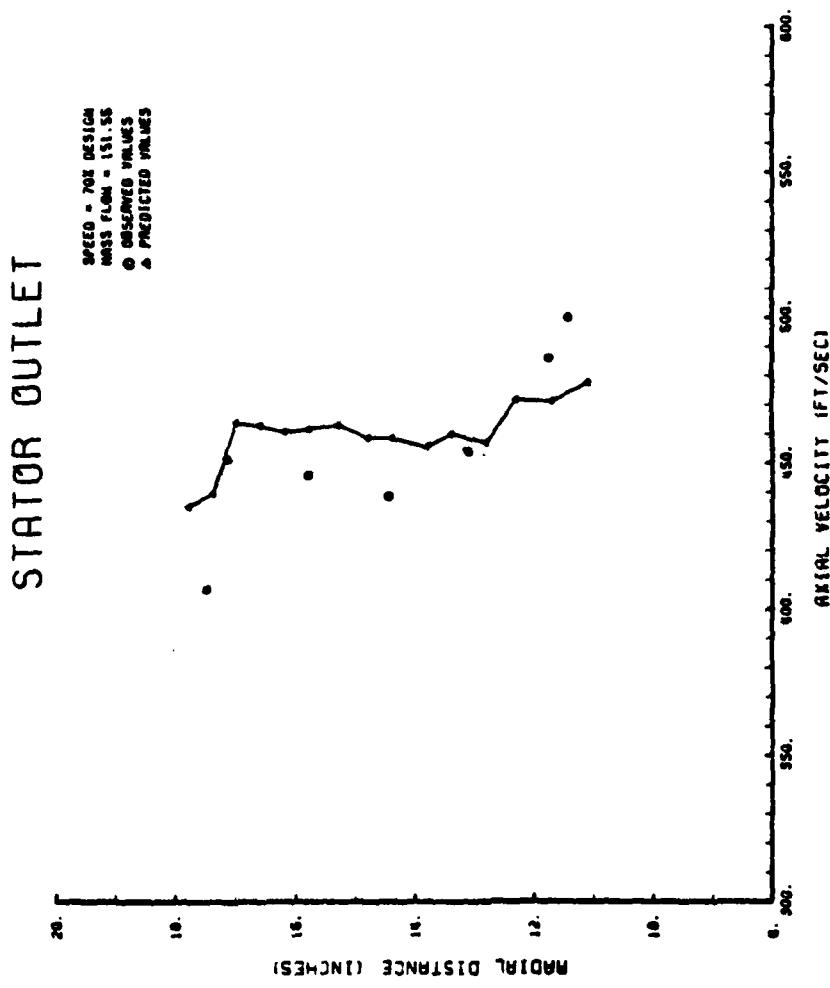


Figure 56. Axial Velocity at the Stator Outlet, 70% Design

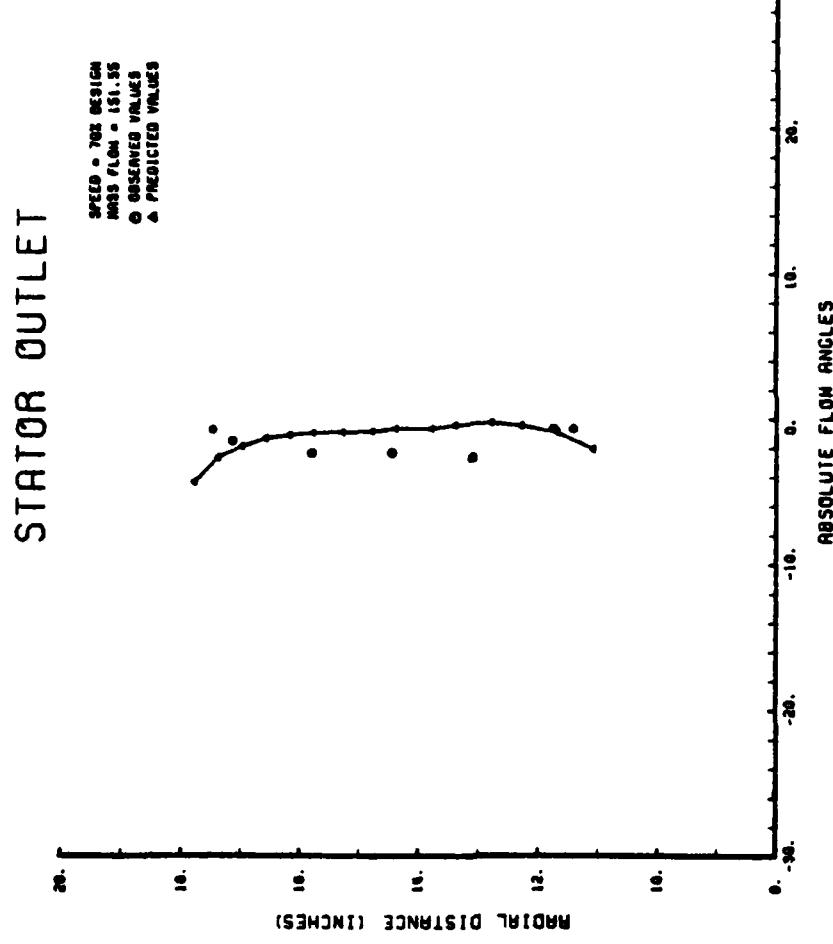


Figure 57. Absolute Flow Angles at Stator Outlet, 70° Design

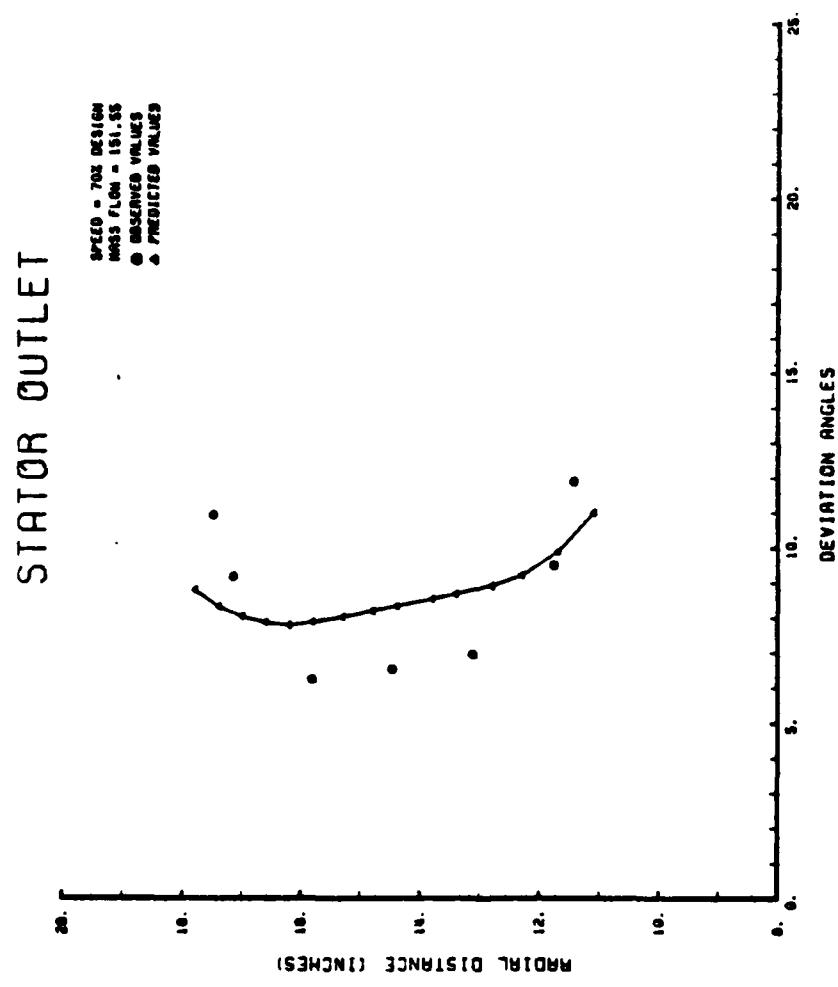


Figure 58. Deviation Flow Angles at Stator Outlet, 70% Design

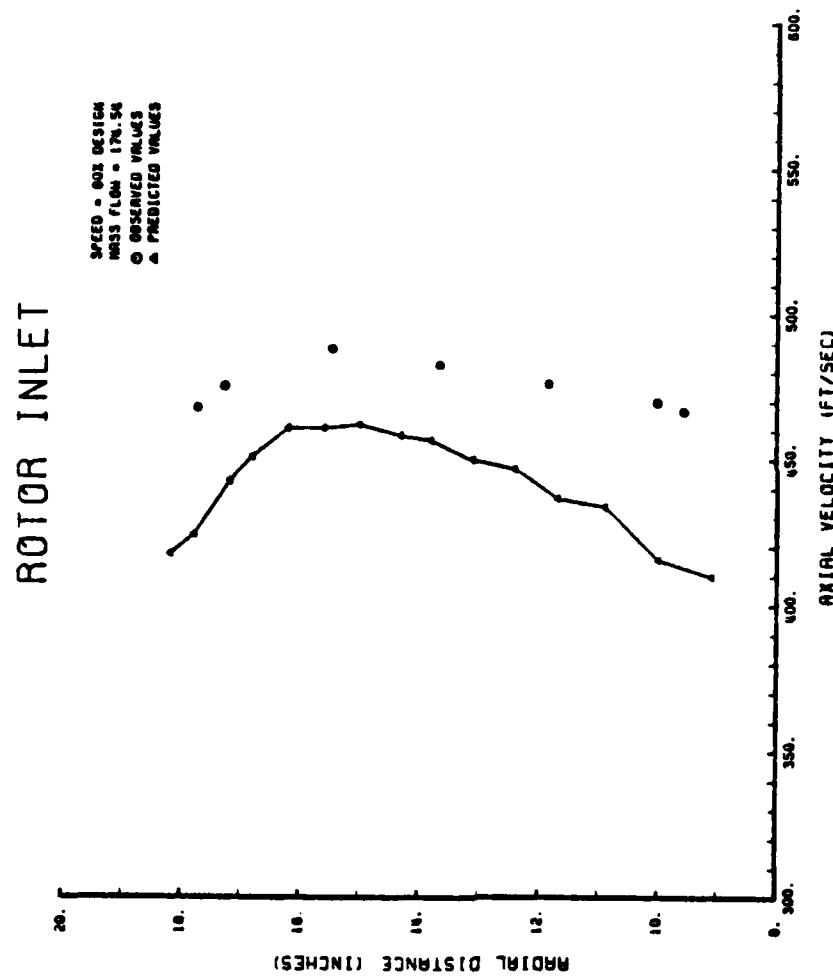


Figure 59. Axial Velocity at the Rotor Inlet, 80% Design

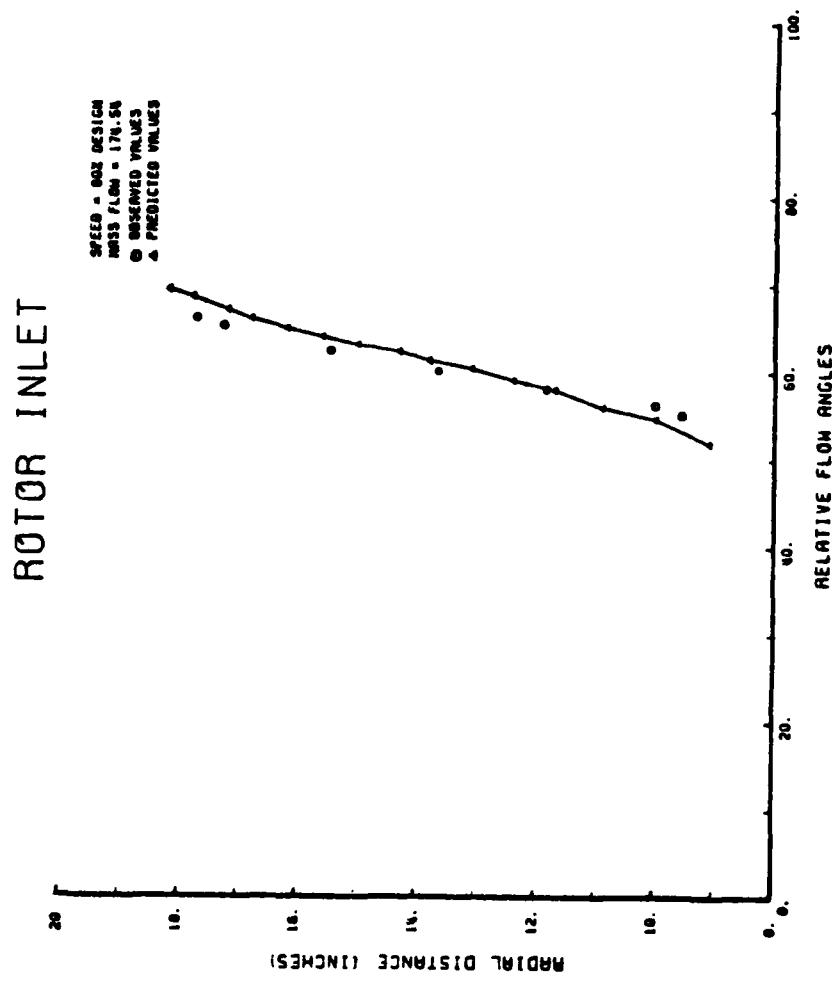


Figure 60. Relative Flow Angles at Rotor Inlet, 80% Design

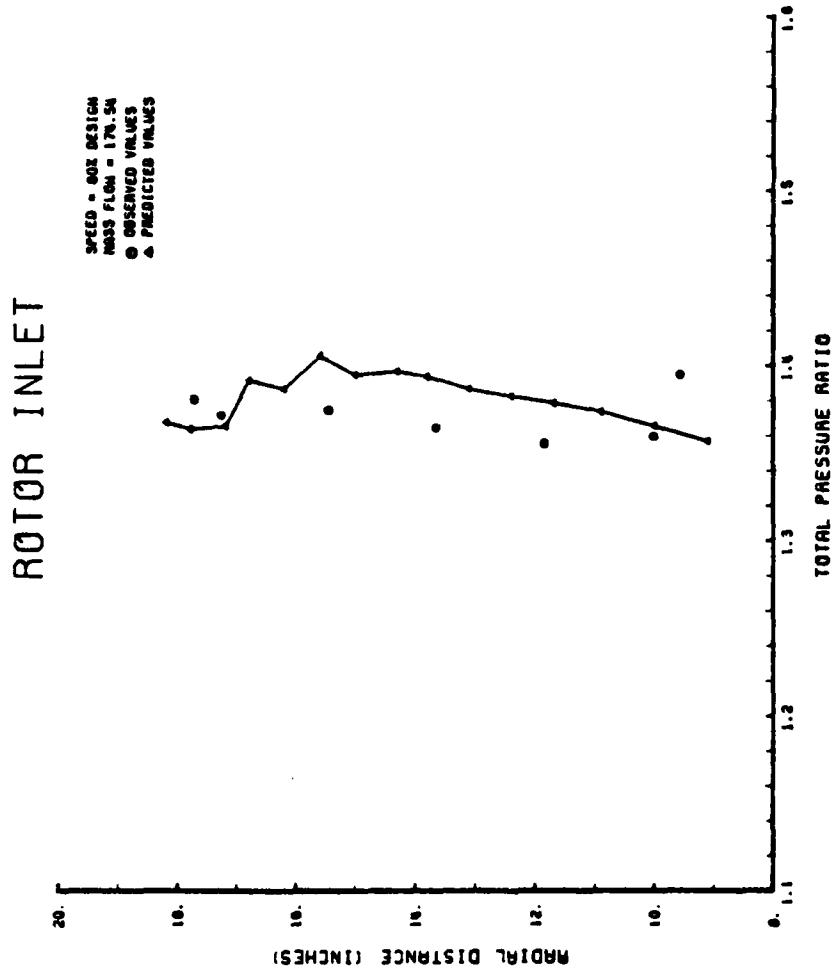


Figure 61. Total Pressure Ratio of the Rotor, 80% Design

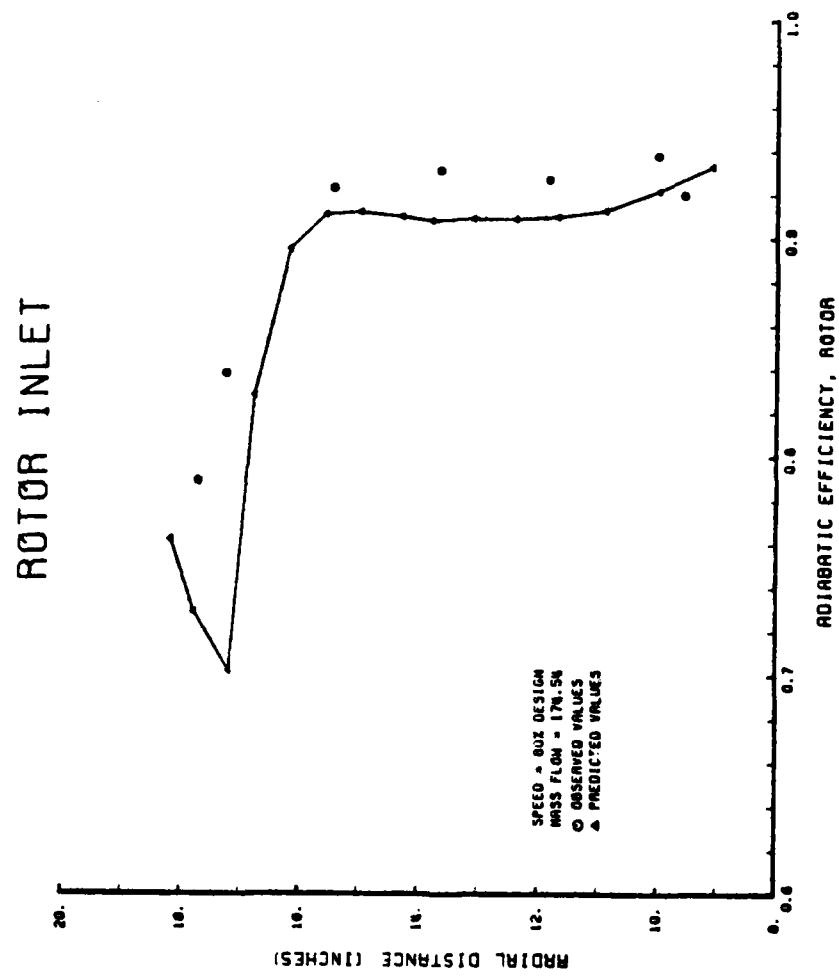


Figure 62. Adiabatic Efficiency of the Rotor, 80% Design

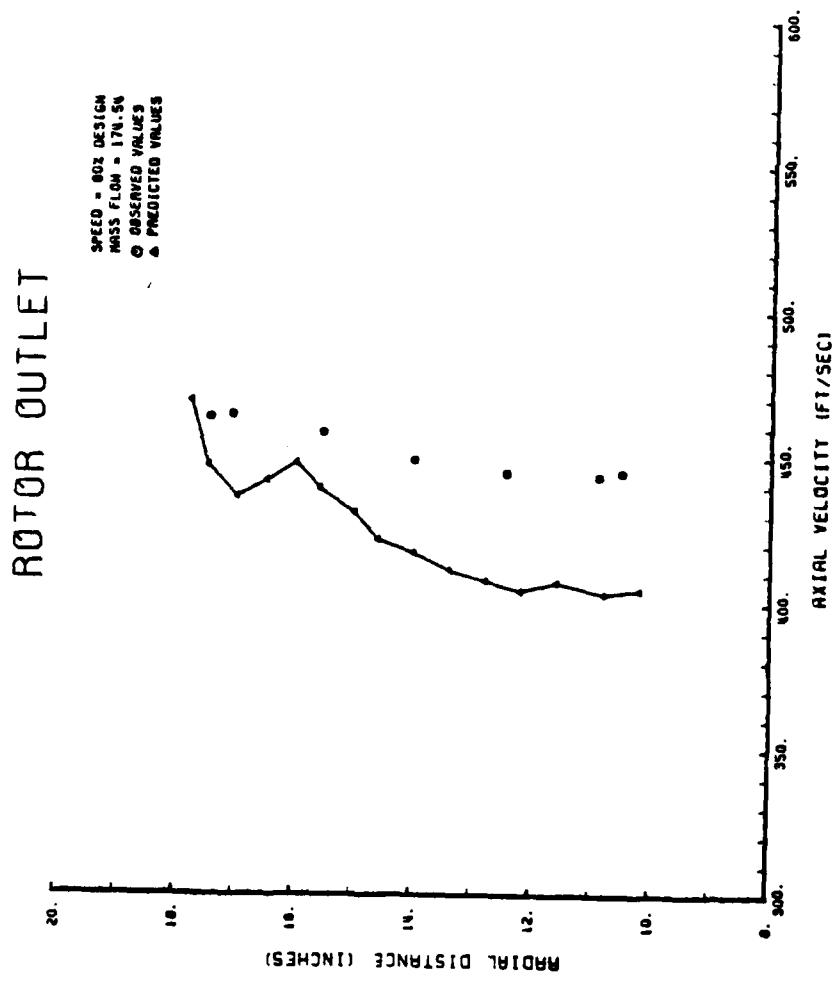


Figure 63. Axial Velocity at the Rotor Outlet, 80% Design

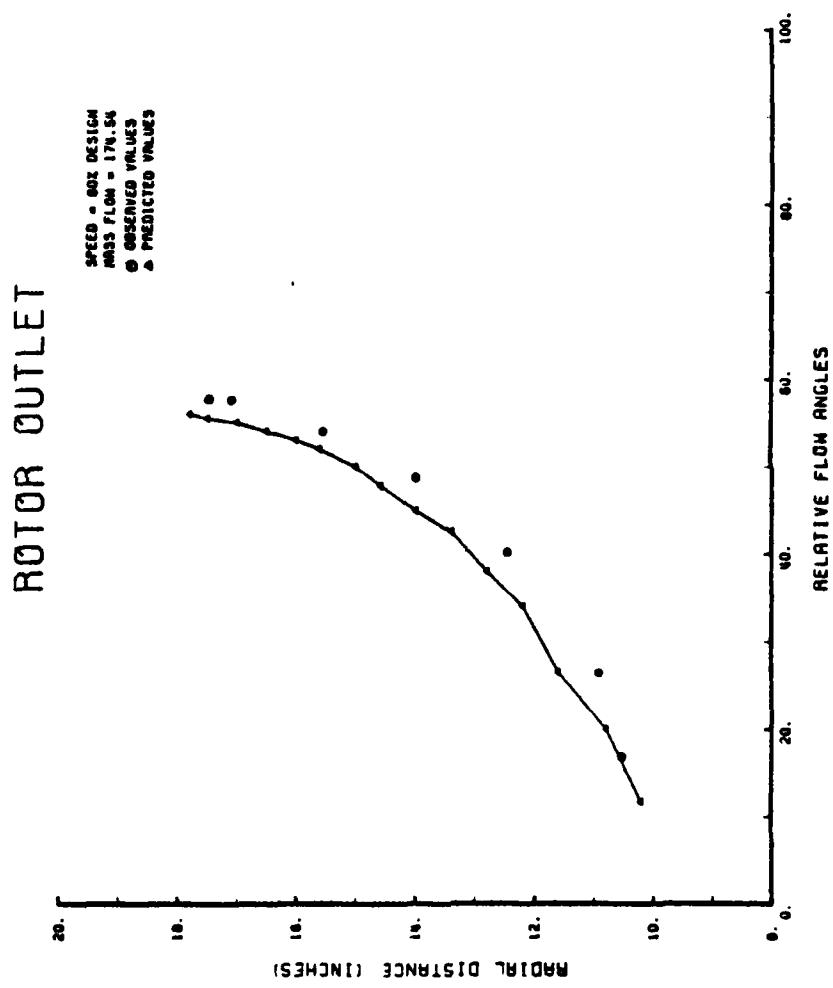


Figure 64. Relative Flow Angles at Rotor Outlet, 80% Design

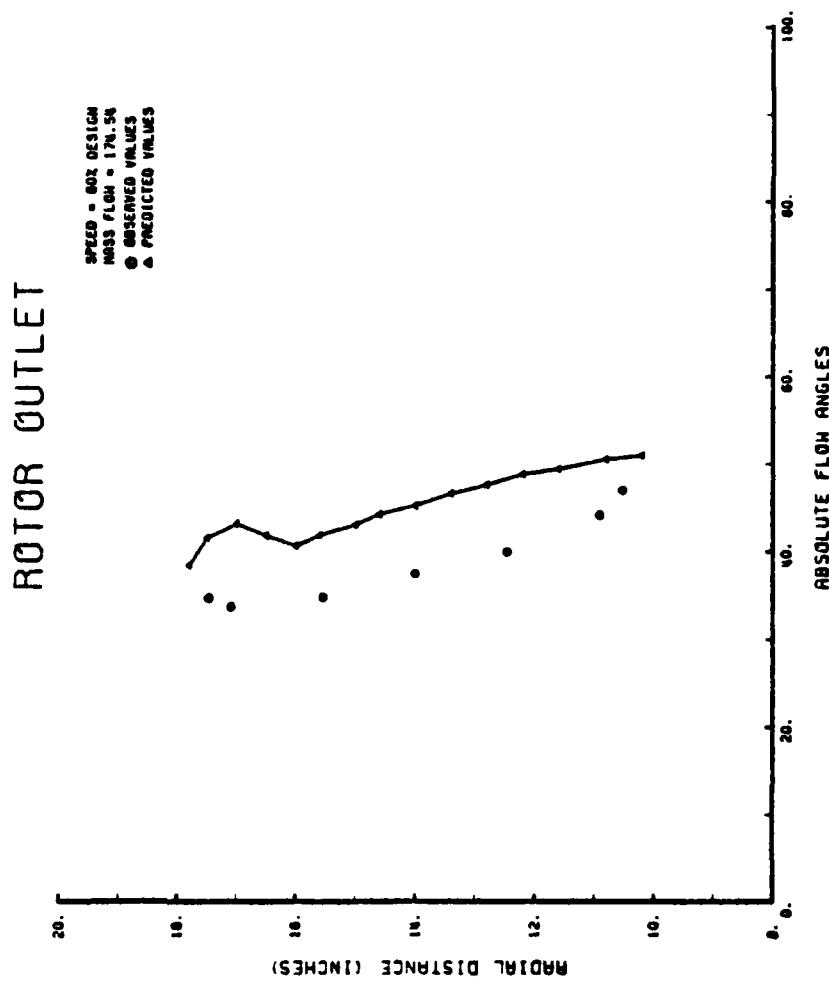


Figure 65. Absolute Flow Angles at Rotor Outlet, 80% Design

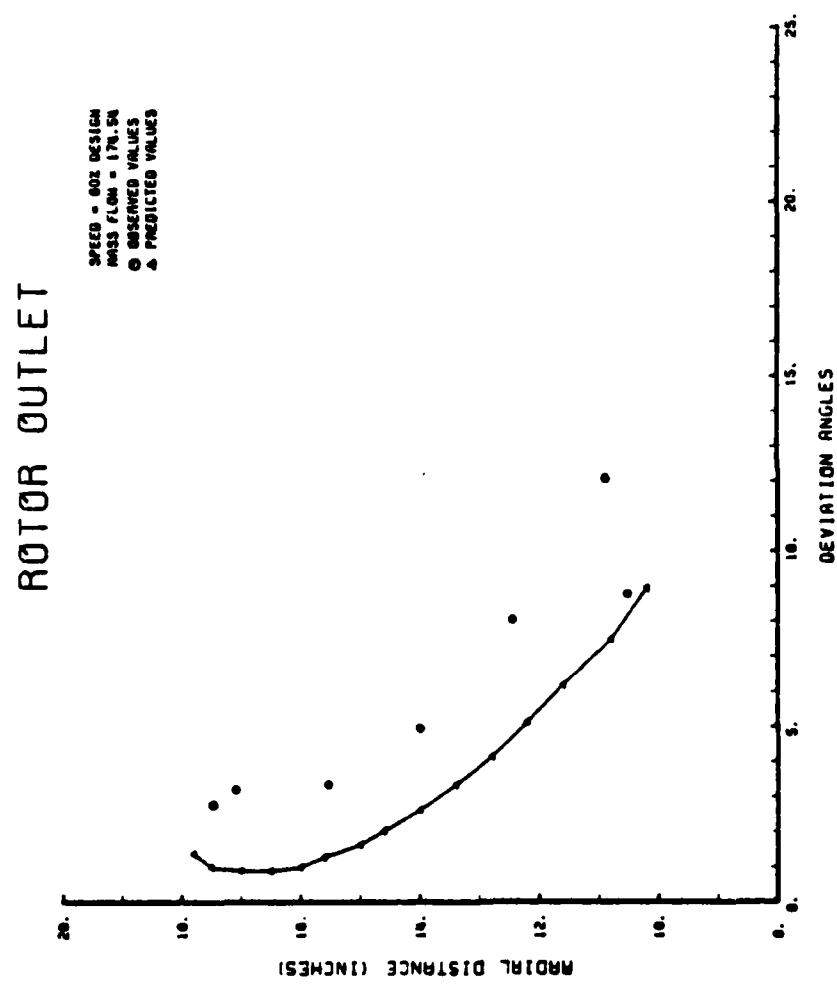


Figure 66. Deviation Flow Angles at Rotor Outlet, 80% Design

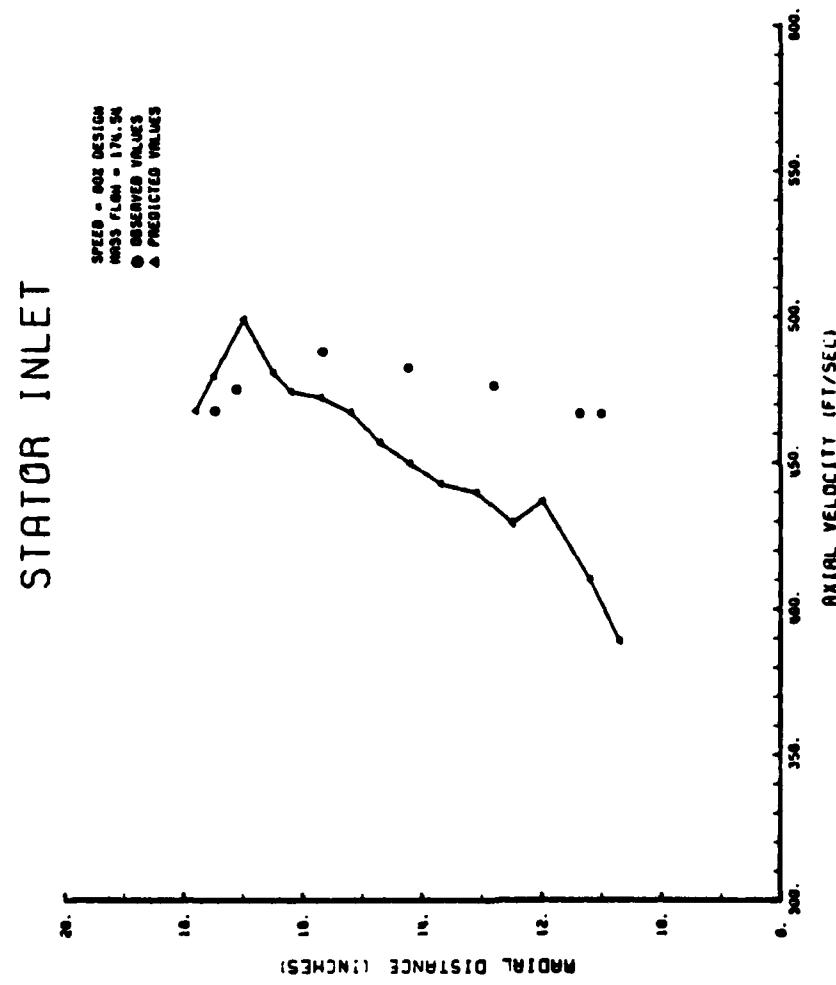


Figure 67. Axial velocity at the Stator Inlet, 80% Design

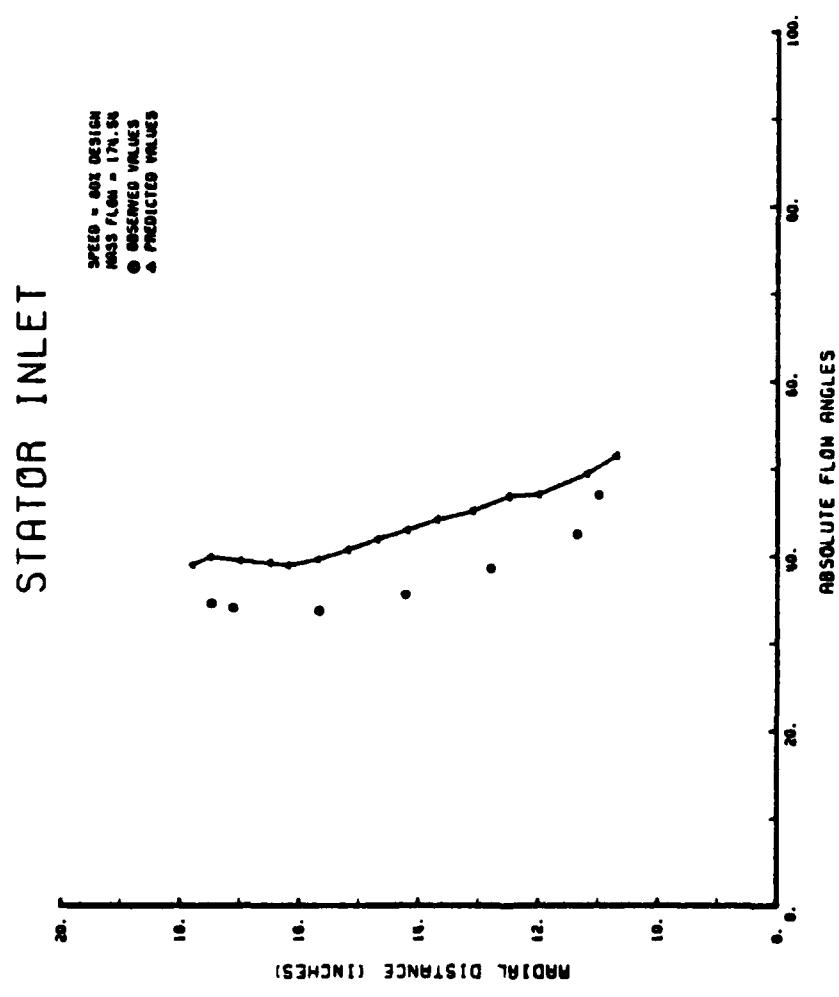


Figure 68. Absolute Flow Angles at Stator Inlet, 80% Design

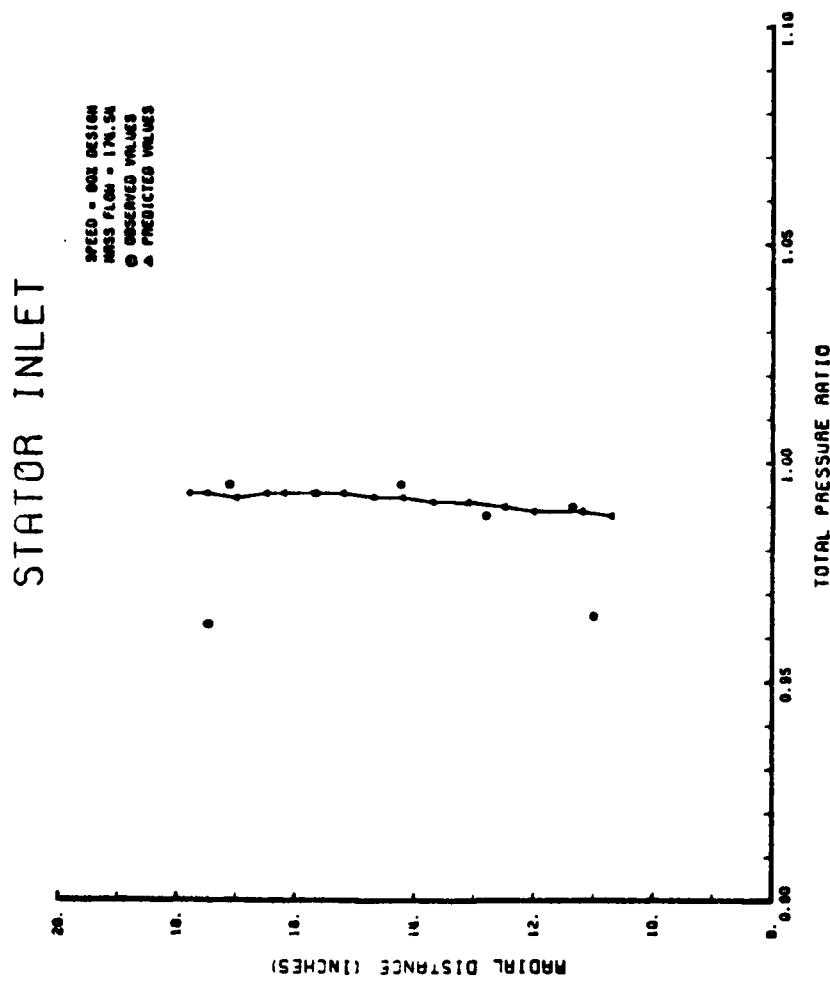


Figure 69. Total Pressure Ratio at Stator Inlet, 80% Design

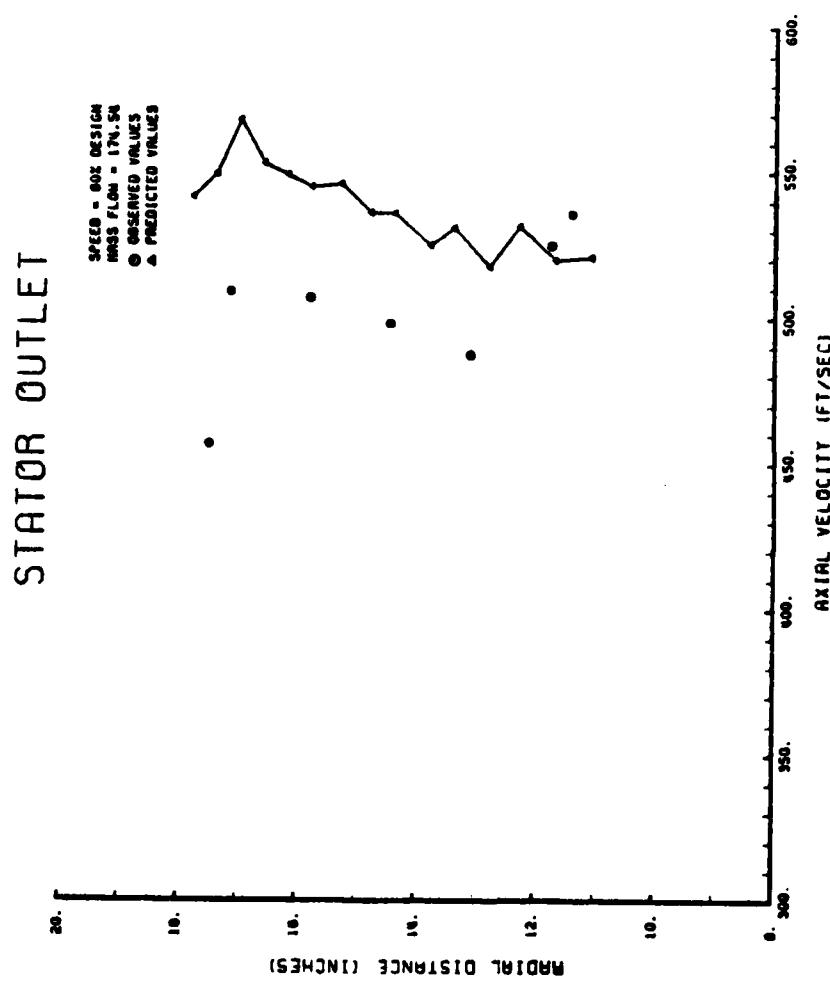


Figure 70. Axial Velocity at the Stator Outlet, 80% Design

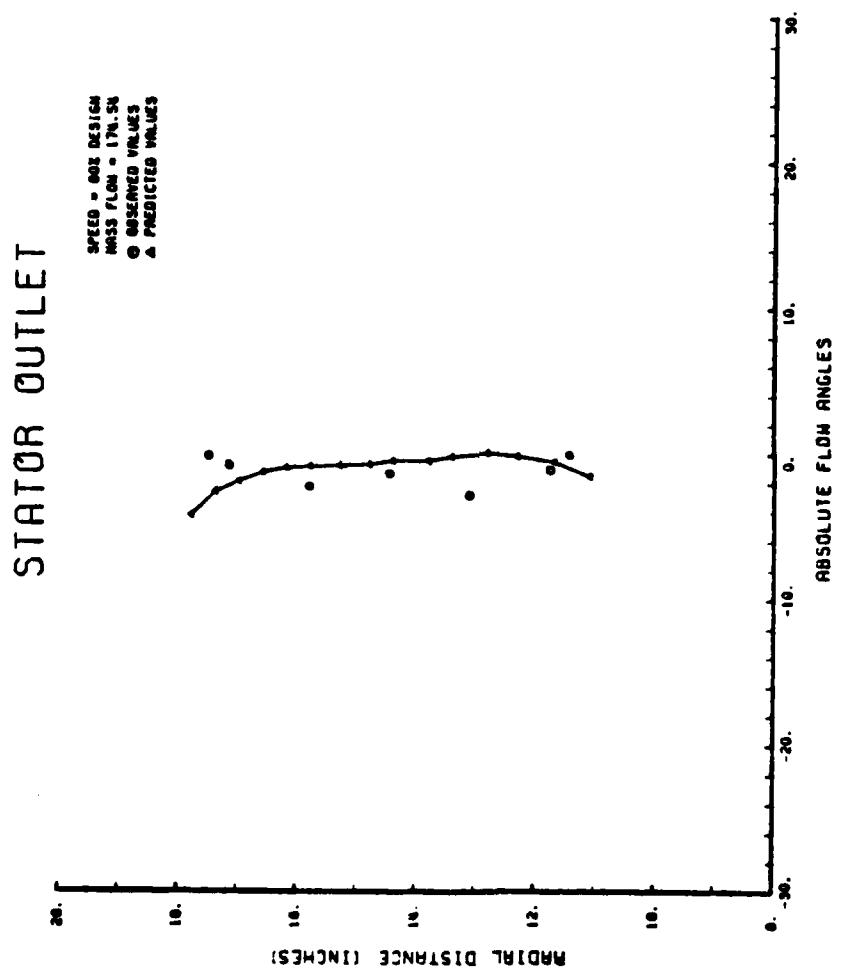


Figure 71. Absolute Flow Angles at Stator Outlet, 80% Design

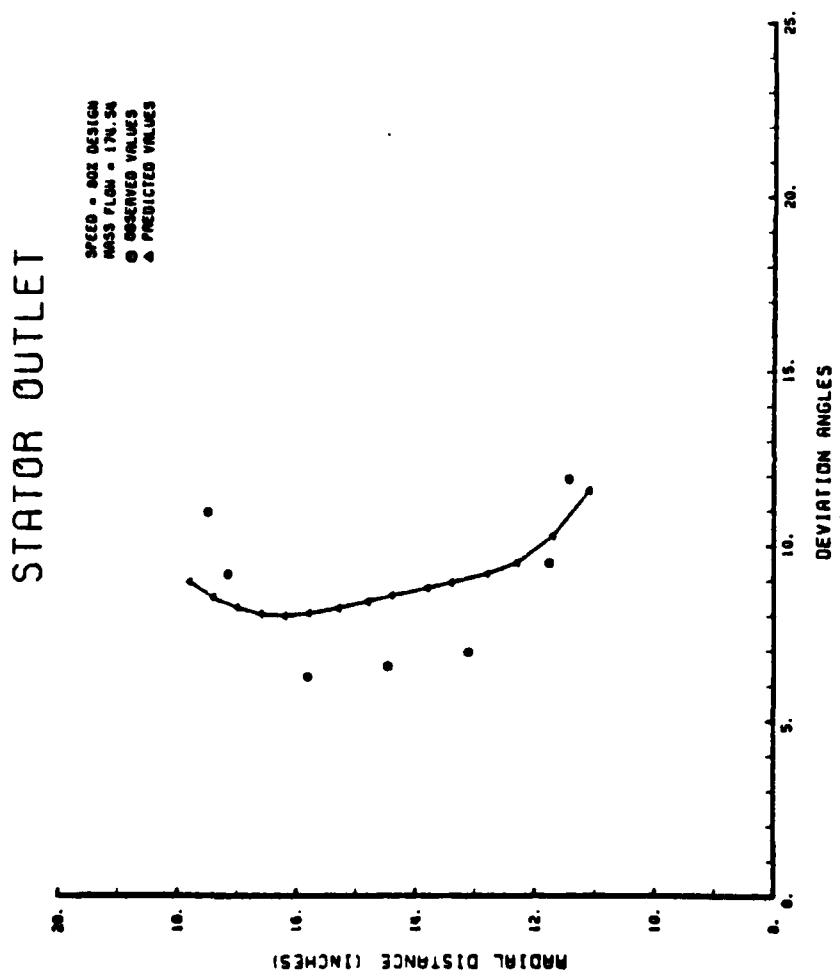


Figure 72. Deviation Flow Angles at Stator Outlet, 80% Design

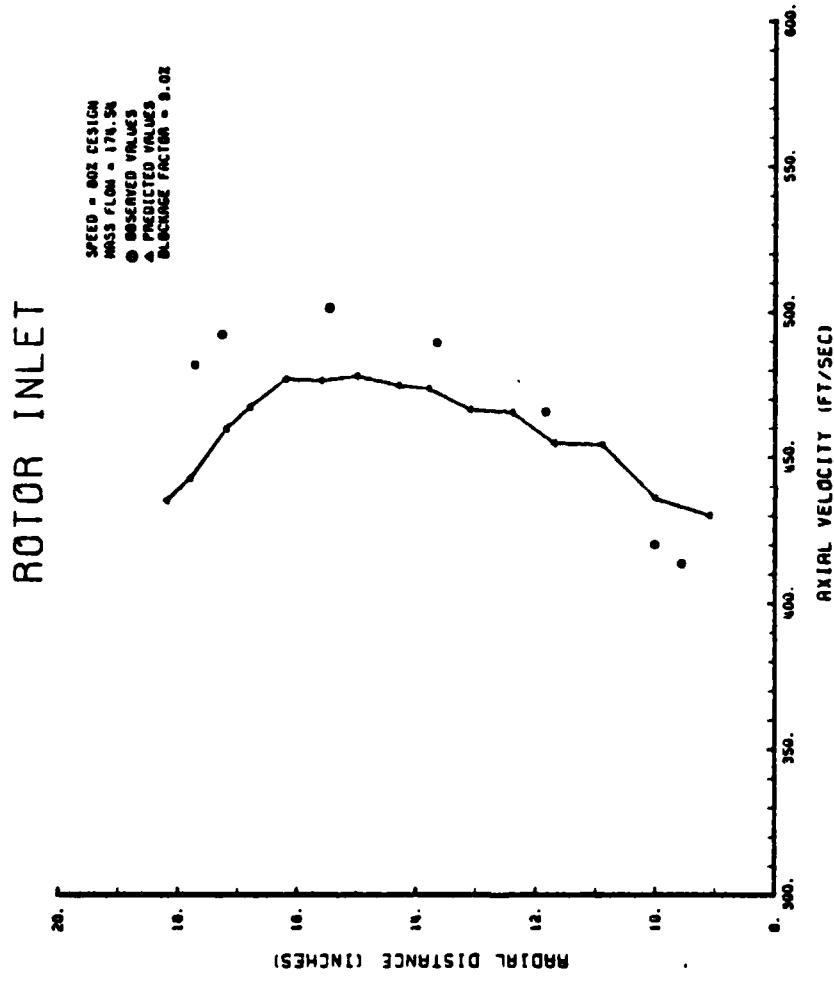


Figure 73. Axial Velocity at the Rotor Inlet, 80% Design, 9% Blockage

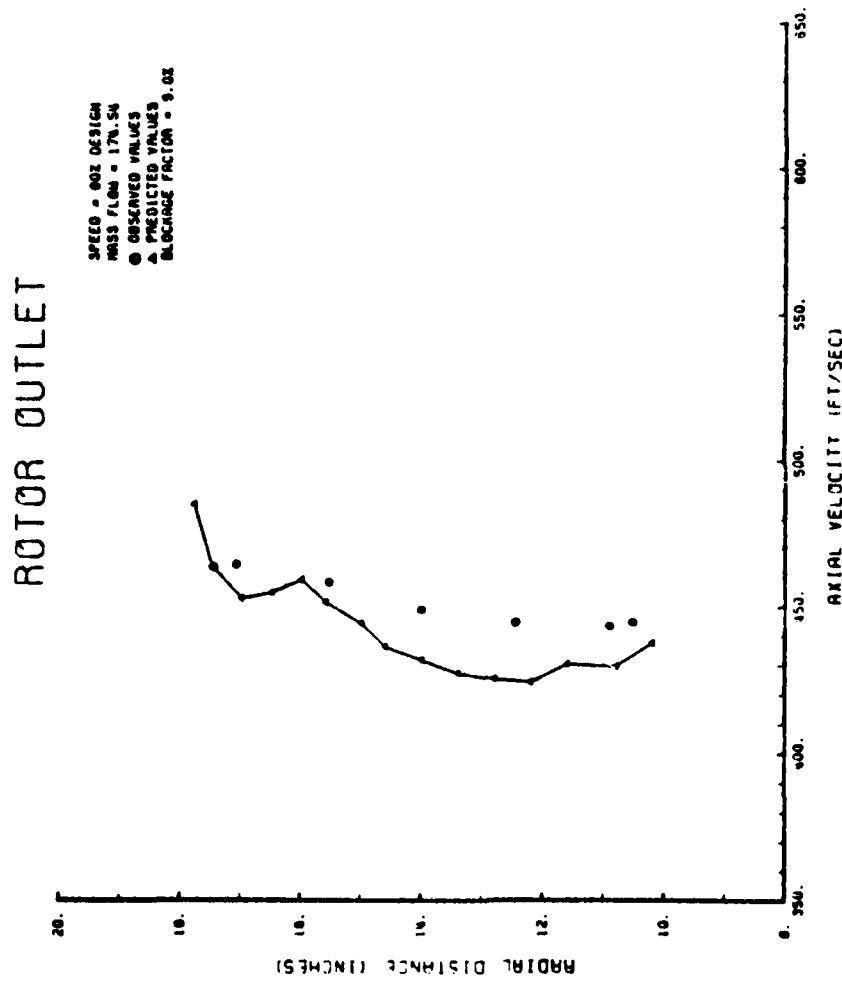


Figure 74. Axial Velocity at the Rotor Outlet, 80% Design, 9% Blockage

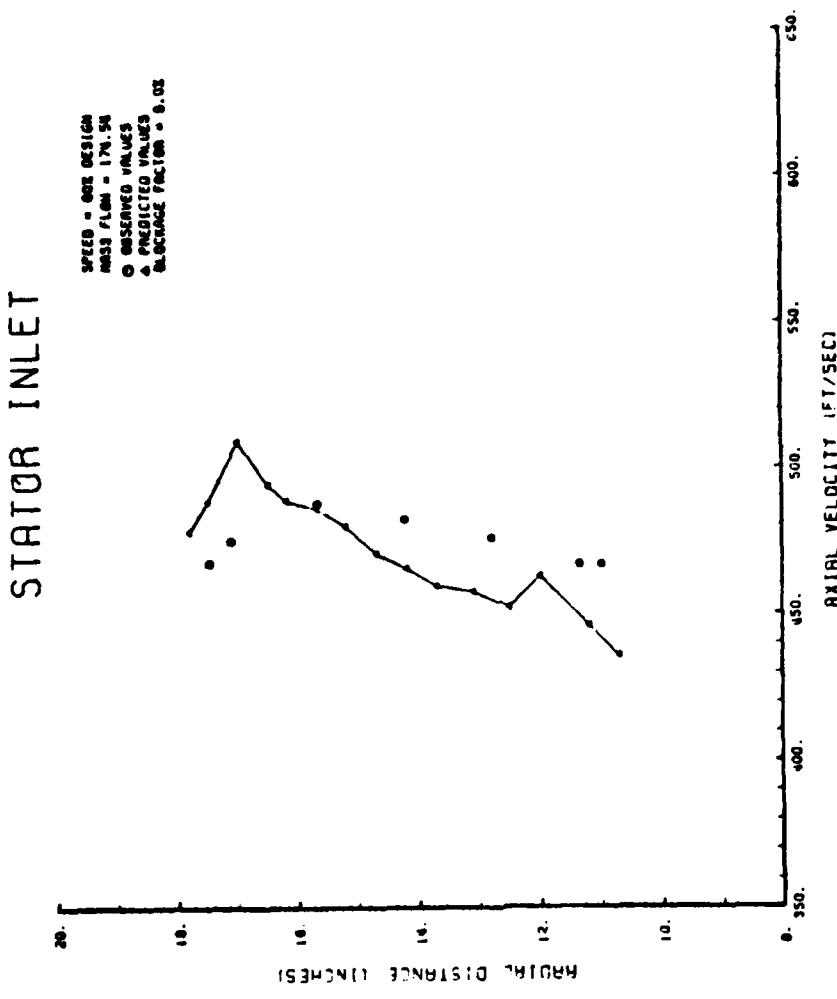


Figure 75. Axial Velocity at the Stator Inlet, 80% Design, 9% Blockage

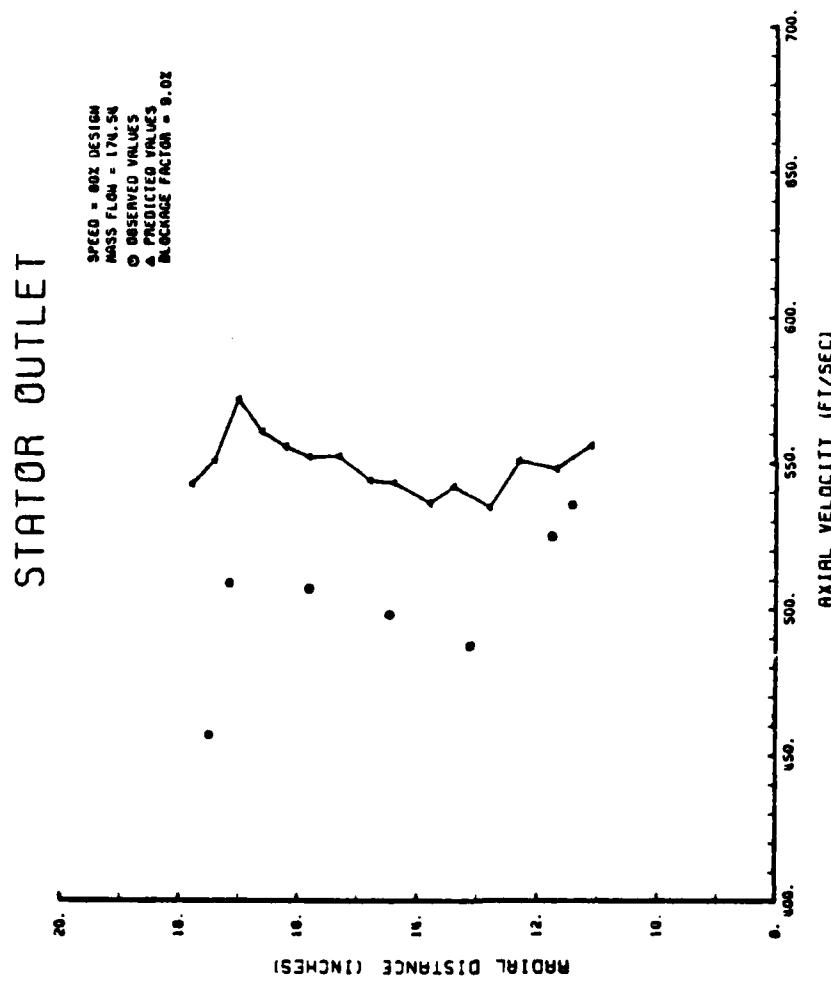


Figure 76. Axial Velocity at the Stator Outlet, 80% Design, 9% Blockage

APPENDIX A  
STORAGE ALLOCATION FOR THE PROGRAM MESHGEN

The following list of pointers and variable names indicates the storage location of the first value of the corresponding array:

**REAL 8 Variables (Array R8)**

| <u>Pointer Name</u> | <u>Array Name</u> | <u>Array Contents</u>   |
|---------------------|-------------------|---|
| J1                  | XSE               | Z Coordinates of the Super Elements                                     |
| J2                  | YSE               | R Coordinates of the Super Elements                                     |
| J3                  | XXSE              | Temporary Storage of the Z Coordinates of the Super Element Subdivision |
| J4                  | YYSE              | Temporary Storage of the R Coordinates of the Super Element Subdivision |
| J5                  | ZC                | Z Coordinates   |
| J6                  | RC                | R Coordinates   |
| J7                  | PSI               | Stream Function   |
| J8                  | B                 | Blockage Factor   |
| J9                  |                   | End of Stored Values  |

**REAL 4 Variables (Array O4)**

| <u>Pointer Name</u> | <u>Array Name</u> | <u>Array Contents</u>                          |
|---------------------|-------------------|--|
| M1                  | OHZC              | Z Coordinates of Streamwise Element Boundaries |
| M2                  | OHRC              | R Coordinates of Streamwise Element Boundaries |
| M3                  | OVZC              | Z Coordinates of Transverse Element Boundaries |
| M4                  | OVRC              | R Coordinates of Transverse Element Boundaries |
| M5                  | OHZC1             | Z Coordinates of Streamwise Element Boundaries |
| M6                  | OHRC1             | R Coordinates of Streamwise Element Boundaries |
| M7                  | OZC               | Real 4 Z Coordinates of Mesh                   |
| M8                  | ORC               | Real 4 R Coordinates of Mesh                   |
| M9                  |                   | End of Stored Values                           |

**INTEGER 4 Variables (Array I2)**

| <u>Pointer Name</u> | <u>Array Name</u> | <u>Array Contents</u>               |
|---------------------|-------------------|-------------------------------------|
| L1                  | NSEC              | Number of Columns per Super Element |
| L2                  | NTSE              | Super Element Type                  |
| L3                  | NTE               | Element Type Identifier             |
| L4                  | NODE              | Connectivity Matrix                 |
| L5                  | NBC               | Boundary Condition Nodes            |
| L6                  |                   | End of Stored Values                |

APPENDIX B  
LISTING OF THE OUTPUT VARIABLES FOR THE PROGRAM MESHGEN

| <u>FORTRAN Name</u> | <u>Variable</u>          | <u>File</u> |
|---------------------|--------------------------|-------------|
| ZC                  | Z Coordinate             | TMESH DATA  |
| RC                  | R Coordinate             | TMESH DATA  |
| B                   | Blockage Factor          | TMESH DATA  |
| PRESS               | Inlet Static Pressure    | FLUID DATA  |
| PTOT                | Inlet Total Pressure     | FLUID DATA  |
| TEMP                | Inlet Static Temperature | FLUID DATA  |
| TTOT                | Inlet Total Temperature  | FLUID DATA  |
| RHOSTA              | Inlet Static Density     | FLUID DATA  |
| RHOTT               | Inlet Total Density      | FLUID DATA  |
| WDOT                | Inlet Mass Flow          | FLUID DATA  |
| CP                  | Specific Heat            | FLUID DATA  |
| R                   | Gas Constant             | FLUID DATA  |
| G                   |                          | FLUID DATA  |
| RPM                 | RPM                      | FLUID DATA  |
| VZI                 | Inlet Axial Velocity     | FLUID DATA  |
| NODE                | Connectivity Matrix      | NODE DATA   |
| NTE                 | Element Type ID          | NODE DATA   |

|     |  |             |
|-----|--|-------------|
| NBC | Boundary Nodes<br>Where $\psi$ Specified | BC DATA     |
| PSI | Stream Function                          | STREAM DATA |

APPENDIX C  
STORAGE ALLOCATION FOR THE PROGRAM TURBO

The following list of pointers and variable names indicates the storage location of the first value of the corresponding array:

**REAL 8 Variables (Array R8)**

| <u>Pointer Name</u> | <u>Array Name</u> | <u>Array Contents</u>         |
|---------------------|-------------------|-------------------------------|
| NP1                 | ZC                | Z Coordinates                 |
| NP2                 | RC                | R Coordinates                 |
| NP3                 | B                 | Blockage Factors              |
| NP4                 | ALP               | Absolute Flow Angles          |
| NP5                 | BE                | Relative Flow Angles          |
| NP6                 | H                 | Total Enthalpy                |
| NP7                 | HS                | Static Enthalpy               |
| NP8                 | VZ                | Axial Velocity                |
| NP9                 | VR                | Radial Velocity               |
| NP10                | VU                | Absolute Tangential Velocity  |
| NP11                | WU                | Relative Tangential Velocity  |
| NP12                | PSI               | Current Streamfunction Values |
| NP13                | PSIO              | Previous Streamfunction Value |
| NP14                | F                 | Right-hand Side Vector        |
| NP15                | RHS               | Temporary Storage for F       |
| NP16                | RHO               | Static Density                |

|      |        |                               |
|------|--------|-------------------------------|
| NP17 | RHON   | Not Used                      |
| NP18 | WRL    | Angular Momentum Vector       |
| NP19 | ETA    | Adiabatic Efficiencies        |
| NP20 | EM     | Stiffness Matrix Elements     |
| NP21 | DEV1   | Rotor/Stator Deviation Angles |
| NP22 | PRAT   | Total-to-Total Pressure Ratio |
| NP23 | TEMP   | Static Temperature            |
| NP24 | TTOT   | Total Temperature             |
| NP25 | PRESS  | Static Pressure               |
| NP26 | PTOT   | Total Pressure                |
| NP27 | RHOVT  | Total Density                 |
| NP28 | HR     | Rhothalpy                     |
| NP29 | ENTROP | Entropy                       |
| NP30 |        | End of Stored Values          |

REAL 4 Variables (Array 04)

| <u>Pointer Name</u> | <u>Array Name</u> | <u>Array Contents</u>       |
|---------------------|-------------------|-----------------------------|
| NPO1                | OVEL              | Velocity at a Given Station |
| NPO2                | ORC               | R Coordinates               |
| NPO3                | OBE               | Relative Flow Angles        |
| NPO4                | OALP              | Absolute Flow Angles        |
| NPO5                |                   | End of Stored Values        |

**INTEGER 4 Variables (Array I2)**

| <u>Pointer Name</u> | <u>Array Name</u> | <u>Array Contents</u>   |
|---------------------|-------------------|-------------------------|
| NPI1                | NFS               | Nodes for F Calculation |
| NPI2                | NBC               | Boundary Nodes          |
| NPI3                | NTE               | Element Type Identifier |
| NPI4                | NODE              | Connectivity Matrix     |
| NPI5                |                   | End of Stored Values    |

**APPENDIX D**

**LISTING OF THE OUTPUT VARIABLES FOR THE PROGRAM TURBO**

| <u>Listing Name</u> | <u>Variable</u>              | <u>Units</u>         |
|---------------------|------------------------------|----------------------|
| PSI                 | Stream Function              | lbm/sec              |
| VZ                  | Axial Velocity               | ft/sec               |
| VR                  | Radial Velocity              | ft/sec               |
| R                   | Radius                       | inches               |
| DENSITY             | Static Density               | lbm/ft <sup>3</sup>  |
| WRL                 | Angular Momentum             | ft <sup>2</sup> /sec |
| HT                  | Total Enthalpy               | BTU/lbm              |
| VT                  | Absolute Tangential Velocity | ft/sec               |
| WT                  | Relative Tangential Velocity | ft/sec               |
| HS                  | Static Enthalpy              | BTU/lbm              |
| TEMP                | Static Temperature           | °R                   |
| TTOT                | Total Temperature            | °R                   |
| PRESS               | Static Pressure              | psia                 |
| PTOT                | Total Pressure               | psia                 |
| RHOT                | Total Density                | lbm/ft <sup>3</sup>  |
| ALPHA               | Absolute Flow Angle          | Degrees              |
| BETA                | Relative Flow Angle          | Degrees              |

## APPENDIX E

### LISTING OF THE PROGRAM MESHGEN

FILES: ~~MECHEN~~ FCR TRAN AI NAVAL POSTGRADUATE SCHOOL

```

PROGRAM MESHGEN
*** THIS PROGRAM FREQUIRES A RECTANGULAR EIGHT-NODED
*** ISOPARAMETRIC GRID OF ARBITRARY DIMENSION M X N GIVEN
*** THE CODE NEEDS OF THE MESH'S "SUPER ELEMENTS".
*** THE PROGRAM ALSO COMPUTES AN INITIAL STREAM FUNCTION
*** EQUATION, TANGENTIAL BLOCKAGE FACTORS, AND THE
*** APPROPRIATE INLET CONDITIONS NEED FOR THE PROGRAM TURBO.
*** THE PROGRAM IS ONLY LIMITED BY THE SPACE ALLOCATED IN
*** THE ARRAYS R9, C4, I2 OR MACHINE LIMITATIONS.
*** IMPLICIT REAL*8(A-G,P-Z), REAL*4(H)
INTEGER*4 M2,MPI1,NC,NC1,IC,MRP1,MPI1,MN1,MN2,MN3,MN4,NSF2,N2
INTEGER*4 IANS
COMMON /INTG/ M2,MPI1,NC,NC1,IC
COMMON /ACCOUNT/ MRCP,MRTWI,MNSFC
COMMON /ACCT/ MN,NFL,NCL1,NE,NROUTC,NSTAT
COMMON /NPINIT/ J1,J2,J3,J2,J4,J5,J7,J3,J5,J10,J11,J12,J13,J14,
L1,L2,L3,L4,L5,L6,M1,M2,M3,M4,M5,M6,M7,M8,M9
COMMON /FFAR/ FA(5000)
COMMON /FFAR2/ I2(5000)
COMMON /FFAR4/ C4(3000)
LNR = 5000
LNI = 3000
LMP4 = 3000

CALL INIT1(M4C62,NCCL2,NSF,NSE2,WDOT,PTCT,TTCT,FPM,G,R,CP,ZMAX,
1ZMIN,RMAX,PMIN,IANS)

CALL INPLT(R8(J1),R8(J2),I2(L1),I2(L2),NE1,NSE2,NSE,MPI1,N2,I4C,
1I4I,L1M4,IANS)

CALL TMESH(R8(J1),R8(J2),R8(J3),R8(J4),R8(J5),R8(J6),I2(L1),
1,I2(L2),I2(L3),R8(J13),NSE2,NSE,N2,MPI1,IANS)

CALL CONECT(I2(L2),I2(L3),I2(L4),NSF,N2,NE1,NROUT3,NROUT6,
INSTATR,NSTATE,IANS)

CALL INIT2(R8(J6),I2(L4),I2(L5),R8(J7),RHCSTA,PHOTT,TTCT,WDCT,
1PTCT,PRESS,TEMP,VZI,R,CP,G,NSF2,MPI1,N2,NSE,NE1,IANS)

IF(IANS.EQ.2) GOTO 100

CALL TASK1(R8(J6),R8(J13),NSE2,MPI1,N2,MRCTCB,NROUT6,NSTATE,
INSTATE)

100 CALL FILGEN(R8(J5),R8(J6),R8(J13),R8(J7),I2(L4),I2(L2),I2(L5),
1PRESS,DTCT,TEMP,TOT,PHCSTA,PHOTT,VZI,WDCT,RPM,CP,P,G,NROUT6,
2NSTATB,NSE2,NSE,N2,MPI1,NE1,IANS)

200 WRITE(15,200)
FORMAT(5X,'DO YOU WANT A PLOT OF THE MESH ? ',/,' 1 = YES
1 2 & NO')

```

FILE: MESHGEN FORTRAN AI NAVAL POSTGRADUATE SCHOOL

```

      FEAD(15,*), FANS
      IF(IANS.EC.2) GTC 300

      CALL MPLECT(R2(J5),F8(J6),ZMAX,ZMIN,RMAX,RMIN,C4(M1),C4(M2),
      1 C4(M3),C4(M4),C4(M5),C4(M6),C4(M7),C4(M8),I2(L5),NSE2,MRF1,
      2 ZA2,NSE,RE1,NC2,MN2)

300   STCP
      EAC

      SUBROUTINE INIT1(MRN2,NCCL2,NSE,NSE2,WDT,POT,TCT,RPM,G,R,CP,
      1 ZMAX,ZMIN,FMAX,PVIN,IANS)
      IMPLICIT REAL*8(A-H,P-Z), REAL*4(H)
      INTEGER*4 MR,NC,NC1,IC,MRCW1,NEL,MN1,MN2,MN3,MN4,NSE2,N2
      INTEGER*4 IANS
      COMMON /INT4/ MR,MN1,NC,NC1,IC
      COMMON /MCOUNT/ MFCW1,NCOL1,MASEC
      COMMON /MCINT/ MN,NCOL1,NE,ROTC,NSTAT
      COMMON /MPCINT/ J1,J2,J3,J4,J5,J6,J7,J8,J9,J10,J11,J12,J13,J14,
      1 L1,L2,L3,L4,L5,Lc,M1,M2,M3,M4,M5,M6,M7,M8,M9
      WRITE(15,10)
      100  FCFMAT(5), 'ENTER INLET CONDITIONS: ././ MASS FLOW(LBM PER SEC)
      1 TCTAL TEMP(DLG 2) AND TOTAL PRESS(PSIA) ./)
      1 READ(15,*), WCT,TCT,POT
      WRITE(15,110)
      110  FCRMAT(5), 'ENTER OPERATING CONSTANTS: ././ RPM:././ RATIO OF
      1 SPECIFIC HEATS (GAMMA):././ GAS CONSTANT R(FT-LBF/LF^4-DEG R1),/
      2 SPECIFIC HEAT CONSTANT PRESSURE CP(BTU/LBM-DEG R1),/
      READ(15,*), RPM,G,R,CH
      WRITE(15,120)
      120  FCRMAT(5), 'ENTER GRAPH SCALING CONSTANTS: ././ Z MAX:././ Z
      1 MIN:././ R MAX:././ R MIN:./)
      READ(15,*), ZMAX,ZMIN,RMAX,RMIN
      WRITE(15,5C)
      50   FORMAT(5A,'DO YOU WANT TO CREATE A NEW MESH ? ./' 1 = YES
      1 2 = NO)
      1 READ(15,*), IANS
      IF(IANS.EC.1) GTC 149
      130  READ(25,120) MR,MFCW,NCCL,MRCW1,NCOL1,ROTCB,NSTATB
      FCRMAT(715)
      GTC 201
      WRITE(15,200)
      145  FCRMAT(5), 'ENTER NO OF SUPER ELEMENTS AND NO ELEMENT ROWS:./)
      150  READ(15,*), NSE,MRCW
      MRCW1 = 2 * MRCW + 1
      MRC1 = MRCW1 - 1
      PR = MRC1
      MRCW2 = MRCW1
      NCCL2 = NCCL1
      NSE2 = 2 * NSE + 2
      J1 = 1
      J2 = J1 + NSE2
      J3 = J2 + NSE2
      L1 = 1
      L2 = L1 + NSE
      L3 = L2 + NSE
      RETURN
      EAC

      SUBROUTINE INPUT(XSE,YSE,NSEC,NTSE,NE1,NSE2,NSE,MRR1,N2,LIM2,
      1 LIM1,LIM4,IANS)
      ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++
      ++
      ++ THIS SUBROUTINE CONTAINS THE DESCRIPTION OF THE SUPER ELEMENTS ++
      ++
      ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++

```

FILE: MESHGEN FORTRAN A1 NAVAL POSTGRADUATE SCHOOL

```

C
      IMPLICIT REAL*8(E14-H,P-2)
      INTEGER*4 MR,MFI,NC,NC1,IC,MRR1,NE1,NN1,MN2,MN3,MN4,ASE2,N2
      INTEGER*4 IANS
      CCPMLN /INT4/ MR,NE1,NC,NC1,IC
      CCPMN /MCOUNT/ MROW,MROW1,MSEC
      CCPMN /ACCOUNT/ NN,ACCL,ACCL1,RE,WOTC,INSTAT
      CCPMN /MCINT/ J1,J2,J3,J4,J5,J6,J7,J8,J9,J10,J11,J12,J13,J14,
      1L1,L2,L3,L4,L5,L6,M1,M2,M3,M4,M5,M6,M7,M8,M9
      DIMENSION XSE(1),YSE(1)
      DIMENSION ASE(1),MTSE(1)

C
      IF(IANS.EQ.2) GOTO 202
      ACCL = 0
      J = 1
      NSE22 = ASE2/2
      CC 100 I = 1,ASE22
      WRITE(15,101)
      101 FCPMAT(15,X,'ENTER Z,R COORDINATE PAIRS FOR STATION ',I2/)
      READ(15,*) XSE(J),YSE(J),XSE(J+1),YSE(J+1)
      J = J + 2
      102 CC CONTINUE
      CC 200 I = 1,ASE
      WRITE(15,201)
      201 FCPMAT(15,X,'ENTER TYPE OF SUPER ELEMENT AND THE NO OF COLUMNS = '
      10R SUPER ELEMENT ',I2/)
      READ(15,*) NESE(I),NSE(I)
      ACCL = NCOL + ASE(I)

C
      202 CC CONTINUE
      NE = MROW * NCCL
      NE1 = NE
      NCCL1 = 2 * NCCL + 1
      NR = NCCL1 * MROW1 - MROW * NCCL
      N2 = NN
      PRRI = MFCW1
      J1 = 1
      J2 = J1 + ASE2
      J3 = J2 + ASE2
      J4 = J3 + MFCW1
      J5 = J4 + MFCW1
      J6 = J5 + N2
      J7 = J5 + N2
      J8 = J7 + N2
      J9 = J8 + N2
      J10 = J9 + N2
      J11 = J10 + N2
      J12 = J11 + N2
      J13 = J12 + N2
      J14 = J13 + N2
      L1 = 1
      L2 = L1 + ASE
      L3 = L2 + ASE
      L4 = L3 + ASE1
      L5 = L4 + ASE1
      L6 = L5 + ASE1
      K1 = 1
      K2 = M1 + NCCL1
      K3 = M2 + NCCL1
      K4 = M3 + NCCL1
      K5 = M4 + NCCL1
      K6 = M5 + NCCL1
      K7 = M6 + NCCL1
      K8 = M7 + N2
      K9 = M8 + N2
      NEXCR = L14 - J14
      NEXCI = L14 - L1
      NEXC4 = L14 - M2
      IF(NEXCR.LT.0) CALL SPRI(NEXCR)
      IF(NEXCI.LT.0) CALL SPRI(NEXCI)
      IF(NEXC4.LT.0) CALL SPRI(NEXC4)
      WRITE(6,200) NEXC4,NEXCI

```

FILE: MESHGEN FORTRAN A1 NAVAL POSTGRADUATE SCHOOL

```

30C  FORMAT(1CX,' MEMORY SPACE AVAILABLE : REAL =',IS,2X,', INTEGER = '
1,I5//)
      RETURN
      END

C
C
C     SUBROUTINE TMEASH(XSE,YSE,XXSE,YYSE,ZC,RC,NSEC,NTSE,NT,
1E,NSE2,NSE,N2,MF4,IANS)
C
C++ THIS SUBROUTINE PRODUCES A RECTANGULAR EIGHT-NODED
C++ ISOPARAMETRIC GRID OF ARBITRARY DIMENSION M X N GIVEN
C++ THE COMMON ACCES OF THE MESH'S "SUPER ELEMENTS"
C
C
C     IMPLICIT REAL*8(A-H,F-Z)
      INTEGER*4 VR,MRI,NC,NC1,IC,MRR1,NEL,MN1,MN2,MN3,MN4,NSE2,N2
      INTEGER*4 IANS
      COMMON /INT4/ MR,MRI,NC,NC1,IC
      COMMON /MCOUNT/ LROW,MROW
      COMMON /MCOUNT/ MR,MRI,NC,NC1,RE,IPUTC,NSTAT
      DIMENSION XSE(1),YSE(1),XXSE(1),YYSE(1),ZC(1),RC(1),B(1)
      DIMENSION NSEC(1),NTSE(1),NT(1)

C
      IF(IANS.EQ.1) GOTO 20
C 11  IF(I.EQ.1) THEN
        READ(20,12) ZC(1),RC(1),B(1)
      12  FORMAT(3F15.11)
C 11  CONTINUE
      20  GOTO 10C
      20  XCIV1 = FLCAT(MR)
      XCIV2 = FLCAT(MR)
      I = 1
      J = 1
      IL = 1
      IR = 1
      NSTAT = C
      AFCTO = C
      DC 10 N = 1,MN
      ZC(N) = J*DC
      RC(N) = 0.0C
      10  CONTINUE
      10  DO J = 1,NT
        IF(J.EQ.1) GOTO 120
        IF(NTSE(J)-NTSE(J-1).LT.0) 120,120,140
      140  IF(IM.LT.0) GOTO 150
        AFCTO = IL
        IM = C - IM
        GFTO = 12C
      15C  NSTAT = IL
        IM = C - IM
      120  IT = NSEC(J) + MROW
        DO 110 IL = 1,IT
          NT(IL) = NTSE(J)
        IL = IL + 1
      110  CONTINUE
        NCOL1 = 2 * NSEC(J) + 1
        NC1 = NCOL1
        NC = NC1 - 1
        XCIV1 = FLCAT(NC1)
        XCIV2 = FLCAT(NC1)
        X1 = XSE(1)
        X2 = XSE(1 + 1)
        X3 = XSE(1 + 2)
        X4 = XSE(1 + 3)
        Y1 = YSE(1)
        Y2 = YSE(1 + 1)
        Y3 = YSE(1 + 2)

```

FILE: MESHGEN FCRTRAN 41 NAVAL POSTGRADUATE SCHOOL

```

Y4 = YSE(1 + 3)
X0IF2 = X3 - X1
X0IF3 = X4 - X2
YCIF2 = Y1 - Y3
YDIF3 = Y2 - Y4
XINT2 = X0IF2 / XDIV2
XINT3 = X0IF3 / XDIV2
YINT2 = YDIF2 / YDIV2
YINT3 = YCIF3 / YDIV2
ISEL = -1
DO 200 K = 1, NCI
    XCIFI = X1 - X2
    YCIFI = Y1 + Y1 - Y2 + Y2
    IF(ISEL) Z1C, 210, 220
    XINT1 = XCIFI / YCIV1
    YINT1 = YCIFI / YDIV1
    MRF = MRCW1
    GTC 230
210    XINT1 = X0IF1 / YDIV2
    YINT1 = YDIF1 / YDIV2
    MRF = MRCW + 1
220    DC 200 IC = 1, MPR
        IC1 = IC - 1
        XXSE(IC1) = X1 - FLOAT(IC - 1)*XINT1
        IF(IC .NE. 1) GOTO 231
        YYSE(IC) = Y1
        GTC 232
231    YYSE(IC1) = DSQRT(YYSE(IC1)*YYSE(IC1) - YINT1)
        ZC(IJ) = XXSE(IC)
        RC(IJ) = YYSE(IC)
        IJ = IJ + 1
200    CONTINUE
        X1 = X1 + XINT2
        X2 = X2 + XINT3
        Y1 = Y1 - YINT2
        Y2 = Y2 - YINT3
        ISEL = -ISEL
20C    CONTINUE
        ISEL = -ISEL
        IJ = IJ - MRCW1
        I = 24J + 1
10C    CONTINUE
        WRITE(6, 400) MRCW, NCCL
400    FORMAT(1CX, ' NUMBER OF ROWS = ', I3, 1EX, ' NUMBER OF COLUMNS = '
     1, I3//)
        WRITE(6, 410) NE, NN
410    FORMAT(5, ' TOTAL NUMBER OF ELEMENTS = ', I4, 10X,
     1' TOTAL NUMBER OF NODES = ', I4//)
        RETURN
        END
C
C      SUBROUTINE CCNEC(NTE, NODE, NSE, N2, NE1, IPOTOB, NROTCE, NSTATB,
     1NSTATE, IANS)
C      ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++
C      ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++
C      ++
C      ++      THIS SUBROUTINE PRODUCES THE CONNECTIVITY MATRIX      ++
C      ++
C      ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++
C      ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++
C
C      IMPLICIT REAL*8(A-H,P-Z)
      INTEGER*4 YR, MRL, AC, NC1, IC, MRR1, NE1, NL1, MN2, MN3, MN4, ASE2, N2
      INTEGER*4 IANS
      COMMON /MCOUNT/ MRCW, MRCW1, MNSEC
      COMMON /MCOUNT/ NE, NCOL, NCBL, NE, NROTC, NSTAT
      COMMON /JNTS/ NH, N1, NC, NC1, IC
      DIMENSION NODE(N1, 1), NE1, NC1, IC
      IF(IANS.EC.2) GTC 601
      K = 1
      MK = 1

```

FILE: PESGEN FORTAN II NAVAL POSTGRADUATE SCHOOL

FILE: MESHGEN FORTRAN A1 NAVAL POSTGRADUATE SCHOOL

```

      INTEGER IANS
      COMMON /ACCOUNT/ MRCH,MRROW1,MNSEC
      COMMON /ACCF1/T/ NR,NCOL,NCOL1,NE,NROTC,NSTAT
      COMMON /INT4/ MR,VPI,NC,NC1,IC
      DIMENSION FC(1),PSI(1)
      DIMENSION NODE(NE1,1),NBC(1)
      G = 1.4CC
      R = 53.2EC
      GC = 32.174D0
      IF(IANS.EC.1) CCTC 10
      NBC = 2 * NCCL1 + MRROW1 - 2
      CC 700 I = 1,MRHC
      READ(25,710) REC(I)
      FORMAT(15)
710  CONTINUE
700  GETC 300
10   CC 100 I = 1,MRR1
      NBC(I) = I
100  CONTINUE
      MM = 1
      MK = 1
      K = MRR1 + 1
      JK = K
      L = MRF1 + 1
      N = 2 * (NCOL1 - 1) + K
      CC 200 J = K,4
      IF(MM.GT.0) CCTO 210
      NBC(K) = L
      K = K + 1
      L = L + 1
      MM = C - MM
      CCTO 200
210  IF(MK.GT.0) GETC 220
      NBC(K) = L
      L = L + MRCH1 - 1
      MK = C - MK
      K = K + 1
      MM = C - MM
      GOTO 200
220  NBC(K) = L
      L = L + MRCH
      MK = C - MK
      K = K + 1
      MM = C - MM
200  CONTINUE
      K = MRR1 + 2 * (NCOL1 - 2) + 1
      JK = N2 - MR21 + 1
      NBC(K) = J
      NBC(K+1) = '2
200  CONTINUE
      GM1 = C - 1.00
      GM1I = 1.00 / GM1
      RFUBZ = FC(MRCH1) * FC(MRCH1)
      ARFA = 3.141592654 * (RC(1) * RC(1) - RHUB2) / 144.000
      CALL FLCFCT(WUC,PTCT,TTCT,RHOT,T,VTDT,XVEL,CP,R,GM1,GMII,
1     AFEA)
      CLANT = 1.00C - XVEL * XVEL
      TEMP = TTCT * QUANT
      PRESS = PTCT * CLANT**G*G+1I
      RFCTSTA = RHOT * QUANT**GM1I
      VZI = VTCT * XVEL
      NC = MRCH1 - 1
      PCII = WUC / (2.000 * 3.141592654)
      PSID = PCII / FLCAT(NC)
      J = 0
      CC 500 I = 1,NC
      PSI(I) = PSI1 - FLCAT(J) * PSID
      J = J + 1
500  CONTINUE
      PC1(MRCH1) = 0.00000000000000
      CC 600 I = 1,NE1
      NII = NODE(I,1)

```

FILE: MSGGEN FORTRAN AI NAVAL POSTGRADUATE SCHOOL

FILE: MESHGEN FORTRAN A1 NAVAL POSTGRADUATE SCHOOL

```

MM7 = MM6 + MRCW
MM8 = MM7 + 1
CC 100 I = 1,NN
E(1) = 1.000
100 CCNTINUE
CC 200 I = NRCTCE,MM1
RCI = RC(1)
RTCL = C.0131143 - 2.45672D-4 * RCI - 1.03091D-5 * RCI * RCI +
14.24339D-10 * PCI * PCI * PCI * PCI * PCI
RSIGL = 1.04573 + 1.22251D-3 * PCI*RCI + 1.46066D-8 * PCI * RCI *
1 RCI * PCI * RCI - 1.33865D-3 * DCOS(RCI) - 7.42494D-4 * DSIN(RCI)
2 + 2.39562D-5 * DTAN(RCI) - 2.12971 * DLOG(RCI)
B(1) = 1.00 - RTCL * RSIGL
20C CCNTINUE
CC 300 I = MM2,MM3
RCI = RC(1)
FTCM = C.140493 - 5.89061D-3 * RCI - 3.28044D-11 * RCI*RCI*RCI
1 * RCI * PCI + 2.15909D-15 * PCI * PCI * PCI * PCI * PCI *
2 RCI * RCI * RCI * RCI - 4.55276D-4 * DCOS(RCI) - 1.2162D-4 *
3 CSIM(RCI) - 3.84042D-6 * DTAN(RCI)
RCI = RC(1)
RSIGM = 10.9594 + .255785 * RCI - 2.62070D-6 * RCI * RCI * RCI *
1 RCI + 4.45133D-3 * DSIN(RCI) - 1.04378D-4 * DTAN(RCI)
2 - 4.83566 * DLG(RCI)
E(1) = 1.00 - FTCM * RSIGM
30C CCNTINUE
CC 400 I = MM4,NRCTCE
RCI = RC(1)
RTCT = 0.0195736 - 1.11132D-3 * RCI + 1.34790D-6 * RCI*RCI*PCI -
1 1.046835D-14 * PCI * RCI * PCI * RCI * RCI * RCI * RCI * RCI
2 + 8.41826D-5 * DCOS(RCI) + 2.68422D-5*DSIN(RCI) - 5.72134D-7 *
3 CTAN(RCI)
RSIGT = 17.9156 + .656044 * RCI - 3.96E91D-6 * RCI * PCI * RCI *
1 RCI + 1.65550D-3 * DTAN(RCI) - 1.36477D-4 * DTAN(RCI)
2 - 6.50E66 * DLG(RCI)
E(1) = 1.00 - RTCT * RSIGT
40C CCNTINUE
CC 500 I = NSTATE, MM5
RCI = RC(1)
STNCI = C.0118089 + 2.31157D-3 * PCI + 1.60096D-5 * RCI * RCI -
17.69E43D-5 * DCOS(RCI) + 4.83218D-5 * DSIN(RCI) + 3.24003D-7 *
2 DTAN(RCI)
SSIGL = 2.77577 - C.350357 * RCI + 9.45492D-3 * PCI * PCI -
1 5.52663D-13 * PCI*RCI*PCI*RCI*PCI*PCI*PCI*PCI*PCI*PCI
2 - 1.31164D-3 * CSIM(RCI) - 1.72756D-5 * DTAN(RCI)
STEML = C.272E02 - 5.25550D-3 * RCI + 1.9243D-11 *
1 RCI*RCI*PCI*RCI*RCI*RCI*RCI + 3.85063D-4 * DCOS(RCI)
2 + 2.25631D-4 * DSIN(RCI) - 0.47245D-5 * DTAN(RCI)
B(1) = 1.000 - STEMCL * STMCL * SSIGL
50C CCNTINUE
CC 600 I = MM6, MM7
RCI = RC(1)
STMCM = C.01158762 + 2.58621D-3*RCI + 9.85191D-6 * PCI * RCI -
1 7.66525D-6 * DCOS(RCI) + 6.89499D-5 * DSIN(RCI) - 4.15541D-5 *
2 CTAN(RCI)
SSIGM = 6.17132 - .401549 * RCI + 1.13424D-2 * RCI * RCI -
1 6.500E5C-13 * PCI*PCI*PCI*PCI*PCI*PCI*PCI*PCI*PCI*PCI +
2 2.61135D-4 * DTAN(RCI)
E(1) = 1.000 - STMCM * SSIGM
60C CCNTINUE
CC 700 I = MM8, NSTATE
RCI = RC(1)
STNCT = C.00994821 + 3.14858D-3 * RCI - 3.60047D-11 * RCI*PCI*PCI
1 * PCI*RCI*RCI*RCI + 6.8320D-5 * DCOS(RCI) - 3.03125D-5 * DSIN(RCI) -
2 1.2412e-5 * DTAN(RCI)
SSIGT = 6.50125 - 5.05655 * PCI + 1.52234D-2 * RCI * RCI -
1 2.16170D-11 * PCI*PCI*PCI*PCI*PCI*PCI*PCI*PCI*PCI*PCI
2 + 1.07767D-4 * CTAN(RCI)
STETMT = C.015182 + 2.5254D-4 * PCI * PCI - 6.34D-14 * PCI *
1 PCI*PCI*PCI*PCI*PCI*PCI*PCI*PCI*PCI*PCI + 1.64098D-3 * DCOS(RCI)
2 - C.194717 * DLG(RCI)
E(1) = 1.000 - STETMT * STNCT * SSIGT

```

FILE: MESHGEN FORTRAN A1 NAVAL POSTGRADUATE SCHOOL

```

70C  CCNTINUE
     RETURN
     END

C
C
C      SUBROUTINE FILGEN(ZC,RC,R,PSI,NODE,NTF,NRC,PRESS,PTOT,TEMP,TTCT,
1PFCSTA,PFCTT,V2I,WQCT,RDV,CP,R,G,NROTCB,NSTATB,
2ASE2,NSE,N2,MRR1,NE1,IA1'S)
C  ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++
C  ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++
C  ++
C  ++ THIS EXECUTIVE WRITES THE OUTPUT OF THE PROGRAM ++
C  ++ INTO DISK STORAGE. THE FILE DEFINITIONS ARE LISTED ++
C  ++ IN THE EXEC FILES TURH01 AND TUR601A. ++
C  ++
C  ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++
C  ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++
C
C      IMPLICIT REAL*8(A-H,P-Z)
      INTEGER*4 NR, VF1, NC, NC1, IC, MRR1, NE1, MN1, MN2, MN3, MN4, NSE2, N2
      INTEGER*4 IANS
      COMMON /INT4/ NR, NE1, NC, AC1, IC
      COMMON /MCOUNT/ VF0A, MROW1
      COMMON /MCOUNT/ MN, NCCL, NCOL1, NE, NROTC, NSTAT
      COMMON /SICN/ ZC(1), RC(1), PSI(1), PTF, PTOT, TEMP, TTCT, RHOCTA, RHOTT, WQCT, CP, R, G, RPM, V2I
      DIMENSION NODE(NF1,1), NTE(1), NEC(1)

C
      IF(IANS.EC.2) GOTO 1C1
      100 I=1,NA
      WRITE(12C,11C) ZC(I),RC(I),E(I)
      11C  FORMAT(2F15.11)
      10C  CCNTINUE
      101  WRITE(25,12C) MN, WQCT, NCCL, MROW1, NCOL1, NROTCB, NSTATB
      102  WRITE(25,12C) PRESS, PTOT, TEMP, TTCT, RHOCTA, RHOTT, WQCT, CP, R, G, RPM, V2I
      12C  FCFMT(715)
      121  FCFMT(4F11.6,2F12.8,/,4F12.7,FE.1,F13.8)
      IF(IANS.EC.2) GOTO 301
      200 I = 1, NE1
      WRITE(12C,21C) NODE(I,1), NCCE(I,2), NCDE(I,3), ACDE(I,4),
1      NCCE(I,5), NCCE(I,6), NCDF(I,7), NODE(I,8), NTE(I)
      21C  FORMAT(15I5)
      20C  CCNTINUE
      201  L = 2 * NCCL1 + 2 * MROW1 - 4
      202  I = 1, L
      203  WRITE(12C,31C) NAC(I)
      31C  CCNTINUE
      301  EC 400 I=1,NA
      WRITE(4C,-11C) PSI(I)
      41C  CCNTINUE
      400  CCNTINUE
      RETURN
      END

C
C
C      SUBROUTINE FLCFCT(WQCT,PTOT,TTCT,RHOCTT,VTCT,XVEL,C2,R,G41,G71,
1PFCB)
C  ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++
C  ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++
C  ++
C  ++ THIS COMPLETES THE NONDIMENSIONAL VELOCITY X ++
C  ++
C  ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++
C  ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++
C
C      IMPLICIT REAL*8(A-G,P-Z), REAL*4(H)
      INTEGER*4 NR, MRR1, NC, NC1, IC, MRR1, NE1, MN1, MN2, MN3, MN4, NSE2, N2
      COMMON /INT4/ NR, MRR1, NC, NC1, IC
      COMMON /MCOUNT/ VF0A, MROW1, MN, NC
      COMMON /MCOUNT/ MN, NCCL, NCOL1, NE, NROTC, NSTAT

```

FILE: PESMGEN FCRTRAN A1 NAVAL POSTGRADUATE SCHOOL

```

IT = 0
XVEL = C_1CO
EPS = 1.0E-06
RJ = 778.2 * 32.174
RHCTT = (F1JT * 144.000) / (R * TTDT)
VTCT = DSCAT(2.0DC * CP * RJ * TTCT)
PHI1 = WCCT / (RHCTT * VTCT * APEA)
PHI = XVEL * (1.000 - XVEL * XVEL) ** GM1I
C1IFF = CABS(PHI1 - PHI)
IF(C1IFF.LT.EPS) GOTO 200
CPHIDX = PHI * (1.0DC/XVEL - (2.0D0*XVEL)/(GM1*(1.000 - XVEL*XVEL)))
XVEL = XVEL + (PHI1 - PHI) / DPHIDX
IT = IT + 1
IF(IT.LT.21) GOTO 100
IF(IT.LT.5) GOTO 110
FCRFORMAT(' CONVERGENCE NOT REACHED.',/)
CCONTINUE
VZI = XVEL * VTCT
RETURN
END

```

SUBROUTINE MPCT(ZC,PC,ZMAX,ZH IN, RHX,RM IN,OHZC,OHPC,  
1CZC,OCFC,CHZC1,CHPC1,CZC,OCPC,KLK,NS,E2,4RR1,N2,NS,E,  
2E,V2,OCFC,CHPC1)

VPL030:

\*\*\* CONVENT THE NCAL COORDINATES TO REAL\*4  
\*\*\* TO MAKE THEM COMPATABLE WITH PLTGT CALLS

CC 30 I = 1, MN  
0ZC(I) = ZC(I)  
CFC(I) = RC(I)

```

30 CONTINUE
CXYL(1) = ZMIN
CXYL(2) = ZMIN
CXYL(3) = ZMAX
CXYL(4) = RMAX
CALL DSINIT
CALL DSEFSE

```

$$\begin{aligned} CC_5^I &= 1,11C_1 \\ C_1ZC(1) &= C_0 \end{aligned}$$

FILE: MSGEN FORTRAN AI NAVAL POSTGRADUATE SCHOOL

```

      CHRC(1) = 0.0
      CONTINUE
      DO 6 I = 1,NFCW1
        CVZC(1) = 0.0
        CVRC(1) = 0.0
6      CONTINUE
      MM1 = VFCW1 + 1
      ACC1 = NCCL1 + 1
      A14 = NA
      NF14 = NFCW1
      AC14 = NCCL1
      AN1 = 'N' + !
      NP1 = NCW + 1
      CALL PLCT('NNNNNNNNNN',A14,CZC,CPC,OXYL,37,TITL1)
      ****
      ** PLCT FOR HORIZONTAL ELEMENT BOUNDARIES
      ** ****
      MM = 1
      KKK = 1
      DO 100 J = 1, NCCL1
        CZC(J) = ZC(MM)
        OHFC(J) = FC(MM)
        KLK(J) = NP
        IF(KKK.LT.C) GOTO 90
        MM = MM + VFCW1
        KKK = 0 - KKK
        GCTC 100
        MM = MM + NP1
        KKK = 0 - KKK
100    CONTINUE
      CALL PLCT('NNNNNNNNNN',NC14,OHZC,CHRC,OXYL,37,TITL1)
      DO 200 I = 1,NFCW
        MLK = 2 * I
        KKK = 1
        DO 800 III = 1,NCCL1
          IF(KKK.LT.C) GOTO 850
          III = KLK(III) + "LK
          OHZC1(III) = CZC(III)
          CHRC1(III) = CRC(III)
          KKK = 0 - KKK
          GCTC 800
          III = KLK(III) + !
          OHZC1(III) = CZC(III)
          CHRC1(III) = CRC(III)
          KKK = C - KKK
800    CONTINUE
        MM = (2 * I) + 1
        MLK = MLK + 1
      CALL PLCT('NNNNNNNNNN',NC14,OHZC1,CHRC1,OXYL,37,TITL1)
200    CONTINUE
      ****
      ** PLOT VERTICAL BOUNDARIES
      ** ****
      MM = 1
      NC11 = NCCL + 1
      DO 400 I = 1,NC11
        DO 300 J = 1,NCW1
          OVZC(J) = ZC(MM)
          OVFC(J) = FC(MM)
          MM = MM + 1
300    CONTINUE
      CALL PLCT('NNNNNNNNNN',MR14,CVZC,CVPC,OXYL,37,TITL1)
      MM = MM + NP1
400    CONTINUE
      CALL DSTEM
      RETURN
      END

```

APPENDIX F  
LISTING OF THE PROGRAM TURBO

FILE: TURBO FORTRAN AI NAVAL POSTGRADUATE SCHOOL

PROGRAM TURBO

THIS MERIDIONAL THROUGH-FLOW ANALYSIS PROGRAM APPLIES A GALERKIN FINITE ELEMENT METHOD TO A STREAM FUNCTION FORMULATION. THE PROGRAM USES EIGHT NODE, ISOPARAMETRIC ELEMENTS AND THREE-POINT GAUSS-LEGENDRE QUADRATURE. NUMERICAL INTEGRATION. SELECTED RESULTS ARE DISPLAYED ON THE TEKTRONIX 619 GRAPHICS TERMINAL.

```

      IMPLICIT FFAL*(A=4,F=2), REAL*4(C)
      INTEGER*4 NREAD,NWRITE,IC,NRCD,NE4,NNE4,NPOWS,LIMR,LIMI
      DIMENSION ALL ARRAY VARIABLES
      CCMON /REAL 2/ 22(56000)
      CCMON /REAL 4/ L-(500)
      CCMON /I,I2/ I2(2000)
      CCMON /ACOUNT/ NCOL,NCOL1,MN,NE
      1,NE4,NPC,ANNE,C
      CCMON /FCOM/ RG,G,GP,PT,TT,AG,WDT,RHCT,RHOSTA,
      1UINLET,UOUTLET,PS,T,RTJ,F2I2,F1I1,GC
      CCMON /LTC/ LTC,LTCT
      CCMON /NFCINT/ NPI1,NPI2,NPI3,NPI4,NPI5,NPI6,NPI7,NPI8,NPI9,
      1NP10,NPI11,NPI12,NPI13,NPI14,NPI15,NPI16,NPI17,NPI18,NPI19,NPI20,NPI21,NPI22,NPI23,
      2NP24,NPI25,NPI26,NPI27,NPI28,NPI29,NPI30,NPI31,NPI32,NPI33,NPI34,NPI35,
      3NP36,NPI37,NPI38,NPI39,NPI40,NPI41,NPI42,NPI43,NPI44,NPI45,NPI46,NPI47,NPI48,NPI49,NPI50,
      CINFS1FA(ZA(1),T(1),E(1),D(1),S(1),E(2),SF(3),
      CINFS1FA(ZA(2),T(2),E(3),D(2),S(2),E(4),SF(5),
      CINFS1FA(ZA(3),T(3),E(5),D(3),S(3),E(6),SF(7),
      CINFS1FA(ZA(4),T(4),E(7),D(4),S(4),E(8),SF(9),
      CATA STCF1/
      CALL RDATA(ZA,EA,W,RELX,KK,LIMI,LIMR,LIM4)
      CALL INITI(LIMI,LIMR,TITLE,PROTOB,NSTATB,LIM4,MNCD,NE4,NF4,
      1NRCS)
      CALL ZEFFC1(R1(NP14),R2(NP12),RE(NP15),RR(NP3),RS(NP9)
      2,RE(NP10),RR(NP2),RR(NP2),RE(NP1),RE(NP5),
      3I2(NP11),I2(NP12),RR(NP13),RR(NP14),RR(NP17),RR(NP5),RS(NP7)
      4,RE(NP4),RE(NP11),R2(NP5),R2(NP21),RE(NP22),R2(NP2),
      5MS,12(NP13),I2(NP14),RR(NP26),RR(NP23),RS(NP19),MNCD,NE4,
      6NF4,NPCVS)
      CALL INF1(R1(NP1),RS(NP2),R2(NP3),RE(NP4),RS(NP5),I2(NP13),
      1T2(NP14),R2(NP22),R2(NP21),RE(NP15),RS(NP2),RS(NP12),RS(NP13),
      2R8(NP23),RS(NP24),RS(NP25),R2(NP26),RE(NP27),
      3I2(NP11),I2(NP12),TITLE,NFOTB,MNCD,NE4,NNE4,NRCS)

```

FILE: TURBO FCRTRAN A1 NAVAL POSTGRADUATE SCHOOL

```
C  
C  
C          CALCULATE 'V' AND 'U' VELOCITIES AND NEW NODEL CENSITY  
ECC  CALL DIST(R8(NP2),I2(NP14),R8(NP1),R8(NP12),R8(NP16),R8(NP17),  
1R8(NP3),R8(NP8),R8(NP9),R8(NP4),R8(NP5),R8(NP13),R8(NP27)  
2,R8(NP23),R8(NP24),R8(NP25),R8(NP7),R8(NP10),R8(NP11),  
3I2(NP13),R8(NP26),R8(NP29),R8(NP21),R8(NP22),R8(NP19),NNOD,  
4NSTATB,NNCD,NE4,NE4,NNROWS)  
C  
C          CALL FCAL(R8(NP14),W,R8(NP6),ZA,EA,R8(NP8),  
1R8(NP2),R8(NP1),R8(NP14),R8(NP10),I2(NP12),I2(NP14),R8(NP11),  
2I2(NP13),R8(NP28),R8(NP29),R8(NP24),NNOD,NE4,NNF4,NNCNS)  
C  
C          CALL STIFF(R8(NP2),R8(NP1),R8(NP3),EM1,ZA  
1,EA,W,I2(NP14),R8(NP16),R8(NP12),R8(NP14),R8(NP15),R8(NP20),I2(NP12),NNOD,NE4,NNE4,NNROWS)  
C  
C          SOLVE SYSTEM OF EQUATIONS  
CALL DSINC(R8(NP2C1),R8(NP14),NNOD,KS)  
C  
C          CALL REPLA(R8(NP12),R8(NP14),R8(NP15),NNOD,NE4,NNE4)  
C  
C          COMPARE NEW AND OLD STREAM FUNCTION DISTRIBUTIONS  
CALL TEST(R8(NP12),R8(NP13),X,NNOD,NE4,NNE4)  
C  
C          CALL RELAX(RELX,R8(NP12),R8(NP13),I2(NP12),NNOD,NE4,NNE4)  
C          TEST FOR STREAM FUNCTION CONVERGENCE  
KK = KK + 1  
IF(X.LE.2.0D-02) GOTO 800  
C  
C          CALL NOCCR(X,R8(NP14),R8(NP12),R8(NP13),R8(NP20),IFL,NNOD,NE4,  
1NE4)  
C          NEXT ITERATION  
C          IF(IFL.EC.C) GOTO 900  
ECC  GOTO 300  
ECC  IFL = 1  
C  
C          CALL OUTFLT(X,KK,R8(NP8),R8(NP9),R8(NP10),R8(NP11),  
1,R8(NP12),R8(NP2),R8(NP1),R8(NP14),R8(NP16),R8(NP17),  
2,R8(NP7),R8(NP23),R8(NP24),R8(NP25),R8(NP26),R8(NP27),  
3R8(NP4),R8(NP5),NNOD,NE4,NE4,NNROWS)  
C          FMAX = 20.0D0  
C          RMIN = 0.0D0  
C          CALL MPLOCT(R8(NP1),RMAX,2*IN,FE(NP2),C4(NP01),C4(NP02),C4(NP03),  
1C4(NP04),NNOTCF,FE(NP4),R8(NP5),R8(NP21),R8(NP22),R8(NP14),  
2C4(NP05),NNOD,NE4,NNE4,NNROWS)  
C  
C          STOP  
ECC
```

FILE: TURBO FORTRAN A1 NAVAL POSTGRADUATE SCHOOL

```
C
C***** BEGINNING OF THE SUBROUTINE SECTION:
C***** CONTAINS FCAL, VEL, SLIVE, JACCB,
C***** SHAPE, MFLCT, AND CSIMC.
C*****+
C
C      SUBROUTINE INITI(LIMI,LIMP,TITLE,NRCTCB,NSTATB,LIN4,NACD,NE4,
1NRF4,NRC(1S)
C
C
C
C
C
C      IMPLICIT REAL*8(I-H,F-2)
C      INTEGER*4 NREAD,NWRITE,IC,NACD,NE4,NN4,NPCWS,LIMP,LIMI
C      COMMON /ACCOUNT/ NCOL,NCOL1,NN,NE
1,NE4,NN4C,NACD
C      COMMON /ACCOUNT/ NROW,NROW1,KK
C      COMMON /FCINT/ NP1,NP2,NP3,NP4,NP5,NP6,NP7,NP8,NP9,NP10,NP11,
NP12,NP13,NP14,NP15,NP16,NP17,NP18,NP19,NP20,NP21,NP22,NP23,
NP24,NP25,NP26,NP27,NP28,NP29,NP30,NPI1,NPI2,NPI3,NPI4,NPI5,
3NFC1,NPC2,NPC3,NPC4,NPC5,NPC6
C      COMMON /LIC/ NRE2C,NWRITE
C      DIMENSION TITLE(1C)
C
C      READ(NREAD,100)TITLE
100     FORMAT(1CA4)
C
C
C      READ IN NUMBER OF NODES AND NUMBER OF ELEMENTS
C
C      READ(25,200) NN,NROW,NCOL,NPCW1,NCOL1,NRCTCB,NSTATB
200     FORMAT(7I5)
C
C
C      NE = 3
C      NE = NROW * NCOL
C      NACD = NN
C      NE4 = NE
C      NRF4 = NE
C      NFCWE = 2 * NROW + 1
C      NPI1 = 1
C      NP1 = NPI1 + NACD
C      NP2 = NPI2 + NACD
C      NP3 = NPI3 + NACD
C      NP4 = NPI4 + NACD
C      NFE = NFE + NACD
C      NPI5 = NPI5 + NACD
C      NPI6 = NPI6 + NACD
C      NPI7 = NPI7 + NACD
C      NPI8 = NPI8 + NACD
C      NPI9 = NPI9 + NACD
C      NPI10 = NPI10 + NACD
C      NPI11 = NPI11 + NACD
C      NPI12 = NPI12 + NACD
C      NPI13 = NPI13 + NACD
C      NPI14 = NPI14 + NACD
C      NPI15 = NPI15 + NACD
C      NPI16 = NPI16 + NACD
C      NPI17 = NPI17 + NACD
C      NPI18 = NPI18 + NACD
C      NPI19 = NPI19 + NACD
C      NPI20 = NPI20 + NACD
```

FILE: TLF80 FORTRESS AI NAVAL POSTGRADUATE SCHOOL

```

SLPRCUTIME JACCB(E1,Z1,D,F,RCS,ZCS,R JAC)
+
+
+ THIS SLPRCUTIME COMPUTES THE JACOBIAN MATRIX
+ REQUIRED FOR THE COORDINATE TRANSFORMATIONS.
+ CALL STATEMENT DEFINITION:
+ E1 = VALUE OF THE ETA INPUT
+ Z1 = VALUE OF THE XCOLES INPUT
+ D(I) = THE DERIVATIVE OF S(I) WRT Z
+ F(I) = THE DERIVATIVE OF S(I) WRT E
+ RCS = R COORDINATES OF THE ELEMENT'S NODES
+ ZCS = Z COORDINATES OF THE ELEMENT'S NODES
+ R JAC = 2X2 JACOBIAN MATRIX OUTPUT
+
+
+

```

```

IMPLICIT REAL*8(A-H,P-Z)
DIMENSION ICN FJAC(2,2),C(5),E(5),PC$(3),ZC$(8)
RJAC(1,1) = X*DC
FJAC(1,2) = Y*DC
RJAC(2,1) = C*DC
RJAC(2,2) = D*DC
DC(1) = (E1 + 2.00*Z1 + 2.00*Z1*E1 + E1*E1)/4.00
DC(2) = -(Z1 + Z1*E1)
DC(3) = (-E1 + 2.00*Z1 + 2.00*Z1*E1 - E1*E1)/4.00
DC(4) = (-1.00 + Z1*E1)/2.00
DC(5) = (E1 + 2.00*Z1 - 2.00*Z1*E1 - E1*E1)/4.00
DC(6) = -Z1 + Z1*E1
DC(7) = (-E1 + 2.00*Z1 - 2.00*Z1*E1 + E1*E1)/4.00
DC(8) = (1.00 - Z1*E1)/2.00
E(1) = (Z1 + 2.00*Z1 + Z1*Z1 + 2.00*Z1*E1)/4.00
E(2) = (1.00 - Z1*Z1)/2.00

```

FILE: TURBO FCRTRAN A1 NAVAL POSTGRADUATE SCHOOL

```
E(3) = (-Z1 + 2.00*E1 + Z1*Z1 - 2.00*Z1*E1)/4.00
E(4) = -E1 + E1*Z1
E(5) = (Z1 + 2.00*E1 - Z1*Z1 - 2.00*Z1*E1)/4.00
E(6) = (-1.00 + Z1*Z1)/2.00
E(7) = (-Z1 + 2.00*E1 - Z1*Z1 + 2.00*Z1*E1)/4.00
E(8) = -(E1 + Z1*E1)
CC 100 I = 1,8
RJAC(1,1) = RJAC(1,1) + D(I)*ZCS(I)
RJAC(1,2) = RJAC(1,2) + D(I)*RCS(I)
RJAC(2,1) = RJAC(2,1) + E(I)*ZCS(I)
RJAC(2,2) = RJAC(2,2) + E(I)*RCS(I)
```

10C CCNTINUE  
RETURN  
END

SLBRGROUTINE DIST(FC,NODE,ZC,PSI,RHC,RFCN,R,VZ,VP,ALP,PE,H,APL,  
1,RFCTT,TEMP,TTCT,PTCT,SPRESS,HS,VU,VJ,ATE,HP,EMTACP,DEV1,PFAT,  
2ETA,AROTCE,ISTATE,MNCD,NF4,MNE4,NRWS)

THIS SUBROUTINE CALCULATES L AND V VELOCITIES AND A NEW  
NODAL DENSITY FROM A KNOWN PSI DISTRIBUTION AT EACH OF  
THE NODES IN THE SYSTEM.

CALL STATEMENT DEFINITIONS:

NE = NUMBER OF ELEMENTS IN THE MESH  
NN = NUMBER OF NODES IN THE MESH  
RC = ARRAY OF R NODAL COORDINATES  
NODE = ARRAY OF ELEMENTAL NODE ASSIGNMENTS  
G = RATIO OF SPECIFIC HEATS  
FG = GAS CONSTANT  
TT = INLET TOTAL TEMPERATURE  
RHCT = INLET TOTAL DENSITY  
RFCN = WORK VECTOR WITH NEW DENSITY DISTRIBUTION  
ZC = ARRAY OF NODAL Z COORDINATES  
FSI = NODAL STREAM FUNCTION VECTOR  
RHC = NODAL STATIC DENSITY VECTOR  
F = ACCAL BLEEDAGE FACTOR  
LINLET = INLET AXIAL VELOCITY  
VZ = NODAL AXIAL VELOCITY  
VP = NODAL RADIAL VELOCITY  
FFCSTA = INLET STATIC DENSITY  
ALP = ACCAL ABSOLUTE FLOW ANGLE ARRAY  
FE = NODAL RELATIVE FLOW ANGLE VECTOR  
F = ACCAL TOTAL ENTHALPY VECTOR  
VG = FCTCF SPEED IN RAD/SEC

```
IMPLICIT REAL*8(A-H,F-Z)
INTEGER*4 NREAD,NWRITE,IC,MNCD,MNE4,NRWS,LIMR,LIMI
COMMON /ACCOUNT/ NCOL,NCOL1,NN,NE
1,NAME,NNPC,NNNAC
1,COMMON /ACCNT/ VRCN,VRDN,KK
COMMON /FCCLT/ FG,G,CF,PT,TT,VZ,WDCT,RFCT,PHOSTA,
1,INLYT,LELT,PSI,THT,F2,F1,1,PS
1,DIMENSION ZC(1),U(1),H(1),E(1),RCS(1),HS(1),HU(1)
1,DIMENSION RBL(1),PHLT(1),EXP(1),TCT(1),PTCT(1)
1,DIMENSION VZT(1),VS(1),PH(1),PSI(1),VU(1)
1,DIMENSION P(F),E(F),SF(S),FC(S),ZC(S),ST(1)
1,DIMENSION F1(8),Z1(8),LCP(1),E(1),EN-E(1),HR(1)
1,DIMENSION R(8),DZ(8),RJAC(2,2),PH(1),DEV1(1),PFAT(1)
DATA Z1/1.00,-1.00,-1.00,-1.00,1.00,1.00,1.00,1.00/
DATA E1/1.00,1.00,1.00,0.00,-1.00,-1.00,-1.00,0.00/
```

BEGIN WITH MID-NODE OF FIRST ELEMENT AND THEN CYCLE  
THROUGH EACH ELEMENT.

FILE: TLFBC FCRTRAN AI NAVAL POSTGRADUATE SCHOOL

```

GM1 = G = 1.CDC
ISTA(1) = 3
ISTA(2) = 2
ISTA(3) = 1
ISTA(4) = 1
ISTA(5) = 1
ISTA(6) = 2
ISTA(7) = 2
ISTA(8) = 3
CC 100 I = 1,MPOW1
    H(I) = CP * (TCT(I) * BTU * GC * 144.000
    HS(I) = CP * TEMP(I) * BTU * GC * 144.000
100  CCCONTINUE
CC 200 II = 1,NE
    NTEL = NTE(II)
    IF(NTEL - 2) 210,220,230
C
C
210  DC 24C N = 1,NE
    ISTA1 = ISTA(M)
    CALL SLINF(FC,PSI,VZ,VE,NCDE,V11,FC11,PC12,ALP1,NTEL,RHC1,
    1RHC1,ALP1,E1,I1,ISTA1,M,PHOTT,TEMP,T1,T2,PRESS,PTOT,B,F1,T1,PT1,
    2FFFT1,H1,FS1,B2,OPSI2,OPSI22,F,HS,Z1,VZ1,VRL,ZC,ENTRCF,ENT1,
    3NACD,NE4,NE4,NRCWS)
    IJK = NE4 - NRCWS
    IF(I1 LE E1) GOTO 211
    IF(I1 GE B2) GOTO 211
    OPSI22 = 0.000
    CALL FCCT(FC11,PC12,T1,P1,TT1,FT1,RHCT1,H1,FS1,
    2RHC1,ALF1,ALF2,OPCI2,OPCI22,ISTA1,T2,TT2,P2,PT2,RHCT2,
    3V1,VZ1,NCDE,NE4,NRCWS)
    IF(ISTA1.EQ.11) GOTO 24C
    NMV = NCDFE(II,N)
    PRESS(NM) = F2
    PTOT(NM) = PT2
    TTCT(NM) = T12
    TEMP(NM) = T2
    F(NM) = H1
    HS(NM) = FS1
    RHCNIM1 = RHC2
    RHOTT(NM1) = RHCT2
    ALP1NM1 = ALF2
    VZ(NM) = 144.000 * OPSI2 / (RHCT2 * B2 * PC12 / 12.000)
    VP(NM) = -144.000 * OPSI22 / (RHC2 * B2 * PC12 / 12.000)
    VL(NM) = VZ2 * DTAN(ALP2)
    WPLRNM1 = PC12 * VL(NM)
    ENTFCF(NM) = ENT1
    CONTINUE
    GOTO 200
C
C
220  DC 25C N = 1,NE
    ISTA1 = ISTA(M)
    RMAX1 = PC(NFCNPB)
    PMINI = PC(NFCNPB + MPOW1 - 1)
    CALL SLINF(FC,PSI,VZ,VE,NCDE,V11,FC11,PC12,ALP1,NTEL,RHC1,
    1RHC1,ALP1,E1,I1,ISTA1,M,PHOTT,TEMP,T1,T2,PRESS,PTOT,B,F1,T1,PT1,
    2FFFT1,H1,FS1,B2,OPCI2,OPCI22,F,HS,Z1,VZ1,VRL,ZC,ENTRCF,ENT1,
    3NACD,NE4,NE4,NRCWS)
    CALL FCCT(FC11,VV1,BETA1,BETA2,RAUCHR,PMAX1,PMINI,
    1CFV,FC12,RH11,FFC2,T1,T2,T3,T4,LP1,RHCT1,PTOT2,B,F1,T1,PT1,
    2OPSI22,OPCI2,ALF2,P2,P12,T1,TT2,TPXES,VM2,RPASS,NCDE,NE4,NE4,
    3NACD)
    NMV = NCDFE(II,N)
    PFACT = (1.00 - RPASS)*(1.00 - RPASS)
    IF(ISTA1 - 2) 251,252
    PRESS(NM) = (F2 + F1) / 2.000
    PTOT(NM) = (PT2 + PT1) / 2.000
    TTCT(NM) = (T12 + TT1) / 2.000
    TEMP(NM) = (T2 + T1) / 2.000
    .

```

FILE: TLFDC FCRTRAN AI NAVAL POSTGRADUATE SCHOOL

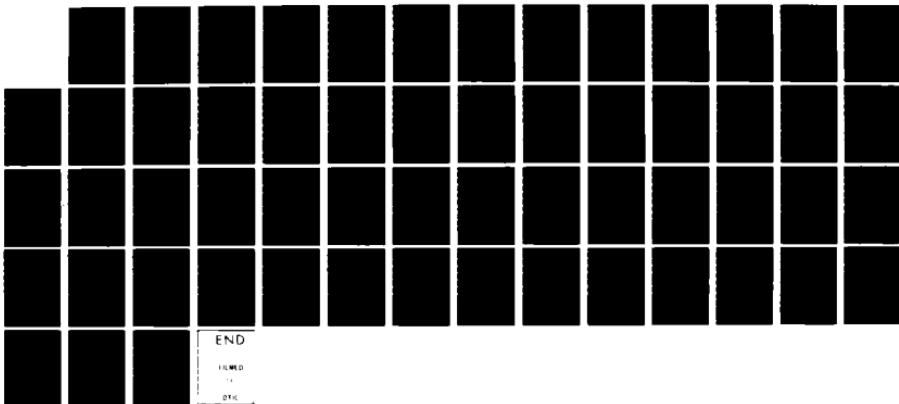
```

H(NIM) = CP * TTCT(NIM) * BTU * GC * 144.000
HS(NIM) = CP * TEMP(NIM) * RTJ * GC * 144.000
PHO(NIM) = (RHC2 + RHC1) / 2.000
PHOTT(NIM) = (PHOT2 + PHOT1) / 2.000
ALP(NIM) = (ALF2 + ALP1) / 2.000
BE(NIM) = (BETA2 + BETA1) / 2.000
VZ2 = 144.000 * CPSIR2 / (RHC2 * R2 * RCI2 / 12.00)
VP2 = -144.000 * CPSIZ2 / (RHC2 * R2 * RCI2 / 12.00)
VZ(NIM) = (VZ1 + VZ2) / 2.000
VR(NIM) = (VR1 + VR2) / 2.000
VU(NIM) = (VM2 + VM1) * CTAN(ALP(NIM)) / 2.000
WU(NIM) = (VM2 + VM1) * CTAN(ALP(NIM)) / 2.000
WPL(NIM) = (PC11+RCI2) * VU(NIM)/2.000
FRIN(NIM) = H(NIM) - WG*(RCI2 + RCI1) * VU(NIM)/2.000
ENTRCF(NIM) = C.5CDC * ENT
GOTO 25C
PRESS(NIM) = P2
PTOT(NIM) = PT2
TTOT(NIM) = TT2
TEMP(NIM) = T2
CEV1(NIM) = DEV
H(NIM) = CP * TT2 * BTU * CC * 144.000
FS(NIM) = CF * T2 * BTU * GC * 144.000
RHC(NIM) = RHC2
RHOTT(NIM) = RHOT2
ALP(NIM) = ALP2
BE(NIM) = BETA2
VZ(NIM) = 144.000 * DPSIZ2 / (RHC2 * R2 * RCI2 / 12.00)
VP(NIM) = -144.000 * DPSIZ2 / (RHC2 * R2 * RCI2 / 12.00)
VU(NIM) = VM2 * CTAN(ALP2)
WU(NIM) = VM2 * CTAN(BETA2)
WPL(NIM) = RCI2 * VU(NIM)
HS(NIM) = H(NIM) - WG * RCI2 * VU(NIM)
ENTRCF(NIM) = ENT
GOTO 25C
BE(NIM) = BETA1
WU(NIM) = VM1 * CTAN(BETA1)
HP(NIM) = H(NIM) - WG * RCI1 * VU(NIM)
ETA(NIM) = (TT1/(TT2 - TT1))*(((PT2/PT1)**(G41/G1) - 1.000)
PRAT(NIM) = PT2/PT1
CONTINUE
GOTO 200
}
25C
26C M = 1,NE
ISTAI = ISTA1(v)
RMAXI = RC(ISTATB)
RMINI = RC(ISTATB + RCM1 - 1)
NIM = RCODE(11,1)
CALL SLIME(FC,FSI,VZ,VE,NCPD,VM1,FC11,PC12,ALP1,ATC1,RHC1,
1RHC1,ALD,E1,I1,IS1,TT1,-HOTT,TEND,TTOT,PRESS,PTOT,B1,F1,T1,TT1,PT1,
2PFCT1,L1,FSI1,B2,PSI12,CPSIZ2,FSI1,VM1,VZ1,VR1,ZC,ENTRCF,ENT1,
3RACCD,VE4,NCPD,ATC1)
IF(ISTA1.EQ.1) GOTO 264
BETAI = ALP1
GOTO 265
264
BETAI = ALF(NIM)
CALL STAT(RC11,VM1,BETA1,BETAI,BETA2,R13CHR,2MAX1,0MIN1,
1CEV,FC12,RHOT1,FSI1,T1,TT1,P1,PT1,ALP1,RHC1,RHOT2,PSI1,STAT,DPSIR2,
2CFCSI12,FCAC,ALF2,P1,RT1,T2,TT2,BU,FG,VZ2,RPASS,WPDRIVE4,WFE4,
3RACCS)
RF4GT = (1.00 - RPASS)*(1.00 - RPASS)
IF((ISTA1.EQ.1)) 263,261,262
PRESS(NIM) = (P2 + PI) / 2.000
PTOT(NIM) = (PT2 + PT1) / 2.000
TTOT(NIM) = (TT2 + TT1) / 2.000
TEMP(NIM) = (T2 + T1) / 2.000
H(NIM) = F1
FS(NIM) = H<1
RHO(NIM) = (RHC2 + RHC1) / 2.000
PHOTT(NIM) = (PHOT2 + PHOT1) / 2.000

```



AD-A124 987 FINITE ELEMENT PROGRAM FOR CALCULATING FLOWS IN  
TURBOMACHINES WITH RESULTS FOR NASA TASK-1 COMPRESSOR  
(U) NAVAL POSTGRADUATE SCHOOL MONTEREY CA J A FERGUSON  
UNCLASSIFIED OCT 82 F/G 28/4 NL 3/3

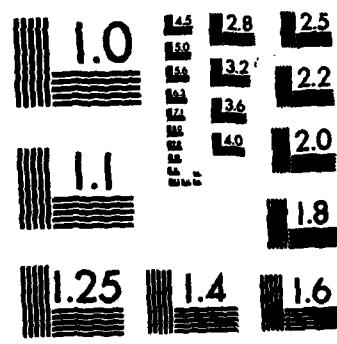


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MICROCOPY RESOLUTION TEST CHART  
NATIONAL BUREAU OF STANDARDS-1963-A

FILE: TLFBO FCRTRAN AI NAVAL FCSTGRADUATE SCHOOL

C REAC NODE NUMBERS, NCAL COORDINATES (IN INCHES), AND  
C NCAL BLOCKAGE FACTOR. INLET STATION ZC(1) MUST BE = 0.00.  
C LAST NCAL ZC(NN) MUST BE AT THE CUTLET STATION.

```

DC 170 I = 1,NN
READ(2C,1CCC)ZC(I),PC(I),E(I)
FORMAT(3F15.11)
10CO
17C CCR TINUE
CC 171 I = 1,NN
E(I) = E(I) * 0.91
171 CCR TINUE
IT1 = MRC(TC8
IT2 = IT1 + MRC(W1
IT3 = IT2 + MRC(W1 + 1
IT4 = IT3 + MRC(W1
IT5 = IT4 + MRC(W1 + 1
IT6 = IT5 + MRC(W1
IT7 = IT6 + MRC(W1
IT1A = IT7 + 1
IT2A = IT8 + 1
IT3A = IT9 + 1
IT4A = IT10 + 1
IT5A = IT11 + 1
IT6A = IT12 + 1
IT7A = IT13 + 1
IT8A = BLCKAGE(I,FCF) 704 DESIGN
P(IT3) = B(I,IT2) * 4 0.4600
P(IT4) = B(I,IT3) * 4 0.4200
P(IT5) = B(I,IT4) * 4 0.4000
P(IT3A) = B(I,IT5) * 4 0.3800
P(IT4A) = B(I,IT6) * 4 0.3600
P(IT5A) = B(I,IT7) * 4 0.3400
TIP BLCKAGE(I,FCF) 504 DESIGN
P(IT2) = B(I,IT1) * 4 0.3500
P(IT4) = B(I,IT3) * 4 0.3200
P(IT5) = B(I,IT4) * 4 0.3000
P(IT3A) = B(I,IT6) * 4 0.2800
P(IT4A) = B(I,IT7) * 4 0.2600
P(IT5A) = B(I,IT8) * 4 0.2400
P(IT3B) = B(I,IT9) * 4 0.2200

```

READ IN CONNECTIVITY MATRIX. LOCAL NODE NUMBERS START AT ELEMENT'S UPPER RIGHT HAND CORNER AND TRAVERSE CCW.  
READ IN THE ELEMENT TYPE IDENTIFICATION

CC 180 J = 1,NE  
READ(3C,101C) NODE(J,1),NCDE(J,2),NODE(J,3),NODE(J,4)

FILE: TURBO FCRTRAN A1 NAVAL POSTGRADUATE SCHOOL

```

1010 1 ,NODE(J,5),NCE(J,6),NOCE(J,7),NOCE(J,8),NTE(J)
      FORMAT(7I5)
1015  CCNTINUE
      READ IN INLET THERMODYNAMIC QUANTITIES; FLOW RATE,INLET
      U VELOCITY,OUTLET U VELOCITY,RHOT,RHGSTA,PTOT,TTCT,N
      UNITS ARE AS FOLLOWS: FLOW RATE (LBM/SEC)
      VELOCITY (FT/SEC) ; RHOT AND RHGSTA (LBW/CU FT) ;
      FTCT (PSF) ; TT (DEGREES RANKINE) ; SPEED (RPM)
1024  READ(125,1025)PRES,PT,TEM,TT,RHGSTA,RHCT
1025  FCFVAT(4F11.6,2F12.8)
      READ IN FLUID/GAS CONSTANTS,R(GAS CONSTANT),GAMMA, CP(RTU/LBM-R)
1027  READ(125,1027)WCCT,CP,RG,G,SPEED,UINLET
      FCPMAT(4F12.7,F9.1,F13.8)
      FIND OMEGA (RAD/SEC)
      WG = SPEED*2.00*3.141593D0/60.00
      LINLET = LINLET*12.000
      COMPLETE THE FIRST ESTIMATES OF UVEL(I) AND RHO(I).
      CC 166 I = 1,NN
      UVEL(I) = UINLET
      PHO(I) = RHGSTA
      TEMO(I) = TEM
      TTOT(I) = TT
      PRESS(I) = PRES
      PTOT(I) = PT
      RHOTT(I) = RHCT
1026  CCNTINUE
      READ NCSES WHERE PSI IS SPECIFIED
      NBC = 2 * NCCL1 + NRGW1 - 2
      CC 190 I = 1,NNBC
      READ(35,102C) NBC(I)
      FCPMAT(15)
1020  CCNTINUE
      READ IN THE FIRST ESTIMATE OF SYSTEM'S PSI DISTRIBUTION
      CC 103 I = 1,NN
      READ(40,1021) FSI(I)
      PSIC(I) = FSI(I)
      FCPMAT(F15.11)
1021  CCNTINUE
      LINLT = LINLET / 12.000
      FIND NCSES WHERE F(F,Z) IS SPECIFIED
      NNABC = NN - NBC
      PRINT ALL INPUT DATA
      CC TO 11C1
1022  WRITE(NWFITE,1C30)TITLE
      FCPMAT(11,11,1,1,2CX,10A4)
1023  WRITE(NWFITE,1C4C)NN,NE
      FCPMAT(' ',1X,'NC. CF MODES = ',I3,27X,
      1'AC. OF ELEMENTS = ',I2,1)
1024  WRITE(NWFITE,1C41)RCN,NCOL
      FCPMAT(S,'NC. CF RCNS = ',I3,28X,
      1'AC. OF COLUMNS = ',I2,1)
      WRITE(NWFITE,1C45)
      FCPMAT(' ',1,2CX,'SUMMARY OF Nodal Coordinates')
      GOTO 9999
1025  WRITE(NWFITE,1C50)
      FCPMAT(' ',1,NCDF,5X,I2(I)',IIX,I2(I)',IIX,'B(I)',I
      7X,'ARS FLOW ANG',3X,'REL FLOW ANG',/)
```

FILE: TLRBO FCRTRAN A1 NAVAL POSTGRADUATE SCHOOL

```

C = 1.
CC 1052 I = 1,NN
      WRITE(NWRITE,1060)(I,ZC(I),RC(I),E(I),ALP(I),RE(I))
      IF(I.EQ.NN) GOTO 1054
      IC =
      QQ = FLCAT(IC) / 40.
      IF(IC.EQ.0) GOTO 1052
      WRITE(NWRITE,1053)
      FORMAT('1')
      WRITE(NWRITE,1045)
      WRITE(NWRITE,1050)
      C = C + 1.
      GOTO 1053.
1054 WRITE(NWRITE,1053)
1055 CCFORMAT('1',I3,2X,E13.6,2X,E13.6,2X,E13.6,2X,E13.6)
1060 WRITE(NWRITE,1062)
1062 FCFORMAT('1',7X,'SYSTEM TCPOLCGY://,2X,'ELEMENT',
11C2,PNTDESI,4C2,'TYPE CF ELEMENT')
      WRITE(NWRITE,1062)(I,NODE(I,1),NODE(I,2),NODE(I,3),NODE(I,4),
1,NCDF(I,5),NODE(I,6),NODE(I,7),NODE(I,8),NTE(I),I=1,NE)
      FCFORMAT('1',3X,I3,3X,I3,3X,I3,3X,I3,3X,I3,3X,I3,3X,I3,
15X,I3)
      WRITE(NWRITE,1064)WDT,RHOT,PT,TT,SPEED,UINLT,PG,C,CP
      FCFORMAT('1',14X,'INLET THERMODYNAMIC VARIABLES ARE AS FOLLOWS',
1//,4X,'FLC',RATF,'1',E13.6,'LEM/SEC',//,4X,'TOT DENSITY = ',
4,E13.6,'LBF CL FT',//,4X,'TCT PRESSURE = ',E13.6,'PSI',//,
3,4X,'TEMPERATURE = ',E13.6,'DEG RANKINE',//,
4,4X,'ROTATIONAL SPEED = ',E13.6,'RPM',//,
5,4X,'INLET ELECTRICALITY = ',E13.6,'FT/SEC',//,
7,4X,'GAS CONSTANT = ',E13.6,//,
8,4X,'STATIC SPECIFIC HEATS = ',E13.6,//,
9,4X,'SPECIFIC HEAT AT CONSTANT PRESSURE = ',E13.6,/1
      WRITE(NWRITE,1064)RHOSTA
1065 FCFORMAT('1',3X,'STATIC CENDSITY AT INLET = ',E13.6)
      WRITE(NWRITE,1066)
      WRITE(NWRITE,1070)
1070 FCFORMAT('1',4X,'IF PSI IS SPECIFIED',//,4X,
1,NCDF,1C80,F5(1)) )
      WRITE(NWRITE,1070)(NCDF(I),PSI(NCDF(I)),I=1,NNBC)
1080 FCFORMAT('1',5X,I3,1C80,E13.6)
5555 WRITE(NWRITE,1053)
      RETURN
      END
C
C
      SUBROUTINE ZERCI(F,PSI,RHS,UVEL,VVEL,TVEL,PRESS,RC,ZC,
1R,AFS,1BC,1RL,RHC,1RCH,1,1IS,ALP,TVEL,RE,DCV1,PPAT,EM,
2EM$,ITE,NCDF,MP,ETATCP,ETA,NNBC,NC4,NC54,AR4NS1)
C ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++
C ++
C ++
C ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++
C ++
C ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++
C ++
C ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++
C
      IMPLICIT REAL*8(A-H,P-Z)
      INTEGER*4 AR4NS1,AR4NS2,ITE,IC,NCDF,NC4,NC54,NC7WS,LIMR,LIMI
      CCNEN/FCFM/PG,G,CP,PT,TT,PG,WDT,RHOT,RHOSTA,
      ITINLET,LCLINLET,PSI,I,ITU,F2I,F1,I,SC
      CCPMN/NCDF,I/NCDF,NCDF,NC,NC
      1,ARF,NNBC,NNBC
      CCPDN/NCDF,I/RHC,MRDW1,KK
      DIMENSION ATE(1),NCDF(1,1)
      DIMENSION E1,(E,1),F(1),P(1),PHS(1),EM(1,1),ETA(1)
      DIMENSION UVEL(1),VVEL(1),TVEL(1),PRESS(1),SC(1),DCV1(1)
      DIMENSION RE(1),HL(1),ZC(1),RHC(1),RF2(1),HP(1),PSAT(1)
      DIMENSION HS(1),ALP(1),EE(1),TVEL(1),ENTROP(1)
      DIMENSION AR4NS1,AR4NS2
C
      INITIALIZE ALL MATRICES

```

FILE: TURBO FCRTRAN A1 NAVAL POSTGRADUATE SCHOOL

```

CC 110 I = 1,NN
F(I) = C,CC
PSI(I) = 0,00
RHS(I) = C,CC
UVEL(1) = 0,00
VVEL(1) = 0,00
TVEL(1) = C,CC
PRESS(1) = C,CC
RC(I) = 0,DC
ZC(I) = 0,DC
B(I) = 0,EO
NFS(I) = C
ETA(I) = C,CC
NBC(I) = J
WFL(I) = C,CC
RHO(I) = 0,00
RHON(1) = 0,00
ENTRCF(I) = C,CC
F(I) = 0,DC
HS(I) = 0,00
MF(I) = 0,DC
ALP(I) = 0,EO
TVEL(1) = C,CC
BF(I) = 0,CC
BEVI(I) = 0,00
PRAT(I) = C,CC
110  CCNTINUE
CC 150 I = 1,NN
DO 150 J = 1,NN
   EN(I,J) = C,DC
150  CCNTINUE
CC 160 I = 1,NNE
DO 160 J = 1,NNE
   EN(I,J) = 0,00
160  CCNTINUE
CC 165 I = 1,NE
   NTE(I) = C
DO 165 J = 1,NNE
   NCCE(I,J) = 0
165  CCNTINUE
RETURN
ENC
CC
CC
      SUBROUTINE OUTPUT(X,KK,UVEL,VVEL,TVEL,TWEL,PSI,FC,ZC,
1     ,RHO,WFL,H,HS,TEMP,TTOT,PRESS,PTCT,FCFT,ALP,BE,
2     ,NNE4,NNE4,NCAS)
CC
CC
      IF(FL.EQ.1) GOTO 450
      IF(IFLT(NHFT,1).NE.0) KK=1
1600  FORMAT(1X,'PROGRAM TERMINATED ON ITERATION NO.',I3,/,1,019.12)
      1' RESULTS WHICH FOLLOW ARE FOR CONVERGENCE EPSILON = 1,019.12)
      GOTO 1310

```

FILE: TLR80 FORTRAN A1 NAVAL POSTGRADUATE SCHOOL

```

C      GCTC1104
4SC   WRITE(NWFITE,1200)KK,X
1200  FORMAT(' ','STOPCAV FUNCTION CONVERGENCE CRITERION SATISFIED ON ITC
          ITRATION NUMBER ',I3,'//',I3,'RESULTS ARE AS FOLLOWS FOR CONVERGENCE
          2EPSILON = ',D15.12)
C      WRITE OUT RESULTS
C      GCTC 131C
C      CHANGE UNITS OF VELOCITY TO FT/SEC
C
1104  CC 600 I = 1,AN
      UVEL(I) = UVEL(I)/12.00
      VVEL(I) = VVEL(I)/12.00
      TVEL(I) = TVEL(I)/12.00
      TWEL(I) = TWEL(I)/12.00
      ALP(I) = ALP(I) * 57.2578
      WRL(I) = WRL(I) / 144.000
      BE(I) = BE(I) * 57.2578
      H(I) = H(I) / (GC*144.000*BTU)
      HS(I) = HS(I) / (GC*144.000*BTU)
C      CONTINUE
      WRITE(NWFITE,1103)
1105  FCFORMAT(' ','')
      WRITE(NWFITE,1105)
1110  FCFORMAT(' ','//','27X,'FINITE ELEMENT RESULTS',//,
          1,'NODEC',5X,'PSI(I)',10X,'VZ',13X,'VR',12X,'R(I)',10X,'DENSITY')
      IC = 1
      CC 1122 I = 1,AN
      WRITE(NWFITE,1120)(I,PSI(I),UVEL(I),VVEL(I),RC(I),RC(I))
      IF(I.EQ.NN) GOTO 1124
      IC = I
      CC = FLCAT(IC)/4C
      IF(C.NE.CC) GCTC 1123
      C = C + 1
      WRITE(NWFITE,1C53)
      WRITE(NWFITE,111C)
      GOTO 1122
      WRITE(NWFITE,1C53)
1123  CCNTINUE
1124  CCNTINUE
1125  CCNTINUE
1126  FCFORMAT(' ',13.2X,C13.6,2X,D13.6,2X,D13.5,2X,D13.6,2X,C13.6)
1127  FCFORMAT(' ','ACCE',5X,'WRL(I)',10X,'HT',13X,'VT',12X,'WT',14X,'HS')
      C = 1
      CC 1322 I = 1,AN
      WRITE(NWFITE,1320)(I,WRL(I),H(I),TVEL(I),TWEL(I),HS(I))
      FCFORMAT(' ',13.2X,C13.5,2X,D13.6,2X,D13.6,2X,D13.5,2X,D13.6)
      IF(I.EQ.NN) GCTC 1324
      IC = I
      CC = FLCAT(IC)/4C
      IF(C.NE.CC) GCTC 1323
      C = C + 1
      WRITE(NWFITE,1C53)
      WRITE(NWFITE,1121)
      GOTO 1322
      WRITE(NWFITE,1C53)
1323  FCFORMAT(' ')
      CCNTINUE
1324  CCNTINUE
1325  CCNTINUE
1326  FCFORMAT(' ','ACCE',5X,'TEMP',11X,'TTOT',11X,'PRESS',11X,
          1,'FTOT',11X,'RHOT')
      C = 1
      CC 1423 I = 1,AN
      WRITE(NWFITE,1400)(I,TEMP(I),TTOT(I),PRESS(I),PTOT(I),RHOT(I))
      FCFORMAT(' ',13.2X,C13.6,2X,D13.6,2X,D13.6,2X,C13.6,2X,D13.6)
      IF(I.EQ.NN) GCTC 1424
      IC = I
      CC = FLCAT(IC)/4C
      IF(C.NE.CC) GCTC 1423
      C = C + 1
      WRITE(NWFITE,1C53)

```

FILE: TLRBO FCRTRAN AI NAVAL POSTGRADUATE SCHOOL

FILE: TLR80 FORTRESS AI NAVAL POSTGRADUATE SCHOOL

EAC

FILE: TURBO FCRTRAN A1 NAVAL POSTGRADUATE SCHOOL

```

EXPO2 = ((1.9828 - 0.344*RPASS + 1.563*RPASS2)/RMACHR)**EXP1
RICIFF = -2.5 + 2.55*RPASS + (5.275 + 7.5*RPASS - 2.5 * RPASS2)
1 * RMACTF**=XPC2
RIREF = R120 + RICIFF
RKCELS = C.70CC
RKCELT = 4.607*TCVLC + 24.45*TCVLC2
CELT'1 = (-0.0C1A3 + 0.0257*SIG + 0.0CC144*SIG2)*BETA1 +
1 (-1.51D-4 - 2.55D-4 * SIG - 3.102E-4 * SIG2)*BETA12 +
2 (-2.07E + 7.343 * SIG + 3.621 * SIG2)*1.00-6*BETA13
SLCPM = 0.25 + 7.06E-4 * BETA1 - 1.29E-5 * BETA12 + 3.10E-7 *
1 BETA13
ACCN = 3.35 - C.C124 * BETA1 - C.000984 * BETA12
BCCN = C.0070 - 0.00070X * BETA1 + 1.3E-5 * BETA12
CCELDI = EXP(-ACCN*SIG) + (ECCN/SIG2)*(DSIN(3.141593*SIG /
2 (1.2*57.29578)))*2
E = 0.56E - 0.00305 * BETA1 + 6.195D-5 * BETA12 - 1.4789D-6 *
1 BETA13
CEL2D = FKDELS * FKDELT * DELTEN + (SLCPM/(SIG**3)) * PHI +
1 RICIFF * CCEDCI
ACCN2 = -1.75 + 3.5*RPASS + RPASS**6.58
BCCN2 = C.29 - 5.5E-5 * RPASS + 31.8E - RPASS2 - 57.2*RPASS*RPASS2
1 + 35.15 * RFAE**2 * RPASS2
CCCN = 0.43 + 5.6 * (RPASS - C.535)**2
CCEDCI = ACJN2 + ECCN2 * RMACHR**CCON
CELRF = CEL2D + CCEDCI
CEV = DELRF + (RINC0 - RIREF)*CCEDCI
BETA2 = BETA1 - RFI - RINC0 + CEV
EPS = 1.0D-6
IT = 1
BETA1 = BETA1 / 57.29578
BETA2 = BETA2 / 57.29578
REAR = (FCII + RC12)/2.000
GM1 = G - 1.0
GM1I = 1.0 / GM1
RFC2 = PFC1*0.75
VM2 = (CSQRT((FSIP2*CPSIR2+DPSIZZ+DPSIZZ) * 144.000 /
4FC2*P2*RC12/12.00)
600 ALF2 = CTAN((LG3*CT2 - V*2*CTAN(BETA2))/VM2)
DFAC = 1 - (VM2*CCOS(BETA11))/(VM1*DCCS(BETA21)) + (VM1*CTAN(BETA11) *
1 RC11 - VM2*CTAN(BETA21)*RC12)*CCOS(BETA11)/(2.00*SIG*VM1*REAR)
DFAC2 = DFAC*DFAC
DFAC3 = DFAC2 * DFAC
DFAC5 = DFAC2 * DFAC3
TCMEG1 = (0.0C215C58 + 0.0304237*DFAC - 0.338447*DFAC2 +
1 (0.55549*DFAC - 0.60274*DFAC5) * (2.000*SIG/DCCS(BETA21))
TCMEG2 = (0.0C215C58 + 0.0319210*DFAC - C.106474*DFAC2 +
1 0.222378*DFAC3 - 0.0333925*DFAC5) * (2.000*SIG/DCCS(BETA21))
TCIFF = (TCMEG1 - TCMEG3)
IF(RPASS.GE.0.30) GOTO 100
IF(DPASS.LE.0.10) GOTO 110
TCMEG = TCMEG3 + (TCMEG1 - TCMEG3) * ((0.3-RPASS)/0.20)**2
GOTO 120
110 TCMEG = TCMEG1
GOTO 120
120 TCMEG = TCMEG3
TE1 = T21 + WG*WG = (FCI2*RCI2 - RCI1*RCI1) / (BCCN3 * 144.0001
FF1 = PT1 * (TF1/IT1)**(G*GM1I)
PR1 = PT1 * (TF1/IT1)**(G*GM1I)
PE2 = PE1 - TCMEG*(PR1 - PI)
W2 = V*2 / DCCS(BETA2)
T2 = TE1 - W2*V2 / (BCCN3 * 144.0001
P2 = PE2 * (T2 / TE1)**(G*GM1I)
RHC2 = (F2*144.0001) / (PG * T2)
VM2N = (CSQRT(CPHEIF2*DPSIR2+EPSIZZ*DPSIZZ) * 144.000 /
1 (FF1*2*RC12/12.00)
CTEST = CARB(VM2 - VM2N)
IT = IT + 1
IF(DTEST.LT.EPS) GOTO 700
VM2 = VM2N
IF(IT.LT.100) GOTO 600
WFITE(5,65)
FCFORMAT(' CONVERGENCE NOT REACHED IN 20 ITERATIONS')

```

FILE: TURBO FORTAN A1 NAVAL POSTGRADUATE SCHOOL

```

CONTINUE
RMACH2 = VM2N*VM2N*(1.0DC + (DTAN(ALF2))**2)/(G*RG*GC*T2*144.0CC)
CLNT = 1.0DC + (GM1/2.0D)*RMACH2
PT2 = P2 * QUANT*(G*GP11)
TT2 = T2 * QUANT
PFTCT2 = (PT2 * 144.0C) / (RG * TT2)
ENT = -RC*DLOG(PT2/PT1) * GC * 144.0CC
RETURN
END

SUBROUTINE STAT(KCT1,VM1,RETA1,RETA2,RMACHR,RMAX1,RMIN1,
1CEV,FCI2,RHOT1,FHC2,T1,TT1,P1,PT1,ALP1,FHC1,RHOT2,E2,ENT,DPSIR2,
2DFES12,EFAC,ALF2,P2,P12,T2,TT2,T1V2G,VM2,RPASS,NACD,NE4,NHE4,
3KPCNS)
+++++*****+++++*****+++++*****+++++*****+++++*****+++++*****+++++*****+
+++++*****+++++*****+++++*****+++++*****+++++*****+++++*****+++++*****+
+++
+++
+++++*****+++++*****+++++*****+++++*****+++++*****+++++*****+++++*****+
IMPLICIT REAL*8(I-H,P-Z)
INTEGER*4 NRSEC,ITE,IC,NCOL,NE4,NHE4,NROWS,LIMR,LIMI
COMMON /ACCOUNT/ NCUL,NCOL1,NA,NE
1,NE,VMPC,NPC,BF
COMMON /FCOM/ FC,E,CP,PT,TT,WG,WDCT,FMCT,RHOSTA,
1INLET,LCLLST,TSETI,STU,F2I2,F1I1,GC
COMMON /MCOUNT/ VROW,NROW1,KK

      FIND HFEL,WR1,AND TWEL AT LCC NODES 3,4,5(PCTOR).

RCI12 = FCI1 * RCI1
RCI13 = RCI1 * FCI12
RCI14 = RCI1 * RCI13
RCI15 = RCI1 * RCI14
RCI16 = RCI1 * RCI15
RCI17 = RCI1 * RCI16
RCI18 = RCI1 * RCI17
RCI19 = RCI1 * RCI18
X1 = VM1 / DCCS1(FETA1)
ECCN2 = 2.0DC + (F * GC * STU
RMACHR = DC*JE*((V1+V1)*(1.0C + DTAN(RETA1)*DTAN(BETA1))) /
1 +(E*GC*G*T1*144.0D))
RKAPP1 = 64.6222 - 1.89349 * RCI1 + 1.68514D-9 * RCI17
1 - .174C26 * DCCS(RCI1) + .42145 * DSIN(RCI1)
TCVC = 0.0188C99 + 2.31157D-3 * RCI1 + 1.60096D-5 * RCI12 -
17.65943D-5 * DCCS(RCI1) + 4.62218D-5 * DSIN(RCI1) + 3.34003D-7 *
2(DTAN(RC11))
SIG = 4.77577 - C.350357 * RCI1 + 0.45492D-3 * FCI12 -
1E-226631-124FCI115 - 1.31164D-3*DSIN(RC11) - 1.72765C6-5*DTAN(RC11)
FF1 = 1.6E.5AD + 1.1117Cn-10 * RCI1 - .925595 + DCOS(RCI1) +
1 1.52365*DSIN(RC11) + .021427*DTAN(RC11) - 45.7748*SLCG(RC11)
TCVC2 = TCVC * TCVC
TCVC3 = TCVC * TCVC2
BETA1 = BETA1 * 5.725578
BETA12 = BETA1 * BETA1
BETA13 = BETA1 * BETA12
BETA14 = BETA1 * BETA13
BETA15 = BETA1 * BETA14
BETA16 = BETA1 * BETA15
SIG2 = SIG * SIG
SIG3 = SIG * SIG2
RPASS = (RMAX1 - RMIN1) / (RMAX1 - RMIN1)
RPASS2 = RPASS * RPASS
RNACD = BETA1 - FKAPP1
RKISH = C.7000
PKIT = 1.0C*TCVC - 78.06*TOVC2 + 199.5*TOVC3
RITEN = SIG * (0.039*BETA1 - 2.337D-1*2*BETA16)
SLCPEN = -C.024*(2.5-SIG) - 0.002264*(1.8 - SIG)*DSRT(CA85(1.8

```

FILE: TLF80 FORTRAN AI NAVAL POSTGRADUATE SCHOOL

```

1-SIG1)*BETA1-2.1F90-8*(26.43-1.0/(SIG*SIG))*BETA13
R12D = RMISH * RKIT + PITHM + SLOPEN + PHI
EXFC1 = 2.7054 - 1.1375*RPASS + 0.4375*RPASS2
EXFC2 = ((1.3828 - 0.344*RPASS + 1.563*RPASS2)/R'MACHR)**EXP01
RICIFF = -2.2 + 2.55*RPASS + (5.275 + 7.5*RPASS - 2.5 * PPASS2)
1 * R'MACHF * EXP2
PIREF = F12D * RICIFF
RKCELS = C.7C0C
RKCELT = 4.6E7*TVC + 24.45*TVC2
CELTIN = (-0.00143 + 0.0257*SIG + 0.000144*SIG2)*BETA1 +
1 (1.517-4 - 9.25-4 * SIG - 3.102D-4 * SIG2)*BETA12 +
2 (-2.07C + 7.042 * SIG + 3.621 * SIG2)*BETA13 + 1.0D-4*BETA13
SLCPEN = 0.25 + 7.06D-4 * BETA1 - 1.28D-5 * BETA12 + 3.109D-7 *
1 BETA13
ACCN = 3.25 - C.0124 * BETA1 - 0.000584 * BETA12
BCCA = 0.0070 - C.003708 * BETA1 + 1.36D-5 * BETA12
CCEDLI = EXPD(-ACCN/SIG) + (BCCA/SIG2)*(DSIN(3.141593*SIG /
2 (1.2*57.29579))**2
E = C.566 - 0.00205 * BETA1 + 6.195D-5 * BETA12 - 1.4788D-6 *
1 BETA13
CFL2D = RKCELS * RKDELT * DELTA + (SLCPEN/(SIG**8)) * PHI +
1 RICIFF * COELCI
ACCN2 = -1.75 + 2.5*PPASS + RPASS**6.58
ACCN4 = C.29 - 5.55*PPASS + 31.84 * PPASS2 - 57.2*RPASS*RPASS2
1 + 25.16 * RPASS2 * RPASS2
CCCN = .43 + 55.6 * (RPASS - C.535) **2
COELDIF = CFL2D + BCDA2 * R'MACHF**CCON
CELDIF = CEL2D + COELDIF
CEV = DELFFF + (FINCD - RIREF)*COELD I
EE132 = EFT1 - FFI - FINCC + CEV
REAR = (ECII + RC12)/2.000
FFS = 1.00-6
11 = 1
BETA1 = EFT1 / E7.29578
BETA2 = EFT2 / E7.29578
GM1 = G - 1.0C
GM1I = 1.00 / GM1
RHC2 = RHC1
TT2 = T1
V1 = DSCRT(V1*V1*1*(1.000 + DTAN(BETA1)*DTAN(BETA1)))
VM2 = (DSCRT(CFSI2*DPSIZ2+CPSIZ2*DPSIZ2)) * 144.000 /
1 (RHC2*E2*FC12/12.00)
60C V2 = DSCRT(V2*V2*V2*(1.000 + DTAN(BETA2)*DTAN(BETA2)))
DFAC = 1 - V2/V1 + (V41*DTAN(BETA1)*DTAN(BETA2))
1 - VM2*(DTA1*(BETA2))/((C.00312054 + 0.0313210*DFC*C - 0.109476*DFAC2 +
1 C.22337E*DFAC3 - 0.0438925*DFAC51 * (2.00*SIG/CCCS(BETA2)))
PT2 = PT1 - TONEG*(PT1 - P1)
T2 = TT1 - 3*V1 * (VM2*VM2*(1.00 + (DTAN(ALP2))**2)) / (2.00 *
1 RG * RG + GC + 144.00C)
P2 = PT2 * (T2 / TT1)**(G*GM1)
RHC2 = (F2*1.44.CC) / (RG * T2)
VM2N = (DSCRT(CPSIZ2*DPSIZ2+DPSIZ2*DPSIZ2)) * 144.000 /
1 (RHC2*E2*RC12/12.00)
CTEST = ABS(VM2 - VM2N)
IT = IT + 1
IF(DTEST.LT.EFS) GOTC 700
VM2 = VM2N
IF(IT.LT.100) GOTC 600
WRITE(6,65C)
65C FCNMAT('CONVERGENCE NOT REACHED IN EC ITERATIONS')
R'MACH2 = VM2N*VM2N*(1.000 + DTAN(BETA2))**2/(G*RG*GC*T2*144.00C)
CL/NT = 1.00C + (GM1/2.00)*R'MACH2
RFCT2 = (PT2 + 1.44.RC) / (RG * T2)
EA1 = -RC*CLCC(FT2/PT1) * GC * 144.00C
RETURN
ENC
C
C
SUBROUTINE RCAT2(ZA, EA, w, FELX, KK, LIMI, LIMR, LIM4)
C ++++++*****+++++*****+++++*****+++++*****+++++*****+++++*****+
```

FILE: TURBO FORTRESS AI NAVAL POSTGRADUATE SCHOOL

FILE: TLF83 FCR TRAN A1 NAVAL POSTGRADUATE SCHOOL

```

      SUBROUTINE ERR1(NEXCR)
      NEXCR = -NEXCR
      WRITE(6,100) NEXCR
      100 FORMAT(' NEXCR EXCEEDED MAXIMUM ALLOWABLE SPACE FOR REAL*8 VARIABLES BY
     1,I10//')
      STCP
      END

      SUBROUTINE ERR2(NEXCI)
      NEXCI = -NEXCI
      WRITE(6,100) NEXCI
      100 FORMAT(' NEXCI EXCEEDED MAXIMUM ALLOWABLE SPACE FOR INT*2 VARIABLES BY
     1,I10//')
      STCP
      END

      SUBROUTINE ERR3(NEXC4)
      NEXC4 = -NEXC4
      WRITE(6,100) NEXC4
      100 FORMAT(' NEXC4 EXCEEDED MAXIMUM ALLOWABLE SPACE FOR REAL*2 VARIABLES BY
     1,I10//')
      STCP
      END

      SUBROUTINE MPLCT(RC,RMAX,FMIN,VZ,CVEL,CRC,PRE,DALEP,NRCTOR,ALP,
     1BF,DEV1,FRAT,ETA,NNCC,CR3T,NE4,NN4,NNWS)      *PLOC04
     100 FORMAT(' NEXC4 EXCEEDED MAXIMUM ALLOWABLE SPACE FOR REAL*2 VARIABLES BY
     1,I10//')
      STCP
      END

      THIS SUBROUTINE CREATES A TEKTRONIX 618 PLOT OF
      THE FOLLOWING PARAMETERS:                                *PLOC04
      AXIAL VELOCITY AT THE ROTOR INLET
      RELATIVE FLOW ANGLES AT THE ROTOR INLET
      AXIAL VELOCITY AT THE ROTOR CUTLET
      RELATIVE FLOW ANGLES AT THE ROTOR CUTLET
      ABSOLUTE FLOW ANGLES AT THE ROTOR INLET
      AXIAL VELOCITY AT THE STATOR INLET
      ABSOLUTE FLOW ANGLES AT THE STATOR INLET
      AXIAL VELOCITY AT THE STATOR OUTLET
      ABSOLUTE FLOW ANGLES AT THE STATOR OUTLET
      ADIABATIC EFFICIENCY AT THE ROTOR INLET
      RELATIVE DEVIATION ANGLE AT THE ROTOR OUTLET
      ABSOLUTE DEVIATION ANGLE AT THE STATOR CUTLET
      STATIC PRESSURE RATIO FOR THE ROTOR
      STATIC PRESSURE RATIO FOR THE STATOR

```

FILE: TURBO FORTRAN A1 NAVAL POSTGRADUATE SCHOOL

```

C **** * **** * **** * **** * **** * **** * **** * **** * **** * **** * **** * ****
C **
C **          FLOT THE ROTOR INLET VELOCITY DISTRIBUTION      **
C **
C **** * **** * **** * **** * **** * **** * **** * **** * **** * **** * ****
C DATA TITLE1/'AXIAL VELOCITY PROFILE AT ROTOR INLET'/
C DATA TITLE2/'AXIAL VELOCITY PROFILE AT ROTOR OUTLET'/
C DATA TITLE3/'AXIAL VELOCITY PROFILE AT STATOR INLET'/
C DATA TITLE4/'AXIAL VELOCITY PROFILE AT STATOR OUTLET'/
C DATA TITLE5/'RELATIVE FLOW ANGLES AT ROTOR INLET'/
C DATA TITLE6/'RELATIVE FLOW ANGLES AT ROTOR OUTLET'/
C DATA TITLE7/'ABSOLUTE FLOW ANGLES AT ROTOR OUTLET'/
C DATA TITLE8/'ABSOLUTE FLOW ANGLES AT STATOR INLET'/
C DATA TITLE9/'ABSOLUTE FLOW ANGLES AT STATOR OUTLET'/
C DATA TITLE10/'AERODYNAMIC EFFICIENCY AT ROTOR INLET'/
C DATA TITLE11/'DEVIATION ANGLE ROTOR OUTLET'/
C DATA TITLE12/'DEVIATION ANGLE STATOR OUTLET'/
C DATA TITLE13/'TOTAL PRESSURE RATIO, ROTOR'/
C DATA TITLE14/'TOTAL PRESSURE RATIO, STATOR'/
C WRITE(6,100)
100   FORMAT(5X,'>DO YOU WISH TO ENTER THE PLOT SEQUENCE? ',/,,
     1 1 = YES; 2 = NO. ',/')
     READ(15,1) NANS
     IF(NANS.EQ.2) GOTC 500
     J = MROTC
     CC 30 I = 1,MRCH1
     ORC(I) = RC(J)
     OVEL(I) = VZ(J)
     CBE(I) = BE(J)
     CALP(I) = E11(J)
     CRAT(I) = PRAT(J)
     J = J + 1
30    CONTINUE
     NR14 = MFCW1
     CXYL(1) = -50.000
     CXYL(2) = RMIN
     CXYL(3) = 700.000
     CXYL(4) = RMAX
     CALL DSINIT
     CALL GSEFSE
     CALL PLCT('MMGNNABNW1',NR14,CVEL,RC,CXYL,37,TITL1)
     CALL PLCT('MMGNNBOW1',NR14,CVEL,RC,CXYL,37,TITL1)
     CALL DSTEAV
     WRITE(6,101)
101   FORMAT(5X,'>DO YOU WISH TO CONTINUE PLOT SEQUENCE? ',/,,
     1 1 = YES; 2 = NO. ',/')
     READ(15,1) NANS
     IF(NANS.EQ.2) GOTC 500
     CALL DSINIT
     CALL GSEFSE
     CALL PLCT('MMGNNABNW1',NR14,CBE,RC,CXYL,35,TITL5)
     CALL PLCT('MMGNNBOW1',NR14,CBE,RC,CXYL,35,TITL5)
     CALL DSTEAV
     WRITE(6,101)
     READ(15,1) NANS
     IF(NANS.EQ.2) GOTC 500
     CALL DSINIT
     CALL GSEFSE
     CALL PLCT('MMGNNABNW1',NR14,CRAT,RC,CXYL,27,TITL13)
     CALL PLCT('MMGNNBOW1',NR14,CRAT,RC,CXYL,27,TITL13)
     CALL DSTEAV
     WRITE(6,101)
     READ(15,1) NANS
     IF(NANS.EQ.2) GOTC 500
     CALL DSINIT
     CALL GSEFSE
     CALL PLCT('MMGNNABNW1',NR14,CALP,RC,CXYL,35,TITL10)
     CALL PLCT('MMGNNBOW1',NR14,CALP,RC,CXYL,35,TITL10)
     CALL DSTEAV
     WRITE(6,101)
     READ(15,1) NANS
     IF(NANS.EQ.2) GOTC 500

```

FILE: TLRBG FCRTRAN A1 NAVAL POSTGRADUATE SCHOOL

```
J = J + MRCH + 1
CC 40 I = 1,MRCH1
OFC(I) = OC(J)
CVEL(I) = VZ(J)
CFAT(I) = PRAT(J)
CALP(I) = CEV1(J)
= J + 1
40 CONTINUE
CALL DSINIT
CALL GSEFSE
CALL PLCT('MGNRAN3NWL',NR14,CVEL,OC,CXYL,38,TITL2)
CALL PLCT('MGNRAN3OWL',NR14,CVEL,OC,CXYL,39,TITL2)
CALL DSTERM
WRITE(6,110)
READ(15,4) NANS
IF(NANS.EC.2) GOTC 500
CALL DSINIT
CALL GSEFSE
CALL PLCT('MGNRAN3NWL',NR14,CHE,OC,CXYL,36,TITL5)
CALL PLCT('MGNRAN3OWL',NR14,CHE,OC,CXYL,35,TITL5)
CALL DSTERM
WRITE(6,110)
READ(15,4) NANS
IF(NANS.EC.2) GOTC 500
CALL DSINIT
CALL GSEFSE
CALL PLCT('MGNRAN3NWL',NR14,CALP,OC,CXYL,28,TITL11)
CALL PLCT('MGNRAN3OWL',NR14,CALP,OC,CXYL,28,TITL11)
CALL DSTERM
WRITE(6,110)
READ(15,4) NANS
IF(NANS.EC.2) GOTC 500
J = J + MRCH1
CC 41 I = 1,MRCH1
OFC(I) = OC(J)
CFAT(I) = PRAT(J)
= J + 1
41 CONTINUE
CALL DSINIT
CALL GSEFSE
CALL PLCT('MGNRAN3NWL',NR14,CHE,OC,CXYL,36,TITL7)
CALL PLCT('MGNRAN3OWL',NR14,CHE,OC,CXYL,35,TITL7)
CALL DSTERM
WRITE(6,110)
READ(15,4) NANS
IF(NANS.EC.2) GOTC 500
J = J + MRCH + 1
CC 50 I = 1,MRCH1
OFC(I) = OC(J)
CVEL(I) = VZ(J)
CFAT(I) = PRAT(J)
CFAT(I) = ALF(J)
= J + 1
50 CONTINUE
CALL DSINIT
CALL GSEFSE
CALL PLCT('MGNRAN3NWL',NR14,CVEL,OC,CXYL,39,TITL3)
CALL PLCT('MGNRAN3OWL',NR14,CVEL,OC,CXYL,38,TITL3)
CALL DSTERM
WRITE(6,110)
READ(15,4) NANS
IF(NANS.EC.2) GOTC 500
CALL DSINIT
CALL GSEFSE
CALL PLCT('MGNRAN3NWL',NR14,CHE,OC,CXYL,34,TITL8)
CALL PLCT('MGNRAN3OWL',NR14,CHE,OC,CXYL,36,TITL8)
CALL DSTERM
WRITE(6,110)
READ(15,4) NANS
IF(NANS.EC.2) GOTC 500
CALL DSINIT
CALL GSEFSE
```

FILE: TURBO FORTRESS A1 NAVAL POSTGRADUATE SCHOOL

BEGIN WITH MID NODE OF FIRST ELEMENT AND THEN CYCLE  
THROUGH EACH ELEMENT.

FILE: TLR80 FCRTRAN A1 NAVAL POSTGRADUATE SCHOOL

```

N1 = NCDE(11,N)
P = FSI(N1)
IT = 1
K = 1
IF(ISTA1.EQ.1.AND.NTE1.EQ.1) GOTO 700
IF(ISTA1.EQ.1) GOTO 200
RCI2 = SC(11)
B2 = E(11)
C 13C      CHECK TO SEE IF P IS WITHIN THE PRESENT ELEMENT
C      IF(P.GE.PSI(NCCE(K,5)))GOTC140
C      CHECK NEXT ELEMENT BELOW THE PRESENT ELEMENT.
K = K + 1
GOTC13C
14C      IF(P.GT.PSI(NCCE(K,4)))GOTC170
EL = E1(4)
ER = E1(5)
E2 = (EL + ER)/2.00
CALL SFAP(E2,-1.00,SF)
PA = SF(3)*FSI(NCCE(K,3)) + SF(4)*PSI(NODE(K,4))
      + SF(5)*FSI(NODE(K,5))
C 1      CHECK FOR STREAMLINE CONVERGENCE
EPS = DABS(FA - P)
IF(EPS.LT.1.0E-6)GOTO190
IT = IT + 1
IF(IT.GT.15)GOTO190
IF(PA.LT.P)GOTC160
EL = E2
GOTC13C
16C      ER = E2
GOTC13C
17C      IF(P.GT.PSI(NCCE(K,3)))GOTC135
EL = E1(3)
ER = E1(4)
GOTC13C
C 185      CHECK NEXT ELEMENT ABOVE PRESENT ELEMENT
K = K - 1
GOTC13C
C      IF CONVERGENCE CRITERIA SATISFIED,
C      CALCULATE WHIRL AND STATIC ENTHALPY
C
19C      NK3 = NCDE(K,3)
NK4 = NCDE(K,4)
NK5 = NCDE(K,5)
RC11 = SF(3) * RC(NK3) + SF(4) * RC(NK4) +
      SF(5) * RC(NK5)
VZ1 = SF(3) * VZ(NK3) + SF(4) * VZ(NK4) +
      SF(5) * VZ(NK5)
VR1 = SF(3) * VR(NK3) + SF(4) * VR(NK4) +
      SF(5) * VR(NK5)
VM1 = CSOPT(VZ1 * VZ1 + VR1 * VR1)
ALP1 = SF(3) * ALP(NK3) + SF(4) * ALP(NK4) +
      SF(5) * ALP(NK5)
RHO1 = SF(3) * RHC(NK3) + SF(4) * RHC(NK4) +
      SF(5) * RHC(NK5)
RHOT1 = SF(3) * RHCTT(NK3) + SF(4) * RHCTT(NK4) +
      SF(5) * RHCTT(NK5)
T1 = SF(3) * TEMP(NK3) + SF(4) * TEMP(NK4) +
      SF(5) * TEMP(NK5)
TT1 = SF(3) * TTCT(NK3) + SF(4) * TTOT(NK4) +
      SF(5) * TTCT(NK5)
PT1 = SF(3) * PTCT(NK3) + SF(4) * PTOT(NK4) +
      SF(5) * PTCT(NK5)
P1 = SF(3) * PRESS(NK3) + SF(4) * PRESS(NK4) +
      SF(5) * PRESS(NK5)
ENT1 = SF(3) * ENTHCP(NK3) + SF(4) * ENTRCP(NK4) +
      SF(5) * ENTRCP(NK5)
H1 = SF(3) * H(NK3) + SF(4) * H(NK4) +
      SF(5) * H(NK5)
HS1 = SF(3) * HS(NK3) + SF(4) * HS(NK4) +
      SF(5) * HS(NK5)

```

FILE: TLFFR0 FCRTRAN A1 NAVAL POSTGRADUATE SCHOOL

```

1      SF(5) = PS(NK5)
      K = 1
      GOTO 210
20C    RCI1 = SC(NI1)
      VZ1 = VZ(NI1)
      VR1 = VR(NI1)
      VM1 = DSOFT(VZ1 + VZ1 + VR1 + VR1)
      ALP1 = ALP(NI1)
      RH01 = RH0(NI1)
      RHOT1 = RHOT(NI1)
      P1 = FRES(NI1)
      PT1 = STUT(NI1)
      TT1 = TTOT(NI1)
      T1 = TEMP(NI1)
      HI = H(NI1)
      HS1 = PS(NI1)
      GCTC 215
      IF(VTE1.EC.1.CR.1STA1.EQ.3) GOTO 300
      IT = 1
      K = 1
      C      CHECK TO SEE IF P IS WITHIN THE PRESENT ELEMENT
      IF(P.GE.PSI(NCCE(K,7)))GOTO 24J
      C      CHECK NEXT ELEMENT BELOW THE PRESENT ELEMENT.
      K = K + 1
      GOTO 23C
23C    IF(P.GT.PSI(NCCE(K,8)))GOTO 270
      EL = E1(S)
      FR = E1(7)
      E2 = (EL + SF)/2.CD
      CALL SFADDE(E2,1.DC,SF)
      PA = SF(1)*PSI(NODE(K,1)) + SF(7)*PSI(NODE(K,7))
      + SF(8)*PSI(NODE(K,3))
      C      CHECK FCP STREAMLINE CONVERGENCE
      EPS = CABS(FA - P)
      IF(EPS.LT.1.D-6)GOTC29C
      IT = IT + 1
      IF(IT.GT.15)GOTC290
      IF(PA.LT.P)GOTC26C
      EL = E2
      GOTO 23C
26C    ER = E2
      GOTC26C
27C    IF(P.GT.PSI(NCCE(K,1)))GOTC285
      EL = E1(1)
      EP = E1(8)
      GOTO 23C
      C      CHECK NEXT ELEMENT ABOVE PRESENT ELEMENT
      K = K - 1
      GOTO 24C
      C      IF CONVERGENCE CRITERIA SATISFIED,
      CALCULATE WHTPL AND STATC ENTHALPY
29C    1      RCI2 = SF(1)*RC(NCCE(K,1)) + SF(7)*RC(NCCE(K,7))
      1      + SF(8)*RC(NCCE(K,3))
      1      S2 = SF(1)*FC(NCCE(K,1)) + SF(7)*FC(NCCE(K,7))
      + SF(8)*FC(NCCE(K,3))
      C      GO TO NEXT ELEMENT.
3CC     N1 = NCCE(K,1)
      N2 = NCCE(K,2)
      N3 = NCCE(K,3)
      N4 = NCCE(K,4)
      N5 = NCCE(K,5)
      N6 = NCCE(K,6)
      N7 = NCCE(K,7)
      N8 = NCCE(K,8)
      FC3(1) = RC(N1)
      FC3(2) = RC(N2)
      FC3(3) = RC(N3)
      FC3(4) = RC(N4)
      FC3(5) = RC(N5)
      FC3(6) = RC(N6)

```

FILE: TLEBG FCRTRAN A1 NAVAL POSTGRADUATE SCHOOL

```

RCS(7) = PC(N7)
RCS(8) = PC(N8)
ZCS(1) = ZC(N1)
ZCS(2) = ZC(N2)
ZCS(3) = ZC(N3)
ZCS(4) = ZC(N4)
ZCS(5) = ZC(N5)
ZFS(E1) = ZC(NE1)
ZCS(7) = ZC(N7)
ZCS(8) = ZC(N8)
DO 500 I = 1, NC
      FIND THE JACOBIAN.
      CALL JACCG(F1(I), Z1(I), O1, E, RCS, ZCS, RJAC)
      DETJ = RJAC(1,1)*RJAC(2,2) - RJAC(1,2)*RJAC(2,1)
      FIND INVERSE OF JACOBIAN.
      DLPI = RJAC(1,1) / DETJ
      RJAC(1,1) = RJAC(2,2) / DETJ
      RJAC(1,2) = -RJAC(1,2) / DETJ
      RJAC(2,1) = -RJAC(2,1) / DETJ
      RJAC(2,2) = DUM!
      FIND ENI/DR AND ENI/DZ
      DC 400 L = 1, NNE
      CZ(L) = RJAC(1,1)*D(L) + RJAC(1,2)*E(L)
      CR(L) = RJAC(2,1)*D(L) + RJAC(2,2)*E(L)
      CONTINUE
      CHECK TO SEE IF SOLUTION IS AT INLET
      FIND C(PSI)/DR AND C(PSI)/DZ
      DPSIR(I) = CR(1)*PSI(N1) + CR(2)*PSI(N2) +
      CR(3)*PSI(N3) + CR(4)*PSI(N4) +
      CR(5)*PSI(N5) + CR(6)*PSI(N6) +
      CR(7)*PSI(N7) + CR(8)*PSI(N8)
      DPSIZ(I) = CZ(1)*PSI(N1) + CZ(2)*PSI(N2) +
      CZ(3)*PSI(N3) + CZ(4)*PSI(N4) +
      CZ(5)*PSI(N5) + CZ(6)*PSI(N6) +
      CZ(7)*PSI(N7) + CZ(8)*PSI(N8)
      CONTINUE
      IF(NTF1.EQ.1) GOTO 500
      IF(ISTF1.EQ.1) GOTO 600
      DPSIRz = SF(1) * DPSIR(1) + SF(7) * DPSIR(7) +
      SF(8) * DPSIR(8)
      DPSIZz = SF(1) * DPSIZ(1) + SF(7) * DPSIZ(7) +
      SF(8) * DPSIZ(8)
      GOTO 700
      DPSTRz = DPSIZ(M)
      CPSIZz = CPSIZ(M)
      CONTINUE
      RETURN
END

```

SERROUTINE FCAL(F,W,F,ZA,E8,VZ,PC,ZC,WL,VU,NRC,NUDE,WU,  
INTE,HA,ENTRNP,TTT,F,NCDO,NE4,NEE4,ROWS)

THIS SUBROUTINE CALCULATES THE RIGHT-HAND SIDE VECTOR  
 $F(R, Z)$  FROM KNOWN RADIAL DISTRIBUTIONS OF WHIRL,  
 ENTHALPY, PCTENTHALPY, AND ENTALPY.

CALL STATEMENT DEFINITION:

|      |                                  |
|------|----------------------------------|
| $F$  | = RIGHT HAND SIDE VECTOR         |
| $R$  | = GAUSSIAN HEIGHT FRACTION ARRAY |
| $Z$  | = MEAN TOTAL ENTHALPY VECTOR     |
| $ZA$ | = ARRAY OF EXCIE GAUSSIAN VALUES |
| $EA$ | = ARRAY OF ETA GAUSSIAN VALUES   |

FILE: TLEBO FORTRAN AI NAVAL POSTGRADUATE SCHOOL

VZ = NCDALE AXIAL VELOCITY  
 FC = ARRAY OF NODAL R COORDINATES  
 ZC = AFRAY OF NODAL Z COORDINATES  
 VPL = NCDALE ANGULAR MOMENTUM VECTOR  
 VU = NCDALE ABSOLUTE TANGENTIAL VELOCITY VECTOR  
 REC = NODES WHERE BOUNDARY CONDITIONS APPLY  
 NCCE = ARRAY OF ELEMENTAL NODE ASSIGNMENTS  
 NR = NUMBER OF NODES IN THE MESH  
 NE = NUMBER OF ELEMENTS IN THE MESH  
 RU = NCDALE RELATIVE TANGENTIAL VELOCITY VECTOR  
 RTE = TYPE OF ELEMENT ARRAY

```

      IMPLICIT REAL*8(A-H,P-Z)
      INTEGER *4 NR,FACT,ALW4ITE,IC,NNCU,NE4,NNE4,NROWS,LIMR,LIMI
      COMMON /MCURRY/ ACCL,NCOL1,NN,NE
      1 NNE,NNBC,ANNPC
      2 CCMN,NN - /MCOUNT/ MRCW,MRCW1,KK
      COMMON /FCC4/ FG,G,CP,PT,TT,WG,WDT,FHT,RHOSTA,
      3 LINLET,LCLLET,PE11,STJ,F2I2,F11,I1,J1
      DIMENSION ZC(1),ZC(1),H(1),F(1),TTT(1)
      DIMENSION VZ(1),VL(1),WL(1),WU(1),SF(8)
      DIMENSION MAC(1),ATE(1),NDE(NE4,1)
      DIMENSION H(4),ZL(9),FA(9),F(8),H(1),ENTROP(1)
      DIMENSION D(3),E(1),SF(3),RCS(8),ZCS(3),PJAC(2,2)

      ZEROIZE OUT FS() .

```

CC 50 I = 1.00E  
FS(I) = 0.00  
C C R A T I N E

CYCLE FOR EACH ELEMENT.

CYCLE FCB EACH LOCAL NODE.

DC 3CO J = 1,9  
 CALL STAPF( ZA(J),ZA(J),SF)  
 CALL JACCB( ZA(J),ZB(J),ZC(J),ZD(J),FC1,FC2,FC3,ZC3,RJAC)  
 DETJ = RJAC(1,1)\*RJAC(2,2) - RJAC(1,2)\*RJAC(2,1)  
 FIND THE INVERSE OF THE JACOBIAN.  
 DUM1 = RJAC(1,1) / DETJ  
 RJAC(1,1) = RJAC(2,2) / CETJ

FILE: TLRBC FORTAN AI NAVAL POSTGRADUATE SCHOOL

```

      RJAC(1,2) = -RJAC(1,2) / DETJ
      RJAC(2,1) = -RJAC(2,1) / DETJ
      RJAC(2,2) = SUM1

      FIND NI*LI,NI*VI,THETAI,NI*RI,D(WRL1)/CR,D(H1)/DR

      SUMU = 0.00
      ELMV = C.00
      SUMK = C.00
      CSOR = C.00
      ELM1 = C.00
      TOSDR = C.00
      CWRLR = 0.00
      DHCR = C.00
      CC 11C KL = 1,NNF
      N41 = NCDE(II,KL)
      SUMU = SUMU + SF(KL)*VZ(N41)
      SUMT = SUMT + SF(KL)*TTDT(N41)
      IF(INTE(II).EQ.2)GOTO105
      SUMV = SUMV + SF(KL)*VU(N41)
      DHCR = DHCR + (RJAC(2,1)*E(KL))
      + RJAC(2,1)*F(KL))*H(N41)
      CSCR = CSOR + (RJAC(2,1)*D(KL))
      + RJAC(2,2)*E(KL))+ENTROP(N41)
      1
      1
      GOTO105
      105
      SUMV = SUMV + SF(KL)*VU(N41)
      DHCR = DFDR + (RJAC(2,1)*E(KL))
      + XJAC(2,2)*E(KL))*HRT(N41)
      CSCR = CSOR + (RJAC(2,1)*D(KL))
      + PJAC(2,2)*E(KL))+ENTROP(N41)
      106
      1
      SUMR = SUMR + SF(KL)*RC(N41)
      DWRLF = DWRLR + (RJAC(2,1)*D(KL))
      + RJAC(2,2)*E(KL))*WRL(N41)
      1
      11C
      CONTINUE

      FIND FS(NCDE(II,II))

      TOSCF = SUMT + OSDR
      CC 20C I = 1,NNF
      XX = (TOSDR - DHCR + (SUMV/SUMR)*DWRLR)*(SF(I)/SUMU)*DETJ
      FS(I) = FS(I) + XX*FS(J)
      CCNTINUE
      CCNTINUE

      ASSEMBLE RIGHT HAND SIDE VECTOR.

      F(N1) = F(N1) + FS(1)
      F(N2) = F(N2) + FS(2)

```

ASSEMBLE RIGHT HAND SIDE VECTOR.

DO 500 = 1,000  
CONTINUE  
RETURN  
END

SUBROUTINE CSIM (MCY-IMSL)  
ACAPTED FOR THE 360 BY RUM EPISIML

**PURPOSE** OBTAIN SOLUTION OF A SET OF SIMULTANEOUS LINEAR EQUATIONS,  $Ax = b$

FILE: TLRB0 FORTRAN A1 NAVAL POSTGRADUATE SCHOOL

```
      USAGE
      CALL DSIMQ(A,B,N,KS)

      DESCRIPTION OF PARAMETERS
      A AND B MUST BE REAL*8
      A - MATRIX OF COEFFICIENTS STORED COLUMNWISE.
      THESE ARE DESTROYED IN THE COMPUTATION. THE
      SIZE OF MATRIX A IS N BY N.
      B - VECTOR OF ORIGINAL CONSTANTS (LENGTH N). THESE
      ARE REPLACED BY FINAL SOLUTION VALUES, VECTOR X.
      N - NUMBER OF EQUATIONS AND VARIABLES
      KS - OUTPUT DIGIT
          0 FOR A NORMAL SOLUTION
          1 FOR A SINGULAR SET OF EQUATIONS

      REMARKS
      MATRIX A MUST BE GENERAL.
      IF MATRIX IS SINGULAR, SOLUTION VALUES ARE MEANING-
      LESS. AN ALTERNATIVE SOLUTION MAY BE OBTAINED BY USING
      MATRIX INVERSION (INV) AND MATRIX PRODUCT (GPPD).

      SUBROUTINES AND FUNCTION SUBPROGRAMS REQUIRED
      NONE
```

SUBROUTINE DSIMQ(A,B,N,KS)

SUBROUTINE DSIMQ NOT INCLUDED NON-IMSL LIBRARY SUBROUTINE

```
SUBROUTINE STIFF(RC,ZC,R,EMS,ZA,EA,N,NNCD,NNCE,NNE4,NRCLS,RHO,
1FSI,F,PHS,EM,KPC,NNCD,NE4,NNE4,NRCLS,LIMR,LIMI
1,NE4,NNPC,NNNEC
1CMON /ACCOUNT/ NCOL1,NCOL2,NN,PE
1LINLET,LCLLT,FSI,TU,F212,F111,GC
1CPMCN /COUNT/ VCOLN,WRGW1,KK
1CENCN /L1/ NFFAC,NKPITE
1CIPNSCN ZC(1),RC(1),ZC(1),RC(1),RC(1)
1CIPNSCN RC(1),RC(1),RC(1),RC(1),RC(1)
1CIPNSCN EA(NNCD,1),RIJAC(2,2),PSI(1),F(1),PHS(1)
1CIPNSCN D(1),E(1),S(1),Z(1),Z(1),Z(1),DNDZ(1)
1CIPNSCN RUE(1),ZCS(1),EME(8,1),RIJAC(2,2),DNDZ(1)
```

```
CC 400 II = 1,NE
N1 = RC(1,1)
N2 = RC(1,2)
N3 = RC(1,3)
N4 = RC(1,4)
N5 = RC(1,5)
N6 = RC(1,6)
N7 = RC(1,7)
N8 = RC(1,8)
RC(1) = RC(N1)
RC(2) = RC(N2)
RC(3) = RC(N3)
RC(4) = RC(N4)
```

FILE: TLRRO FORTRAN A1 NAVAL POSTGRADUATE SCHOOL

```

RC5(5) = RC(N5)
RC5(6) = RC(N6)
RC5(7) = RC(N7)
RC5(8) = RC(N8)
ZCS(1) = ZC(N1)
ZCS(2) = ZC(N2)
ZCS(3) = ZC(N3)
ZCS(4) = ZC(N4)
ZCS(5) = ZC(N5)
ZCS(6) = ZC(N6)
ZCS(7) = ZC(N7)
ZCS(8) = ZC(N8)

```

PERFORM GAUSSIAN QUADRATURE INTEGRATION

```

DO 320 I = 1,C
    CALL SHAPE(EA(I),ZA(I),SF)
    CALL JACCP(EA(I),ZA(I),C,E,RC5,ZCS,RJAC)
    DETJ = P*JAC(1,1)*RJAC(2,2) - RJAC(1,2)*RJAC(2,1)
    FIND INVERSE OF JACCBIAN
    DUM1 = RJAC(1,1) / DETJ
    RJAC(1,1) = RJAC(2,2) / DETJ
    RJAC(1,2) = -RJAC(1,2) / DETJ
    RJAC(2,1) = -RJAC(2,1) / DETJ
    RJAC(2,2) = DUM1
    DC 321 J = 1,P
        DNDZ(J) = RJAC(1,1)*D(J) + RJAC(1,2)*E(J)
        DNDR(J) = RJAC(2,1)*D(J) + RJAC(2,2)*E(J)
321  CCNTINUE
100  IF(I>N,E,I) GOTO 100
100  CCNTINUE

```

FIND RHC, R, AND B FOR NUMERICAL INTEGRATION.

```

RFCRE = C,CC0
DC 330 L = 1,NNE
    NIEL = NCDE(I,I,L)
    PH(RA = RHOR8 + SF(L)*PHD(NIEL)+RC(NIEL)*B(NIEL))
330  CCNTINUE
    SPP = (1.00/(RFCRE)*144.00)*12.000

```

```

DC 340 J = 1,NNE
    IF(I>N,E,I) GOTO 309
309  CCNTINUE
    DC 340 K = 1,NNE
        EMS(J,K) = EMS(J,K) + W(I) * SYK * (DNDZ(J)*DNCZ(K)
1      + UNDR(J)*DNDR(K)) * DETJ
2      * 144.000

```

CCNTINUE  
CCNTINUE  
CCNTINUE

ASSEMBLE SYSTEM INFLUENCE MATRIX W/OUT REGARD FOR  
BOUNDPY CCNTITIONS

```

N(1) = N1
N(2) = N2
N(3) = N3
N(4) = N4
N(5) = N5
N(6) = N6
N(7) = N7
N(8) = N8

```

```

DO 350 IS = 1,NNE
    ! = N(IS)
    DC 350 JS = 1,NNE
    J = N(JS)
        EM(I,J) = EM(I,J) + EMS(IS,JS)
350  CCNTINUE

```

ZERCIZE CUT EMS( ) FOR NEXT ELEMENT

FILE: TLPBC FCRTRAN A1 NAVAL POSTGRADUATE SCHOOL

```
      DO 370 I2 = 1,NNE
      DC 370 J2 = 1,NNE
      EMS(I2,J2) = C.00
370  CCNTINUE
      CC          RECYCLE FOR NEXT ELEMENT
400  CCNTINUE
      CC          MODIFY SYSTEM OF EQUATIONS TO INCLUDE BOUNDARY CONDITIONS
      CC 410 I = 1,NN
      DC 410 J = 1,NBBC
      JX = NEC(J)
      F(I) = F(I) - EM(I,JX)*PSI(JX)
      EM(I,JX) = C.00
      EM(JX,I) = C.00
      EM(JX,JX) = 1.00
      F(JX) = PSI(JX)
410  CCNTINUE
      C          RETURN
      END
      C
      C
      C          SUBROUTINE REPLA(PSI,F,RHS,NNBC,NE4,NNE4)
      ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++
      ++
      ++
      ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++
      ++
      ++
      ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++
      ++
      ++
      IMPLICIT REAL*8(A-H,P-Z)
      INTEGER*4 NREAD,NWRITE,IC,NNBC,NE4,NNE4,NROWS,LIMR,LIMI
      CCPMN /ACCOUNT/ NCOL,NCOL1,NN,AF
      1,NE4,NNBC,NNBC
      CCPMN /ACCOUNT/ NROW,MPGW1,KK
      DIMENSION PSI(1),PSIC(1),F(1),RHS(1)
      C          REPLACE PSI(I) WITH SOLUTION VECTOR AND RESET F(I) WITH RHS(I)
      C 100 I = 1,NN
      PSI(I) = F(I)
      F(I) = RHS(I)
100  CCNTINUE
      C          RETURN
      END
      C
      C          SUBROUTINE RELAX(FELX,PSI,PSIC,NBC,NNBC,NE4,NNE4)
      ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++
      ++
      ++
      ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++
      ++
      ++
      IMPLICIT REAL*8(A-H,P-Z)
      INTEGER*4 NREAD,NWRITE,IC,NNBC,NE4,NNE4,NROWS,LIMR,LIMI
      CCPMN /ACCOUNT/ NCOL,NCOL1,NN,NE
      1,NE4,NNBC,NNBC
      CCPMN /ACCOUNT/ NROW,MPGW1,KK
      DIMENSION PSI(1),PSIC(1),F(1),RHS(1)
      C          IF KK GE 1, PERFORM UNDER RELAXATION BEFORE
      C          COMPUTING NEW VELOCITY AND DENSITY DISTRIBUTION.
```

FILE: TLF80 FORTRAN AI NAVAL POSTGRADUATE SCHOOL

```
C      EC 100 I = 1,NN
      DO 200 J = 1,NBC
         LTEST = I - NBC(J)
         IF(LTEST.EQ.0) GOTO 100
200      CONTINUE
         PSI(I) = PSIC(I) + RELX*(PSI(I) - PSIO(I))
100      CONTINUE
         RETURN
      END

C
C      SUBROUTINE NOCCN(X,F,PSI,PSIC,EM,IFL,NNOD,NNE4,NNE)
C      ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++
C      ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++
C      ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++
C      ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++
C
C      IMPLICIT REAL*8(A-H,P-Z)
      INTEGER*4 NRFAE,NWRITE,IC,NNOD,NNE4,NNE,NROWS,LIMR,LIMI
      COMMON /ACOUNT/ NCOL,NCOLI,NN,NB
      1,NNE,NNEC,NN,NC
      CCPMCN /ACOUNT/ NROW,NROW1,KK
      CCPMCN /ACOUNT/ NCOL,NCOLI
      CIPENS(ICA PS1(1),PSIC(1),EM(NNOD,1),F(1)
      WRITE (NWRITE,1400) KK,X
1400  FCPMAT(' ','LARGEST',EPS FOR ITERATION ',I2,' IS ',016.12)
      IF (KK.LT.22) GOTO 1500
      IFL = 0
      GOTO 451
1500  WRITE(NWRITE,1102)KK
1102  FCPMAT(' ','ITERATION NO. ',I2,' COMPLETE','/',' STREAM FUNCTION CON
      VERGENCE NOT YET SATISFIED.',/,,'NEXT ITERATION IS IN PROGRESS')
      PREPARE FOR NEXT ITERATION. ZEROIZE
      STIFFNESS MATRIX AND RIGHT HAND SIDE VECTOR
      REPLACE PSIO() WITH CURRENT VALUE OF PSI.
      IFL = 2
      EC 460 I = 1,NN
      F(I) = C0.DC
      PSIO(I) = PSI(I)
      DO 460 J = 1,NN
         EM(I,J) = C0.DC
460      CONTINUE
451      RETURN
      END

C
C      SUBROUTINE TEST(EST,PSIO,X,NNOD,NES,NNE,NROWS)
C      ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++
C      ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++
C      ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++
C      ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++ ++++++
C
C      IMPLICIT REAL*8(A-H,P-Z)
      INTEGER*4 NRFAE,NWRITE,IC,NNOD,NNE4,NNE,NROWS,LIMR,LIMI
      1,NNE,NNEC,NN,NC
      CIPENS(ICA PSI(I),FSIC(1))

C      COMPARE NEW AND OLD STREAM FUNCTION DISTRIBUTIONS
      X = 0.0C
```

FILE: TURBO FCRTRAN A1 NAVAL POSTGRADUATE SCHOOL

```
DC 100 I = 1,NN
IF(PSI(I) .EQ. C,D0) GOTO 110
EPS = ABS((FS10(I) - PSI(I))/PSI(I))
GOTO 12C
11C EPS = ABS(FS1C(I) - PSI(I))
12C IF (X.GT.EPS) GOTO 100
      X = EPS
10C CONTINUE
      RETURN
      END
```

APPENDIX G

SAMPLE PROGRAM OUTPUT

MEMORY SPACE AVAILABLE :  
 REAL 8 = 499 INTEGER = 988 REAL 4 = 424

| NASA TASK-1 TRANSONIC COMPRESSOR |               |                              |               | NO. OF ELEMENTS = 63 | NO. OF COLUMNS = 9 |
|----------------------------------|---------------|------------------------------|---------------|----------------------|--------------------|
| NOCE                             | Z(I)          | SUMMARY OF NODAL COORDINATES |               | ABS FLOW ANG         | REL FLOW ANG       |
|                                  |               | R(I)                         | S(I)          |                      |                    |
| 1                                | C.0.C         | C.16E78CD+02                 | C.9100000D+00 | 0.0                  | 0.0                |
| 2                                | C.0.C         | C.132900D+02                 | C.9100300D+00 | 0.0                  | 0.0                |
| 3                                | C.0.C         | C.176924D+02                 | C.9100000D+00 | 0.0                  | 0.0                |
| 4                                | C.0.C         | C.170E32D+02                 | C.9100000D+00 | 0.0                  | 0.0                |
| 5                                | C.0.C         | C.162999D+02                 | C.9100000D+00 | 0.0                  | 0.0                |
| 6                                | C.0.C         | C.157194D+02                 | C.9100000D+00 | 0.0                  | 0.0                |
| 7                                | C.0.C         | C.150091D+02                 | C.9100000D+00 | 0.0                  | 0.0                |
| 8                                | C.0.C         | C.142814D+02                 | C.9100000D+00 | 0.0                  | 0.0                |
| 9                                | C.0.C         | C.134734D+02                 | C.9100000D+00 | 0.0                  | 0.0                |
| 10                               | C.0.C         | C.126343D+02                 | C.9100000D+00 | 0.0                  | 0.0                |
| 11                               | C.0.C         | C.117347D+02                 | C.9100000D+00 | 0.0                  | 0.0                |
| 12                               | C.0.C         | C.107646D+02                 | C.9100000D+00 | 0.0                  | 0.0                |
| 13                               | C.0.C         | C.970091D+01                 | C.9100000D+00 | 0.0                  | 0.0                |
| 14                               | C.0.C         | C.8600100D+01                | C.9100000D+00 | 0.0                  | 0.0                |
| 15                               | C.0.C         | C.7093000D+01                | C.9100000D+00 | 0.0                  | 0.0                |
| 16                               | C.1500000C+01 | C.14686100D+02               | C.9100000D+00 | 0.0                  | 0.0                |
| 17                               | C.1500000C+01 | C.1750212D+02                | C.9100000D+00 | 0.0                  | 0.0                |
| 18                               | C.1500000C+01 | C.1628579D+02                | C.9100000D+00 | 0.0                  | 0.0                |
| 19                               | C.1500000C+01 | C.1433656D+02                | C.9100000D+00 | 0.0                  | 0.0                |
| 20                               | C.1500000C+01 | C.1116493D+02                | C.9100000D+00 | 0.0                  | 0.0                |
| 21                               | C.1500000C+01 | C.964624D+01                 | C.9100000D+00 | 0.0                  | 0.0                |
| 22                               | C.1500000C+01 | C.709344D+01                 | C.9100000D+00 | 0.0                  | 0.0                |
| 23                               | C.1500000C+01 | C.179124D+02                 | C.9100000D+00 | 0.0                  | 0.0                |
| 24                               | C.1500000C+01 | C.173216D+02                 | C.9100000D+00 | 0.0                  | 0.0                |
| 25                               | C.1500000C+01 | C.167166D+02                 | C.9100000D+00 | 0.0                  | 0.0                |
| 26                               | C.1500000C+01 | C.1607610D+02                | C.9100000D+00 | 0.0                  | 0.0                |
| 27                               | C.1500000C+01 | C.154154D+02                 | C.9100000D+00 | 0.0                  | 0.0                |
| 28                               | C.1500000C+01 | C.1400100D+02                | C.9100000D+00 | 0.0                  | 0.0                |
| 29                               | C.1500000C+01 | C.1320772D+02                | C.9100000D+00 | 0.0                  | 0.0                |
| 30                               | C.1500000C+01 | C.124265D+02                 | C.9100000D+00 | 0.0                  | 0.0                |
| 31                               | C.1500000C+01 | C.1160291D+02                | C.9100000D+00 | 0.0                  | 0.0                |
| 32                               | C.1500000C+01 | C.1061212D+02                | C.9100000D+00 | 0.0                  | 0.0                |
| 33                               | C.3000000C+01 | C.9591218D+01                | C.9100000D+00 | 0.0                  | 0.0                |
| 34                               | C.3000000C+01 | C.943402D+01                 | C.9100000D+00 | 0.0                  | 0.0                |
| 35                               | C.3000000C+01 | C.70940CD+01                 | C.9100000D+00 | 0.0                  | 0.0                |
| 36                               | C.4500000C+01 | C.184463D+02                 | C.9100000D+00 | 0.0                  | 0.0                |
| 37                               | C.4500000C+01 | C.172972D+02                 | C.9100000D+00 | 0.0                  | 0.0                |

| NOCE | Z(I)          | SUMMARY OF NODAL COORDINATES |              |     | ABS FLOW ANG | REL FLOW ANG |
|------|---------------|------------------------------|--------------|-----|--------------|--------------|
|      |               | R(I)                         | B(I)         |     |              |              |
| 41   | C.450000CE+01 | C.150449D+02                 | C.910000D+00 | 0.0 | 0.0          |              |
| 42   | C.450000CE+01 | C.146975D+02                 | C.910000D+00 | 0.0 | 0.0          |              |
| 43   | C.450000CE+01 | C.132144D+02                 | C.910000D+00 | 0.0 | 0.0          |              |
| 44   | C.450000CE+01 | C.115418D+02                 | C.910000D+00 | 0.0 | 0.0          |              |
| 45   | C.450000CE+01 | C.958145D+02                 | C.910000D+00 | 0.0 | 0.0          |              |
| 46   | C.450000CE+01 | C.799000D+01                 | C.910000D+00 | 0.0 | 0.0          |              |
| 47   | C.600000CE+01 | C.1840400D+03                | C.910000D+00 | 0.0 | 0.0          |              |
| 48   | C.600000CE+01 | C.1780340D+03                | C.910000D+00 | 0.0 | 0.0          |              |
| 49   | C.600000CE+01 | C.1720240D+03                | C.910000D+00 | 0.0 | 0.0          |              |
| 50   | C.600000CE+01 | C.1664460D+02                | C.910000D+00 | 0.0 | 0.0          |              |
| 51   | C.600000CE+01 | C.160137D+02                 | C.910000D+00 | 0.0 | 0.0          |              |
| 52   | C.600000CE+01 | C.155656D+02                 | C.910000D+00 | 0.0 | 0.0          |              |
| 53   | C.600000CE+01 | C.142707D+02                 | C.910000D+00 | 0.0 | 0.0          |              |
| 54   | C.600000CE+01 | C.135680D+02                 | C.910000D+00 | 0.0 | 0.0          |              |
| 55   | C.600000CE+01 | C.131917D+02                 | C.910000D+00 | 0.0 | 0.0          |              |
| 56   | C.600000CE+01 | C.123872D+02                 | C.910000D+00 | 0.0 | 0.0          |              |
| 57   | C.600000CE+01 | C.115524D+02                 | C.910000D+00 | 0.0 | 0.0          |              |
| 58   | C.600000CE+01 | C.105528D+02                 | C.910000D+00 | 0.0 | 0.0          |              |
| 59   | C.600000CE+01 | C.957131D+01                 | C.910000D+00 | 0.0 | 0.0          |              |
| 60   | C.600000CE+01 | C.842615D+01                 | C.910000D+00 | 0.0 | 0.0          |              |
| 61   | C.600000CE+01 | C.705990D+01                 | C.910000D+00 | 0.0 | 0.0          |              |
| 62   | C.600200CE+01 | C.134025D+02                 | C.910000D+00 | 0.0 | 0.0          |              |
| 63   | C.890200CE+01 | C.172474D+02                 | C.910000D+00 | 0.0 | 0.0          |              |
| 64   | C.890200CE+01 | C.150022D+02                 | C.910000D+00 | 0.0 | 0.0          |              |
| 65   | C.890200CE+01 | C.145664D+02                 | C.910000D+00 | 0.0 | 0.0          |              |
| 66   | C.890200CE+01 | C.131886D+02                 | C.910000D+00 | 0.0 | 0.0          |              |
| 67   | C.890200CE+01 | C.1155221D+02                | C.910000D+00 | 0.0 | 0.0          |              |
| 68   | C.890200CE+01 | C.1055221D+01                | C.910000D+00 | 0.0 | 0.0          |              |
| 69   | C.890200CE+01 | C.705995D+01                 | C.910000D+00 | 0.0 | 0.0          |              |
| 70   | C.118034CE+02 | C.133567D+02                 | C.910000D+00 | 0.0 | 0.0          |              |
| 71   | C.118034CE+02 | C.173524D+02                 | C.910000D+00 | 0.0 | 0.0          |              |
| 72   | C.118034CE+02 | C.166251D+02                 | C.910000D+00 | 0.0 | 0.0          |              |
| 73   | C.118034CE+02 | C.160346D+02                 | C.910000D+00 | 0.0 | 0.0          |              |
| 74   | C.118040CE+02 | C.155486D+02                 | C.910000D+00 | 0.0 | 0.0          |              |
| 75   | C.118040CE+02 | C.146630D+02                 | C.910000D+00 | 0.0 | 0.0          |              |
| 76   | C.118040CE+02 | C.136436D+02                 | C.910000D+00 | 0.0 | 0.0          |              |
| 77   | C.118040CE+02 | C.131546D+02                 | C.910000D+00 | 0.0 | 0.0          |              |
| 78   | C.118040CE+02 | C.123507D+02                 | C.910000D+00 | 0.0 | 0.0          |              |
| 79   | C.118040CE+02 | C.115519D+02                 | C.910000D+00 | 0.0 | 0.0          |              |
| 80   | C.118040CE+02 | C.115519D+02                 | C.910000D+00 | 0.0 | 0.0          |              |

| NODE | Z(I)           | SUMMARY OF NODAL COORDINATES<br>R(I) | S(I)           | ABS FLOW ANG | REL FLOW ANG |
|------|----------------|--------------------------------------|----------------|--------------|--------------|
| 81   | 0. 1180340E+02 | 0. 105993D+02                        | 0. 7130000D+00 | 0.0          | 0.0          |
| 82   | 0. 1150400E+02 | 0. 550962D+01                        | 0. 9120000D+00 | 0.0          | 0.0          |
| 83   | 0. 1180400E+02 | 0. 942622D+01                        | 0. 9100000D+00 | 0.0          | 0.0          |
| 84   | 0. 1180400E+02 | 0. 711000D+01                        | 0. 9100000D+00 | 0.0          | 0.0          |
| 85   | 0. 1180400E+02 | 0. 183835D+02                        | 0. 9100000D+00 | 0.0          | 0.0          |
| 86   | 0. 1200400E+02 | 0. 171000D+02                        | 0. 9100000D+00 | 0.0          | 0.0          |
| 87   | 0. 1200400E+02 | 0. 171000D+02                        | 0. 9100000D+00 | 0.0          | 0.0          |
| 88   | 0. 1200400E+02 | 0. 171000D+02                        | 0. 9100000D+00 | 0.0          | 0.0          |
| 89   | 0. 1200400E+02 | 0. 171000D+02                        | 0. 9100000D+00 | 0.0          | 0.0          |
| 90   | 0. 1200400E+02 | 0. 171000D+02                        | 0. 9100000D+00 | 0.0          | 0.0          |
| 91   | 0. 1200400E+02 | 0. 171000D+02                        | 0. 9100000D+00 | 0.0          | 0.0          |
| 92   | 0. 1200400E+02 | 0. 171000D+02                        | 0. 9100000D+00 | 0.0          | 0.0          |
| 93   | 0. 1200400E+02 | 0. 171000D+02                        | 0. 9100000D+00 | 0.0          | 0.0          |
| 94   | 0. 1200400E+02 | 0. 171000D+02                        | 0. 9100000D+00 | 0.0          | 0.0          |
| 95   | 0. 1200400E+02 | 0. 171000D+02                        | 0. 9100000D+00 | 0.0          | 0.0          |
| 96   | 0. 1200400E+02 | 0. 171000D+02                        | 0. 9100000D+00 | 0.0          | 0.0          |
| 97   | 0. 1200400E+02 | 0. 171000D+02                        | 0. 9100000D+00 | 0.0          | 0.0          |
| 98   | 0. 1200400E+02 | 0. 171000D+02                        | 0. 9100000D+00 | 0.0          | 0.0          |
| 99   | 0. 1200400E+02 | 0. 171000D+02                        | 0. 9100000D+00 | 0.0          | 0.0          |
| 100  | 0. 1200400E+02 | 0. 171000D+02                        | 0. 9100000D+00 | 0.0          | 0.0          |
| 101  | 0. 1200400E+02 | 0. 171000D+02                        | 0. 9100000D+00 | 0.0          | 0.0          |
| 102  | 0. 1200400E+02 | 0. 171000D+02                        | 0. 9100000D+00 | 0.0          | 0.0          |
| 103  | 0. 1200400E+02 | 0. 171000D+02                        | 0. 9100000D+00 | 0.0          | 0.0          |
| 104  | 0. 1200400E+02 | 0. 171000D+02                        | 0. 9100000D+00 | 0.0          | 0.0          |
| 105  | 0. 1200400E+02 | 0. 171000D+02                        | 0. 9100000D+00 | 0.0          | 0.0          |
| 106  | 0. 1200400E+02 | 0. 171000D+02                        | 0. 9100000D+00 | 0.0          | 0.0          |
| 107  | 0. 1200400E+02 | 0. 171000D+02                        | 0. 9100000D+00 | 0.0          | 0.0          |
| 108  | 0. 1200400E+02 | 0. 171000D+02                        | 0. 9100000D+00 | 0.0          | 0.0          |
| 109  | 0. 1200400E+02 | 0. 171000D+02                        | 0. 9100000D+00 | 0.0          | 0.0          |
| 110  | 0. 1200400E+02 | 0. 171000D+02                        | 0. 9100000D+00 | 0.0          | 0.0          |
| 111  | 0. 1200400E+02 | 0. 171000D+02                        | 0. 9100000D+00 | 0.0          | 0.0          |
| 112  | 0. 1200400E+02 | 0. 171000D+02                        | 0. 9100000D+00 | 0.0          | 0.0          |
| 113  | 0. 1200400E+02 | 0. 171000D+02                        | 0. 9100000D+00 | 0.0          | 0.0          |
| 114  | 0. 1200400E+02 | 0. 171000D+02                        | 0. 9100000D+00 | 0.0          | 0.0          |
| 115  | 0. 1200400E+02 | 0. 171000D+02                        | 0. 9100000D+00 | 0.0          | 0.0          |
| 116  | 0. 1200400E+02 | 0. 171000D+02                        | 0. 9100000D+00 | 0.0          | 0.0          |
| 117  | 0. 1200400E+02 | 0. 171000D+02                        | 0. 9100000D+00 | 0.0          | 0.0          |
| 118  | 0. 1200400E+02 | 0. 171000D+02                        | 0. 9100000D+00 | 0.0          | 0.0          |
| 119  | 0. 1200400E+02 | 0. 171000D+02                        | 0. 9100000D+00 | 0.0          | 0.0          |
| 120  | 0. 1236500E+02 | 0. 161236D+02                        | 0. 900352D+00  | 0.0          | 0.0          |

| NODE | Z(I)           | SUMMARY OF NODAL COORDINATES |              |      | ABS FLOW ANG | REL FLOW ANG |
|------|----------------|------------------------------|--------------|------|--------------|--------------|
|      |                | R(I)                         | S(I)         | T(I) |              |              |
| 121  | 0.183295E+02   | 0.155706D+02                 | 0.900157D+00 | 0.0  | 0.0          | 0.0          |
| 122  | 0.1822520E+02  | 0.149315D+02                 | 0.399463D+00 | 0.0  | 0.0          | 0.0          |
| 123  | 0.1822520E+02  | 0.143400D+02                 | 0.394627D+00 | 0.0  | 0.0          | 0.0          |
| 124  | 0.1822520E+02  | 0.137617D+02                 | 0.397659D+00 | 0.0  | 0.0          | 0.0          |
| 125  | 0.1822520E+02  | 0.131054D+02                 | 0.394626D+00 | 0.0  | 0.0          | 0.0          |
| 126  | 0.1822520E+02  | 0.124101D+02                 | 0.395389D+00 | 0.0  | 0.0          | 0.0          |
| 127  | 0.181095E+02   | 0.116758D+02                 | 0.362246D+00 | 0.0  | 0.0          | 0.0          |
| 128  | 0.180733E+02   | 0.108210D+02                 | 0.360216D+00 | 0.0  | 0.0          | 0.0          |
| 129  | 0.180365E+02   | 0.100750D+02                 | 0.347743D+00 | 0.0  | 0.0          | 0.0          |
| 130  | 0.180000E+02   | 0.912500D+01                 | 0.370122D+00 | 0.0  | 0.0          | 0.0          |
| 131  | 0.19445317E+02 | 0.182305D+02                 | 0.963326D+00 | 0.0  | 0.0          | 0.0          |
| 132  | 0.19446361E+02 | 0.150190D+02                 | 0.349761D+00 | 0.0  | 0.0          | 0.0          |
| 133  | 0.19446361E+02 | 0.150192D+02                 | 0.335566D+00 | 0.0  | 0.0          | 0.0          |
| 134  | 0.19446361E+02 | 0.148620D+02                 | 0.219544D+00 | 0.0  | 0.0          | 0.0          |
| 135  | 0.19446361E+02 | 0.126162D+02                 | 768639D+00   | 0.0  | 0.0          | 0.0          |
| 136  | 0.19446361E+02 | 0.112324D+02                 | 742266D+00   | 0.0  | 0.0          | 0.0          |
| 137  | 0.19446361E+02 | 0.963300D+01                 | 722375D+00   | 0.0  | 0.0          | 0.0          |
| 138  | 0.19446361E+02 | 0.176710D+02                 | 901992D+00   | 0.0  | 0.0          | 0.0          |
| 139  | 0.19446361E+02 | 0.174325D+02                 | 901446D+00   | 0.0  | 0.0          | 0.0          |
| 140  | 0.19446361E+02 | 0.165215D+02                 | 902035D+00   | 0.0  | 0.0          | 0.0          |
| 141  | 0.19446361E+02 | 0.160470D+02                 | 901244D+00   | 0.0  | 0.0          | 0.0          |
| 142  | 0.19446361E+02 | 0.155678D+02                 | 900278D+00   | 0.0  | 0.0          | 0.0          |
| 143  | 0.19446361E+02 | 0.154528D+02                 | 900666D+00   | 0.0  | 0.0          | 0.0          |
| 144  | 0.19446361E+02 | 0.153671D+02                 | 937793D+00   | 0.0  | 0.0          | 0.0          |
| 145  | 0.19446361E+02 | 0.153671D+02                 | 900756D+00   | 0.0  | 0.0          | 0.0          |
| 146  | 0.19446361E+02 | 0.153671D+02                 | 900756D+00   | 0.0  | 0.0          | 0.0          |
| 147  | 0.19446361E+02 | 0.153671D+02                 | 900756D+00   | 0.0  | 0.0          | 0.0          |
| 148  | 0.19446361E+02 | 0.128338D+02                 | 495680D+00   | 0.0  | 0.0          | 0.0          |
| 149  | 0.19446361E+02 | 0.122616D+02                 | 954372D+00   | 0.0  | 0.0          | 0.0          |
| 150  | 0.19446361E+02 | 0.120560D+02                 | 900740D+00   | 0.0  | 0.0          | 0.0          |
| 151  | 0.19446361E+02 | 0.116765D+02                 | 190541D+00   | 0.0  | 0.0          | 0.0          |
| 152  | 0.19446361E+02 | 0.113535D+02                 | 367706D+00   | 0.0  | 0.0          | 0.0          |
| 153  | 0.19446361E+02 | 0.110000D+02                 | 910000D+00   | 0.0  | 0.0          | 0.0          |
| 154  | 0.19446361E+02 | 0.106450D+02                 | 910000D+00   | 0.0  | 0.0          | 0.0          |
| 155  | 0.19446361E+02 | 0.102970D+02                 | 910000D+00   | 0.0  | 0.0          | 0.0          |
| 156  | 0.19446361E+02 | 0.102970D+02                 | 910000D+00   | 0.0  | 0.0          | 0.0          |
| 157  | 0.19446361E+02 | 0.102970D+02                 | 910000D+00   | 0.0  | 0.0          | 0.0          |
| 158  | 0.19446361E+02 | 0.102970D+02                 | 910000D+00   | 0.0  | 0.0          | 0.0          |
| 159  | 0.19446361E+02 | 0.102970D+02                 | 910000D+00   | 0.0  | 0.0          | 0.0          |
| 160  | 0.220347E+02   | 0.111752D+02                 | 910000D+00   | 0.0  | 0.0          | 0.0          |

| ACCE | Z(I)          | SUMMARY OF MODAL COORDINATES |               |     | ABS FLOW ANG | REL FLOW ANG |
|------|---------------|------------------------------|---------------|-----|--------------|--------------|
|      |               | R(I)                         | B(I)          |     |              |              |
| 161  | C.22105E+02   | C.1C393CD+02                 | C.9100CCD+00  | 0.0 |              | 0.0          |
| 162  | C.22950E+02   | C.179300D+02                 | C.3990680+00  | 0.0 |              | 0.0          |
| 163  | C.28E711C+E02 | C.1752110+02                 | C.8829320+00  | 0.0 |              | 0.0          |
| 164  | C.289011C+E02 | C.169950D+02                 | C.8987750+00  | 0.0 |              | 0.0          |
| 165  | C.44911E+E02  | C.1666010+02                 | C.3445932+00  | 0.0 |              | 0.0          |
| 166  | C.229564E+E02 | C.1611220+02                 | C.8683830+00  | 0.0 |              | 0.0          |
| 167  | C.229764E+E02 | C.155180D+02                 | C.4091430+00  | 0.0 |              | 0.0          |
| 168  | C.20076E+E02  | C.1517720+02                 | C.3978730+00  | 0.0 |              | 0.0          |
| 169  | C.20176E+E02  | C.1468740+02                 | C.3975720+00  | 0.0 |              | 0.0          |
| 170  | C.20176E+E02  | C.141P060+02                 | C.8372400+00  | 0.0 |              | 0.0          |
| 171  | C.20176E+E02  | C.131C850+02                 | C.86648780+00 | 0.0 |              | 0.0          |
| 172  | C.20176E+E02  | C.1253810+02                 | C.9260510+00  | 0.0 |              | 0.0          |
| 173  | C.20176E+E02  | C.1194040+02                 | C.955730+00   | 0.0 |              | 0.0          |
| 174  | C.20176E+E02  | C.1131130+02                 | C.850620+00   | 0.0 |              | 0.0          |
| 175  | C.20176E+E02  | C.1064500+02                 | C.84427L+00   | 0.0 |              | 0.0          |
| 176  | C.20176E+E02  | C.1783600+02                 | C.8212890+00  | 0.0 |              | 0.0          |
| 177  | C.20176E+E02  | C.1701660+02                 | C.32177+00    | 0.0 |              | 0.0          |
| 178  | C.20176E+E02  | C.1655570+02                 | C.32240+00    | 0.0 |              | 0.0          |
| 179  | C.20176E+E02  | C.152410+02                  | C.3233610+00  | 0.0 |              | 0.0          |
| 180  | C.20176E+E02  | C.1427330+02                 | C.3229730+00  | 0.0 |              | 0.0          |
| 181  | C.20176E+E02  | C.1324120+02                 | C.9219560+00  | 0.0 |              | 0.0          |
| 182  | C.20176E+E02  | C.1211490+02                 | C.32291100+00 | 0.0 |              | 0.0          |
| 183  | C.20176E+E02  | C.1087250+02                 | C.3948920+00  | 0.0 |              | 0.0          |
| 184  | C.20176E+E02  | C.1783400+02                 | C.3937310+00  | 0.0 |              | 0.0          |
| 185  | C.20176E+E02  | C.1764130+02                 | C.3935620+00  | 0.0 |              | 0.0          |
| 186  | C.20176E+E02  | C.1703740+02                 | C.393440+00   | 0.0 |              | 0.0          |
| 187  | C.20176E+E02  | C.1442400+02                 | C.3933660+00  | 0.0 |              | 0.0          |
| 188  | C.20176E+E02  | C.1616440+02                 | C.89916E0+00  | 0.0 |              | 0.0          |
| 189  | C.20176E+E02  | C.1571421+02                 | C.8979620+00  | 0.0 |              | 0.0          |
| 190  | C.20176E+E02  | C.1531630+02                 | C.8779740+00  | 0.0 |              | 0.0          |
| 191  | C.20176E+E02  | C.1455430+02                 | C.3673260+00  | 0.0 |              | 0.0          |
| 192  | C.20176E+E02  | C.14337860+02                | C.3673260+00  | 0.0 |              | 0.0          |
| 193  | C.20176E+E02  | C.13283600+02                | C.06973260+00 | 0.0 |              | 0.0          |
| 194  | C.20176E+E02  | C.13337530+02                | C.3695260+00  | 0.0 |              | 0.0          |
| 195  | C.20176E+E02  | C.1294430+02                 | C.8660610+00  | 0.0 |              | 0.0          |
| 196  | C.20176E+E02  | C.12339040+02                | C.8735500+00  | 0.0 |              | 0.0          |
| 197  | C.20176E+E02  | C.1171340+02                 | C.1949910+00  | 0.0 |              | 0.0          |
| 198  | C.20176E+E02  | C.1110000+02                 | C.1943220+00  | 0.0 |              | 0.0          |
| 199  | C.20176E+E02  | C.1783600+02                 | C.9100600+00  | 0.0 |              | 0.0          |

| NODE | Z(I)         | SUMMARY OF NODAL COORDINATES |              |              | ABS FLOW ANG | REL FLOW ANG |
|------|--------------|------------------------------|--------------|--------------|--------------|--------------|
|      |              | R(I)                         | B(I)         | C(I)         |              |              |
| 201  | C.2647000+C2 | C.1704080+02                 | C.9100000+00 | C.9100000+00 | 0.0          | 0.0          |
| 202  | C.2964000+C2 | C.1920670+02                 | C.9100000+00 | C.9100000+00 | 0.0          | 0.0          |
| 203  | C.2610000+C2 | C.1932720+02                 | C.9100000+00 | C.9100000+00 | 0.0          | 0.0          |
| 204  | C.2624000+C2 | C.1934410+02                 | C.9100000+00 | C.9100000+00 | 0.0          | 0.0          |
| 205  | C.2655000+C2 | C.1936010+02                 | C.9100000+00 | C.9100000+00 | 0.0          | 0.0          |
| 206  | C.2654000+C2 | C.1937710+02                 | C.9100000+00 | C.9100000+00 | 0.0          | 0.0          |
| 207  | C.2630000+C2 | C.1783600+02                 | C.9100000+00 | C.9100000+00 | 0.0          | 0.0          |
| 208  | C.2630000+C2 | C.1784410+02                 | C.9100000+00 | C.9100000+00 | 0.0          | 0.0          |
| 209  | C.2630000+C2 | C.1784410+02                 | C.9100000+00 | C.9100000+00 | 0.0          | 0.0          |
| 210  | C.2630000+C2 | C.1784410+02                 | C.9100000+00 | C.9100000+00 | 0.0          | 0.0          |
| 211  | C.2630000+C2 | C.1663400+02                 | C.9100000+00 | C.9100000+00 | 0.0          | 0.0          |
| 212  | C.2630000+C2 | C.1574190+02                 | C.9100000+00 | C.9100000+00 | 0.0          | 0.0          |
| 213  | C.2630000+C2 | C.1574190+02                 | C.9100000+00 | C.9100000+00 | 0.0          | 0.0          |
| 214  | C.2630000+C2 | C.1574190+02                 | C.9100000+00 | C.9100000+00 | 0.0          | 0.0          |
| 215  | C.2630000+C2 | C.1440950+02                 | C.9100000+00 | C.9100000+00 | 0.0          | 0.0          |
| 216  | C.2630000+C2 | C.1352210+02                 | C.9100000+00 | C.9100000+00 | 0.0          | 0.0          |
| 217  | C.2630000+C2 | C.1341550+02                 | C.9100000+00 | C.9100000+00 | 0.0          | 0.0          |
| 218  | C.2630000+C2 | C.1341550+02                 | C.9100000+00 | C.9100000+00 | 0.0          | 0.0          |
| 219  | C.2630000+C2 | C.1341550+02                 | C.9100000+00 | C.9100000+00 | 0.0          | 0.0          |
| 220  | C.2630000+C2 | C.1341550+02                 | C.9100000+00 | C.9100000+00 | 0.0          | 0.0          |
| 221  | C.3300000+C2 | C.1117000+02                 | C.9100000+00 | C.9100000+00 | 0.0          | 0.0          |

**SYSTEM TOPOLOGY**

| ELEMENT | TYPE OF ELEMENT |
|---------|-----------------|
| 1       | 1               |
| 2       | 1               |
| 3       | 1               |
| 4       | 1               |
| 5       | 1               |
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| 221     | 1               |

INLET THERMODYNAMIC VARIABLES ARE AS FOLLOWS

FLOW RATE = 0.17454CD+03 LB/M/SEC

TOT DENSITY = 0.251884D-01LB/M CU FT

TOT PRESSURE = 0.17500CD+02 PSI

TOT TEMPERATURE = 0.555000D+03 DEG RANKINE

ROTATIONAL SPEED = 0.697520D+04 RPM

INLET U VELOCITY = 0.306522D+03 FT/SEC

GAS CONSTANT = 0.52300CD+02

RATIO OF SPECIFIC HEATS = 0.140000D+01

SPECIFIC HEAT AT CONSTANT PRESSURE = 0.240000D+00

STATIC DENSITY AT INLET = 0.922124D-01

#### **NOTES WHERE PSI IS SPECIFIED**

LARGEST EPS FCF ITERATION 1 IS 0.3467595592C6D+00  
ITERATION NO. 1 COMPLETE STREAM FUNCTION CONVERGENCE NOT YET SATISFIED.  
NEXT ITERATION IS IN PROGRESS  
LARGEST EPS FCF ITERATION 2 IS 0.2C4B02273624D+00  
ITERATION NO. 2 COMPLETE STREAM FUNCTION CONVERGENCE NOT YET SATISFIED.  
NEXT ITERATION IS IN PROGRESS  
LARGEST EPS FCF ITERATION 3 IS 0.135922144523D+00  
ITERATION NO. 3 COMPLETE STREAM FUNCTION CONVERGENCE NOT YET SATISFIED.  
NEXT ITERATION IS IN PROGRESS  
LARGEST EPS FCF ITERATION 4 IS 0.941426824100D-01  
ITERATION NO. 4 COMPLETE STREAM FUNCTION CONVERGENCE NOT YET SATISFIED.  
NEXT ITERATION IS IN PROGRESS  
LARGEST EPS FCF ITERATION 5 IS 0.662198640489D-01  
ITERATION NO. 5 COMPLETE STREAM FUNCTION CONVERGENCE NOT YET SATISFIED.  
NEXT ITERATION IS IN PROGRESS  
LARGEST EPS FCF ITERATION 6 IS 0.469498753800D-01  
ITERATION NO. 6 COMPLETE STREAM FUNCTION CONVERGENCE NOT YET SATISFIED.  
NEXT ITERATION IS IN PROGRESS  
LARGEST EPS FCF ITERATION 7 IS 0.342379259092D-01  
ITERATION NO. 7 COMPLETE STREAM FUNCTION CONVERGENCE NOT YET SATISFIED.  
NEXT ITERATION IS IN PROGRESS  
FRGPM TERMINATED ON ITERATION NO. 9  
RESULTS WHICH FOLLOW ARE FOR CONVERGENCE EPSILON = 0.189017741652D-01  
STREAM FUNCTION CONVERGENCE CRITERION SATISFIED ON ITERATION NUMBER 9  
RESULTS ARE AS FOLLOWS FCR CONVERGENCE EPSILON = 0.189017741652D-01

FINITE ELEMENT RESULTS

**FINITE ELEMENT RESULTS**

|  | DENSITY | RILL | VR   | V2   | PSI (1) | KCDE |
|--|---------|------|------|------|---------|------|
|  | 0.02    | 0.02 | 0.02 | 0.02 | 0.01    | 0.01 |
|  | 0.04    | 0.04 | 0.04 | 0.04 | 0.02    | 0.02 |
|  | 0.06    | 0.06 | 0.06 | 0.06 | 0.03    | 0.03 |
|  | 0.08    | 0.08 | 0.08 | 0.08 | 0.04    | 0.04 |
|  | 0.10    | 0.10 | 0.10 | 0.10 | 0.05    | 0.05 |
|  | 0.12    | 0.12 | 0.12 | 0.12 | 0.06    | 0.06 |
|  | 0.14    | 0.14 | 0.14 | 0.14 | 0.07    | 0.07 |
|  | 0.16    | 0.16 | 0.16 | 0.16 | 0.08    | 0.08 |
|  | 0.18    | 0.18 | 0.18 | 0.18 | 0.09    | 0.09 |
|  | 0.20    | 0.20 | 0.20 | 0.20 | 0.10    | 0.10 |
|  | 0.22    | 0.22 | 0.22 | 0.22 | 0.11    | 0.11 |
|  | 0.24    | 0.24 | 0.24 | 0.24 | 0.12    | 0.12 |
|  | 0.26    | 0.26 | 0.26 | 0.26 | 0.13    | 0.13 |
|  | 0.28    | 0.28 | 0.28 | 0.28 | 0.14    | 0.14 |
|  | 0.30    | 0.30 | 0.30 | 0.30 | 0.15    | 0.15 |
|  | 0.32    | 0.32 | 0.32 | 0.32 | 0.16    | 0.16 |
|  | 0.34    | 0.34 | 0.34 | 0.34 | 0.17    | 0.17 |
|  | 0.36    | 0.36 | 0.36 | 0.36 | 0.18    | 0.18 |
|  | 0.38    | 0.38 | 0.38 | 0.38 | 0.19    | 0.19 |
|  | 0.40    | 0.40 | 0.40 | 0.40 | 0.20    | 0.20 |
|  | 0.42    | 0.42 | 0.42 | 0.42 | 0.21    | 0.21 |
|  | 0.44    | 0.44 | 0.44 | 0.44 | 0.22    | 0.22 |
|  | 0.46    | 0.46 | 0.46 | 0.46 | 0.23    | 0.23 |
|  | 0.48    | 0.48 | 0.48 | 0.48 | 0.24    | 0.24 |
|  | 0.50    | 0.50 | 0.50 | 0.50 | 0.25    | 0.25 |
|  | 0.52    | 0.52 | 0.52 | 0.52 | 0.26    | 0.26 |
|  | 0.54    | 0.54 | 0.54 | 0.54 | 0.27    | 0.27 |
|  | 0.56    | 0.56 | 0.56 | 0.56 | 0.28    | 0.28 |
|  | 0.58    | 0.58 | 0.58 | 0.58 | 0.29    | 0.29 |
|  | 0.60    | 0.60 | 0.60 | 0.60 | 0.30    | 0.30 |
|  | 0.62    | 0.62 | 0.62 | 0.62 | 0.31    | 0.31 |
|  | 0.64    | 0.64 | 0.64 | 0.64 | 0.32    | 0.32 |
|  | 0.66    | 0.66 | 0.66 | 0.66 | 0.33    | 0.33 |
|  | 0.68    | 0.68 | 0.68 | 0.68 | 0.34    | 0.34 |
|  | 0.70    | 0.70 | 0.70 | 0.70 | 0.35    | 0.35 |
|  | 0.72    | 0.72 | 0.72 | 0.72 | 0.36    | 0.36 |
|  | 0.74    | 0.74 | 0.74 | 0.74 | 0.37    | 0.37 |
|  | 0.76    | 0.76 | 0.76 | 0.76 | 0.38    | 0.38 |
|  | 0.78    | 0.78 | 0.78 | 0.78 | 0.39    | 0.39 |
|  | 0.80    | 0.80 | 0.80 | 0.80 | 0.40    | 0.40 |
|  | 0.82    | 0.82 | 0.82 | 0.82 | 0.41    | 0.41 |
|  | 0.84    | 0.84 | 0.84 | 0.84 | 0.42    | 0.42 |
|  | 0.86    | 0.86 | 0.86 | 0.86 | 0.43    | 0.43 |
|  | 0.88    | 0.88 | 0.88 | 0.88 | 0.44    | 0.44 |
|  | 0.90    | 0.90 | 0.90 | 0.90 | 0.45    | 0.45 |
|  | 0.92    | 0.92 | 0.92 | 0.92 | 0.46    | 0.46 |
|  | 0.94    | 0.94 | 0.94 | 0.94 | 0.47    | 0.47 |
|  | 0.96    | 0.96 | 0.96 | 0.96 | 0.48    | 0.48 |
|  | 0.98    | 0.98 | 0.98 | 0.98 | 0.49    | 0.49 |
|  | 1.00    | 1.00 | 1.00 | 1.00 | 0.50    | 0.50 |

## FINITE ELEMENT RESULTS

DENSITY

R111

VR

VZ

POS111

ACCE

### FINITE ELEMENT RESULTS

|  |         |   |   |   |   |   |   |
|--|---------|---|---|---|---|---|---|
|  | DENSITY | 0.7810060-0.7815910+0.7814250-0.7817768                     | 0.7810060-0.7815910+0.7814250-0.7817768                     | 0.7810060-0.7815910+0.7814250-0.7817768                     | 0.7810060-0.7815910+0.7814250-0.7817768                     | 0.7810060-0.7815910+0.7814250-0.7817768                     | 0.7810060-0.7815910+0.7814250-0.7817768                     |
|  | R(11)   | 0.70360+0.70370-0.70360+0.70370                             | 0.70360+0.70370-0.70360+0.70370                             | 0.70360+0.70370-0.70360+0.70370                             | 0.70360+0.70370-0.70360+0.70370                             | 0.70360+0.70370-0.70360+0.70370                             | 0.70360+0.70370-0.70360+0.70370                             |
|  | V(1)    | 0.132100+0.12000-0.14480+0.12000                            | 0.132100+0.12000-0.14480+0.12000                            | 0.132100+0.12000-0.14480+0.12000                            | 0.132100+0.12000-0.14480+0.12000                            | 0.132100+0.12000-0.14480+0.12000                            | 0.132100+0.12000-0.14480+0.12000                            |
|  | V(2)    | 0.613200+0.03000-0.613200+0.03000                           | 0.613200+0.03000-0.613200+0.03000                           | 0.613200+0.03000-0.613200+0.03000                           | 0.613200+0.03000-0.613200+0.03000                           | 0.613200+0.03000-0.613200+0.03000                           | 0.613200+0.03000-0.613200+0.03000                           |
|  | PSI(11) | 0.00177573E+02-0.00177698E+02+0.00177615E+02-0.00177665E+02 | 0.00177573E+02-0.00177698E+02+0.00177615E+02-0.00177665E+02 | 0.00177573E+02-0.00177698E+02+0.00177615E+02-0.00177665E+02 | 0.00177573E+02-0.00177698E+02+0.00177615E+02-0.00177665E+02 | 0.00177573E+02-0.00177698E+02+0.00177615E+02-0.00177665E+02 | 0.00177573E+02-0.00177698E+02+0.00177615E+02-0.00177665E+02 |
|  | ACDE    | 1.2116667E+02-1.2116667E+02+1.2116667E+02-1.2116667E+02     | 1.2116667E+02-1.2116667E+02+1.2116667E+02-1.2116667E+02     | 1.2116667E+02-1.2116667E+02+1.2116667E+02-1.2116667E+02     | 1.2116667E+02-1.2116667E+02+1.2116667E+02-1.2116667E+02     | 1.2116667E+02-1.2116667E+02+1.2116667E+02-1.2116667E+02     | 1.2116667E+02-1.2116667E+02+1.2116667E+02-1.2116667E+02     |

FINITE ELEMENT RESULTS

FINITE ELEMENT RESULTS

The image shows a decorative page border. The top and right edges feature a repeating pattern of small, stylized cross-like motifs. Along the bottom edge, there is a horizontal band of decorative dots. On the far left, there is a vertical column of text and symbols. The text consists of three rows of numbers: '1901 1902 1903' stacked vertically. Between the first and second rows of numbers are two rows of plus signs ('+') and between the second and third rows are two rows of minus signs ('-').

三

A decorative horizontal border consisting of a repeating pattern of small circles and dots.

2

କାନ୍ତିର ପଦମାଲା ପଦମାଲା ପଦମାଲା ପଦମାଲା ପଦମାଲା

HT

A decorative horizontal border at the bottom of the page. It consists of a repeating pattern of small circles and dots, alternating in size and position, set against a dark background.

卷八

400

בְּנֵי יִשְׂרָאֵל וְבְנֵי יִהוָה כַּאֲשֶׁר  
יְהוָה אֱלֹהֵינוּ נִצְחָנוּ בְּנֵי יִשְׂרָאֵל

A decorative horizontal border at the bottom of the page. It consists of a repeating pattern of small circles and vertical lines. The pattern is composed of two main elements: a solid circle followed by a vertical line, which then repeats across the width of the border.

A decorative horizontal border consisting of two rows of small circles, centered at the bottom of the page.

၁၂

A decorative horizontal border at the bottom of the page. It consists of a repeating pattern of small circles and dots arranged in a grid-like structure. The pattern is composed of two main elements: a larger circle with a smaller circle inside it, and a single dot. These elements are arranged in rows and columns to create a textured, woven appearance.

A decorative horizontal border pattern consisting of a repeating sequence of small, stylized geometric shapes. The pattern includes small crosses, dots, and other abstract motifs arranged in a grid-like fashion.

URL (1)

ପାଦମୁଖରେ କାହାର ପାଦମୁଖରେ କାହାର ପାଦମୁଖରେ କାହାର ପାଦମୁଖରେ କାହାର ପାଦମୁଖରେ

This block contains a decorative border pattern. The inner border consists of a series of small circles arranged in a rectangular frame. The outer border is formed by a continuous line that creates a series of diamond-shaped patterns, each containing a small circle at its center. The entire pattern is rendered in a dark color against a light background.

॥ १ ॥

A decorative horizontal border at the bottom of the page. The top half consists of a repeating pattern of small circles in a light beige color. The bottom half features a decorative band with a central floral or geometric motif, flanked by vertical columns of small circles.

WT

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A decorative horizontal border at the bottom of the page, consisting of a repeating pattern of small, stylized floral or geometric motifs, possibly representing a traditional Islamic or Persian design.

MONTE

PLOT-C1  
PLOT  
PRESSURE  
PLOT  
TEMP  
ACCE



শ প্রেরণার পথের অন্তর্ভুক্ত কোর্টের বিষয়ে নির্ণয় দেওয়া হল।  
১০০০০০০ টাকা প্রেরণা করা হল।

I ALPH-A PET-A

|     |    |          |     |        |
|-----|----|----------|-----|--------|
| 151 | 47 | 352873   | 26. | 945351 |
| 152 | 46 | 225312   | 2C. | 13777C |
| 153 | 48 | 57344E   | 11. | 63E745 |
| 154 | 37 | 144172   | C.O |        |
| 155 | 36 | 22208C   | O.O |        |
| 156 | 36 | 49535    | C.O |        |
| 157 | 40 | 264267   | O.O |        |
| 158 | 42 | 644644   | C.O |        |
| 159 | 44 | E37178   | C.O |        |
| 160 | 47 | 177568   | C.O |        |
| 161 | 46 | E63456   | O.O |        |
| 162 | 36 | 662038   | C.O |        |
| 163 | 37 | 175301   | O.O |        |
| 164 | 37 | 450088   | C.O |        |
| 165 | 37 | 26953    | C.O |        |
| 166 | 37 | C60664   | O.O |        |
| 167 | 37 | 781893   | O.O |        |
| 168 | 38 | 750196   | O.O |        |
| 169 | 39 | E22428   | O.C |        |
| 170 | 40 | 5777704  | O.O |        |
| 171 | 42 | 142144   | C.O |        |
| 172 | 43 | 150334   | O.O |        |
| 173 | 44 | 470770   | O.O |        |
| 174 | 44 | 142851   | C.O |        |
| 175 | 46 | 482382   | C.O |        |
| 176 | 47 | E57274   | O.O |        |
| 177 | 16 | 1E450E   | C.O |        |
| 178 | 17 | 779586   | O.O |        |
| 179 | 18 | C1631C   | C.O |        |
| 180 | 18 | 617928   | C.O |        |
| 181 | 20 | 26766C   | O.O |        |
| 182 | 21 | 428172   | C.O |        |
| 183 | 23 | C20279   | O.O |        |
| 184 | -4 | 33047    | C.O |        |
| 185 | -2 | 724489   | C.O |        |
| 186 | -1 | 542182   | C.O |        |
| 187 | -1 | 21081    | C.O |        |
| 188 | -1 | C48351   | C.O |        |
| 189 | -1 | 929789   | C.O |        |
| 190 | -0 | F2656    | C.O |        |
| 191 | -0 | 810042   | C.O |        |
| 192 | -0 | E3789    | C.O |        |
| 193 | -0 | E56746   | O.O |        |
| 194 | -0 | 296937   | C.O |        |
| 195 | -0 | 135356   | C.O |        |
| 196 | -0 | 3381202  | O.O |        |
| 197 | -0 | 8346922  | C.O |        |
| 198 | -0 | 8566694  | C.O |        |
| 199 | -1 | 576527   | O.O |        |
| 200 | -4 | 776527   | O.O |        |
| 201 | -2 | C276C2   | C.O |        |
| 202 | -1 | 128587   | C.O |        |
| 203 | -0 | E50967   | O.O |        |
| 204 | -0 | E9791    | C.O |        |
| 205 | -0 | 252444   | O.O |        |
| 206 | -0 | C410C    | C.O |        |
| 207 | -2 | E2594C   | O.O |        |
| 208 | -4 | 656014   | O.O |        |
| 209 | -2 | 194146   | C.O |        |
| 210 | -1 | 5C12887  | C.O |        |
| 211 | -1 | 25830318 | C.O |        |
| 212 | -0 | 5863226  | C.O |        |
| 213 | -0 | 612828C  | O.O |        |
| 214 | -0 | E23388C  | C.O |        |
| 215 | -0 | 6859226  | O.O |        |
| 216 | -0 | 556639   | C.O |        |
| 217 | -0 | 266610   | O.O |        |
| 218 | -0 | 202307   | C.O |        |
| 219 | -1 | 220231   | C.O |        |
| 220 | -3 | C18592   | C.O |        |

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